



Article Regression Rate and Combustion Efficiency of Composite Hybrid Rocket Grains Based on Modular Fuel Units

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Abstract: This study investigated combustion characteristics of composite fuel grains designed based on a modular fuel unit strategy. The modular fuel unit comprised a periodical helical structure with nine acrylonitrile-butadiene-styrene helical blades. A paraffin-based fuel was embedded between adjacent blades. Two modifications of the helical structure framework were researched. One mirrored the helical blades, and the other periodically extended the helical blades by perforation. A laboratory-scale hybrid rocket engine was used to investigate combustion characteristics of the fuel grains at an oxygen mass flux of $2.1-6.0 \text{ g/(s} \cdot \text{cm}^2)$. Compared with the composite fuel grain with periodically extended helical blades, the modified composite fuel grains exhibited higher regression rates and a faster rise of regression rates as the oxygen mass flux increased. At an oxygen mass flux of $6.0 \text{ g/(s \cdot cm^2)}$, the regression rate of the composite fuel grains with perforation and mirrored helical blades increased by 8.0% and 14.1%, respectively. The oxygen-to-fuel distribution of the composite fuel grain with mirrored helical blades was more concentrated, and its combustion efficiency was stable. Flame structure characteristics in the combustion chamber were visualized using a radiation imaging technique. A rapid increase in flame thickness of the composite fuel grains based on the modular unit was observed, which was consistent with their high regression rates. A simplified numerical simulation was carried out to elucidate the mechanism of the modified modular units on performance enhancement of the composite hybrid rocket grains.

Keywords: modular unit; modification; regression rate; combustion efficiency; hybrid rocket engine

1. Introduction

The hybrid rocket engine (HRE) is progressively emerging as a compelling propulsion system with significant potential for development and commercial viability [1-5]. In comparison with conventional liquid or solid rocket engines, the HRE offers the benefits of adjustable thrust, enhanced safety protocols, environmental friendliness, ease of repeated ignition, and low manufacturing cost [6–9]. However, the following issues with HRE are still unresolved: the different phases of the propellants and the diffuse combustion characteristics result in the low combustion efficiency of fuel grains [10-13]. Both low regression rates and oxygen-to-fuel ratio shift will affect engine performance, and proper regression rates can reduce oxygen-to-fuel ratio shift [14]. Therefore, research has focused on improving the regression rates. The use of complex single-port structures and multiport fuel grains have proven effective in increasing regression rates of fuel grains [15–20]. At a constant oxygen mass flow rate, there is an antagonism between the rise rate of regression and enhancement of flux enhancement, which often involves employing multiport structures to augment the burning surface. This carries certain risks, such as the collapse of adjacent ports caused by the increased volumetric load [12,21]. For this reason, it is more attractive to use a modified single port to improve the fuel grains and resulting



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Copyright: © 2024 by the authors. Licensee MDPI, Basel, Switzerland. This article is an open access article distributed under the terms and conditions of the Creative Commons Attribution (CC BY) license (https:// creativecommons.org/licenses/by/ 4.0/). turbulence to increase the regression rate [22]. Typically, the work of Kumar et al. [23,24] showed that the use of protrusion and bluff bodies in the single port fuel grain significantly improved the combustion efficiency and regression rate of the engine.

In recent years, three-dimensional (3D) printing technology has become sufficiently developed and successfully applied to the manufacturing of hybrid rocket fuel grains [25–32]. This technique allows for direct integral molding of polymeric fuel grains with a complex single port, which is difficult to accomplish using traditional manufacturing methods [25]. Representative work was published by Whitmore et al. [26,27], who printed a spiral singleport acrylonitrile-butadiene-styrene (ABS) fuel grain in one piece. Centrifugal swirl flow, induced by the helical port, promoted convective heat transfer and increased the regression rates. Another strategy for applying 3D printing to fuel grains has been to embed the printed skeleton reinforcement structure in paraffin-based fuels. Paraffin-based fuels have gained significant attention due to their high regression rates, despite their poor mechanical properties. Embedding 3D printed skeletons, such as the "armored grain" proposed by Bisin et al. [28,29], enhanced the mechanical properties of the paraffin-based grains at the expense of regression rate. In the work of our group, both mechanical properties as well as the regression rate of paraffin-based fuels with embedded ABS helical structure framework proposed by Wang et al. [30] were significantly improved. Varying regression rates of fuels generate groove structures that facilitate oxidant vortex flow to improve turbulence and combustion efficiency. Lin et al. [31,32] subsequently modified the helical structure framework by perforation and used metallic materials to strengthen the mechanical properties and combustion characteristics of the fuel grains. These studies validated the potential of embedded skeletal reinforcement.

Single-port stepped fuel grains with adjustable internal ballistics were recently investigated by Glaser et al. [33,34]. Their design is similar to the stacking of multiple fuel grains, each of which is a separate single-port paraffin-based grain. Flexible modification of multiple single-port paraffin-based grains, and controlling their inner diameter and length to obtain the desired ballistic design within the fuel grains, improves engine performance. However, the stacking may cause interface instability and requires a complicated manufacturing process. Owing to the above-mentioned issues, integrally molded skeleton reinforcement structures produced by 3D printing could be expected to simplify the manufacturing process of fuel grains and enable targeted improvements in HRE.

This study proposed a modular fuel design strategy that can modify the engine combustion. A periodic helical structure framework is the smallest unit of the modular unit. The modular fuel unit comprises a periodical helical structure framework that can be flexibly regulated and integrated by 3D printing. A paraffin-based fuel was embedded in the framework. Two modifications of the helical structure framework were investigated. One was mirroring the helical structure (MS grain); the other was the periodic extension of the helical structure with perforations (PS grain). A fuel grain with periodically extended helical structure (HS grain) and paraffin-based grain (PP grain) were tested for comparison. In the tests, the mass flow rate of oxygen was 9.5–28.1 g/s, corresponding to a mass flux of 2.1–6.0 g/(s·cm²). The combustion chamber pressure, oxygen-to-fuel ratio (O/F) distribution, regression rate, and combustion efficiency of the engines were studied. Radiation imaging was used to analyze the characteristic flame structure in the combustion chamber during the experiments. Simplified 3D simulations were carried out to reveal the mechanism of the modified modular units on performance enhancement of the HRE.

2. Materials and Methods

2.1. Modular Unit Framework

A nested framework is an important support part of the modular fuel unit. As shown in Figure 1, a periodic helical structure framework is the smallest unit of the modular framework that can be flexibly regulated and integrated by 3D printing. The modular unit framework was periodically extended from the helical structural of the smallest unit and consisted of nine 1.5-mm-thick blades with axial helical extension and a 2-mm-thick outer wall. The helical pitch of the blades was 100 mm with 0.5 turns. The length of the modular fuel unit was 50 mm, which was half the length of the manufactured fuel grain. Its outer and inner diameters were 60 mm and 20 mm, respectively. The blades and outer wall were integrally manufactured.



Figure 1. Structures of modular fuel unit, HS grain, PS grain, and MS grain.

Three composite fuel grains based on the modular unit were studied. The structure of the HS grain was a periodically extended modular unit framework with one periodic expansion. The PS grain improved the helical blades based on the HS grain. There were numerous pores spread over the helical blades of the PS grain, which enhanced adhesion of the paraffin-based fuel to the helical blades. The perforations were carried out along the path of a screw thread rotating in the opposite direction to that of the helical blades, with a pitch of 25 mm. The pores had a diameter of 1 mm and were spaced 2 mm apart. The MS grain design adjusted the helical direction of the helical blades. The structure of the MS grain was obtained by mirroring the helical blades along one end face of the modular unit framework. This particular mirror structure is used to explore the impact of the modular unit design strategy.

Manufacturing of the composite fuel grains comprised two main steps. The first was to print out the designed structure using a commercially available 3D printer (Raiser 3D, Pro2 Plus, Bakersfield, CA, USA). The second was embedding the paraffin-based fuel by means of centrifugal casting technology. Details of the fabrication of the composite columns can be found in previous studies [30].

2.2. Laboratory-Scale Hybrid Rocket Engine

A schematic of the laboratory-scale HRE is shown in Figure 2. The engine consisted of two combustion chambers at the head and tail of the engine, and a main combustion chamber and a laval nozzle with 5 mm throat diameter. The pre- and post-combustion chambers had lengths of 30 mm and 50 mm, respectively, and shared the same diameter of 50 mm. The dimensions of the main combustion chamber correspond to that of the fuel grains. Interfaces were provided in the combustion chambers to connect pressure sensors. Further details can be found in our previous studies [30,35]. The engine was ignited by a torch-type igniter activated by gaseous methane and oxygen. Oxygen from

the combustion chamber entered through the annular port of the coaxial injector at an average oxygen mass flow rate of 9.5 to 28.1 g/s. A mass flow controller (Bronkhorst, model F-203AV, Ruurslo, The Netherlands) was used to regulate the oxygen mass flow rate. The coaxial core section of the injector was a fiber optic viewport, which was connected via fiber optics to a high-speed camera (iX Cameras, i-Speed 220, London, UK) and acted as a carrier of radiation luminescence. The frame rate of the camera was 1 kHz and its exposure time was 0.28 ms. A narrowband filter was used to eliminate optical signal clutter. The narrowband filter had a center wavelength of 430 nm and a bandwidth of 10 nm. A digital delay generator (Stanford Research Systems, DG645, Sunnyvale, CA, USA) was connected between the high-speed camera and host computer to provide precise signal pulses and stable trigger times for the camera.



Figure 2. General schematic of laboratory-scale hybrid rocket engine.

3. Results and Discussion

Table 1 lists the recorded results of fifty experiments, pertaining to average oxygen mass flow rate, \dot{m}_o , average oxygen mass flux, G_{ox} , average chamber pressure, \overline{P} , regression rate, \dot{r} , and O/F. These values were calculated as weight averages.

Table 1. Test results of fuel grains.

| No. | Grain | \dot{m}_o (g/s) | G_{ox} g/(s·cm ²) | – P (MPa) | <i>r</i> (mm/s) | O/F |
|-----|----------|-------------------|---------------------------------|--------------|-----------------|------|
| 1 | | 10.06 | 2.34 | 1.02 | 0.76 | 1.95 |
| 2 | | 12.03 | 2.73 | 1.11 | 0.88 | 1.95 |
| 3 | HS grain | 13.76 | 3.11 | 1.28 | 0.88 | 2.22 |
| 4 | | 15.33 | 3.44 | 1.49 | 0.94 | 2.29 |
| 5 | | 18.37 | 3.98 | 1.74 | 1.1 | 2.22 |
| 6 | | 18.25 | 4.01 | 1.77 | 1.04 | 2.45 |
| 7 | | 18.50 | 4.01 | 1.83 | 1.11 | 2.29 |
| 8 | | 19.09 | 4.16 | 1.81 | 1.05 | 2.44 |
| 9 | | 20.71 | 4.47 | 1.97 | 1.12 | 2.51 |
| 10 | | 22.36 | 4.74 | 2.08 | 1.21 | 2.44 |
| 11 | | 24.11 | 5.08 | 2.23 | 1.26 | 2.50 |
| 12 | | 24.40 | 5.28 | 2.13 | 1.16 | 2.81 |
| 13 | | 26.27 | 5.55 | 2.39 | 1.27 | 2.73 |
| 14 | | 28.09 | 5.98 | 2.45 | 1.23 | 3.01 |

| No. | Grain | \dot{m}_o (g/s) | G_{ox} g/(s·cm ²) | – P (MPa) | <i>r</i> (mm/s) | O/F |
|-----|-----------|-------------------|---------------------------------|--------------|-----------------|------|
| 15 | PS grain | 9.61 | 2.18 | 0.95 | 0.84 | 1.65 |
| 16 | | 11.60 | 2.64 | 1.10 | 0.86 | 1.95 |
| 17 | | 12.11 | 2.70 | 1.14 | 0.94 | 1.81 |
| 18 | | 13.79 | 3.04 | 1.33 | 0.99 | 1.95 |
| 19 | | 16.75 | 3.64 | 1.59 | 1.07 | 2.14 |
| 20 | | 17.45 | 3.78 | 1.74 | 1.08 | 2.20 |
| 21 | | 18.47 | 3.95 | 1.82 | 1.15 | 2.16 |
| 22 | | 18.59 | 4.01 | 1.80 | 1.14 | 2.23 |
| 23 | | 20.87 | 4.44 | 1.97 | 1.23 | 2.26 |
| 24 | | 22.40 | 4.76 | 2.11 | 1.22 | 2.44 |
| 25 | | 23.73 | 4.93 | 2.27 | 1.34 | 2.27 |
| 26 | | 24.06 | 5.06 | 2.22 | 1.26 | 2.50 |
| 27 | | 24.66 | 5.21 | 2.40 | 1.28 | 2.57 |
| 28 | | 25.70 | 5.37 | 2.41 | 1.35 | 2.51 |
| 29 | | 9.49 | 2.24 | 0.92 | 0.71 | 1.98 |
| 30 | | 12.12 | 2.76 | 1.20 | 0.86 | 2.01 |
| 31 | | 13.34 | 2.96 | 1.36 | 0.96 | 1.93 |
| 32 | | 14.73 | 3.23 | 1.49 | 1.02 | 1.96 |
| 33 | MS grain | 16.96 | 3.74 | 1.66 | 1.04 | 2.21 |
| 34 | wis grain | 18.43 | 3.95 | 1.80 | 1.16 | 2.13 |
| 35 | | 21.18 | 4.53 | 2.03 | 1.19 | 2.34 |
| 36 | | 23.51 | 4.87 | 2.33 | 1.36 | 2.21 |
| 37 | | 25.69 | 5.42 | 2.48 | 1.32 | 2.43 |
| 38 | | 27.02 | 5.63 | 2.62 | 1.37 | 2.54 |
| 39 | | 9.77 | 2.32 | 0.89 | 0.68 | 2.15 |
| 40 | | 11.42 | 2.66 | 1.00 | 0.75 | 2.22 |
| 41 | | 13.90 | 3.19 | 1.17 | 0.79 | 2.52 |
| 42 | | 17.46 | 4.03 | 1.46 | 0.82 | 3.08 |
| 43 | PP grain | 18.74 | 4.27 | 1.56 | 0.88 | 3.03 |
| 44 | | 18.92 | 4.28 | 1.69 | 0.91 | 2.93 |
| 45 | | 19.60 | 4.38 | 1.66 | 0.88 | 3.13 |
| 46 | | 19.17 | 4.38 | 1.65 | 0.96 | 2.84 |
| 47 | | 20.58 | 4.54 | 1.78 | 1.00 | 2.84 |
| 48 | | 21.42 | 4.81 | 1.92 | 0.95 | 3.21 |
| 49 | | 24.64 | 5.44 | 2.00 | 1.02 | 3.34 |
| 50 | | 25.87 | 5.68 | 2.23 | 1.03 | 3.46 |

Table 1. Cont.

3.1. Combustion Chamber Pressure

The chamber pressure can reflect the operating status of the engine [36]. Figure 3 provides the histories of the chamber pressure and oxygen mass flow rate with time for the four fuel grain designs under three average oxygen mass fluxes, and the distribution of the average chamber pressures at different oxygen mass fluxes. The ignition time of the engine was defined as the zero point. Differences in ignition times between fuel grains are due to valve switching delays. The entire ignition process lasted approximately 6 s. As the pressure stabilized, the oxygen flow rate stabilized. The variation in chamber pressure was significantly higher than that of the pure paraffin-based grain, which is consistent with the previous conclusions of our group [31]. The pressure in the combustion chamber increased when composite fuel grains consisted of regulated modular units. The combustion chamber pressures of the MS and PS grains were higher than the chamber pressures of the HS grain.



Figure 3. Pressure and the oxygen mass flow rate as a function of time for the four grains for average oxygen mass flux of: (a) $3.1 \text{ g/(s \cdot cm^2)}$, (b) $4.0 \text{ g/(s \cdot cm^2)}$, (c) $5.5 \text{ g/(s \cdot cm^2)}$, and (d) plot of average pressure as a function of average oxygen mass flux.

3.2. Regression Rate and Combustion Efficiency

As an important parameter for HRE, regression rate is generally determined by measuring the mass of the grain before and after testing, using the following expression:

$$\dot{r} = \frac{d_f - d_0}{2t} = \frac{\sqrt{d_0^2 + \frac{4(m_0 - m_f)}{\pi\rho L} - d_0}}{2t} \tag{1}$$

where d_f , d_0 refer to the inner diameter of the fuel grains before and after combustion, respectively. m_f , m_0 refer to the mass of the fuel grains before and after combustion. ρ refers to the density of fuel, L refers to the length of fuel grains, and t refers to the actual time of engine operation. This value \dot{r} can be represented as a function of G_{OX} :

$$= aG_{OX}^n \tag{2}$$

where \dot{r} denotes the regression of composite grains, G_{ox} refers to the mass flux of the oxidizer, and *a* and *n* refer to the constants obtained by fitting.

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Figure 4 illustrates the distribution of regression rates and their fitting results. Modified composite grains exhibited higher regression rates compared to paraffin-based fuel. The different modifications of the helical structure also different degrees of increase in the regression rates. At the lowest oxygen mass flux, the regression rate of the PS grain was significantly higher than that of the HS grain, while those of the HS and MS grains were

similar. However, as the oxygen mass flux increased, the regression rate of the MS grain with the modified modular fuel unit significantly increased, and the rise rate was higher than that of the conventional HS grain. At an average oxygen mass flux of $6.0 \text{ g/(s} \cdot \text{cm}^2)$, the regression rate of the PS grain was consistently 8.0% and 30.7% higher than that of the HS and PP grains, respectively. Similarly, at corresponding positions, the MS grain was 14.1% and 38.1% higher than those of the HS and PP grains, respectively. The high regression rates of two modified composite grains may be due to turbulence enhancement resulting from the designs of the helical structure frameworks.



Figure 4. Variation in regression rates with oxygen mass flux for different fuel grains.

Characteristic velocity is another key parameter of HRE. Combustion efficiency is expressed by:

$$\eta = \left(\frac{C_e^*}{C_t^*}\right) \tag{3}$$

where C_t^* denotes the theoretical characteristic velocity of pure paraffin calculated by the Chemical Equilibrium with Applications software (CEA) [37]. C_e^* denotes the experimentally obtained characteristic velocity:

$$C_e^* = \frac{\overline{P}A}{\overline{m}_f + \overline{m}_g} \tag{4}$$

where \overline{P} denotes the average value of the chamber pressure, *A* denotes the throat area of the nozzle, which is considered to be constant due to small variations, and \overline{m}_{f} , \overline{m}_{g} denote the average mass flow rates of the oxidizer and fuel, respectively.

Figure 5 shows the characteristic velocity distributions of the four fuel grains and their O/F as a function of average oxygen mass flux. The combustion efficiency was significantly improved by the modular design and modifications of the helical structure. The combustion efficiencies of the modified fuels were more concentrated than the paraffin-based grain. The MS grain exhibited higher characteristic velocities and combustion efficiencies, and demonstrated smaller fluctuations of these values across a wide range of average oxygen mass fluxes, compared with the HS grain. The characteristic velocity of the PS grains did not exhibit a notable increase, possibly attributed to the significant increase in regression rate, which reduced the corresponding O/F value. In practical applications, over-dispersion of the O/F will increase the launching cost of the rocket [38]. Figure 5b shows that the O/F of the MS grain changed less compared with other fuel grains for the same average oxygen mass flux. Combined with the above analysis of regression rates, these results show that the increase in regression rate can inhibit changes in the O/F ratio.



Figure 5. (a) Distribution of characteristic velocities and the oxidizer-to-fuel ratio (O/F) for four grains and (b) average O/F as a function of average oxygen mass flux.

3.3. Radiation Luminescence

Figure 6 plots the radiation luminescence intensity curves of the different fuel grains and corresponding pressure changes. The average mass flow rate of the oxidant was 18.5 g/s. The radiative intensity used in this study was based on the average pixel value of each frame from the visual area of initial grayscale images, arranged in chronological order according to the capture time. Following ignition, the combustion chamber pressures and radiation intensities of all engines rose at the same time. The combustion chamber pressure reached a steady state first. The upward trend of radiation intensity did not weaken, which may be caused by flame propagation to the pre-combustion chamber, resulting in a larger flame area in the field of view, until the supply of oxidizer was stopped, at which point both the combustion chamber pressure and radiation intensity were reduced to zero. These experiments showed that the composite fuel grains with the modified modular fuel units had higher radiative intensity than the paraffin-based grain.

Figure 7 shows the four grains of Figure 6 before and after firing. Prior to combustion, the inner surfaces of all grains were smooth and flat. However, after combustion, a distinct difference can be observed. While the surface of the paraffin-based grain remains flat, the inner surfaces of the other composite grains exhibit prominent grooves. These helical grooves played a crucial role during the combustion process by continuously inducing oxidizer swirl. The presence of these grooves post-combustion highlights their significance in optimizing the performance of composite fuel grains.

Images capturing the flame within the chamber were obtained to examine alterations in the flame behavior. Values of t_1 - t_6 correspond to 0.5–5.5 s after engine ignition in Figure 6 with an interval of 1 s. Twenty-five images before and after the six times values were selected in turn, and the average gray values of the pixels were determined. Pseudo-color processing results after overall averaging are shown in Figure 8.

The average images of the various fuel grains at different reaction times exhibit similar characteristics. During the initial stage of the reaction (t_2, t_3) , the flame primarily accumulated in the center of the image. As the burning surface increased, flame diffusion and propagation were further enhanced until the flame penetrated the entire combustion chamber and occupied the entire sight window. At t_5 and t_6 , when the combustion chamber pressure had stabilized, the images indicate that the flames in the combustion chamber continued to develop. This observation is consistent with the analysis of radiation intensity. The PP grain exhibited a slower rate of flame development and smaller flame coverage area compared with the composite fuel grains. Additionally, the composite fuel grains produced an obvious filamentous flame structure that was caused by swirling during the combustion process. The average flame images indicate that the composite fuel grains

with modified helical structures facilitated flame development. The MS grain exhibited the fastest flame development and thickest flame due to turbulence enhancement. The periodically extended helical structure with perforations increased the swirling flow of the flame and the flame distribution of the PS grain was more concentrated.



Figure 6. Radiative intensity and pressure changes with time for the four grains: (**a**) MS grain, (**b**) PS grain, (**c**) HS grain, and (**d**) PP grain.



Figure 7. The inner surface of the fuel grains before and after combustion.



Figure 8. Radiant luminescence images as a function of time from ignition for different grains.

The pulsation characteristics of the flame in the combustion chamber were analyzed using proper orthogonal decomposition (POD) [39]. This process of POD is used to transform a sequence of image datasets into a set of linearly uncorrelated variables (modes) to classify the dominant modes of the flame and provide characterization of the flame dynamics and associated imaging time series. Flame images can be linearly superimposed by multiple orthogonal modalities, the formular for which is as follows:

$$\phi(\xi, t) - \overline{q}_0 = \sum_{i=1}^{M} \alpha_i(t) \phi_i(\xi)$$
(5)

where ξ is a spatial coordinate (in units of pixels and each image has a fixed number of pixels 200 × 200), *t* is the time sequence, \overline{q}_0 is the mean value of the images, α_i is the time coefficient of the temporal mode, ϕ_i is the spatial distribution characteristic of the mode, and *M* is the total number of modes.

The method of decomposition used in this paper is the singular value decomposition (SVD) [40,41]. The images can then be expressed as an m × n of data matrix $A_{m\times n}$, where n is the total number of images, the value of m is obtained by multiplying the pixels in each row of the image by the pixels in each column. The matrix can be decomposed into the product of three matrices via SVD:

$$A_{m \times n} = U_{m \times m} S_{m \times n} V_{n \times n}^T \tag{6}$$

where *U*, *S*, and *V* represent the orthogonal modes of the flame series and their corresponding mode energy and time coefficients, *T* is the transpose of a matrix.

$$\lambda = S^2 \tag{7}$$

where λ represents the eigenvalue.

$$E_i = \frac{\lambda_i}{\sum_i^M \lambda_j} \tag{8}$$

where *E* represents the normalized energy ratio of each mode. Summing the normalized energy ratio of each mode can obtain the cumulative energy distribution of all modes as shown in Figure 9a.





Figure 9. (a) Proper orthogonal decomposition cumulative energy distribution of modes and (b) first six modes of the images.

Figure 9a demonstrates the cumulative energy distributions of the modes. Energy accumulations of the MS and PS grains were greater than those of the HS and PP grains. Figure 9b shows the first six modes of the combustion chamber flame, with the red and blue colors indicating areas where the gas-phase reaction dramatically changed. They are listed in order of decreasing energy. The first modality of the three composite fuel grains is similar due to their common modular unit. The composite fuel grains based on modular units had significantly more layers and obvious swirling characteristics in the flame change region compared with the pure paraffin-based grain. A significant change in the number of flame layers indicates a change in intensity of the reaction; more layers indicate stronger flame oscillation characteristics and a more intense combustion [37]. These phenomena indicate that the helical structure of the modified modular units increased turbulent flow in the combustion chamber and improving regression rates of modified fuel grains.

4. Combustion Simulation

4.1. Model Description and Precision Estimates

An evaluation of the simplified flow field within the combustion chamber was conducted based on ANSYS Fluent software (ANSYS 2022 R1). This is a simplified steady-state combustion simulation designed to elucidate the phenomena observed in the experiment. The regression rates of fuel grains are not taken into account. This study performed combustion simulations on simplified versions of three different engine models by solving the Navier–Stokes equations, while also enabling energy transfer and species transport. The development of turbulence was taken into account by setting the turbulence model to SST k– ω . Ethylene(C₂H₄) was used instead of a paraffin-based fuel [17] and a 10-step C₂H₄ reaction mechanism was adopted [42], which is shown in Table 2. Oxygen was employed as the oxidizing agent. The computational fluid domains for the three fuel models are shown in Figure 10. The walls of the fuel grains and the engine in the fluid domain were adiabatic and non-slippery. The nozzle outlet was a pressurized outlet, and the wall surface of the fuel grains and oxygen inlet section were set as mass flow inlets. The oxidizer and fuel mass flow rates were set to 18.59 g/s and 8.33 g/s, respectively. The PS grain model was modified to simplify calculations. The porous model at the groove was reduced to half of the actual pore cross-section (with a radius of 0.5 mm), and the groove surface was modeled with a single layer of porosity. The inner diameter and groove depth of the grain model were set at 26 mm and 1 mm, respectively, aligning with the inner cavities of the engines.

Different grids with a total number of grids of 2 million, 3 million, and 4.5 million were used for the three composite fuel grains during simulation to eliminate the effect of grid size during the numerical simulation. As shown in Figure 11a, the combustion chamber pressures under different grids were very similar for the same grain, with a maximum

relative error of 8.6%. Experimental values of the chamber pressure are compared with the numerical simulation results, as presented in Figure 11b. Error estimates of the chamber pressure range from 6.21% to 8.33%. The final model utilized a mesh count of 3 million.

Table 2. Global reaction mechanism for ethylene.

| No. | Reaction | | |
|-----|---|--|--|
| 1 | $O_2 + C_2H_4 \leftrightarrow 2CO + 2H_2$ | | |
| 2 | $O + CO(+M) \leftrightarrow CO_2 (+M)$ | | |
| 3 | $OH + CO \leftrightarrow H + CO_2$ | | |
| 4 | $O_2 + H_2 \leftrightarrow OH + OH$ | | |
| 5 | $O_2 + H \leftrightarrow OH + O$ | | |
| 6 | $OH + H_2 \leftrightarrow H + H_2O$ | | |
| 7 | $O + H_2 + M \iff H_2O + M$ | | |
| 8 | $2H+ \leftrightarrow H_2 + M$ | | |
| 9 | $C_2H_4 \leftrightarrow C_2H_2 + H_2$ | | |
| 10 | $C_2H_2 + 2OH \leftrightarrow 2H_2 + 2CO$ | | |



Figure 10. Simplified three-dimensional computational models for the engine inner flow field of three composite fuel grains.



Figure 11. (a) Combustion chamber pressure of composite fuel grains and (b) comparison of chamber pressure for simulation and experimental results.

4.2. Flow Characteristics

Figure 12 illustrates the surface streamlines at different locations of the HRE. As shown in Figure 12a, the cross-sectional velocity distribution of the HRE indicates that oxygen flow in the MS grain was faster in the post-chamber region compared with that of the HS and PS grains. Figure 12b shows that the swirl flow field mainly occurred at the near wall of the three composite grains. In particular, at a distance of 25 mm from the front surfaces of the three fuel grains, streamlines in the near wall were significantly disordered. This disorder may be attributed to turbulent flow induced by the reactive force of the blades on the oxygen, with high initial resultant velocity. Additionally, curved streamlines re-emerged at the 75 mm position from the front face of the MS grain. This phenomenon seems to resemble the superposition of two vortex flows moving in opposite directions.



Figure 12. (**a**) Surface velocity distributions of hybrid rocket engine and (**b**) streamline diagrams at 25 mm axially equidistant positions for the three composite fuel grains.

To further investigate surface streamline variations and the swirl flow on the burning effect of the MS grain, the tangential velocity was analyzed. The distribution of the surface tangential velocity vectors is shown in Figure 13a. The red and blue colors indicate areas of strong swirl flow. The color bars of tangential velocities in Figure 13b are scalar, and the arrows on the surface visualize the change in swirl direction.

A reflux area formed on the pre-chamber of the engines, which was opposite to the direction of helical blades due to the high initial flow rate of oxygen. At a distance of 25 mm from the front face on the near wall of the grains, the reflux area of the oxidizer overlapped with the positive swirling flow directed by the blades, two streams of swirling flow with opposite directions. Subsequently, under induction of the blade, the swirl direction of the near walls of the PS and HS grains was mainly guided by the helical blades, and the area of the recirculation zone continuously reduced. Interestingly, this single direction of swirling flow did not persist within the MS grain model. Figure 13 shows that the central area of the MS grain exhibited swirling flows with opposite directions due to the guiding effect of the mirrored module blades and the inertia of oxygen flow. The vector diagram of tangential velocity corroborates the earlier conjecture regarding the curvature of the streamlines. The presence of two opposite vortices significantly fosters the development of turbulence, which in turn facilitates the mixing of the fuel and combustion gases, consequently elevating the combustion efficiency.



Figure 13. (**a**) Tangential velocity distribution of hybrid rocket engine and (**b**) surface tangential vector distribution of grains at 25 mm axially equidistant positions.

An analysis of the swirling distribution on the near-wall surface of the three fuel grains was conducted (refer to black dotted lines in cutaway views of fuel grains in Figure 9). The resulting eddy swirling intensity distribution on the near-wall surface is presented in Figure 14. It is evident that the swirling intensity at the wall of the MS grain was not greatly weakened, which still promoted the shear effect on the inner surface of the grain. The swirling intensity in the blade region of the PS fuel grain was no longer zero, and that of the near wall was higher than for the HS grain. Swirling enhancement is thought to be one reason for the high regression rate of the PS grain.



Figure 14. Tangential velocity distributions along the Z-axis of three different fuel grains.

Figure 15 illustrates the distribution of CO_2 and H_2O within the HRE and average product mass fractions at the nozzle outlet. Complete combustion of ethylene produces CO_2 and H_2O ; thus, the mass fractions of these products can characterize adequacy of combustion. The average mass fractions of CO_2 and H_2O in the HRE post-chambers of the MS and PS grains were higher than those of the HS grain, which indicates that the



composite fuel grains with a modified modular fuel unit can promote the mixing of the oxidizer and fuel to improve combustion efficiency.

Figure 15. Mass fractions of: (a) CO_2 and (b) H_2O , and (c) average product mass fractions at nozzle outlets.

Overall, the present results confirm that composite fuel grains based on a modular fuel unit are capable of improving the combustion characteristics of HRE. The excellent regression rate and combustion efficiency of the MS grain validate the potential of this strategy. Future research in our group will focus on optimization and exploration of this modular fuel design strategy, such as combining structural features of the MS and PS grains. Research on combustion efficiency over time will also be discussed. In addition, a reconstruction technique with time-resolved calculations will be introduced to analyze the data in more detail in the future [43,44]. More complex turbulence modeling and numerical calculations will be taken into account in subsequent studies.

5. Conclusions

A modular fuel strategy for a composite hybrid rocket grain is proposed. The combustion characteristics of the composite fuel grain based on the modular fuel units with modifications were experimentally and numerically studied. The parameters of the engines with different fuel grains were analyzed for average mass flow fluxes of 9.5–28.1 g/s. The mechanism of the performance enhancement of the composite fuel grains with the modified modular units was analyzed using radiation imaging technology and 3D numerical simulation. The specific conclusions are as follows:

- (1) Composite fuel grains with the modified modular units are more conducive to increasing the regression rates. Under an oxygen mass flux of 6.0 g/(s·cm²), the regression rates of the MS and PS grains increased by 8.0% and 14.1%, respectively, compared with that of the HS grain.
- (2) Experimental results show that the composite fuel grains have the advantage of rapidly increasing the radiant intensity and flame thickness. The MS grain had the most significant improvement on combustion performance. The centralized *O/F* distribution and stable combustion efficiency further illustrated superiority in combustion of the mirrored helical structure.
- (3) Simulation results showed that the pores of the PS grain promoted the development of swirl flow, which increased the regression rate. Two opposite swirling flows resulting from the mirroring configuration significantly fostered the development of turbulence in the MS grain, which improved the mixing of propellant and gas and thus promoted combustion efficiency.

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Nomenclature

| HRE | hybrid rocket engine |
|------------------|---|
| ABS | acrylonitrile-butadiene-styrene |
| MS/PS/HS grain | composite fuel grains with mirrored/perforated/helical blades, respectively |
| PP grain | paraffin-based grain |
| 3D | three-dimensional |
| POD | proper orthogonal decomposition |
| O/F | oxidizer to fuel ratio |
| \dot{m}_o | average oxygen mass flow rate |
| G _{ox} | oxidizer mass flux |
| \overline{P} | average chamber pressure |
| ŕ | average regression rate |
| d_0 | the initial inner diameter of the fuel grain |
| d_f | the inner diameter of the fuel grain after firing experiments |
| m_0 | initial mass of the fuel grain |
| m_f | the mass of the fuel grain after firing experiments |
| η | combustion efficiency |
| C_t^* | theoretical characteristic velocity |
| C_e^* | experimentally obtained characteristic velocity |
| \overline{m}_g | average mass flow rate of the oxidizer |
| d_t | throat diameter of the nozzle |
| t | working time of the engine |
| U | orthogonal modes of the flame series |
| S | corresponding mode energy |
| V | time coefficients |
| Т | transpose of a matrix |
| λ | eigenvalue |
| Ε | normalized energy ratio of each mode |

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