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Analysis of a Thin Layer Formation of Third-Grade Fluid

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Abstract: In present learning, surface protection layer progression of a third-grade fluid (TGF) is examined. Fluid transport within the micro passage made by the firm blade has been presented. Main system of equations of fluidity have been narrated and streamlined by means of lubrication approximation theory (LAT). Here, approximate solutions of velocity, pressure gradient, and coating depth have been presented. Results of coating and layer forming have been tabulated and discussed as well. It is observed that the transport properties of third-order fluid delivers an instrument to regulate flow velocity, pressure, and affect the final coated region.

Keywords: optimal homotopy asymptotic method; non-newtonian fluid; coating; lubrication approximation

1. Introduction

Third-grade fluids fit into the category of well-ordered flowing-particles. These have thermo-viscoelastic properties and are amongst the non-Newtonian fluids (NNF) originated from the viscous constituents and elastic materials. Some of their specimens are polymeric-paints, DNA fluids, bio-organic solutions, and other synthetic materials. Polymeric fluids are practically ubiquitously exist and are used as thin layer deposition materials. Although these organic solutions and colloids demonstrate thermo-viscoelastic behavior. For these coating systems, applied stress takes into the mathematical relationship that is not simply existing in a single equation as described in [1–5]. In this work, Carapau et al. [2] based constitutive model for a third-order fluids is presented. In the present order, beta (β) is taken as a third-order type material factor. Phenomena of shear thickening or shear thinning are largely governed by its mathematical assessment. If material factor beta is larger than zero, the physical system performs similar to a shear thickening substance. In cases where the material factor beta is a smaller than zero, the physical system acts similar to shear thinning

substance. Liquid properties of blade surface protection coatings are mesmerizing, mainly owing to significant engineering solicitations. Application of coating to blade is a progression through which polymeric-particle is coated uninterruptedly to the non-stationary web, and micro-coating thickness is applied on the inflexible part. Blade coatings are largely castoff in broadsheet coats, as it delivers promising evenness to the broadsheets. Other solicitations contain metal oxide based coatings on magnetic recording or adhesive tape, in addition to, suspension glaze on photo layers. Many researchers [2–7] investigated coating flows of Newtonian fluids. Most liquids castoff in manufacturing and mechanical applications have non-linear mathematical behaviors between applied pressure and induced deformation. Most of surface protection coatings belong to NNF. Non-Newtonian category liquids are categorized bestowing to their constitutive models. Applied pressure or stress in case of these liquids is a non-linear strain, and proceeding for the answers of these models is not so straightforward. This is applicable and correct for both exact and for approximate results. It has been found that a second-order liquids do not exhibit the shear thinning or thickening tendency, TGF can exhibit such occurrences. TGF model characterizes inconclusively effort for all-inclusive explanation as NNF presentation owing to the prominence, here we deliberate the TGF based surface coating model. Basics of thermionics and stability of TGF have been given in [8].

Some readings about research work in coating efficaciously indulge the non-linear work and their comparisons in leading TGF [8–18]. Sullivan et al [12] premeditated the coating depth in surface coating size/width of TGF by implementing lubrication estimates with numerical and investigational outcomes for NNF. The influence of elasto-plastic material of blade surface protector with weaker viscoelastic fluids has been investigated as well in [13]. The performance of power-law for liquid in surface protector thin film-geometry has been studied with its behavior for pressure dissemination [14].

Hwang [15] and Dien et al. [16] also premeditated NNF in the blade thin film and projected estimated stream studies, Maxwellian flow model in surface protection layer, and studied the fragile viscoelastic performance. This investigation articulates the statement that viscoelastic characteristics of TGF may affect pressure [17,18] so they espoused LAT and associated the modelling and investigational outcomes. Current efforts on layer examination draw on [18–21]. Studies of Sajid et al. [19] has motivated to study the TGF with non-Newtonian factors. Moreover, Ali et al. [21] figured the transport properties for a diverse coating flow-design, and by comparable composite liquids. Here, the resolution and objective of contemporary investigation is to originate the thin film making device for TGF and to examine in what way the liquid characteristics influence the blade coating process. In this work, optimal homotopy asymptotic method (OHAM) [20–28] based solution is presented. The manuscript is categorized in four sections. In Section 2, the governing equation based upon the heat transfer equation is formulated. In Section 3, computational remarks for solution based on OHAM are given. The results are discussed in Section 4, narrating some cases as examples. Finally, in Section 5, the paper is summarized.

2. Materials and Methods

A two-dimensional blade coater model is taken which is isothermal and steady-state, as expressed in the Figure 1. The geometry comprises of a plane substrate at the level of $s = 0$, which travels with fixed speed U_b in r -direction and a stiff blade with the blade surface as described by $s = h_b(r)$. The stiff blade with length L_b and the edges with heights A_0 and A_1 at $r = 0$ and $r = L_b$ respectively, are held fixed at an angle ϕ such that $\tan \phi = \frac{A_1 - A_0}{L_b}$. A gap originated through a narrow channel within the blade and non-stationary lower-surface to apply coating material on it, would be filled by dragging an incompressible TGF due to non-Newtonian property of fluid and that formulates a thin coating on non stationary substrate.

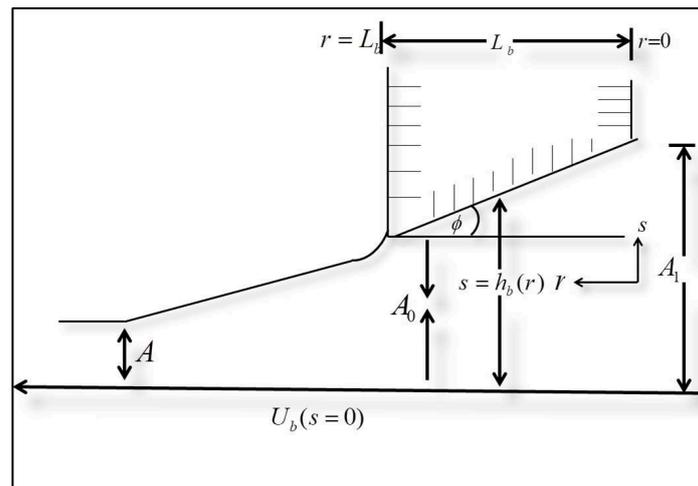


Figure 1. Blade coater geometry.

LAT is mainly designated for this flow based field. An NNF and incompressible TGF with elastic properties crawled in voids originated within narrow route with in unmovable blade and the movable substrate, and hence carved a homogeneous coating of width A on non-stationary surface. Principal models which administrate fluidity of NNF. Principle models which administrate stream of NNF involve the velocity profile

$$V_b = [u(r, s), v(r, s)]$$

where V_b is the velocity vector. This study begins with the LAT based approach. Least gap at the nip from the web and the surface is insignificant as matched to web measurement. it would be expedient to presume a parallel flow. All-purpose liquid drive is principally in r -track, although the liquid speed in s -direction is minor. Here, it is rational to adopt $v \ll u$ and $\frac{\partial}{\partial r} \ll \frac{\partial}{\partial s}$. The fact that the divergence of V_b , i.e., $\nabla \cdot V_b = 0$ implies $\frac{\partial u}{\partial r} = 0$ which implies $V_b = [u(s), 0]$, fulfilling continuity equation, acceleration portion of the momentum

$$\rho \frac{dv}{dt} = -\nabla p + \text{div} \tau,$$

and new form is

$$\nabla p + \text{div} \tau = 0, \tag{1}$$

where ρ denotes the density, p is the pressure, and τ represents the extra tensor for the third grade fluid which is

$$\tau = \mu B_1 + \alpha_1 B_2 + \alpha_2 B_2^2 + \beta_1 B_3 + \beta_2 (B_1 B_2 + B_2 B_1) + \beta_3 (\text{tr}(B_1^2)) B_1,$$

where μ is viscosity and α_1 is the plasticity, α_2 is cross viscosity and $\beta_1, \beta_2, \beta_3$ are material constants. Also $B_1, B_2,$ and B_3 are Rivlin Ericksen tensors. Here

$$\begin{aligned} B_1 &= (\nabla V_b) + (\nabla V_b)^T, \\ B_2 &= \frac{d}{dt} B_1 + B_1 (\nabla V_b) + (\nabla V_b)^T B_1, \\ B_3 &= \frac{d}{dt} B_2 + B_2 (\nabla V_b) + (\nabla V_b)^T B_2 \end{aligned}$$

The Equation (1) clues to momentum equation in constituent formula as

$$\frac{d\tau_{rs}}{ds} - \frac{\partial p}{\partial r} = 0, \tag{2}$$

$$\frac{d\tau_{ss}}{ds} - \frac{\partial p}{\partial s} = 0, \quad (3)$$

where

$$\tau_{rs} = \tau_{sr} = \frac{du}{ds} + 2(\beta_2 + \beta_3)\left(\frac{du}{ds}\right)^3, \quad (4)$$

and

$$\tau_{ss} = (2\alpha_1 + \alpha_2)\left(\frac{du}{ds}\right)^2 \quad (5)$$

Now the generalized pressure P is given

$$P(r, s) = p(r, s) - (2\alpha_1 + \alpha_2)\left(\frac{du}{ds}\right)^2 \quad (6)$$

Using Equations (4)–(6), Equations (2) and (3) take the form

$$\mu \frac{d^2u}{ds^2} + 2(\beta_2 + \beta_3) \frac{d}{ds} \left(\frac{du}{ds}\right)^3 = \frac{\partial P}{\partial r}. \quad (7)$$

$$\frac{\partial P}{\partial s} = 0. \quad (8)$$

Equation (8) depicts that P depends on r alone. Thus, Equation (7) is written

$$\mu \frac{d^2u}{ds^2} + 2\beta \frac{d}{ds} \left(\frac{du}{ds}\right)^3 = \frac{dP}{dr} \quad (9)$$

where $\beta = \beta_2 + \beta_3$. In light of Physics, the boundary conditions are

$$u = \begin{cases} U_b & \text{at } s = 0, \\ 0 & \text{at } s = h_b(r). \end{cases} \quad (10)$$

For the governing equations which are dimensionless for the analysis of blade coating, consider the following dimensionless variables

$$r^* = \frac{r}{L_b}, s^* = \frac{s}{L_b}, u^* = \frac{u}{U_b}, P^* = \frac{pA_0^2}{\mu U_b L_b}, \tilde{h}_b = \frac{h_b}{A_0}, \quad (11)$$

$$\beta^* = \frac{U_b^2 \beta}{\mu A_0}, \lambda = \frac{Q_b}{U_b W_b A_0}.$$

The dimensional form of the volumetric flow rate Q_b is

$$\frac{Q_b}{W_b} = \int_0^{h_b} u ds,$$

where W is thickness of web. Dimensionless representation is

$$\lambda = \int_0^{\tilde{h}_b} u ds. \quad (12)$$

From above variables by neglecting the asterisks signs using Equation (11), the equation of motion (9) with the boundary condition (10) is

$$\frac{d^2u}{ds^2} + 2\beta \frac{d}{ds} \left(\frac{du}{ds}\right)^3 = P_r. \quad (13)$$

where $P_r = \frac{dP}{dr}$.

3. OHAM Formulation

In the light of OHAM [22–30], the differential equation has the form

$$D(v(s)) + f(s) = 0, \quad s \in \Omega, \quad (14)$$

where Ω refers to domain. Now in Equation (14), the operator $D(v)$ is chosen as

$$D(v) = L(v) + N(v).$$

The construction in light of OHAM of an optimal homotopy is following

$$\phi(s; q) : \Omega \times [0, 1] \rightarrow \mathbb{R}$$

satisfying

$$(1 - q)\{L(\phi(s; q)) + f(s)\} - H(q)\{D(\phi(s; q)) + f(s)\} = 0, \quad (15)$$

where parameter $q \in [0, 1]$ is called an embedding parameter, and

$$H(q) = qC_1 + q^2C_2 + q^3C_3 + \dots$$

is called an auxiliary function in optimal homotopy Equation (15), with properties that $H(q) \neq 0$ for $q \neq 0$, $H(0) = 0$. Here the constants C_1, C_2, \dots are to be determined. Taylor's series about parameter q for expanding $\phi(s; q, C_i)$ to show estimated results are

$$\phi(s; q, C_i) = v_0(r_b, t) + \sum_{k=1}^{\infty} v_k(s; C_i)q^k, \quad i = 1, 2, \dots \quad (16)$$

It can be observed that the series convergence in Equation (16) depends mainly upon the constants C_1, C_2, \dots . If at $q = 1$, the series is convergent, then

$$\tilde{v}(s; C_i) = v_0(s) + \sum_{k \geq 1} v_k(s; C_i). \quad (17)$$

Substitution of Equation (17) into (14) gives following residual expression

$$R(s; C_i) = L(v(s; C_i)) + f(s) + N(\tilde{v}(s; C_i)).$$

If $R(s; C_i) = 0$, then $\tilde{v}(s; C_i)$ will give the exact solution. It does not happen in general mostly in case of nonlinear problems. Using the method as mentioned in [20–28]. One can determine the values of constants $C_i, i = 1, 2, \dots, m$.

4. Solution and Main Results

In this section, we will apply the OHAM to nonlinear ordinary differential Equation (13). According to the OHAM, we can construct homotopy of Equation (13) as

$$(1 - q) \left[\frac{d^2 u}{ds^2} - P_r \right] - (qC_1 + q^2C_2 + q^3C_3) \left[\frac{d^2 \mu}{ds^2} + 2\beta \frac{d}{ds} \left(\frac{du}{ds} \right)^3 - P_r \right] = 0. \quad (18)$$

We consider $u(s)$ as

$$u(s) = u_0(s) + qu_1(s) + q^2u_2(s) + q^3u_3(s). \quad (19)$$

Substituting $u(s)$ from Equation (19) into Equation (18), and some simplifications and rearranging based on powers of q -terms, we have

$$q^0 : u_0''(s) - P_r = 0, u_0(0) = 1, u_0(1) = 0. \quad (20)$$

$$q^1 : C_1 P_r - C_1 u_0''(s) - 6\beta C_1 u_0'(s)^2 u_0''(s) + P_r - u_0''(s) + u_1''(s), \\ u_1(0) = 0, u_1(1) = 0. \quad (21)$$

$$q^2 : C_2 P_r - C_2 u_0''(s) - 6\beta C_1 u_0'(s)^2 u_1''(s) - 12\beta C_1 u_0'(s) u_0''(s) u_1'(s) \\ - 6\beta C_2 u_0'(s)^2 u_0''(s) - C_1 u_1''(s) - u_1''(s) + u_2''(s), \\ u_2(0) = 0, u_2(1) = 0. \quad (22)$$

$$q^3 : C_3 P_r - 6\beta C_1 u_0''(s) u_1'(s)^2 - C_3 u_0''(s) - 6\beta C_2 u_0'(s)^2 u_1''(s) \\ - 12\beta C_1 u_0'(s) u_1'(s) u_1''(s) - 6\beta C_1 u_0'(s)^2 u_2''(s) - 12\beta C_2 u_0'(s) u_0''(s) u_1'(s) \\ - 12\beta C_1 u_0'(s) u_0''(s) u_2'(s) - 6\beta C_3 u_0'(s)^2 u_0''(s) - C_2 u_1''(s) - C_1 u_2''(s) \\ - u_2''(s) + u_3''(s), u_3(0) = 0, u_3(1) = 0. \quad (23)$$

Solving the Equations (20)–(23) with boundary conditions, we have

$$u_0(r) = \frac{1}{2}(s-1)(sP_r - 2). \quad (24)$$

$$u_1(r) = \frac{1}{4}\beta C_1 P_r (s-1)s(P_r^2(2s^2 - 2s + 1) + P_r(4 - 8s) + 12). \quad (25)$$

$$u_2(r) = \frac{1}{8}\beta P_r (s-1)s[2C_1(P_r^2(2s^2 - 2s + 1) + P_r(4 - 8s) + 12) \\ + 2C_2(P_r^2(2s^2 - 2s + 1) + P_r(4 - 8s) + 12) + C_1^2\{24(6\beta + 1) + \beta P_r^4 \\ (16s^4 - 32s^3 + 28s^2 - 12s + 3) - 4\beta P_r^3(24s^3 - 36s^2 + 22s - 5) \\ + P_r^2(96\beta + 4(60\beta + 1)s^2 - 4(60\beta + 1)s + 2) - 8(18\beta + 1)P_r(2s-1)\}]. \quad (26)$$

With $q = 1$, Equation (19) becomes

$$u(s) = u_0(s) + u_1(s) + u_2(s) + u_3(s). \quad (27)$$

Substituting values from Equations (24)–(26) in Equation (27), we get the first-order approximate solution of (13) as follows

$$u(r) = \frac{1}{4}(s-1)\{\beta C_1 s P_r (P_r^2(2s^2 - 2s + 1) + P_r(4 - 8s) + 12) + 2sP_r - 4\}. \quad (28)$$

For finding value of the constant C_1 shown in Equation (28), using the method of least squares as described in [17–19] implies that setting

$$\frac{\partial J}{\partial C_1} = 0 \quad (29)$$

gives the values of constant C_1 , where

$$J = \int_0^1 R^2 ds \quad (30)$$

and here R for the Equation (13) of motion is

$$R = \frac{d^2\mu}{ds^2} + 2\beta \frac{d}{ds} \left(\frac{du}{ds} \right)^3 - P_r$$

after substituting the values, we get

$$\begin{aligned}
 R = & \frac{1}{16}P_r\{3\beta(3\beta C_1(P_r(2s-1)-2)^2+2)\{\beta C_1P_r(P_r^2(2s-1)^3 \\
 & -4P_r(6s^2-6s+1)+24s-12)+P_r(4s-2)-4\}^2+8(\beta C_1 \\
 & (P_r^2(6s^2-4s+1)+P_r(4-16s)+12)+2) \\
 & +16\beta C_1P_r(s-1)(P_r(3s-1)-4)-16\}.
 \end{aligned}
 \tag{31}$$

Thus with the choice of $\beta = 0.03$ and $P_r = 2$, the Equation (30) gives

$$\begin{aligned}
 J = & 0.0000515515C_1^6 + 0.0014886C_1^5 + 0.0245795C_1^4 + 0.218005C_1^3 \\
 & + 1.35459C_1^2 + 1.41597C_1 + 0.41472.
 \end{aligned}$$

Finally using Equation (29), we get the following values of C_1

$$\begin{aligned}
 & \{C_1 \rightarrow -7.81945 - 7.39559i\}, \{C_1 \rightarrow -7.81945 + 7.39559i\}, \\
 & \{C_1 \rightarrow -3.91078 - 7.09001i\}, \{C_1 \rightarrow -3.91078 + 7.09001i\}, \\
 & \{C_1 \rightarrow -0.602773\}.
 \end{aligned}$$

Choosing the real value of C_1 , i.e., $C_1 = -0.6027727875127079$; similarly for different values of β , the values of constant C_1 are shown in the Table 1.

Table 1. Values of β and C_1 .

β and C_1	
β	C_1
0.03	-0.6027727875127079
0.04	-0.5367678225757836
0.05	0.4849617709110862
0.06	-0.44305755184121864

Corresponding these values, the values of u are calculated as shown in the Equations (32)–(35).

$$u_{(\beta=0.03)} = 1 + s[s\{s(0.289331 - 0.0723327s) + 0.566004\} - 1.783], \tag{32}$$

$$u_{(\beta=0.04)} = 1 + s[s\{s(0.343531 - 0.0858829s) + 0.484703\} - 1.74235], \tag{33}$$

$$u_{(\beta=0.05)} = 1 + s[s\{s(0.387969 - 0.0969924s) + 0.418046\} - 1.70902], \tag{34}$$

$$u_{(\beta=0.06)} = 1 + s[s\{s(0.425335 - 0.106334s) + 0.361997\} - 1.681]. \tag{35}$$

Figure 2 shows the values of u at different values of β . Also, Figure 3 gives the nature of u at different values of β and s .

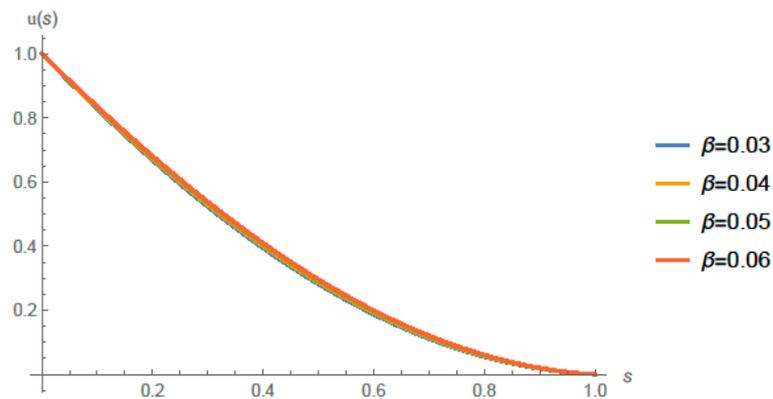


Figure 2. $u(s)$ at different values of β .

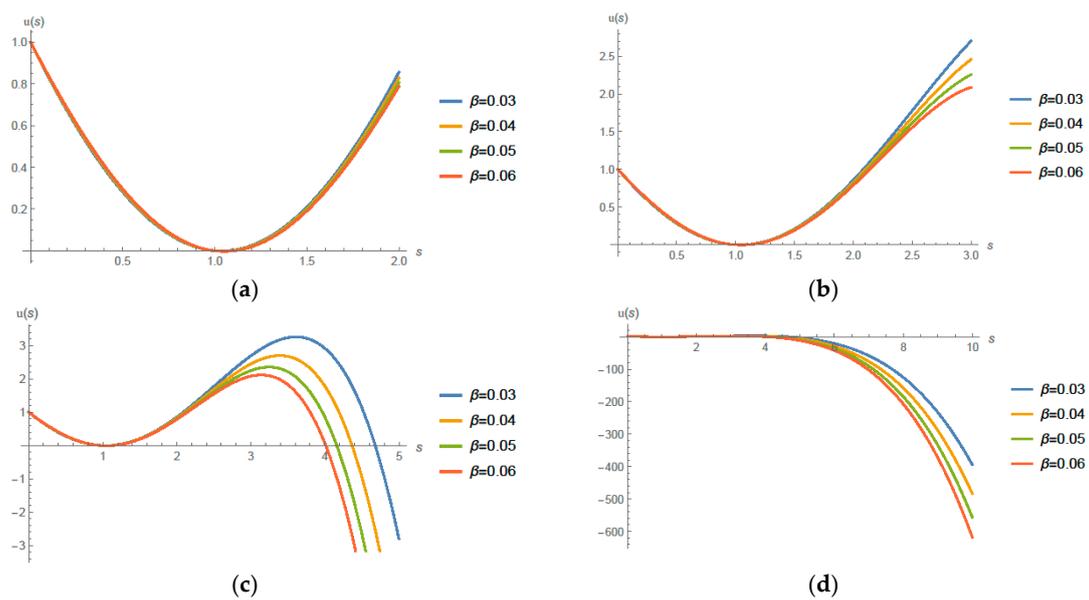


Figure 3. $u(s)$ at different values of β and varying ranges of s . (a) with the s range of $[0-2]$; (b) with the s range of $[0-3]$; (c) with the s range of $[0-5]$; (d) with the s range of $[0-10]$.

Now using from Equation (12)

$$\lambda = \int u \, ds$$

gives

$$\lambda = \frac{1}{120} s [3\beta C_1 s P_r \{P_r^2 (4s^3 - 10s^2 + 10s - 5) - 20P_r (s - 1)^2 + 40s - 60\} + 10(2s^2 P_r - 3(P_r + 2)s + 12)].$$

For fixed value of $\beta = 0.03$ and for different values of P_r , the values of constant C_1 are shown in the Table 2.

Table 2. Values of P_r and C_1 .

For Fixed $\beta = 0.03$	
P_r	C_1
1	-0.7389837541589072
1.5	-0.6670404279185341
2	-0.6027727875127079
2.5	-0.5456887819699325

Corresponding to these values, the values of λ are calculated as shown in the Equations (36)–(39).

$$\lambda_{(P_r=1)} = \frac{1}{120}s\{10(2s^2 - 9s + 12) - 0.0665085s(4s^3 - 10s^2 + 50s - 20(s - 1)^2 - 65)\}. \tag{36}$$

$$\lambda_{(P_r=1.5)} = \frac{1}{120}s\{10(3.s^2 - 10.5s + 12) - 0.0900505s(2.25(4s^3 - 10s^2 + 10s - 5) - 30(s - 1)^2 + 40s - 60)\}. \tag{37}$$

$$\lambda_{(P_r=2)} = \frac{1}{120}s\{10(4s^2 - 12s + 12) - 0.108499s(4(4s^3 - 10s^2 + 10s - 5) - 40(s - 1)^2 + 40s - 60)\}. \tag{38}$$

$$\lambda_{(P_r=2.5)} = \frac{1}{120}s\{10(5.s^2 - 13.5s + 12) - 0.12278s(6.25(4s^3 - 10s^2 + 10s - 5) - 50(s - 1)^2 + 40s - 60)\}. \tag{39}$$

Figure 4 shows the values of λ at different values of P_r . Also, Figure 5 gives the nature of λ at different values of P_r and s .

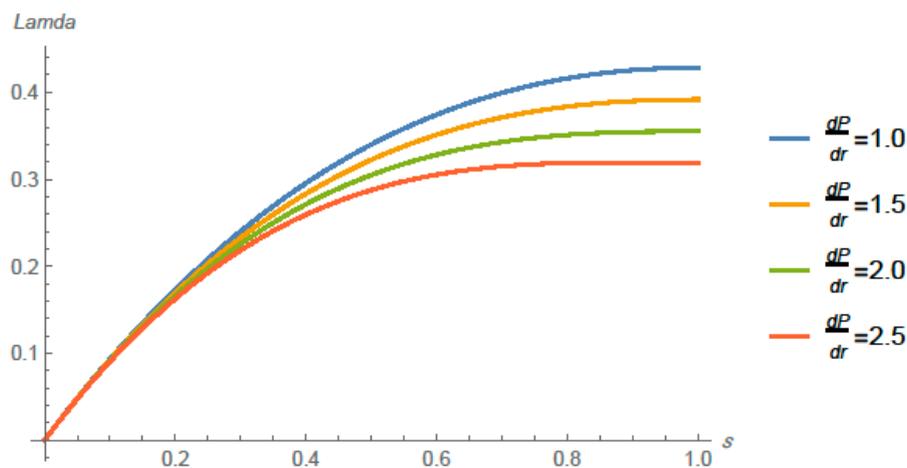


Figure 4. λ at different values of $P_r = \frac{dP}{dr}$.

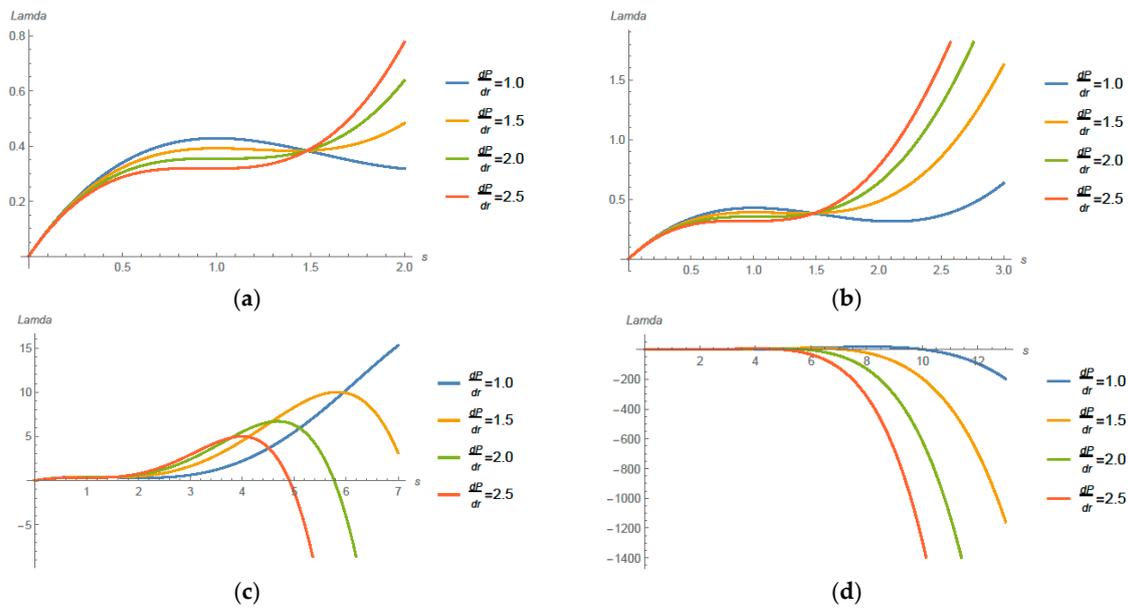


Figure 5. λ at different values of P_r and s . (a) with the s range of [0–2]; (b) with the s range of [0–3]; (c) with the s range of [0–7]; (d) with the s range of [0–12].

For fixed value of $\beta = 0.03$ and for different values of P_r , the values of constant C_1 are shown in the Table 2. the values of u are calculated as shown in the Equations (40)–(43).

$$u_{(P_r=1)} = 1 + s[s\{s(0.0665085 - 0.0110848s) + 0.350356\} - 1.40578], \tag{40}$$

$$u_{(P_r=1.5)} = 1 + s[s\{s(0.157588 - 0.0337689s) + 0.47422\} - 1.59804], \tag{41}$$

$$u_{(P_r=2)} = 1 + s[s\{s(0.289331 - 0.0723327s) + 0.566004\} - 1.783], \tag{42}$$

$$u_{(P_r=2.5)} = 1 + s[s\{s(0.460425 - 0.127896s) + 0.628426\} - 1.96096]. \tag{43}$$

Figure 6 shows the values of u at different values of P_r . Also, Figure 7 gives the nature of u at different values of P_r and s .

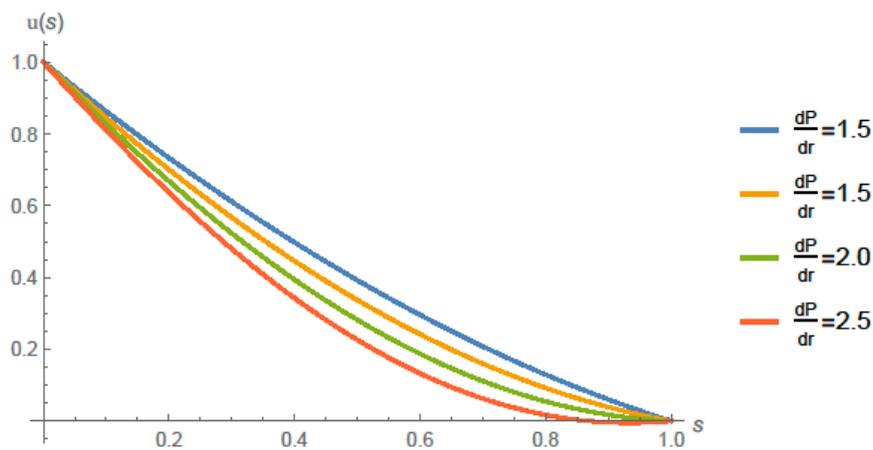


Figure 6. $u(s)$ at different values of $\frac{dP}{dr}$.

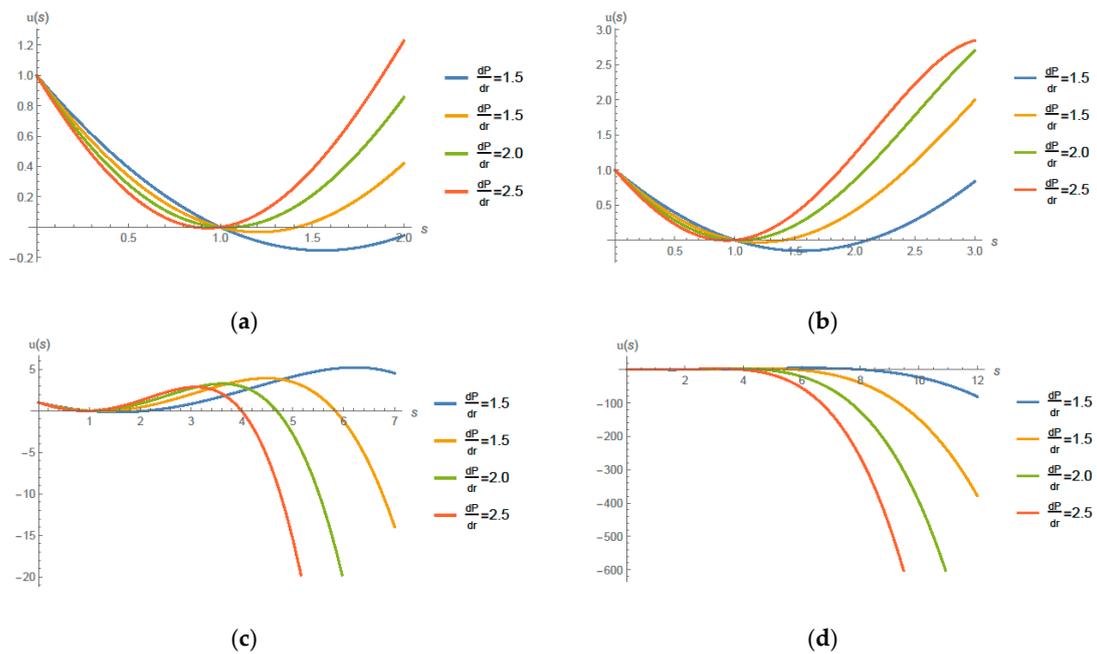


Figure 7. $u(s)$ at different values of $\frac{dp}{dr}$ and s . (a) with the s range of [0–2]; (b) with the s range of [0–3]; (c) with the s range of [0–7]; (d) with the s range of [0–12].

The Equation (6) for the stress, after some manipulation becomes in dimensionless form as

$$p(r, s) = P_r(r, s) + \alpha \left(\frac{du}{ds} \right)^2 \tag{44}$$

which is as

$$p = \alpha \left[\frac{1}{4}(s-1) \{ \beta C_1 P_r (P_r^2 (2s^2 - 2s + 1) + P_r (4 - 8s) + 12) + \beta C_1 s P_r (P_r^2 (4s - 2) - 8P_r) + 2P_r \} + \frac{1}{4} (\beta C_1 s P_r (P_r^2 (2s^2 - 2s + 1) + P_r (4 - 8s) + 12) + 2sP_r - 4) \right]^2 + P_r.$$

For fixed values of $\beta = 0.03$, $P_r = 2$ in

$$u = \frac{1}{4}(s-1) \{ \beta C_1 s P_r (P_r^2 (2s^2 - 2s + 1) + P_r (4 - 8s) + 12) + 2sP_r - 4 \};$$

give the value of constant $C_1 = -0.6027727875127079$, then for different values of α , the values of stress p are calculated as shown in the Equations (45)–(48).

$$p_{(\alpha=0.2)} = 2 + 0.2 \left[\frac{1}{4}(s-1) \{ -0.0361664 (4(2s^2 - 2s + 1) + 2(4 - 8s) + 12) - 0.0361664s(4(4s - 2) - 16) + 4 \} + \frac{1}{4} \{ -0.0361664 (4(2s^2 - 2s + 1) + 2(4 - 8s) + 12)s + 4s - 4 \} \right]^2. \tag{45}$$

$$p_{(\alpha=0.3)} = 2 + 0.3 \left[\frac{1}{4}(s-1) \{ -0.0361664 (4(2s^2 - 2s + 1) + 2(4 - 8s) + 12) - 0.0361664s(4(4s - 2) - 16) + 4 \} + \frac{1}{4} \{ -0.0361664 (4(2s^2 - 2s + 1) + 2(4 - 8s) + 12)s + 4s - 4 \} \right]^2. \tag{46}$$

$$p_{(\alpha=0.4)} = 2 + 0.4 \left[\frac{1}{4}(s-1) \{ -0.0361664 (4(2s^2 - 2s + 1) + 2(4 - 8s) + 12) - 0.0361664s(4(4s - 2) - 16) + 4 \} + \frac{1}{4} \{ -0.0361664 (4(2s^2 - 2s + 1) + 2(4 - 8s) + 12)s + 4s - 4 \} \right]^2. \tag{47}$$

$$\begin{aligned}
 p_{(\alpha=0.5)} = & 2 + 0.5\left[\frac{1}{4}(s-1)\{-0.0361664(4(2s^2-2s+1) + 2(4-8s) + 12)\right. \\
 & -0.0361664s(4(4s-2) - 16) + 4\} + \frac{1}{4}\{-0.0361664(4(2s^2-2s+1) \\
 & \left. + 2(4-8s) + 12)s + 4s - 4\}^2. \tag{48}
 \end{aligned}$$

Figure 8 shows the values of stress p at different values of α . Also the Figure 9 gives the nature of p at different values of α and s . Stratagems the normal stress properties at altered locations of TGF coating progression in dissimilar standards it is perceived that strain upsurges with growing α for constant β . These results are in accordance with [29–38].

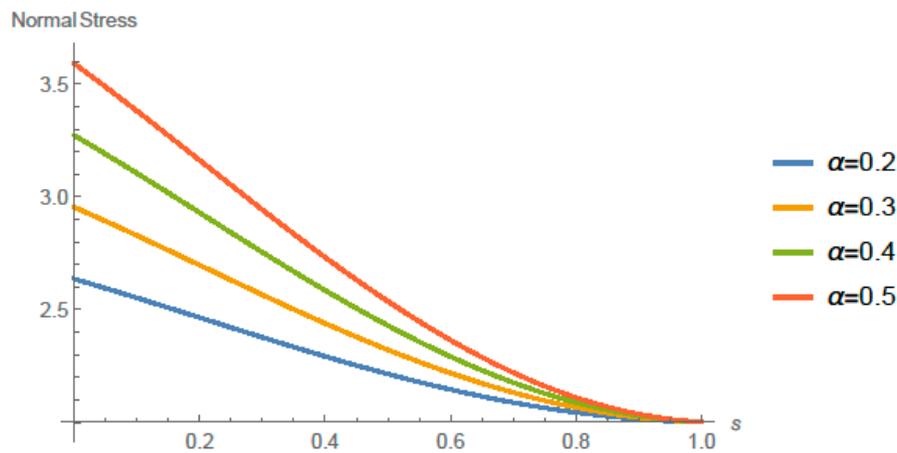


Figure 8. Normal Stress at different values of α .

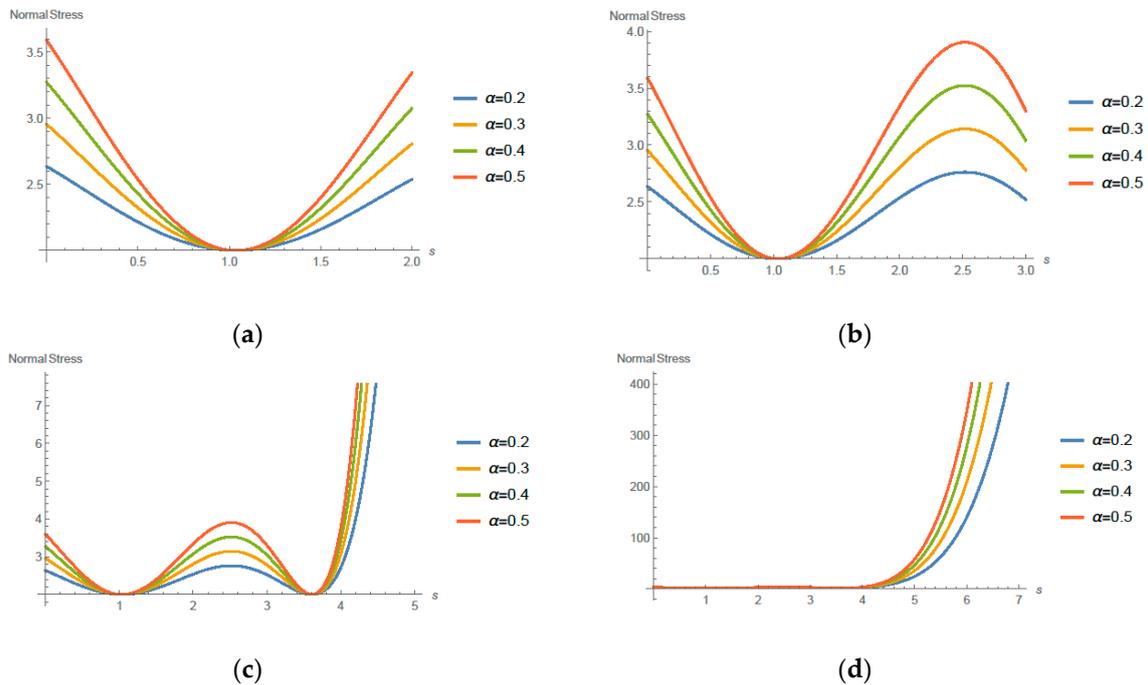


Figure 9. Normal stress at different values of α and s . (a) with the s range of [0–2]; (b) with the s range of [0–3]; (c) with the s range of [0–5]; (d) with the s range of [0–7].

Figure 9 shows normal stress at different values of α . Stratagems the normal stress properties at altered locations of TGF coating process in various perspective. It is perceived that strain is increasing through the area of coating with growing α . Figures 2–9 provide a TGF implementation of the blade thin film and in what way the dissimilar restrictions and physical constraints. Some results are represented in the form of graphs, though results are given in a tabularized arrangement.

Figures 2–7 are the graphical representation of velocity for dissimilar non-Newtonian fluids' parameters. These graphical representations designate that the velocity contours is the combination of Poiseuille and Couette kind of flow streams. In graphical representations of Figures 2, 5 and 6, velocity contours reduce with enhancing NNF parameter. Upsurge in the NNF factor β resembles the shear condensing consequence that rises the liquid viscidness and declines liquid speed as supported by [37–43]. Figure 8 shows behavior of normal stresses at dissimilar values of α . Figure 9 shows behavior of shear stresses at varying values of α and s . Results of Figures 2–5 obviously display β upsurges the NNF character upsurges, i.e., the shear thickening escalates that decreases the liquid flow rate.

5. Summary and Conclusions

In this work, TGF based coating model is investigated and its transport behavior on the blade thin film where the stream is lying within the inflexible edge and the movable web. This effort examines the blade surface coating procedure for TGF. Lubrication approximation theory is employed to progress the main mathematical model for the TGF in the thin and slim conduit. Estimated results based on OHAM for velocities, pressure, and volumetric current rate. The thin film width, maximum pressure, and normal stresses are also been studied comprehensively. Our results strongly show that a third-order fluid performs as the surface coatings where the TGF transport is within inflexible blade and non-stationary system. Lubrication theory is employed to mature the major equation for the TGF in a thin conduit.

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