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Enhancement of Vehicle Eco-Driving Applicability through Road Infrastructure Design and Exploitation

Alex Coiret ^{1,*} , Pierre-Olivier Vandanjon ²  and Romain Noël ³ 

¹ COSYS-SII, Gustave Eiffel University, F-44344 Bouguenais, France

² SPLOTT, Gustave Eiffel University, F-44344 Bouguenais, France

³ I4S, COSYS-SII, INRIA, Gustave Eiffel University, F-44344 Bouguenais, France

* Correspondence: alex.coiret@univ-eiffel.fr

Abstract: Energy moderation of the road transportation sector is required to limit climate change and to preserve resources. This work is focused on the moderation of vehicle consumption by optimizing the speed policy along an itinerary while taking into account vehicle dynamics, driver visibility and the road's longitudinal profile. First, a criterion is proposed in order to detect speed policies that are impeding drivers' eco-driving ability. Then, an energy evaluation is carried out and an optimization is proposed. A numerical application is performed on a speed limiting point with 20 usage cases and 5 longitudinal slope values. In the hypothesis of a longitudinal slope of zero, energy savings of 27.7 liter per day could be realized by a speed sign displacement of only 153.6 m. Potential energy savings can increase to up to 308.4 L per day for a −4% slope case, or up to 70.5 L per day for an ordinary −2% slope, with a sign displacement of only 391.5 m. This results in a total of 771,975 L of fuel savings over a 30 year infrastructure life cycle period. Therefore a methodology has been developed to help road managers optimize their speed policies with the aim of moderating vehicle consumption.

Keywords: road infrastructure; energy moderation; eco-driving; speed policy



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1. Introduction

This work aims to propose simple strategies to reduce road vehicle consumption by enhancing eco-driving applicability with the help of small modifications to road management. The context is linked to the energy moderation needed in terms of both current energy availability issues and long-term climate change impacts.

The energy context is highly critical, as since 2021, the world has faced an energy crisis initiated by a large economic rebound after COVID-19 pandemic lockdowns, which was amplified in 2022 by the conflict between Ukraine and Russia, leading to the discontinuation of Russian gas and oil deliveries to Europe.

Lower availability and higher prices of energy sources, such as gas, petrol and electricity require strategies to limit all sorts of energy demands. Moreover this is in line with the needed global action to mitigate climate change, mainly through the moderation of fossil energy use.

Even if the energy context is so significant that the International Energy Agency is qualifying it as a “global energy crisis” [1], the tensions over energy availability are, however, linked to long-term processes. It has been more than half a century since the risks of oil availability were anticipated within the “peak oil” theory [2]. Oil availability was a major concern in the 1970s, but gas prices were impacted less. The present energy crisis concerns all major energy sources even electricity from nuclear plants and gas because energy economic models are more interlinked now than in the 1970s [1].

Energy moderation is necessary in this context, especially when considering the impact of oil-based energy on climate change [3], which is another major threat to worldwide populations. Indeed, concerning climate change associated with global fossil energy

consumption, every Intergovernmental Panel on Climate Change (IPCC) special report is more alarming [4] and a massive cut to global CO₂ emissions is recommended in order to limit global warming below 2 or 3 degrees.

Nevertheless global CO₂ emissions are rising almost every year, as observed in 2018 and 2019 [5].

The road transportation sector accounts for 28% of worldwide energy use [5]. Attempts at energy reduction have been performed for infrastructure, drivers and vehicle parts. The infrastructure can be optimized in the construction phase by optimizing its longitudinal profile [6], with a possible energy moderation of up to 6%, and by considering the construction and usage phases over a 10 year period.

Energy reduction can be indirectly achieved by urban policies and planning favoring active mobility and public transportation [7]. In this field, in [8], the concept of urban regeneration was applied to the case study of a canal port in Italy, with the joint benefits of active mobility promotion and tree and vegetable implantation, leading to energy moderation and air pollution mitigation.

If considering the vehicles, their energy use still relies on non-renewable energy sources (primarily oil) due to its high energy density and ease of storage. While electric batteries and hydrogen fuel cells offer promising alternatives, they currently face challenges such as a lower energy density in batteries, leading to heavier vehicles or shorter displacement ranges, and hydrogen storage tanks with leakage issues in long-term storage, despite ongoing development of various hydrogen thermodynamic and material-based storage techniques [9].

Therefore this work is centered on the energy reduction of gasoline or diesel vehicles, but this methodology could be profitable for energy reduction in the global meaning for electric or hydrogen vehicles too, since it relies on the minimization of total mechanical energy demand from the road infrastructure.

The energy reduction of vehicles can be seen as the result of a given traffic model. Such information can be studied at different time and space scales, leading to different classification of traffic models. From smaller scales to larger scales, microscopic models describe each vehicle individually [10,11], the mesoscopic models rely on the statistical dynamics of drivers [12,13], and the macroscopic scale focuses on the mean variables regardless of each vehicle's dynamics [14]. Our study belongs to microscopic models, which take into account interactions between vehicles and roads.

Energy use of a vehicle can be related to the vehicle's characteristics, such as its mass, frontal area, motor efficiency or tire width, but it also depends on the driver's behavior or route planning. Additionally, energy consumption can be influenced by some road properties such as the macrotecture, longitudinal slopes, or variations in allowed speeds along an itinerary.

This study takes into account three categories of impact on energy use: vehicle, driver and road infrastructure. Particular attention is paid to the road impact, from the road management point of view, because this energy moderation mean has been investigated little up to now. In particular, this work aims to reduce the energy use of vehicles by optimizing road signalization. Thus, the context is the road energy demand, which is expected to increase by a factor of 1.4–2.3 by 2030 compared with the levels observed in 2020 and 2010 in China [15].

The subcontext is the eco-driving ability, with for example a reduction of more than 10% of energy that could be achieved by teaching eco-driving rules to a group of more than 100 participants [16]. Therefore, the motivation and novelty of our research are to induce eco-driving behavior from usual drivers, only by suggesting speed profiles close to those followed by eco-drivers. A supplementary advantage of our methodology is giving drivers an opportunity to reduce their energy consumption without having to deal with multiple goals in driving behavior, such as safety, time saving and fuel saving [17].

Lastly it is important to note that road speed management has two primary aims: road safety and mobility efficiency. Our work endeavors to add the energy moderation

aim to these while keeping the same or a higher road safety level by always enlarging a lower-speed road section over the neighboring higher-speed section. By doing so, mobility is affected and the road manager should be attentive to excessive length of speed sign displacement proposed by the model. Therefore, the real novelty of this research is to add the energy savings aim to the usual objectives of safety, comfort and mobility. For example, in [18] the speed distribution over an itinerary generally takes into account the high slopes for braking safety, but no attention is paid to less marked slopes in the energy moderation concern, from the road manager point of view. Individually, for a given vehicle, an onboard optimization could be provided to adapt the speed profile to the road profile [19], and specific look-ahead control system could allow up to 3.5% reduction in fuel consumption [20]. The present work is extending this energy moderation opportunity to a whole vehicle traffic facing a given road, without the need of onboard systems.

Our study methodology is detailed in the following part, and then an experimental case is described and results are given.

2. Methodology

2.1. Improper Road Signalization

There are situations on roads that force drivers to brake mechanically, whereas most eco-driving rules focus on decelerating without braking.

If we consider a given road route, we will call the road speed-sectioning the succession of allowed speed along an itinerary, and each point modifying the allowed speed will be called a speed-sectioning point.

Therefore, in the general case, situations that force drivers to brake are induced by a speed-sectioning point which can be the presence of a conventional speed sign, such as the sign visible in Figure 1, with a specific speed limit on it, or a roundabout, with a virtual speed limitation associated with its diameter or visibility field, or a stop sign with an implicit zero km/h limit, a road speed bump, a village entrance panel, and so on.

To study the decelerating maneuvers, the considered speed-sectioning point will be referred to as the Misplaced Speed Panel (MSP) position, and the upstream point from which the driver effectively sees the corresponding sign and starts to decelerate will be referred to as the Starting Point of Deceleration (SPD) point.

The necessity to use mechanical brakes will then rely on the difference in speed between the sections that are separated by the speed-sectioning point, the driver's visibility distance, reaction time, and the natural deceleration rate of the vehicle.

MSP and SPD positions can be reported in worsened road cases with the presence of a longitudinal slope, as illustrated in Figure 2, or in the presence of a sharp turn, as illustrated in Figure 3.

Other situations impeding eco-driving can be a combination of these two situations, such as the slope case without height variation or the turn case with a curvilinear slope.

In an "optimal situation", eco-driving should be assured if the driver starts to decelerate without braking at the SPD point and then reaches the MSP point at the exact indicated speed on this speed sign.

Our methodology is based on the determination of the optimal maneuver distance from the decelerating point to the sign position or speed-sectioning point, to be able to propose a technical modification of the road signalization that optimizes eco-driving. The optimization involves the opportunity to decelerate without using brakes and reach the sign at the required speed. However, this is just an opportunity and this work does not consider the diversity of driver behaviors.

Two evaluations of the eco-driving road level are proposed. First, a simple criterion, independent of the vehicle characteristics, is based on the mean loss of total mechanical energy of a vehicle facing a speed-sectioning point. This criterion would easily be used to roughly identify speed-sectioning points that are not allowing eco-driving since it will be expressed only in terms of allowed speed, sight distance and elevation difference. Second a more complex energy expression based on the vehicle characteristics, such as

mass, aerodynamic and rolling coefficients, will be developed. This expression will allow determining the optimal maneuver distance for each type of vehicle. These two evaluations means are detailed in the next sections.



Figure 1. Conventional speed sign beside a road

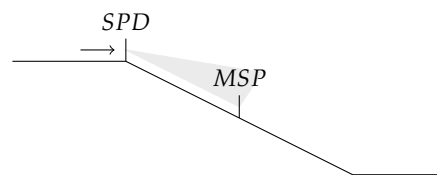


Figure 2. Slope situation.

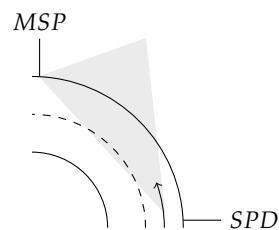


Figure 3. (Left-) turn situation.

2.2. Vehicle-Independent Criterion

In a first approach, a criterion is established to evaluate if a speed sectioning point can to impede eco-driving. A vehicle-independent criterion is proposed to be easily used by road managers without considering road traffic data too much.

For any case depicted in Figures 1–3, with or without a slope or turn, the approaching speed of the vehicle is defined as V_{SPD} , and the restricted speed of the sign V_{MSP} (with the hypothesis of required deceleration, i.e. $V_{MSP} < V_{SPD}$).

In an attempt to eco-drive, it is considered that the driver decelerates without braking over the d_{man} distance separating SPD and MSP positions, and that he reaches the MSP position at the exact V_{MSP} speed.

The criterion is built on the dissipation of mechanical energy along the decelerating distance. For a given vehicle of mass m , this dissipation of energy is decomposed into kinematic and gravitational potential energies:

$$\Delta E_m = \Delta E_k + \Delta E_p \quad (1)$$

$$\Delta E_k = \frac{1}{2} m (V_{SPD}^2 - V_{MSP}^2) \quad (2)$$

$$\Delta E_p = mg(h_{SPD} - h_{MSP}) \quad (3)$$

ΔE_k and ΔE_p the kinematic and potential energies are then expressed only as a function of vehicle mass and speeds, V_{SPD} and V_{MSP} , and vehicle altitudes, h_{SPD} and h_{MSP} , at the SPD and MSP positions.

To be more conveniently used by road managers, the criterion should be independent of the considered vehicle.

If we consider that the car-specific dissipation forces are related to the mg weight, particularly the rolling and air resistance, a dimensionless criterion can be built by dividing the total energy by a decreasing part of the weight over the maneuver distance.

This criterion, named Energy Alert Speed Management (EASM) and noted χ_{EASM} , is then proposed by the following relation:

$$\chi_{EASM} = \frac{\Delta E_m}{mg \log_{10}(d_{man})} . \quad (4)$$

The logarithm function is used to take into account the dominant aerodynamic drag, which decreases as the vehicle decelerates.

The criterion can be developed as a sum of a kinetic part and potential energy part:

$$\chi_{EASM} = \frac{1}{2} \frac{V_{SPD}^2 - V_{MSP}^2}{g \log_{10}(d_{man})} + \frac{h_{SPD} - h_{MSP}}{\log_{10}(d_{man})} . \quad (5)$$

For road managers, a low value of this criterion indicates a better eco-driving potentiality of a given speed sectioning point. Nevertheless, in the following section, a more accurate energy evaluation is developed for specific vehicles.

2.3. Energy Evaluation of a Speed-Sectioning Point

In this section, the energy-saving assessment is estimated for an optimal speed-sectioning point. This E_{MSP} energy takes into account the vehicle dissipating forces and the traffic composition.

The vehicle is modelled as a point undergoing natural deceleration from V_{SPD} to V_{MSP} . The applied forces are:

- The aerodynamic drag: $\frac{1}{2} \rho S C_d w_a^2$, with ρ being the air density, S being the frontal surface, C_d being the drag coefficient, w_a being the apparent wind (assumed to be equal to the vehicle speed);
- Rolling resistance: $mg C_{rr}$, with m being the mass, g being the gravity, C_{rr} being the rolling resistance coefficient;
- Internal forces of the vehicle: F_i (frictions and motor resistance, auxiliaries);
- Gravity forces: $mg \sin(\alpha_r)$, with α_r being the slope angle in radians.

By applying the fundamental principle of dynamics, the acceleration can be calculated as:

$$\gamma = -\frac{1}{m} \left[\frac{1}{2} C_d S v^2 + (mg[C_{rr} + \sin(\alpha)] + F_i) \right] , \quad (6)$$

that can be of the form:

$$\gamma = av^2 + c , \quad (7)$$

with $a = -\frac{1}{2} \rho S C_d / m$ and $c = -g[C_{rr} + \sin(\alpha)] - F_i / m$.

Without variation in the slope, the analytical solution is:

$$v(t) = -B \tan(A(t + K)) , \quad (8)$$

with $B = \sqrt{c/a}$, $A = \sqrt{ac}$, $K = -\arctan(V_{SPD}/B)/A$.

The position x is given by integration:

$$x(t) = E(\log[\cos(A(t + K))] - \log(\cos(AK))) , \quad (9)$$

with $E = 1/|a|$.

With the condition $x(t) = d_{\text{man}}$ the time t_f taken by the vehicle to travel the distance d_{man} is:

$$t_f = \frac{1}{A} \arccos\left(e^{d_{\text{man}}/E} \cos(AK)\right) - K. \quad (10)$$

At last, the energy that can be saved by an optimal speed-sectioning is:

$$E_{\text{MSP}} = \frac{1}{2} m \left(v^2(t_f) - V_{\text{MSP}}^2 \right). \quad (11)$$

In this ideal eco-driving friendly situation, the vehicle reaches V_{MSP} only through natural deceleration. This saved energy can be converted into saved CO₂ emissions, either by using classical conversion factor or by taking into account the traffic impacted by this configuration. This saved CO₂ quantity is denoted as C_{MSP} .

3. Experimental Case: A New Road Project in Nantes Métropole

In a research collaboration with the French Nantes Métropole city, our methodology has been applied in a real road project in the suburbs. This project is taken as a simple application example, since its realization is not yet validated. It gives us some general hypothesis, such as speeds, distances, and traffic.

After the first phase of variants selection, the retained geometry is given in Figure 4. Over this geometry, the present work is focusing on two sections: the existing section from A to B in Figure 4 and the projected section from B to C. The junction at the B point is supporting a principal traffic from A to B of around 7700 vehicles per day and a secondary traffic of 2100 vehicles per day on the crossing road at the B point. Two hypotheses for the junction are considered, either a roundabout or a simple crossroad with a stop sign on the transverse branch.

At first approximation, there is no longitudinal slope on the two roads in the junction vicinity. Therefore, our modeling effort will first be focused on vehicle speed, road layouts in horizontal plane, and related eco-driving opportunities. Nonetheless, simulations will be worked out for longitudinal slopes varying from -4% to 4% by 2% increments, in order to apprehend eco-driving assessments in case of possible road geometry of the definitive project.

Both the criterion and the energy-saving evaluation require knowledge of the variation in elevation of the vehicle, either explicitly for the criterion in Equation (5) or indirectly for the energy evaluation via the longitudinal slope α in Equation (6).

Our research is applied to a road project without slopes and to project variations with fixed slopes from -4% to 4% . To apply our methodology on a yet already built road, the variation in elevation could be measured from onboard systems. Usually, GNSS systems are used for vehicle position and speed acquisition. In [21], a differential GPS-RTK system has been used for an experimental campaign in France, and a simple non-differential GPS for a campaign in Bosnia. The z-axis precision of a differential GNSS system is decimeter (about 0.3 m), which is sufficient, but the z-axis data for the Bosnian campaign are reported to be filtered due to a lack of precision.

Nevertheless GNSS systems may not be sufficiently reliable in some situations because satellite signals are easily obstructed by trees and buildings and multipath signals may affect position computation. Therefore, this absolute sensor could be associated with dead-reckoning sensors such as inertial measurement units (IMU). Signal obstruction of the absolute GNSS system and measurement drift of the inertial system are weaknesses that can be overcome by a complementary usage of these two techniques, by data fusion [22]. Inertia and GNSS data fusion is especially important for autonomous vehicle to ensure a safe driving [23]. By extension other sensors can be added to the data fusion process, such as cameras or LiDAR [24]. These means should be considered for the application of our methodology to hilly terrains or dense urban areas.

In the next section, the criterion and energy gains are computed for these two geometrical hypotheses and several usage cases, such as allowed speed and signalization positions,

and for various slopes cases. Results are given in energy gains E_{MSP} and in CO₂ emission equivalent C_{MSP} .

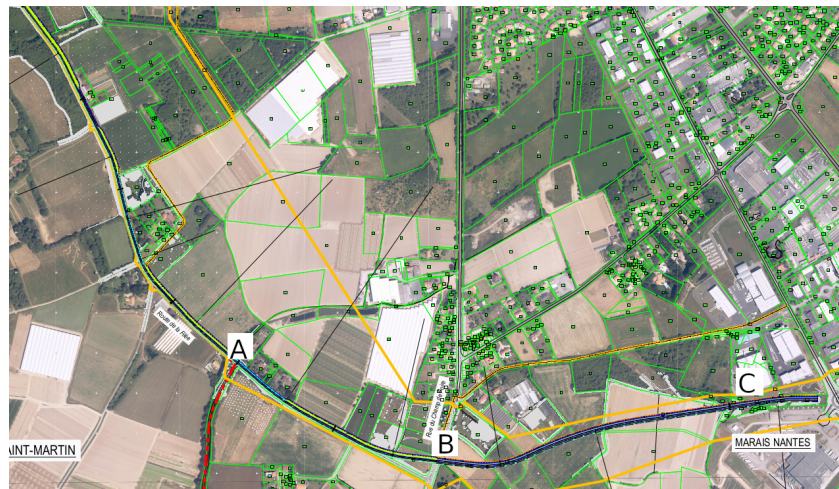


Figure 4. General plan of the circumvention road project.

4. Modeling Parameters From Experimental Conditions

In this section the road junction (“B” point on Figure 4) is detailed in terms of allowed speeds and sign implementations, that is, usage and infrastructure configuration terms. Then, the eco-driving potential associated with each of these configuration/usage sets will be evaluated in first approximation by the means of the simple criterion χ_{EASM} .

4.1. Manoeuvre Distance Hypothesis

Two distances between the speed sign and the intersection are considered: 100 m and 200 m, chosen from French Road rules. Two sight distances are taken into account: 100 m and 300 m.

4.2. Traffic Hypothesis

The traffic is considered to be 4000 vehicles per day on the new road segment, 7700 on the old road segment, and 2100 on the two segments of the transverse road.

Computations were performed for a diesel passenger car since heavy vehicles are not permitted on this circumvention, and diesel vehicles are dominant in the French vehicle market. The characteristics of this vehicle are recalled in Table 1 with data from [25].

Table 1. Characteristics of an average diesel vehicle.

Characteristic	Symbol	Value
mass	m	1200 kg
rolling resistance coefficient	C_{rr}	0.01
frontal surface of the vehicle	S	2 m ²
drag coefficient of the vehicle	C_d	0.3
internal forces	F_i	120 N
air density	ρ	1.2 kg/m ³
fuel consumption at 50 km/h	$c_f(50 \text{ km/h})$	4.4 L/100 km
fuel consumption at 70 km/h	$c_f(70 \text{ km/h})$	4.5 L/100 km
emission of CO ₂ per diesel L consumed	T_f	3.17 kg/L

4.3. Case Study Definition

The existing segment on the principal road, from A to B is named ES. The new segment from B to C is named NS. Then the transverse road is named TR.

The crossroad case is named CR and the roundabout case is named RA. The first speed case is of 50 km/h on ES and NS segments; the second speed case is of 70 km/h on ES and 70 km/h on NS. These cases are named V5050 and V5070.

Then, the eight geometrical configurations:

- C1: RA (roundabout), V5050, position 100 m on ES, NS and TR;
- C2: RA, V5050, position 200 m on ES, NS and TR;
- C3: RA, V5070, position 100 m on ES, NS and TR;
- C4: RA, V5070, position 200 m on ES, NS and TR;
- C5: CR (crossroad), V5050, position 100 m on TR;
- C6: CR, V5050, position 200 m on TR;
- C7: CR, V5070, position 100 m on TR;
- C8: CR, V5070, position 200 m on TR.

The indicated position is the distance from the information sign to the junction (crossing or roundabout). The manoeuvre distance, effectively used by the driver to decelerate, is the sum of this “position” and the sight distance.

Facing these junction configurations, vehicles can have several usages. The most noticeable usages are presented in Table 2. The first usage hypothesis considers the driver’s sight distance, whose values considered here are 100 m and 300 m. The vehicle can reach a roundabout or a crossing, so it should lower its speed to 20 km/h (speed hypothesis at the entrance of a roundabout) or to 0 km/h (stop panel case), or to 50 km/h while initially travelling at 70 km/h.

In this table, the usages that do not require a deceleration are not specified. Turning situations are generally not taken into account either, since the set of geometry/usage combinations is already large for a single junction, and its aim is to illustrate the optimization method. Larger applications could justify the use of micro-traffic simulations [26].

Table 2. Definition of the usage cases.

Sight Distance	Vehicle Situation	C1	C2	C3	C4	C5	C6	C7	C8
300 m sight distance	Vehicle slowing from 50 km/h to 20 (roundabout)	C1.U1	C2.U1	C3.U1	C4.U1				
	Vehicle slowing from 70 km/h to 20 (roundabout)			C3.U2	C4.U2				
	Vehicle slowing from 50 km/h to 0 (crossing)					C5.U3	C6.U3		
	Vehicle slowing from 70 km/h to 50 (crossing)							C7.U4	C8.U4
100 m sight distance	Vehicle slowing from 50 km/h to 20 (roundabout)	C1.U5	C2.U5	C3.U5	C4.U5				
	Vehicle slowing from 70 km/h to 20 (roundabout)			C3.U6	C4.U6				
	Vehicle slowing from 50 km/h to 0 (crossing)					C5.U7	C6.U7		
	Vehicle slowing from 70 km/h to 50 (crossing)							C7.U8	C8.U8

4.4. Criterion Values Facing Road Usages

In this part, each usage combination of the junction is evaluated by the criterion χ_{EASM} . As previously indicated, this criterion has the unique aim of helping the road manager in the predetermination of geometrical configurations and usages of speed-sectioning points that can impede eco-driving.

Table 3 shows results for the 20 usage cases. Its columns are constituted by the configuration and usage label, the approaching and limited speeds V_{SPD} and V_{MSP} , the

distance of the pre-signalization sign d_{sign} from of the speed-sectioning point, the visibility distance of the driver on this sign d_{visi} , the manoeuvre distance d_{man} , and lastly, the criterion χ_{EASM} .

It appears that cases involving high V_{SPD} speed and low V_{MSP} speed are associated with high criterion values, ranging from 6.8 to 7.7 for the C3.U2, C4.U6, and C3.U6 usage cases, all of them with V_{SPD} speed of 70 km/h and V_{MSP} speed of 20 km/h.

Reductions in speed from 50 km/h to 0 km/h or 20 km/h are associated with intermediate criterion values varying from 3.1 to 4.3. The lowest values are associated with longer maneuver distances d_{man} , for otherwise common speed cases.

Reductions in speed from 70 km/h to 50 km/h lead to similar values as for the last group of speed cases, with values varying from 3.5 to 4.1.

As a first conclusion, the criterion χ_{EASM} is able to separate speed cases into several value classes. Nonetheless, the relevance of the criterion will be verified by the means of an energy-saving assessment in the next section.

In anticipation of a possible non-flat road project and to illustrate the criterion's relevance in the face of slope variations, Figure 5 presents the criterion value evolution for slopes varying from -4% to 4% with 2% increments for the entire set of configuration/usage cases.

As a result, increasing the slope from -4% to 4% always decrease the criterion value for each slope increment of 2% . However, compared to the flat case (yellow curve), the variation of the slope does not have a homothetic effect on each usage case. For example, for the C4.U2 case, the criterion value is increasing in a larger proportion with negative slopes than for the C7.U8 case. Indeed, for this example, C4.U2 consists of deceleration from 70 km/h to 20 km/h over a maneuver distance of 500 m, and this large deceleration requirement, even over a long distance, remains difficult to achieve on a -4% slope without braking. The C7.U8 consists of a smaller deceleration from 50 km/h to 20 km/h over a short maneuver distance of 200 m, so it appears to be less sensitive to the slope intensity.

Table 3. Criterion values for each usage case and for the first hypothesis of a zero slope.

Usage	Input Case Parameters					Simple Criterion
	V_{SPD} (km/h)	V_{MSP} (km/h)	d_{sign} (m)	d_{visi} (m)	d_{man} (m)	χ_{EASM}
C1.U1	50	20	100	300	400	3.2
C1.U5	50	20	100	100	200	3.6
C2.U1	50	20	200	300	500	3.1
C2.U5	50	20	200	100	300	3.3
C3.U1	50	20	100	300	400	3.2
C3.U5	50	20	100	100	200	3.6
C3.U2	70	20	100	300	400	6.8
C3.U6	70	20	100	100	200	7.7
C4.U1	50	20	200	300	500	3.1
C4.U5	50	20	200	100	300	3.3
C4.U2	70	20	200	300	500	6.6
C4.U6	70	20	200	100	300	7.1
C5.U3	50	0	100	300	400	3.8
C5.U7	50	0	100	100	200	4.3
C6.U3	50	0	200	300	500	3.6
C6.U7	50	0	200	100	300	4.0
C7.U4	70	50	100	300	400	3.6
C7.U8	70	50	100	100	200	4.1
C8.U4	70	50	200	300	500	3.5
C8.U8	70	50	200	100	300	3.8

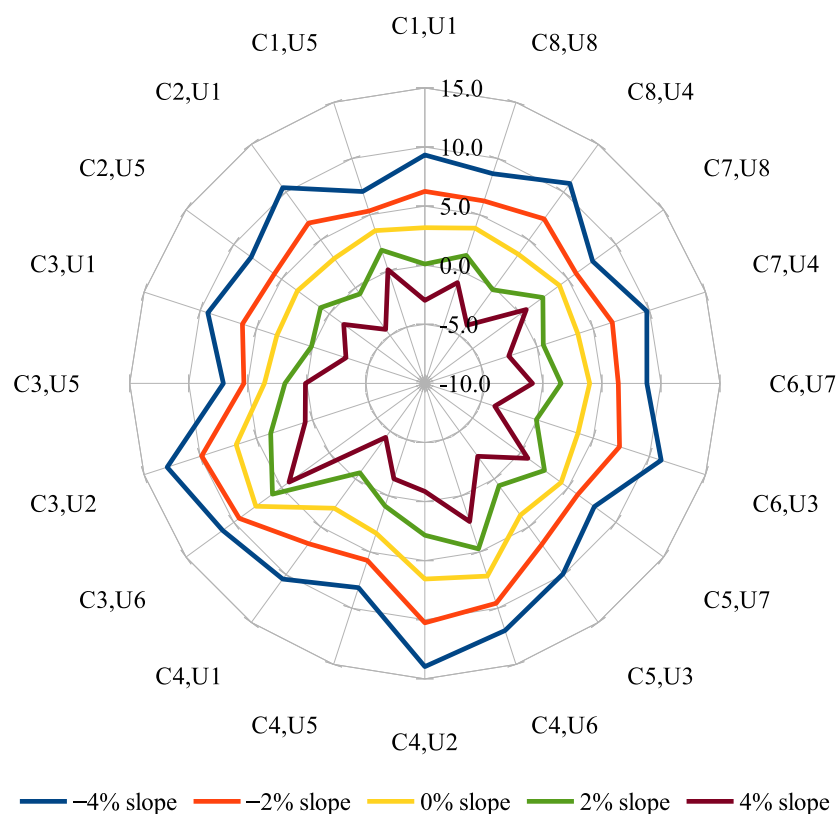


Figure 5. Criterion values for each usage case and for five slope hypotheses from -4% to 4% with 2% increments.

5. Results: Energy Savings Assessment for Several Slopes Projects

5.1. Modeling Results for a Road Set without Slopes

In this section, the 20 configurations and usage combinations from Table 2 are considered in the project's first hypothesis of horizontal road segments, with a zero slope.

For each usage case, Table 4 lists at first the input parameters such as speeds and maneuver distance.

Then, below the "initial scenario" label, computed values are given for the time t_f taken by the vehicle to travel the distance d_{man} , the $v(t_f)$, and the E_{MSP} energy value. These quantities are defined in Equations (10) and (11).

Labels "NC" in the table indicate that, while decelerating, the vehicle has reached a 0 km/h speed before reaching the speed sign. Such situations are relative to large maneuver distances and/or small required reduction of speed. In these cases, t_f , $v(t_f)$ and E_{MSP} are all linked and not calculable since the speed-sectioning sign cannot be reached.

If the vehicle reaches the speed sign, its $v(t_f)$ speed can be either lower, equal or greater than the required V_{MSP} speed, with a respective value of C_{MSP} being negative, zero, or positive. Then, the C_{MSP} is a difference in mechanical energy between the required speed and the actual speed corresponding to the input case parameters (speeds and distances).

Finally, in the "optimization and gain" column of the Table 4, the computed optimal decelerating duration and distance and given, as well as the corresponding energy gain expressed in terms of fuel liters and potential CO_2 emission equivalent, C_{MSP} . These values correspond to the optimal situation in terms of eco-driving, in which the vehicle decelerates over the maneuvering distance $d_{man, opti}$ and reaches the speed-sectioning point at the exact required V_{MSP} speed.

The corresponding gain, in liters per day, is based on the fuel consumption hypothesis of a light vehicle, ranging from 4.4 to 4.5 L/100 km , while covering the $d_{man, opti}$ and d_{man} distance difference, and multiplied by a conventional vehicle traffic of 4000 vehicles per

day. Then, the associated emissions C_{MSP} are taken as a linear proportion of the energy gain (fixed emissions factor of 3.17).

Table 4. Eco-driving assessment results for the whole set of configuration/usage: 0 % longitudinal slope case.

Usage	Input Case Parameters					Initial Scenario			Optimization and Gain			
	V_{SPD} (km/h)	V_{MSP} (km/h)	d_{sign} (m)	d_{visi} (m)	d_{man} (m)	t_f (s)	$v(t_f)$ (km/h)	E_{MSP} (kJ)	$t_{man, opti}$ (s)	$d_{man, opti}$ (m)	Gain (L/d)	C_{MSP} (kg)
C1,U1	50	20	100	300	400	NC	NC	NC	18.1	174.4	−39.7	−125.9
C1,U5	50	20	100	100	200	24.1	10.5	−13.4	18.1	174.4	−4.5	−14.3
C2,U1	50	20	200	300	500	NC	NC	NC	18.1	174.4	−57.3	−181.7
C2,U5	50	20	200	100	300	NC	NC	NC	18.1	174.4	−22.1	−70.1
C3,U1	50	20	100	300	400	NC	NC	NC	18.1	174.4	−39.7	−125.9
C3,U5	50	20	100	100	200	24.1	10.5	−13.4	18.1	174.4	−4.5	−14.3
C3,U2	70	20	100	300	400	NC	NC	NC	28.9	353.6	−8.3	−26.5
C3,U6	70	20	100	100	200	12.3	47.3	85.1	28.9	353.6	27.7	87.7
C4,U1	50	20	200	300	500	NC	NC	NC	18.1	174.4	−57.3	−181.7
C4,U5	50	20	200	100	300	NC	NC	NC	18.1	174.4	−22.1	−70.1
C4,U2	70	20	200	300	500	NC	NC	NC	28.9	353.6	−26.3	−83.5
C4,U6	70	20	200	100	300	21.4	32.0	28.8	28.9	353.6	9.7	30.6
C5,U3	50	0	100	300	400	NC	NC	NC	30.9	209.8	−33.5	−106.1
C5,U7	50	0	100	100	200	24.1	10.5	5.1	30.9	209.8	1.7	5.5
C6,U3	50	0	200	300	500	NC	NC	NC	30.9	209.8	−51.1	−161.9
C6,U7	50	0	200	100	300	NC	NC	NC	30.9	209.8	−15.9	−50.3
C7,U4	70	50	100	300	400	NC	NC	NC	10.8	179.2	−39.7	−126.0
C7,U8	70	50	100	100	200	12.3	47.3	−12.1	10.8	179.2	−3.7	−11.8
C8,U4	70	50	200	300	500	NC	NC	NC	10.8	179.2	−57.7	−183.0
C8,U8	70	50	200	100	300	21.4	32.0	−68.4	10.8	179.2	−21.7	−68.9

As a result, the configurations C3.U6, C4.U6, and C5.U7 are the 3 usage cases for which the initial speed-sectioning impedes eco-driving and they lead to respective energy gains of 27.7, 9.7, and 1.7 L per day.

In terms of speeds, the vehicle reaches the speed-sectioning point at 47.3 km/h instead of 20 for C3.U6, at 32 km/h instead of 20 for C4.U6, and at 10.5 km/h instead of 0 for C5.U7.

For these three cases, the $d_{man, opti}$ is greater than the initial d_{man} distance, and their difference correspond to the length that has to be added to the maneuver distance in order to allow the eco-driving by respecting the speed regulation without braking.

Other usage cases show negative E_{MSP} values, indicating that the initial d_{man} distance is more than sufficient to reach the V_{MSP} speed at the speed-sectioning point without braking. For example, the negative values of E_{MSP} with the higher absolute values, such as for the C8.U8 usage case, have the more excessive d_{man} values. Some usage cases have no E_{MSP} values since the vehicle stopped before reaching the MSP point, so the d_{man} obviously exceeds $d_{man, opti}$ too.

The fact that only three usage cases lead to potential energy savings is linked to the chosen ratio between vehicle mass, which is 1200 kg for the chosen light vehicle, and the engine brake intensity, which is chosen at 400 N based on experimental measurements. It would be different if the energy evaluation were applied for heavy vehicles or with proportionally smaller engine brake values.

As an illustration, Figure 6 represents the three cases of effective potential energy gains in blue bars, and the cases with no potential energy gains in red bars. This figure shows the impact of the maneuver distance d_{man} and the impact of the difference in speed from V_{SPD} to V_{MSP} , with the arbitrary form of the square root of the difference of squared speeds. As expected, low maneuver distances and large differences in speed are more prone to lead to possible optimization energy gains.

In the next section, an investigation will be done on the effect of ordinary longitudinal slopes to evaluate this additional effect on speed-sectioning optimization.

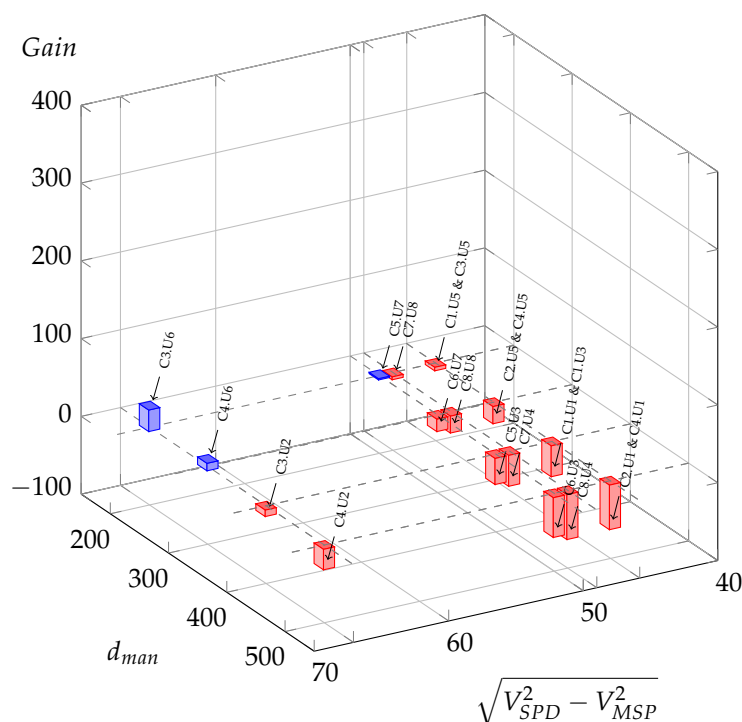


Figure 6. Possible oil gains (L per day) for all meaningful usage cases in function of manoeuvre distance and the difference of squared speed at 0%.

5.2. Modeling Results for Roads with Various Longitudinal Slopes

As previously indicated for the criterion, in the anticipation of a possible non-flat road project, and to discuss the methodology's application over several slope cases, the potential energy gains and associated parameters are evaluated for several representatives slopes for ordinary road conditions, ranging from -4% to 4% with 2% increments, and again for the entire set of configuration/usage cases.

The synthesis of initial situations and optimization results is given in Appendix A in Tables A1, A2, A3, and A4, respectively for the -4% , -2% , 2% , and 4% slope hypothesis.

Table A1 corresponds to a slope case of -4% , which is a relatively steep descent, but neither rare nor exceptional. For that slope scenario, the vehicle never stops before the speed-sectioning point and, moreover, it never reaches it at a lower speed than the mandatory speed of V_{MSP} . Therefore, the $d_{man, opti}$ optimal distance is always longer than the initial d_{man} distance. The case that requires the lighter optimization is the C8.U4 usage case, which involves deceleration from 70 km/h to 50 km/h over a 10.5 m maneuver distance. The optimal maneuver distance is 754.6 m , and without optimization the initial configuration leads to a 56.5 km/h speed instead of 50 km/h at the end of the maneuver. This case has a significant energy-saving potential of $45.8\text{ liter per day}$.

In contrast, the C3.U6 case requires the stronger optimization: the initial maneuver involves deceleration from 70 km/h to 20 km/h over a short distance of 200 m , leading to a speed of 64.5 km/h at the speed-sectioning point and a potential energy savings of 308.4 L/d , by enlarging the deceleration segment up to 1913.6 m . This case illustrates the joint importance of the slope, speed and speed-sectioning layout.

In Table A2, for a very ordinary slope intensity of -2% , results are more divided, with eleven usage cases leading to effective energy saving potential ranging from 0.4 to 70.5 L/d , and nine usage cases that do not need optimization and lead to “virtual” gain values of -1.9 to -37.9 L/d . Six of these cases even correspond to vehicles stopping with reaching the end of the d_{man} distance.

In Table A3, for an ordinary ascent of 2% , eighteen cases are leading to the stopping of the vehicle within the maneuver distance. The two other cases, C3.U6 and C7.U8, both correspond to a deceleration from 70 km/h and a d_{man} length of 200 m , but with mandatory

speeds V_{MSP} of 20 and 50 km/h, respectively, whereas the common computed v speed is of 35.8 km/h. Therefore, in the first case, the energy-saving potential is 9.4 L/d, and there is a potential gain of -12.6 L/d in the second case, indicating no effective potential, and no need to optimize the initial situation.

In Table A4, for a relatively important ascent of 4%, eighteen cases again lead to the stopping of the vehicle within the maneuver distance, and the two remaining cases lead to negative energy potential gains of -0.7 and -17.7 “virtual” liters per day, meaning that the uphill allows eco-driving deceleration in any situation, with no overspeeding at the speed-sectioning point.

Figures 7–10 are a graphical representation of the Tables A1–A4 results.

Compared to the zero slope case (Figure 6), for which three situations led to potential energy savings, it is clear that all situations could benefit from optimization for the -4% slope hypothesis (blue bars), and around half of them for the -2% slope hypothesis, with a majority of them for high Euclidean speed differences and larger potential gains for the shorter maneuver distance, d_{man} .

When considering a positive slope, that is to say uphill, the 2% slope case shows only one usage case with a potential energy saving (C3.U6), and for the 4% slope case all usage cases lead to negative computed gain, meaning that the initial maneuver distance is always sufficient to allow eco-driving.

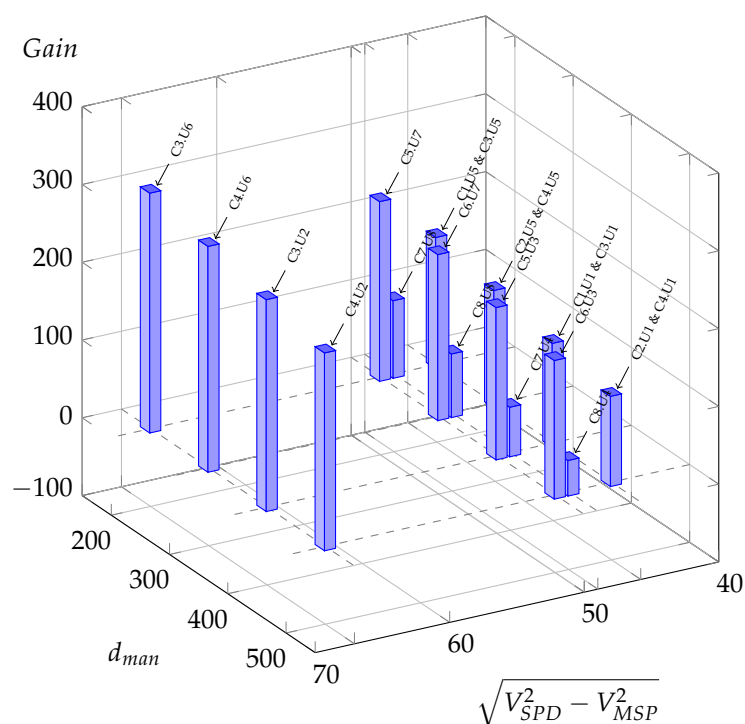


Figure 7. Possible oil gains (L per day) for all meaningful usage cases in function of manoeuvre distance and the difference of squared speed at -4% .

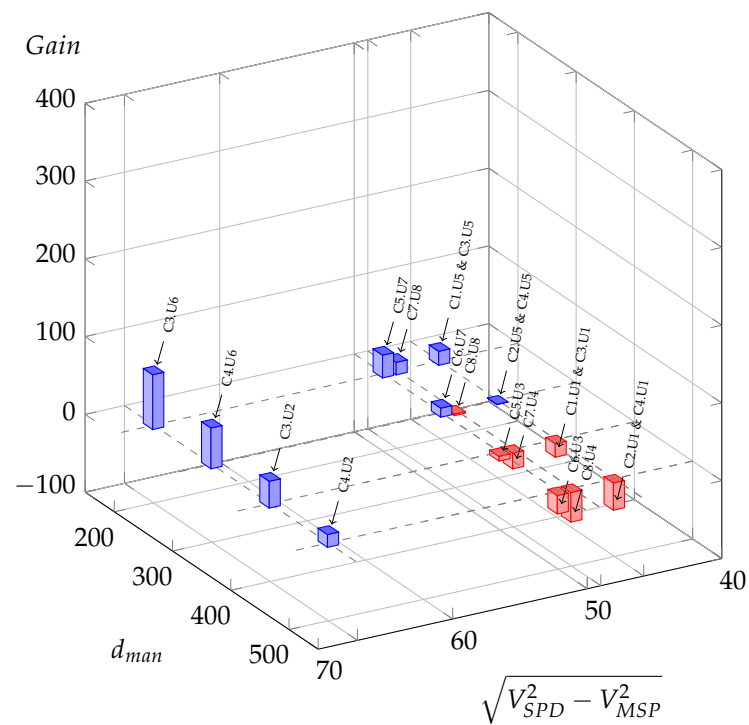


Figure 8. Possible oil gains (L per day) for all meaningful usage cases in function of manoeuvre distance and the difference of squared speed at -2% .

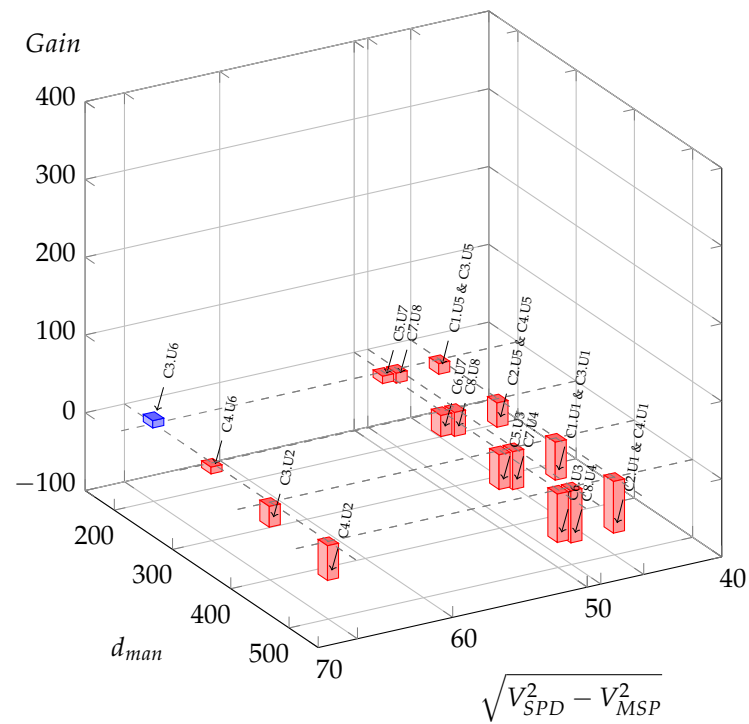


Figure 9. Possible oil gains (L per day) for all meaningful usage cases in function of manoeuvre distance and the difference of squared speed at 2% .

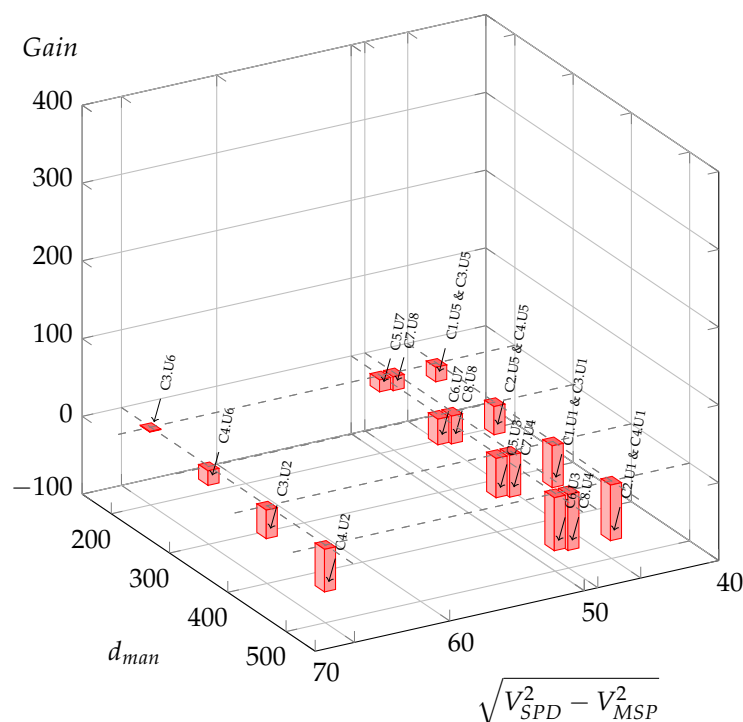


Figure 10. Possible oil gains (L per day) for all meaningful usage cases in function of manoeuvre distance and the difference of squared speed at 4%.

6. Discussion

In an attempt to verify whether speed-sectioning of the road could impede eco-driving, several tools are proposed in this work for road managers. Firstly, a criterion based on road's geometrical configuration and speed policy is proposed as a simple means of eco-driving impeding detection. Then, evaluation of the energy waste linked to the initial configuration is introduced by the E_{MSP} expression. Lastly, an optimization is computed, leading to the computation of an optimal maneuver distance, $d_{man, opti}$, with an associated gain value expressed in liters per day and proportional C_{MSP} CO₂ emissions.

The criterion and modeled energetic quantities have been applied to a specific speed-sectioning point in a road project, with variations in imposed speed deceleration and maneuver distance. Although the road project is expected to be on a global horizontal plane, the criterion and the modeled quantities have been evaluated for several longitudinal slope hypotheses, within ordinary values ranging from -4% to 4% .

The criterion alone is able to distinguish the usage case that are more susceptible to impede eco-driving, such as the C3.U6, C3.U2, C4.U2 and C4.U6 cases, which can be identified as external curvature discontinuities visible on the yellow curve of Figure 5 or as high values in Table 3. On the polar plot, the influence of road slope is easily apprehended too: the slope variation induces a rise of the criterion for each positive slope increment and for each usage case, up to values of 14 (C4.U2 case, blue curve of Figure 5); and a decrease of the criterion for each negative slope increment, even towards negative values such as -4.4 for C4.U1 and C2.U1 usage cases (brown curve of Figure 5).

By using the energetic model relying on Equations (8), (9) and (11) an energetic evaluation of the speed-sectioning is given to road managers, considering the variety of speeds, distances and longitudinal slopes. As a result Figures 6–8 and 10 allow distinguishing the usage cases needing a speed-sectioning optimization, in blue bars, from the situations that do not impede eco-driving, in red bars. These figures show the crucial impact of maneuver distances, requirements in speed reduction, and slopes.

Moreover, Tables A1–A4 provide optimal maneuver distances, $d_{man, opti}$, for which a driver, using only the engine brakes, without mechanical brakes, would need to reach the speed-sectioning point at the exact mandatory speed. For example, the three cases needing

optimization in the zero slope case (Table 4) lead to $d_{\text{man, opti}}$ distances of 353.6; 353.6; 209.6 m instead of the respective initial d_{man} distances of 200; 300; 200 m. So, for the first of these cases (C3.U6), a speed sign displacement of 153.6 m upstream could allow an energy saving evaluated at 27.7 L/d in that case with a reduction of 87.7 kg of CO₂ emissions.

If optimization exists in the zero slope case, it becomes very important and systematic for descent cases, as for the −4% slope case (Figure 7), C3.U6 usage case, with energy savings gains up to 308.4 L/d and an associated reduction of 977.8 kg of CO₂ emissions. Nevertheless, the corresponding speed-sign displacement should be of 1713.6 m, so an intermediate solution should be adopted, in a mix between mechanical brake requirements and maneuver distance extension.

Generally speaking, over an life cycle of an infrastructure, for the C3.U6 case and the zero slope hypothesis, more than 300,000 L of fuel could be saved over 30 years—that is nearly a tonne of CO₂—if we consider the traffic of 4000 vehicles per day, and simply by moving a speed sign of 153.6 m.

It is even more significant with an ordinary slope of −2%, with 70.5 L/d for the C3.U6 usage case, corresponding to a 391.5 m sign displacement and 771,975 L of potential fuel savings over a 30 year infrastructure life cycle period, or 2447 t of CO₂ emissions. This represents considerable savings for a relatively small and acceptable sign displacement.

In the eco-construction field, optimizing a road profile could lead to a 6% global energy gain over a 10 year period [6], taking into account both the building energy and the vehicle usage energy. Here the energy gain is either a given amount from the infrastructure point of view (771,975 L over 30 years in the preceding case), without associated building cost, or a fixed value for each vehicle. It is not possible to compare this percentage with vehicle mean consumption since it depends on the number of speed-sectioning points that can be optimized in the vehicle route.

The model allows finding the limits of acceptance for drivers, with up to 1713 m sign displacement needed to apply eco-driving rules in the C3.U6 case and −4% slope hypothesis. A trade-off should be found in such cases between acceptable sign displacement and mechanical brake requirements.

Finally, it should be reminded that optimizing speed relies on driver behavior. A driver who practices eco-driving will benefit from better speed-sectioning, but the advantage will be lower for a defensive driver (typically elderly drivers) or aggressive driver. For example, in [21], energy savings were estimated to reach 5.5% for eco-drivers but less than 1% for aggressive drivers. Therefore, our methodology, validated on slope variations, should be extended to various traffic conditions and driver behaviors by integration into a micro-traffic simulation.

7. Conclusions

This work deals with the infrequently studied influence of road speed variations on fuel consumption. Typically, speed policies are based on road safety and mobility efficiency criteria. This work aims to reduce vehicle fuel consumption by slight modifying a given road speed-sectioning and allowing eco-driving, without degrading road safety levels. The major eco-driving rule that is taken into account here is to avoid any mechanical braking during a decelerating maneuver, from the point of view of a speed sectioning point to its overpassing at the exact mandatory speed.

Firstly, a proposed criterion based on road geometrical configuration and road speed policy, is a simple means of detecting eco-driving impeding inconsistencies. It has the advantages of being vehicle-independent and quick to work out, allowing for a fast detection of road speed-sectioning inconsistency.

Then, the evaluation of the energy waste due to the initial configuration is introduced and computed. Lastly, an optimization is proposed, with an updated maneuver distance, mainly consisting of the displacement of a speed sign or the addition of an extra sign. The energetic gains associated with this optimization are expressed in liters per day and in proportional CO₂ emissions.

This criterion and the optimization process have been applied to a road project in the Nantes-Metropole area to evaluate different configurations of a city road circumvention. For this practical application, 20 configuration and usage cases have been built using speed and distance parameters.

In the initial road project hypothesis, with a zero longitudinal slope, an energy saving of 27.7 L/d could be achieved with a single speed-sectioning point optimization by a speed sign displacement of only 153.6 m.

While considering alternative project slopes, potential energy savings become very large and systematic for descent cases as for a -4% slope case with energy savings gains up to 308.4 L/d. Nevertheless, the corresponding speed-sign displacement should be more than 1700 m, so a compromise should be adopted, mixing between mechanical brake requirement and maneuver distance extension.

An intermediate scenario with an ordinary slope of -2% leads to consistent energy savings of 70.5 L/d by a sign displacement of only 391.5 m, resulting in 771,975 L of potential fuel savings over a 30 year infrastructure life cycle period. It appears to be considerable for an acceptable sign displacement.

As a global comparison, the eco-conception of a road could lead to a global energy gain of 6%, by limiting the resulting longitudinal slopes, over a 10 year period. Here, the gain is localized to a given position, so the impact on mean vehicle consumption should require a micro-traffic simulation to extend the local gain to several gains over a route.

A study limitation is linked to the driver behavior: a driver prone to eco-driving will benefit from a better speed-sectioning, but the advantage will be lower for a defensive driver (typically elderly drivers) or aggressive driver. In perspective, our methodology, validated on slope variations, should be extended to various traffic conditions and driver behavior by integration into a micro-traffic simulation, such as the DLR-SUMO simulator. Another limitation is our hypothesis of decelerating without braking: in a constrained traffic, braking could be required to avoid vehicle interactions, and the energy savings would be lower. Again a traffic simulation could be used to take into account various traffic compositions.

So far, this research proposes a simple criterion, an energetic evaluation, and an optimization process to help road managers adapt a given road speed-sectioning to reduce the vehicles consumption while considering longitudinal slopes and sight distances.

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Data Availability Statement: Data are available upon reasonable request to the authors.

Conflicts of Interest: The authors declare no conflict of interest.

Appendix A. Eco-Driving Assessment: Results for Various Longitudinal Slopes

Table A1. Eco-driving assessment results: -4% longitudinal slope case.

Usage	Input Case Parameters					Initial Scenario			Optimization and Gain			
	VSPD (km/h)	VMSP (km/h)	d _{sign} (m)	d _{visi} (m)	d _{man} (m)	t _f (s)	v(t _f) (km/h)	E _{MSP} (kJ)	t _{man, opti} (s)	d _{man, opti} (m)	Gain (L/d)	C _{MSP} (kg)
C1,U1	50	20	100	300	400	32.1	40.1	55.8	125.4	1158.9	133.6	423.4
C1,U5	50	20	100	100	200	15.2	45.0	75.3	125.4	1158.9	168.8	535.0
C2,U1	50	20	200	300	500	41.4	37.6	46.9	125.4	1158.9	116.0	367.6
C2,U5	50	20	200	100	300	23.4	42.5	65.3	125.4	1158.9	151.2	479.2

Table A1. Cont.

Usage	Input Case Parameters					Initial Scenario			Optimization and Gain			
	V_{SPD} (km/h)	V_{MSP} (km/h)	d_{sign} (m)	d_{visi} (m)	d_{man} (m)	t_f (s)	$v(t_f)$ (km/h)	E_{MSP} (kJ)	$t_{man,opti}$ (s)	$d_{man,opti}$ (m)	Gain (L/d)	C_{MSP} (kg)
C3,U1	50	20	100	300	400	32.1	40.1	55.8	125.4	1158.9	133.6	423.4
C3,U5	50	20	100	100	200	15.2	45.0	75.3	125.4	1158.9	168.8	535.0
C3,U2	70	20	100	300	400	22.4	59.1	143.2	171.3	1913.6	272.4	863.6
C3,U6	70	20	100	100	200	10.7	64.5	173.8	171.3	1913.6	308.4	977.8
C4,U1	50	20	200	300	500	41.4	37.6	46.9	125.4	1158.9	116.0	367.6
C4,U5	50	20	200	100	300	23.4	42.5	65.3	125.4	1158.9	151.2	479.2
C4,U2	70	20	200	300	500	28.6	56.5	129.2	171.3	1913.6	254.4	806.6
C4,U6	70	20	200	100	300	16.4	61.8	158.1	171.3	1913.6	290.4	920.7
C5,U3	50	0	100	300	400	32.1	40.1	74.3	257.6	1512.8	195.9	620.9
C5,U7	50	0	100	100	200	15.2	45.0	93.8	257.6	1512.8	231.1	732.5
C6,U3	50	0	200	300	500	41.4	37.6	65.5	257.6	1512.8	178.3	565.1
C6,U7	50	0	200	100	300	23.4	42.5	83.8	257.6	1512.8	213.5	676.7
C7,U4	70	50	100	300	400	22.4	59.1	46.0	45.9	754.6	63.8	202.4
C7,U8	70	50	100	100	200	10.7	64.5	76.6	45.9	754.6	99.8	316.5
C8,U4	70	50	200	300	500	28.6	56.5	32.0	45.9	754.6	45.8	145.3
C8,U8	70	50	200	100	300	16.4	61.8	60.8	45.9	754.6	81.8	259.4

Table A2. Eco-driving assessment results: −2% longitudinal slope case.

Usage	Input Case Parameters					Initial Scenario			Optimization and Gain			
	V_{SPD} (km/h)	V_{MSP} (km/h)	d_{sign} (m)	d_{visi} (m)	d_{man} (m)	t_f (s)	$v(t_f)$ (km/h)	E_{MSP} (kJ)	$t_{man,opti}$ (s)	$d_{man,opti}$ (m)	Gain (L/d)	C_{MSP} (kg)
C1,U1	50	20	100	300	400	NC	NC	NC	31.5	302.2	−17.2	−54.6
C1,U5	50	20	100	100	200	17.5	32.7	30.9	31.5	302.2	18.0	57.0
C2,U1	50	20	200	300	500	NC	NC	NC	31.5	302.2	−34.8	−110.3
C2,U5	50	20	200	100	300	31.1	20.3	0.7	31.5	302.2	0.4	1.2
C3,U1	50	20	100	300	400	NC	NC	NC	31.5	302.2	−17.2	−54.6
C3,U5	50	20	100	100	200	17.5	32.7	30.9	31.5	302.2	18.0	57.0
C3,U2	70	20	100	300	400	26.2	41.1	59.5	48.9	591.5	34.5	109.3
C3,U6	70	20	100	100	200	11.4	56.5	129.5	48.9	591.5	70.5	223.4
C4,U1	50	20	200	300	500	NC	NC	NC	31.5	302.2	−34.8	−110.3
C4,U5	50	20	200	100	300	31.1	20.3	0.7	31.5	302.2	0.4	1.2
C4,U2	70	20	200	300	500	36.1	31.6	27.6	48.9	591.5	16.5	52.2
C4,U6	70	20	200	100	300	18.2	49.2	93.5	48.9	591.5	52.5	166.3
C5,U3	50	0	100	300	400	NC	NC	NC	54.8	366.6	−5.9	−18.7
C5,U7	50	0	100	100	200	17.5	32.7	49.4	54.8	366.6	29.3	92.9
C6,U3	50	0	200	300	500	NC	NC	NC	54.8	366.6	−23.5	−74.4
C6,U7	50	0	200	100	300	31.1	20.3	19.2	54.8	366.6	11.7	37.1
C7,U4	70	50	100	300	400	26.2	41.1	−37.7	17.4	289.3	−19.9	−63.2
C7,U8	70	50	100	100	200	11.4	56.5	32.3	17.4	289.3	16.1	50.9
C8,U4	70	50	200	300	500	36.1	31.6	−69.6	17.4	289.3	−37.9	−120.2
C8,U8	70	50	200	100	300	18.2	49.2	−3.8	17.4	289.3	−1.9	−6.1

Table A3. Eco-driving assessment results: 2% longitudinal slope case.

Usage	Input Case Parameters					Initial Scenario			Optimization and Gain			
	V _{SPD} (km/h)	V _{MSP} (km/h)	d _{sign} (m)	d _{visi} (m)	d _{man} (m)	t _f (s)	v(t _f) (km/h)	E _{MSP} (kJ)	t _{man,opti} (s)	d _{man,opti} (m)	Gain (L/d)	C _{MSP} (kg)
C1,U1	50	20	100	300	400	NC	NC	NC	12.7	122.6	−48.8	−154.8
C1,U5	50	20	100	100	200	NC	NC	NC	12.7	122.6	−13.6	−43.2
C2,U1	50	20	200	300	500	NC	NC	NC	12.7	122.6	−66.4	−210.6
C2,U5	50	20	200	100	300	NC	NC	NC	12.7	122.6	−31.2	−99.0
C3,U1	50	20	100	300	400	NC	NC	NC	12.7	122.6	−48.8	−154.8
C3,U5	50	20	100	100	200	NC	NC	NC	12.7	122.6	−13.6	−43.2
C3,U2	70	20	100	300	400	NC	NC	NC	20.5	252.5	−26.6	−84.2
C3,U6	70	20	100	100	200	13.7	35.8	40.7	20.5	252.5	9.4	29.9
C4,U1	50	20	200	300	500	NC	NC	NC	12.7	122.6	−66.4	−210.6
C4,U5	50	20	200	100	300	NC	NC	NC	12.7	122.6	−31.2	−99.0
C4,U2	70	20	200	300	500	NC	NC	NC	20.5	252.5	−44.6	−141.2
C4,U6	70	20	200	100	300	NC	NC	NC	20.5	252.5	−8.6	−27.1
C5,U3	50	0	100	300	400	NC	NC	NC	21.5	147.0	−44.5	−141.2
C5,U7	50	0	100	100	200	NC	NC	NC	21.5	147.0	−9.3	−29.6
C6,U3	50	0	200	300	500	NC	NC	NC	21.5	147.0	−62.1	−196.9
C6,U7	50	0	200	100	300	NC	NC	NC	21.5	147.0	−26.9	−85.4
C7,U4	70	50	100	300	400	NC	NC	NC	7.8	129.9	−48.6	−154.1
C7,U8	70	50	100	100	200	13.7	35.8	−56.5	7.8	129.9	−12.6	−40.0
C8,U4	70	50	200	300	500	NC	NC	NC	7.8	129.9	−66.6	−211.2
C8,U8	70	50	200	100	300	NC	NC	NC	7.8	129.9	−30.6	−97.1

Table A4. Eco-driving assessment results: 4% longitudinal slope case.

Usage	Input Case Parameters					Initial Scenario			Optimization and Gain			
	V _{SPD} (km/h)	V _{MSP} (km/h)	d _{sign} (m)	d _{visi} (m)	d _{man} (m)	t _f (s)	v(t _f) (km/h)	E _{MSP} (kJ)	t _{man,opti} (s)	d _{man,opti} (m)	Gain (L/d)	C _{MSP} (kg)
C1,U1	50	20	100	300	400	NC	NC	NC	9.8	94.5	−53.8	−170.4
C1,U5	50	20	100	100	200	NC	NC	NC	9.8	94.5	−18.6	−58.8
C2,U1	50	20	200	300	500	NC	NC	NC	9.8	94.5	−71.4	−226.2
C2,U5	50	20	200	100	300	NC	NC	NC	9.8	94.5	−36.2	−114.6
C3,U1	50	20	100	300	400	NC	NC	NC	9.8	94.5	−53.8	−170.4
C3,U5	50	20	100	100	200	NC	NC	NC	9.8	94.5	−18.6	−58.8
C3,U2	70	20	100	300	400	NC	NC	NC	15.9	196.4	−36.7	−116.2
C3,U6	70	20	100	100	200	16.6	17.9	−3.6	15.9	196.4	−0.7	−2.1
C4,U1	50	20	200	300	500	NC	NC	NC	9.8	94.5	−71.4	−226.2
C4,U5	50	20	200	100	300	NC	NC	NC	9.8	94.5	−36.2	−114.6
C4,U2	70	20	200	300	500	NC	NC	NC	15.9	196.4	−54.7	−173.2
C4,U6	70	20	200	100	300	NC	NC	NC	15.9	196.4	−18.7	−59.1
C5,U3	50	0	100	300	400	NC	NC	NC	16.5	113.2	−50.5	−160.0
C5,U7	50	0	100	100	200	NC	NC	NC	16.5	113.2	−15.3	−48.4
C6,U3	50	0	200	300	500	NC	NC	NC	16.5	113.2	−68.1	−215.8
C6,U7	50	0	200	100	300	NC	NC	NC	16.5	113.2	−32.9	−104.2
C7,U4	70	50	100	300	400	NC	NC	NC	6.1	101.9	−53.7	−170.1
C7,U8	70	50	100	100	200	16.6	17.9	−100.8	6.1	101.9	−17.7	−56.0
C8,U4	70	50	200	300	500	NC	NC	NC	6.1	101.9	−71.7	−227.2
C8,U8	70	50	200	100	300	NC	NC	NC	6.1	101.9	−35.7	−113.1

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