A Multi-Scale Conceptual Model of Flood-Tide Delta Morphodynamics in Micro-Tidal Estuaries
Re-Aeration on Stepped Spillways with Special Consideration of Entrained and Entrapped Air

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Abstract: As with most high-velocity free-surface flows, stepped spillway flows become self-aerated when the drop height exceeds a critical value. Due to the step-induced macro-roughness, the flow field becomes more turbulent than on a similar smooth-invert chute. For this reason, cascades are oftentimes used as re-aeration structures in wastewater treatment. However, for stepped spillways as flood release structures downstream of deoxygenated reservoirs, gas transfer is also of crucial significance to meet ecological requirements. Prediction of mass transfer velocities becomes challenging, as the flow regime differs from typical previously studied flow conditions. In this paper, detailed air-water flow measurements are conducted on stepped spillway models with different geometry, with the aim to estimate the specific air-water interface. Re-aeration performances are determined by applying the absorption method. In contrast to earlier studies, the aerated water body is considered a continuous mixture up to a level where 75% air concentration is reached. Above this level, a homogenous surface wave field is considered, which is found to significantly affect the total air-water interface available for mass transfer. Geometrical characteristics of these surface waves are obtained from high-speed camera investigations. The results show that both the mean air concentration and the mean flow velocity have influence on the mass transfer. Finally, an empirical relationship for the mass transfer on stepped spillway models is proposed.

Keywords: gas transfer; stepped spillways; skimming flows; self-aeration; air-water interface

1. Introduction

Stepped spillways and cascades are known to be effective energy dissipaters downstream of reservoirs due to the increased flow resistance created by the steps [1]. In addition, the step-induced macro-roughness leads to higher turbulence compared to smooth-invert chutes, which in turn significantly affects the flow structure. Similar to smooth-invert spillways, stepped spillway flows become self-aerated depending on the discharge, slope and spillway extension [1–3]. Once the self-aeration has been triggered, the water surface becomes more and more disturbed, and bubbles, entrained in water, as well as droplets, ejected to the air, are transported along the chute. This involves a significant specific air-water interface across which gas transfer can take place. In particular, oxygen transfer is of high interest as water in reservoirs often suffer from deoxygenation, there being a level of dissolved oxygen (DO) content required to keep good ecological conditions in downstream rivers and streams.

Chanson [4] stated that the re-aeration potential of stepped spillways is higher than for smooth-invert chutes with an identical slope. Moreover, the re-aeration in a skimming flow regime (i.e. a flow regime where water flows down the chute as a coherent stream over the pseudo-bottom formed by the step edges)
becomes higher for lower discharges. Several studies have been conducted in the past to describe this re-aeration potential $E$, which relates the oxygen deficit upstream of the structure to the remaining deficit at the downstream end, as functions of the structure’s geometry [5–12].

However, all of these studies only focus on the global aeration performance and do not investigate the gas transfer mechanism itself. In order to physically describe the gas transfer process, knowledge of the specific air-water interface and a feasible gas transfer model are essential. The latter typically depends on flow conditions, for example, mean air concentration, velocity and turbulence, and needs to be estimated by means of experiments.

In this paper, the gas transfer velocity is determined for stepped spillway flows by conducting detailed measurements of the air-water flow properties and time series of dissolved oxygen content. In contrary to earlier studies, the air-water mixture is assumed to consist of a bubbly flow region (entrained air) below a wavy surface (entrapped air), following insights of [13–15].

2. Background

2.1. Gas Diffusion

Gas transfer between water and air is driven by an existing concentration gradient $\nabla (C)_{gas}$ and may be described by Fick’s 1st law, according to which the diffusive flux $I_{gas}$ (in $\text{mg/(m}^2\text{s)}$) is proportional to this gradient and directed from high concentrations to low concentrations:

$$I_{gas} = -D_{gas} \times \nabla (C)_{gas},$$

with $D_{gas}$ the diffusion coefficient (in $\text{m}^2$/s). $D_{gas}$ depends on the temperature and is a measure for gas transfer velocity. Considering mass conservation and assuming an isotropic diffusivity, a diffusion equation is found as

$$\frac{\partial C_{gas}}{\partial t} = D_{gas} \times \nabla^2 C_{gas}. \tag{2}$$

For the one-dimensional diffusion perpendicular to the air-water interface, Equation (2) becomes

$$\frac{\partial C_{gas}}{\partial t} = D_{gas} \times \frac{\partial^2 C_{gas}}{\partial n^2}. \tag{3}$$

To solve Equation (3), the diffusion length $n$ needs to be known, which, however, depends on the viscosity and the diffusivity itself [16]. The following empirical gas transfer model becomes thus better suited for practical applications:

$$\frac{dC_L}{dt} = k \times \frac{A}{V} \times (C_S - C_L), \tag{4}$$

with $C_L$ the gas concentration in the liquid phase, $C_S$ the saturation concentration, $A$ the interface available for gas transfer in the water volume $V$ and $k$ the mass transfer coefficient (in $\text{m/s}$). The reciprocal value of $k$ can be considered as the resistance involved from both phases (index “L” for liquid and “G” for gas) [17]:

$$\frac{1}{k} = \frac{1}{k_L} + \frac{1}{H_G \times k_G}, \tag{5}$$

with $H_G = C_G/C_L$ the dimensionless Henry coefficient, depending on salinity, temperature and pressure. It is worth noting that $k_G$ is about 40 to 1000 larger than $k_L$, according to [18], for water and air (depending on turbulence), and $H_G$ is approximately 32 for oxygen. The mass transfer coefficient $k$ in Equation (4) can thus be replaced by $k_L$ of the water phase only:

$$\frac{dC_L}{dt} = k_L \times \frac{A}{V} \times (C_S - C_L), \tag{6}$$

with $a = A/V$ the specific air-water interface.
2.2. Mass Transfer Coefficient

Besides some classical, conceptional approaches, such as the two-film model of Lewis & Whitman [17], the penetration model of Higbie [19] and the surface-renewal model of Danckwerts [20], more complex, hydrodynamic models were developed later, such as the Large-Eddy model of Fortescue & Pearson [21], the Small-Eddy model by Lamont & Scott [22] and the Modified-Small-Eddy model by Moog & Jirka [23], or the turbulence-based model by Gualtieri & Gualtieri [24].

The experimental studies of other researchers show a significant influence on the mean flow velocity, Reynolds number and mean air concentration [25], while others pinpointed the influence of bubble sizes [26].

It must be noted that all of these mass transfer models have been developed for specific flow conditions that hardly compare to stepped spillway flows. The chaotic flow regime in skimming flows on a stepped spillway involves significantly higher turbulence and a more intense gas exchange than, for instance, rising bubbles in vertical pipes (case for some of these models). As pointed out by Demars & Manson [27], estimation of the air-water interface in self-aerated flows becomes non-trivial due to bubbles and spray. A more detailed experimental investigation of the re-aeration on stepped spillways and derivation of a suitable mass transfer equation is necessary.

2.3. Specific Air-Water Interface

With consideration of a simple continuity concept and assumption of spherical bubbles, Chanson [28] proposed to estimate the specific air-water interface by:

$$a(z) = 4 \times \frac{F_B(z)}{u(z)}.$$  (7)

Derivation of Equation (7) is based on a statistical relation between the Sauter diameter, typically measured by an air concentration probe, and the diameter of a spherical bubble. $F_B$ is the number of detected bubbles (or droplets) per unit of time (in Hz), which can be directly extracted from the phase detection probe signals, and $u(z)$ is the streamwise flow velocity. Toombes [29] and Toombes & Chanson [30] suggested applying this approach for larger bubbles as well, despite the fact that the shape can differ from an ideal sphere.

However, it is known that the air-water mixture in a stepped spillway flow may be regarded in different ways. While classical approaches assume a homogenous continuum between the bottom and $h_{90}$, that is, the flow depth where the time-averaged air concentration is 90% [31,32], some recent studies distinguish a lower region with entrained air bubbles from an upper region where the air content mainly comes from air pockets being entrapped between surface waves [13,33–35]. Similar flow structures were reported for smooth-invert chutes [14,15]. In different studies, with different geometrical configurations, some characteristic elevations of the maximum wave trough extensions on stepped spillways are proposed. For the setup analyzed in this paper, Bung [13] suggested $h_{75}$, that is, the flow depth with 75% mean air concentration, as the lower extension of the surface waves. Figure 1a shows an exemplary frame taken from a high-speed video in the quasi-uniform flow region of a model test with 1:2 slope ($\varphi = 26.6^\circ$), $s = 6$ cm step height and a specific discharge of $q = 0.07$ m$^2$/s (compare [36]). The level of $h_{75}$, detected with an air concentration probe, is also indicated. It can be seen that for higher elevations, the flow structure changes and the application of Equation (7) becomes questionable for higher levels. Figure 1b shows the distribution of the dimensionless specific air-water interface $a/\max(a)$ assuming Equation (7) along the full z-coordinate up to the level corresponding to 99% air concentration. The change of the air-water flow structure for the surface wave region becomes obvious again.
With the assumption that (1) both tips pierce the same bubbles and (2) bubbles are transported in a quasi-uniform flow conditions set in. For the larger steps, data is gathered at step edges and above the centre of the step cavity. This probe consists of two needle tips with a diameter of 0.13 mm. Both tips are aligned in flow direction with a lateral spacing of 1 mm which aim to pierce up entrained bubbles and ejected droplets. By a change of resistivity depending on the surrounding medium, phase changes can be detected by applying a thresholding technique [44]. Further information on this type of probe can be found in [45].

3. Methodology

3.1. General Comments (Data Base)

The current paper presents a re-analysis of air-water flow data and oxygenation data published in [37–40], respectively. While these studies considered a continuous air-water mixture up to $h_{75}$, the current study accounts for $h_{75}$ as the upper extent of this bubbly layer. In order to account for the air-water surface roughness due to the surface waves [13], an idealized, simple sinusoidal surface is considered. Air bubbles being entrained within these surface waves are not regarded. An analytical derivation of relevant equations and necessary approximations to describe the surface geometry and its volume beneath the waves is given in Appendix A.

All data has been gathered on a physical model with slopes 1:3 and 1:2, step heights $s$ of 3 cm and 6 cm and discharges of $q = 0.07$, 0.09 and 0.11 m$^2$/s in the Hydraulics laboratory of Wuppertal University [37]. The chute width was 30 cm and its total drop height was 2.34 m. Skimming flow was found for all model tests (compare [41–43]). Water discharge was regulated with a flap valve, controlled with a flow meter and pumped into an open head tank before being conveyed into the spillway. The chute was part of a closed water circuit.

3.2. Air-Water Flow Measurements

Air-water flow measurements are conducted using a double-tip conductivity probe in the flume centreline. This probe consists of two needle tips with a diameter of 0.13 mm. Both tips are aligned in flow direction with a lateral spacing of 1 mm which aim to pierce up entrained bubbles and ejected droplets. By a change of resistivity depending on the surrounding medium, phase changes can be detected by applying a thresholding technique [44]. Further information on this type of probe can be found in [45]. With the assumption that (1) both tips pierce the same bubbles and (2) bubbles are transported in a slip-free way, cross-correlation of both signals yields an estimate of the flow velocity. Relevant parameters, such as the local, time-averaged air concentration, the Sauter diameter and bubble count rate can be directly determined from the signals. Additional parameters, for example, the specific air-water interface according to Equation (7), can be indirectly estimated from this data. For all configurations, tests are conducted from the inception point of self-aeration to farther downstream where quasi-uniform flow conditions set in. For the larger steps, data is gathered at step edges and above the centre of the step cavity. For the smaller steps, all data is obtained at step edges only.

As the extraction of surface waves (Figure 1a) from conductivity probe data and the determination of the available air-water interface above $h_{75}$ is not directly possible, a continuum up to $h_{75}$ is considered in this study and a three-dimensional sinusoidal wave field (with waveheights $H$ and wavelengths $\lambda$),
leading to an interfacial surface $\mathcal{S}$ of the air-water mixture. The latter is added to the interface $A_{\text{ent}}$ formed by the entrained air bubbles. A schematic illustration of the employed concept is shown in Figure 2. In order to determine the specific air-water interface, $A_{\text{tot}} = A_{\text{ent}} + \mathcal{S}$ needs to be related to the total volume (up to the upper wave extent).

![Figure 2](image uploading)

**Figure 2.** Schematic illustration of the considered flow structure over the step cavities consisting of a continuous air-water mixture up to $h_{75}$ and non-aerated, sinusoidal surface waves over $h_{75}$.

### 3.3. Surface Wave Characterization

To estimate the interface being available for gas transfer due to surface waves, a simple three-dimensional sinusoidal wave field is assumed. This wave field is characterized by equal wavelengths $\lambda$ in both, streamwise and transverse direction, and a waveheight $H$. Similar surface waves characteristics were assumed to analytically describe the self-aeration process by [3], shedding light on the fluid mechanics of self-aeration. Hence, it is herein proposed that surface waves are connected to the upstream non-aerated region, holding similar structure. As no closed solution for the surface area of the supposed wave field exists, a numerical integration was performed to find an empirical fitting function. The description of the underlying assumptions and solutions can be found in Appendix A.

Characteristic wavelengths and waveheights are obtained from high-speed videos analyzed in [46], which extracted the free surface of the air-water mixture applying an image processing technique. From this data, the median wavelengths and waveheights, given in Table 1, have been obtained (Table 1) and considered representative of the whole structure to determine the air-water mixture surface in Section 4.1.

<table>
<thead>
<tr>
<th>Step Height [cm]</th>
<th>3</th>
<th>6</th>
<th>3</th>
<th>6</th>
</tr>
</thead>
<tbody>
<tr>
<td>Discharge [m$^3$/s]</td>
<td>0.07</td>
<td>0.09</td>
<td>0.11</td>
<td>0.07</td>
</tr>
<tr>
<td>$\lambda$ [cm]</td>
<td>6.1 (1.6)</td>
<td>7.5 (1.4)</td>
<td>8.9 (2.3)</td>
<td>7.1 (1.7)</td>
</tr>
<tr>
<td>$H$ [cm]</td>
<td>1.4 (0.6)</td>
<td>1.3 (0.8)</td>
<td>1.3 (0.6)</td>
<td>2.6 (1.0)</td>
</tr>
</tbody>
</table>

### 3.4. Dissolved Oxygen Measurements

The re-aeration potential of the different stepped spillway model setups has been determined by direct oxygen measurements. The absorption method, according to [47], has been applied by reducing the dissolved oxygen content through sodium solfite addition. In order to speed up the deoxygenation process, cobalt sulfate was added as a catalyst. Before starting the tests, accurate mixing of the chemicals was ensured by circulating the water in the tanks with small pumps. Besides, more sodium sulfite was added than theoretically needed to ensure full deoxygenation. Oxygen concentration was measured with two identical optical probes (Hach HQ10) upstream and downstream of the spillway.
and all tests were repeated at least once. Time series of instantaneous dissolved oxygen concentrations $C_{DO}(t)$ are expected to follow an exponential function given by

$$C_{DO}(t) = C_{S,p,T} - (C_{S,p,T} - C_0) \times e^{-k_L a_T \times t}, \quad (8)$$

where $C_{S,p,T}$ is the saturation concentration at local conditions (with atmospheric pressure $p$ and temperature $T$) and $k_L a_T$ is the aeration coefficient at temperature $T$. For comparison of aeration performance tests with different local conditions, $C_{S,p,T}$ and $k_L a_T$ (yielded by a nonlinear regression analysis) can be normalized to a standard temperature of $20 \, ^\circ C$ and a pressure of 1013 hPa as

$$k_L a_{20} = k_L a_T \times 1.024^{20-T} \quad (9)$$

and

$$C_{S,20} = C_{S,p,T} \times \frac{C_{S,SL,20}}{C_{S,SL,T}} \times \frac{1013}{p}, \quad (10)$$

respectively. In Equation (10), $C_{S,SL,T}$ refers to the standardized saturation concentration for $p = 1013$ hPa and temperature $T$, while $C_{S,SL,20} = 9.09$ mg/L for full standard conditions. An exemplary result from a single oxygenation test is shown in Figure 3a. Figure 3b shows the residuals between the fitting and the laboratory data, proving a good fitting.

**Figure 3.** Exponential regression of oxygenation curves for one exemplary model run with 1:2 slope, $s = 6$ cm and $q = 0.11$ m$^2$/s: (a) Fitting to the experimental data for local conditions and resulting standardized oxygen transfer parameters (note: only every 2nd data point is shown for better legibility); (b) Residuals resulting from the curve fitting.

### 4. Results

#### 4.1. Specific Air-Water Interface

The specific air-water interface according to Equation (7) has been determined between the inception point of self-aeration and the quasi-uniform flow region for $0 \leq h \leq h_{75}$ in all configurations. The results in dimensional numbers are illustrated in Figure 4 for 1:2 slope and Figure 5 for 1:3 slope. It can be observed that the total amount strongly depends on the flow regime. While for smaller step heights and larger discharges a more stable skimming flow regime is found, a more tumbling flow is found for larger step heights and lower discharges. This leads to a more chaotic flow regime and stronger aeration. For all configurations, the specific air-water interface follows a distribution as shown in Figure 1b with a maximum slightly below $h_{75}$. The highest air-water interfaces being observed in the current tests are in the order of 350 m$^2$/m$^3$.

In order to determine the gas transfer velocity from the aeration efficiencies (shown subsequently), the total air-water interface from entrained air bubbles is required and can be calculated by
where \( L_i \) is the distance from the spillway crest to the inception point of self-aeration, \( L_{tot} \) is the total length of the spillway and \( b \) is the width of the flume. The resulting surface area \( S \) of the supposed sinusoidal air-water mixture surface according to Equation (A7) is added to \( A_{ent} \) from Equation (11) to determine the total available air-water \( A_{tot} \) interface. Description of the assumed surface wave geometry and derivation of relevant functions is given in the Appendix A. Results for both interfaces \( A_{ent} \) and \( S \), as well as the total air-water interface \( A_{tot} \) from all the tested configurations, are listed in Table 2.

| Table 2. Total air-water interface for all configurations. |
|-----------------|---|---|---|---|---|---|---|---|---|---|
| **Step Height [cm]** | 3 | 6 | 3 | 6 | 3 | 6 | 3 | 6 | 3 | 6 |
| **Discharge [m²/s]** | 0.07 | 0.09 | 0.11 | 0.07 | 0.09 | 0.11 | 0.07 | 0.09 | 0.11 | 0.07 | 0.09 | 0.11 |
| **Entrained air bubbles \( A_{ent} [m²] \) (Equation (11))** | 7.37 | 7.31 | 6.93 | 12.67 | 12.60 | 13.21 | 5.65 | 7.05 | 17.58 | 18.32 | 16.06 |
| **Air-water mixture surface \( S [m²] \) (Equation (A7))** | 1.95 | 1.58 | 1.34 | 3.24 | 2.29 | 2.58 | 2.55 | 2.62 | 2.06 | 10.67 | 7.88 | 5.71 |
| **\( A_{tot} [m²] \)** | 9.32 | 8.89 | 8.27 | 15.91 | 14.89 | 15.79 | 8.20 | 9.67 | 8.95 | 28.25 | 26.20 | 21.77 |

It is interesting to note that for less stable skimming flow regimes (i.e., for larger steps and lower discharges), the total air-water interface increases significantly with a relevant contribution of the surface waves. With consideration of this total surface \( A_{tot} \) in Table 2 and the total water volume, consisting of the air-water mixture volume

\[
V_{mix} = \int_{x=0}^{x=L_{tot}} \int_{y=0}^{y=b} h_{75} dx dy \quad (12)
\]

plus the fluid volume \( V \) beneath the surface waves for \( z > h_{75} \) according to Equation (A8) \( V_{tot} = V_{mix} + V \), the total specific air-water interface \( \alpha_{tot} \) is found as follows in Table 3:

| Table 3. Total specific air-water interface for all configurations. |
|-----------------|---|---|---|---|---|---|---|---|
| **Step Height [cm]** | 3 | 6 | 3 | 6 | 3 | 6 | 3 | 6 |
| **Discharge [m²/s]** | 0.07 | 0.09 | 0.11 | 0.07 | 0.09 | 0.11 | 0.07 | 0.09 | 0.11 |
| **\( \alpha_{tot} [m²/m³] \)** | 150 | 132 | 106 | 197 | 190 | 177 | 91 | 98 | 81 | 223 | 196 | 153 |

![Figure 4. Cont.](image-url)
Figure 4. Specific air-water interface $\sigma$ (in $m^2/m^3$) for all configurations with slope 1:2; downstream of the inception point of self-aeration and from $z = 0$ (pseudo-bottom) to $h_{75}$; data in quasi-uniform flow region assumed similar to the last two profiles: (a) $s = 3$ cm, $q = 0.07 m^2/s$; (b) $s = 3$ cm, $q = 0.09 m^2/s$; (c) $s = 3$ cm, $q = 0.11 m^2/s$; (d) $s = 6$ cm, $q = 0.07 m^2/s$; (e) $s = 6$ cm, $q = 0.09 m^2/s$; (f) $s = 6$ cm, $q = 0.11 m^2/s$. 
Figure 5. Specific air-water interface $a$ (in m$^2$/m$^3$) for all configurations with slope 1:3; downstream of the inception point of self-aeration and from $z = 0$ (pseudo-bottom) to $h_{75}$; data in quasi-uniform flow region assumed similar to the last two profiles: (a) $s = 3$ cm, $q = 0.07$ m$^2$/s; (b) $s = 3$ cm, $q = 0.09$ m$^2$/s; (c) $s = 3$ cm, $q = 0.11$ m$^2$/s; (d) $s = 6$ cm, $q = 0.07$ m$^2$/s; (e) $s = 6$ cm, $q = 0.09$ m$^2$/s; (f) $s = 6$ cm, $q = 0.11$ m$^2$/s.
4.2. Re-Aeration Performance

Relevant re-aeration performance parameters are obtained by nonlinear regression of the oxygenation data, as shown in Figure 3. As the experiments have been conducted on different days with different local conditions, the standardized re-aeration parameters are always presented in the following. For each configuration, four independent model runs were performed and the averaged results for \( k_{L,a20} \) and \( C_{S,20} \) are listed in Table 4.

Table 4. Standardized re-aeration performance parameters for all configurations (each averaged from four independent model runs, corresponding minimum and maximum values given in brackets).

<table>
<thead>
<tr>
<th>Step Height [cm]</th>
<th>Slope 1:2</th>
<th>Slope 1:3</th>
</tr>
</thead>
<tbody>
<tr>
<td>Discharge ([\text{m}^2/\text{s}])</td>
<td>0.07</td>
<td>0.09</td>
</tr>
<tr>
<td>( k_{L,a20} ) ([1/\text{h}])</td>
<td>(6.71)</td>
<td>(9.27)</td>
</tr>
<tr>
<td>( C_{S,20} ) ([\text{mg/L}])</td>
<td>(8.44)</td>
<td>(8.57)</td>
</tr>
</tbody>
</table>

4.3. Mass Transfer Velocity

The mass transfer coefficient can be directly deduced from Tables 3 and 4. The results are shown in Table 5.

Table 5. Deduced mass transfer velocities for all configurations.

<table>
<thead>
<tr>
<th>Step Height [cm]</th>
<th>Slope 1:2</th>
<th>Slope 1:3</th>
</tr>
</thead>
<tbody>
<tr>
<td>Discharge ([\text{m}^2/\text{s}])</td>
<td>0.07</td>
<td>0.09</td>
</tr>
<tr>
<td>( k_L ) ([\text{cm/h}])</td>
<td>4.6</td>
<td>7.1</td>
</tr>
</tbody>
</table>

Obviously, \( k_L \) increases for higher discharges and is thus a function of the Reynolds number. The flow regime (i.e., the stability of the skimming flow), which was affecting the specific air-water interface, is found to have no significant influence on \( k_L \). In addition, it is noticed that the air concentration strongly affects the mass transfer as well (as it was also described in [38] under consideration of an air-water mixture up to \( h_{90} \)). With the average flow velocity \( u \) being found in the aerated flow region and the average air concentration on the structure (up to \( h_{75} \)), the following relationship is found:

\[
\frac{k_L}{u} = 2.342 \times e^{-5.579 \times C} \times 10^{-5}.
\] (13)

The curve fitting reaches \( R^2 = 0.96 \) when neglecting a single obvious outlier, as indicated in Figure 6.
Weber number of approximately 100. However, bubble sizes must be considered to be strongly affected by scale effects due to the large model scale. Distinguishing between bubbles and waves may help in understanding proper scaling of the flow structure and more accurate extrapolations to prototype scale.

5. Discussion and Conclusions

The results underline that re-aeration on stepped spillways depends on the spillway geometry and discharge. While the amount of entrained air or specific air-water interface becomes higher for less stable skimming flow regimes, where the flow structure is more chaotic, the mass transfer is mainly a function of the mean flow velocity (being a measure of the Reynolds number and thus, the turbulence) and the mean air concentration. This result is supported by a similar finding published in [25]. Application of any mass transfer model with constant $k_L$ seems to be inaccurate for highly-aerated flows at hydraulic structures. It is shown that a proper consideration of the waved surface is essential, particularly for lower discharges. While the difference of specific air-water interface $A_{ent}$ between the present concept (mixture up to $h_{75}$) and the concept applied in [38] (mixture up to $h_{90}$ and neglecting surface waves) was only in the order of 5%, the increase of $A_{ent}$ is found to be up to 65% when considering surface waves. The assumption of a sinusoidal, symmetrical wave field needs to be further investigated and improved in future by considering a wide spectrum of wavelengths and refined geometries.

All tests were conducted on a large scale model meeting the scale recommendations given in different studies (e.g., [48]) with a minimum Reynolds number of approximately $10^5$ and a minimum Weber number of approximately 100. However, bubble sizes must be considered to be strongly affected by scale effects [49]. Thus, the presented results on air-water interface due to entrained air bubbles cannot be directly transferred to different scales. Scale effects in regard to surface waves have not been investigated yet and require further attention. It is not clear if the mass transfer equation presented, that is, Equation (13), is affected by scale effects as well. The flow velocity and air concentration, which were found to be related to mass transfer, may be considered to be unaffected by scale effects due to the large model scale. Distinguishing between bubbles and waves may help in understanding proper scaling of the flow structure and more accurate extrapolations to prototype scale.

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Conflicts of Interest: The authors declare no conflict of interest.
Appendix A. Assumption of a Three-Dimensional, Sinusoidal Surface Wave Field

In this study, a three-dimensional sinusoidal wave field is assumed with a geometry given by

\[ \eta(x, y) = \frac{H}{2} \left[ \frac{1}{2} \cos \left( \frac{2\pi x}{\lambda_x} \right) + \frac{1}{2} \cos \left( \frac{2\pi y}{\lambda_y} \right) \right], \] (A1)

being \( H \) the waveheight. \( \eta \) is periodic in \( x \) and \( y \) (see Figure A1). For simplicity, \( \eta \) is assumed to be homogeneous in both coordinates, thus leading to \( \lambda_x = \lambda_y = \lambda \).

![Free surface waved profile simplification.](image)

Figure A1. Free surface waved profile simplification.

Applying simple trigonometric considerations, the arc length in the plane \( x-z \) is given by:

\[ dr = \sqrt{(dx)^2 + (d\eta)^2} = dx \sqrt{1 + \left( \frac{d\eta}{dx} \right)^2}. \] (A2)

Likewise, for the \( y-z \) plane:

\[ ds = \sqrt{(dy)^2 + (d\eta)^2} = dy \sqrt{1 + \left( \frac{d\eta}{dy} \right)^2}. \] (A3)

Altogether, the surface area will be given by:

\[ S = \int dS = \int dr \; ds = \int dx \; dy \sqrt{1 + \left( \frac{d\eta}{dx} \right)^2} \sqrt{1 + \left( \frac{d\eta}{dy} \right)^2} = \int dx \; dy \sqrt{1 + \left( \frac{d\eta}{dx} \right)^2 + \left( \frac{d\eta}{dy} \right)^2}. \] (A4)

With the last term (cross product of the differences) being of higher order (herein neglected) and \( d\eta/dx \): \[ \frac{d\eta}{dx} = -H \frac{\pi}{\lambda} \sin \left( \frac{2\pi x}{\lambda} \right). \] (A5)

The integral providing the surface area (Equation (A4)) has no closed form, but it can be approximated as a sum of finite differences (\( \Delta S \)):

\[ \Delta S = \Delta x \Delta y \sqrt{1 + \left( \frac{d\eta}{dx} \right)^2 + \left( \frac{d\eta}{dy} \right)^2}. \] (A6)

It is noteworthy that wave surface derivatives are exact (Equation (A3)) and only finite differences are used for \( \Delta x \) and \( \Delta y \). Integrating over a periodic length-scale in both directions (\( x \in [-0.5\lambda, 0.5\lambda] \) and \( y \in [-0.5\lambda, 0.5\lambda] \)); result of Figure A2 can be obtained.
Being the approximation:
\[
\frac{S}{\lambda^2} \approx 39.68 \left( \frac{H}{2\lambda} \right)^2 + 1
\]  \hspace{1cm} (A7)

Note that \( H = 0 \) yields \( S = \lambda^2 \). Due to the symmetry in the vertical direction, the fluid volume \( V \) beneath this surface wave field is given by Equation (2)
\[
V = 0.5H\lambda^2.
\]  \hspace{1cm} (A8)

References


42. Chanson, H.; Toombes, L. Hydraulics of stepped chutes: The transition flow. J. Hydraulic Res. 2004 42, 43–54. [CrossRef]


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