

Article



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Abstract: Additive manufacturing (AM or "3D printing") has become a powerful tool for the rapid prototyping and manufacturing of complex part geometries. Especially interesting for the world of particle accelerators is the process of the 3D printing of stainless steel (and copper) parts. We present a first prototype of a 433 MHz IH-type linac cavity with an internal drift tube structure manufactured by metal 3D printing. The prototype cavity has been constructed to act as a proof-of-concept for the technology. In this paper we present the concept of the cavity as well as first results of vacuum testing and materials testing.

Keywords: additive manufacturing; particle accelerator; linac; selective laser melting; selective laser sintering; direct metal laser sintering; 3D printing; drift tube; IH-DTL; cavity



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# 1. Introduction

Additive manufacturing (AM) of metal parts may provide an interesting new way to manufacture accelerator components. As technology is evolving, the quality and accuracy of parts manufactured this way is ever improving. Recently, a number of studies on the topic of AM for linear accelerator components have been published [1-4]. The manufacturing of single cell 3.9 GHz Nb-cavities using Electron Beam Melting was investigated by Frigola et al. in 2015. The cavities were machined and processed by BCP after manufacturing and successful cold tests were performed up to 3 MV/m [1]. Recently, there have also been efforts to investigate the material and vacuum properties of stainless steel parts made by AM for accelerator applications. An investigation of Jenzer et al. from 2017 shows the first promising results for vacuum and outgassing properties of printed beam pipes [2]. In 2019, they expanded on these results including material strength tests (AM parts show a higher yield strength than conventional 316L steel) and further vacuum investigations [3]. A short summary of efforts was published in [4]. Based on these promising results, we aim to evaluate the suitability of AM parts for direct manufacturing of normal conducting linac structures. To that end, a reproduction of the beam pipe vacuum tests in [2,3] was performed and upon success, a prototype cavity with a fully printed drift tube structure was constructed. The cavity is designed to be UHV capable and includes cooling channels reaching into the stems of the drift tube structure. In the following, the concept and first experimental results are reported. This paper is an extended version of the conference proceeding in [5], providing updated vacuum measurements and material properties investigations of the printed parts.

# 2. Prototype Design and Concept

The prototype cavity was designed for a resonance frequency of 433.632 MHz, which is a harmonic of the GSI UNILAC operation frequency [6]. In combination with a targeted proton beam energy of 1.4 MeV this scenario allows for a compact accelerator at the limits of feasibility and is therefore a good benchmark for the new approach. The internal drift tube structure is fully 3D printed from stainless steel (1.4404), see Figure 1a. Due to the lower complexity of the cavity frame and lids, they are manufactured by CNC milling of bulk stainless steel.

The outer dimensions of the cavity are only  $221 \times 206 \times 261$  mm (without flanges). A center frame acts as the foundation for the cavity. This 7 cm high center frame provides the precision mount points for the girder-drift tube structures and end-drift tubes. While the end-drift tubes are mounted in vacuum, the girders have a vacuum sealing surface at the bottom. Two half shells are mounted on the top and bottom of the center frame. The cavity is equipped with four CF40-Flanges for vacuum, rf-coupler and tuner, as well as KF40 flanges for the beamline and smaller ports for diagnostics. Rf simulations show that the bulk of the rf losses during the operation of this cavity is concentrated on the drift tube structure and the cavity frame. Therefore, water channels are included in the girders up to the stems and also in the center frame. A 3D CAD view of the full construction is shown in Figure 1b.



**Figure 1.** Overview of the cavity geometry and printed parts. (**a**) 3D printed girder drift tube structure. (**b**) Cross section of the assembled cavity model.

### **RF** Simulations

The cavity design was optimized for a frequency of 433.632 MHz. To minimize the need for support structures during the manufacturing process, the shape of the girder-drift tube structure was optimized to reduce overhang. Simulations of electromagnetic fields in the cavity were performed with the CST Microwave Studio eigenmode solver. Figure 2 shows the resulting electric field distribution in the cavity, with the typical characteristics of a  $\beta\lambda/2$  IH-type structure. The simulated dissipated power for the full 2.4 MV acceleration is  $P_{loss} = 24.8$  kW, which corresponds to an effective shunt impedance of  $Z_{eff} = 41.32 \text{ M}\Omega/\text{m}$ . Future shape optimizations will provide opportunity for improved shunt impedance and reduced maximum fields along the structure.



Figure 2. Electric field distribution in the prototype cavity.

#### 3. Mechanical Accuracy

Mechanical accuracy of the printed girder-drift tube structure, especially the drift tube geometry, is critical for the field distribution in the cavity and therefore the beam dynamics. The bottom sides of the structures were milled after the printing process to provide a flat sealing surface and o-ring seals for the water channels. In the first step, however, the stems and drift tubes are not machined to evaluate the accuracy provided solely by the printing process.

While all the individual drift tube lengths and gap lengths are within an acceptable range ( $\Delta L < 100 \,\mu$ m, see Table 1), all deviations have the same sign and therefore add up. This may be fixed by printing with excess material and shortening the drift tubes after printing using conventional machining. Another possible mitigation would be to simulate the warping of the part during manufacturing and producing a compensating model. This will be studied further, as it would reduce the necessary machining needed after printing.

Girder 1		Girder 2	
L <sub>DT,1</sub> L <sub>gap,1-3</sub> L <sub>DT,3</sub> L <sub>gap,3-5</sub> L <sub>DT,5</sub>	+70 μm +50 μm +70 μm +30 μm +60 μm	L <sub>DT,2</sub> L <sub>gap,2-4</sub> L <sub>DT,4</sub>	+20 μm +40 μm +90 μm

Table 1. Deviation of Measured Dimensions from the Original Design Values.

# 4. Water Flow

Both girder-drift tube structures, as well as the center frame of the cavity are watercooled. While the watercooling in the center frame is quite conventional and realized with deep-hole drilled channels, the water cooling in the girders is more complex. To ensure, that the water channels in the girders are working as intended and not blocked by, for example, residual metal powder, some flow measurements were performed. The setup is provided with a water pressure of 8 bar. The tubing used is Festo tubing with an outer diameter of 6 mm. Measurements were performed with a Kobold digital inductive flow meter. Measurement results are summarized in Table 2.

Scenario	Water Flow
Source	10.8 L/min
6 mm tubing	7.2 L/min
Girder 1	4.8 L/min
Girder 2	5 L/min
Girder 1&2 parallel	7.6 L/min

Table 2. Waterflow Measurement.

As expected, the complex inner structure of the cooling channels leads to a reduction in flow. However, the measurements show that significant cooling can be expected for these structures. Future tests with thermal loads will help quantify the cooling capabilities.

#### 5. Preliminary Vacuum Tests

To evaluate the performance of 3D printed components in a UHV environment, preliminary tests were performed with 3D printed beam pipes with KF40 flanges (see Figure 3). These pipes were printed in 1.4404 stainless steel and the sealing surfaces on both ends were turned down on a lathe at the IAP workshop to provide a good vacuum seal. The inside and outside surfaces of the pipes were left as manufactured. A commercially available conventionally manufactured beam pipe of identical dimensions was acquired from Pfeiffer Vacuum. Both the printed and the conventional pipes were then connected to a small 18 L/s turbomolecular pump and repeatedly pumped and flushed with nitrogen gas.



Figure 3. KF 40 pipe printed from stainless steel.

Since these pipes were not cleaned and were used as manufactured, some initial outgassing was expected. The experiment showed some interesting results. For one, as expected with each pumping and flushing cycle, the pump down time to a certain pressure point was taking less time with each iteration. In the end, both pipes performed very similar and the pumping curve as well as the achieved final pressure were indistinguishable within the accuracy limitations of the experiment. Pressure data of the fourth pumping cycle are shown in Figure 4.

The final pressure achieved was about  $2 \times 10^{-7}$  mbar for both the conventional and the printed pipe. Similarly promising results were reported in, for example, [3,7]. These results certainly show promise for the UHV capability of printed parts. To discern more minute differences between the two samples, a test stand to accurately measure the outgassing rate via the rate-of-rise method is currently being constructed at IAP Frankfurt.



Figure 4. Pumpdown curve of the fourth vacuum cycle of the KF40 test pipes.

### 6. Printed Material Properties

To assess the material properties, one of the printed stainless steel pipes was sent to a materials testing lab. The manufactured material was confirmed to be within the compositional limits of 1.4404 stainless steel. Surface roughness was found to be in the order of  $R_z$  = 16.05–37.52 µm dependent on location and orientation of the measurement. Material porosity was determined to be 0.04% by optical analysis of material cross-sections (see e.g., Figure 5a,c). Larger cavities in the material were found in the overhang area of the flange, but nowhere within the bulk material. The melt-pool structure of the bulk material, which is caused by the manufacturing process, can be seen in the etched cross-sections in Figure 5b,d. Overall, the material seems well suited for vacuum applications as the degree of the porosity is not concerning. Depending on the application, further surface treatment will be necessary to reduce surface roughness.





Figure 5. Cont.





**Figure 5.** Cross sections of printed KF40 pipe. (**a**) material homogeneity of the pipe wall (untreated cut); (**b**) material structure of the pipe wall (etched cut); (**c**) material homogeneity of the flange overhang (untreated cut); (**d**) material structure of the flange overhang (etched cut) Courtesy of GSI Darmstadt, pLinac project.

### 7. Cavity Vacuum Test

The cavity was fully assembled in early May 2021 (see Figure 6). For first vacuum tests, the cavity was attached to a turbo-molecular pump (Pfeiffer HiPace80) via one of the top CF40 flanges. A vacuum gauge (Pfeiffer PKR261) was used to measure and log the cavity vacuum. The cavity lids, as well as the girder drift tube structures were sealed using 1.5 mm aluminum wire. Following the publication of [5], where a chamber pressure of  $2.97 \times 10^{-6}$  mbar was reached, the cavity was disassembled and reassembled a couple of times. An additional pump was attached via a long KF40 bellow pipe. Additional pressure gauges were attached to the cavity. Due to the addition of some new components, additional outgassing was observed during pumpdown. In the end, the result of the vacuum test was improved: At the time of writing, a pressure of  $1.19 \times 10^{-6}$  mbar was achieved after about 140 h of pumping. This latest pumpdown curve is shown in Figure 7.



Figure 6. Top view of the cavity during assembly with installed drift tube structure.

While this result is already promising and the end-pressure is not reached yet, it is believed that the results can be further improved. For reliable accelerator operation, a cavity pressure of  $<1 \times 10^{-7}$  mbar is needed. The main source for the remaining leak rates is suspected to be a minor defect in one of the conventionally milled lids of the cavity. Additional leak tests of the internal water channels in the girder drift tube structures were performed with a helium leak tester. They were found to be vacuum tight without any leakage through the bulk material of the printed structure. In the future, a second turbopump will be attached to the cavity. Additional steps will include the revision of

the cavity lid sealing surface and the addition of a o-ring sealed secondary vacuum (prevacuum) behind the metal wire seals. Furthermore, the cavity components may also be baked to reduce outgassing.



Figure 7. Pumpdown curve of the fully assembled cavity.

### Further Steps

In addition to the above mentioned planned improvements to the vacuum performance, the next steps will be the surface preparation and copper plating of printed components. Rf performance will be evaluated by low level rf measurements, comparing the printed parts in unpolished, polished and copper plated states. Cavity lids and center frames will also be copper plated. Finally, the structure will be tested at full power with a 30 kW pulsed rf amplifier.

## 8. Discussion

The first vacuum tests show promising results for this proof-of-concept cavity. A cavity pressure of  $1.19 \times 10^{-6}$  mbar was reached so far. Additional tests showed that the printed structures are not a source of measurable leak rates. Mechanical accuracy of the printed parts may be improved by predictive strain simulations or after the fact machining. This prototype is a first step, indicating that stainless steel 3D printed parts can indeed be used as the main components for accelerator structures. This opens up many possibilities for more complex designs and possibly previously impossible geometries for particle accelerators.

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