

## Article

# Continuous Estimation of the Crack Growth Rate during Rotating-Bending Fatigue Testing

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**Abstract:** A method for estimating the crack growth rate in steel during rotating-bending fatigue testing is presented. Constant deflection tests were conducted in which the initial load remained constant prior to crack nucleation, when it decreased as the crack grew. In the proposed approach, steel samples were sharp-notched to produce a characteristic circular fracture upon loading and the final fracture area was correlated with a ratio of the load prior to fracture and the initial load. In this method, the deflection imposed is a function of a material's elastic modulus rather than its yield strength and the correlation obtained to estimate the average crack length as a function of the instantaneous load is independent of the applied stress or steel grade.

**Keywords:** fatigue test; fracture; crack growth; steel; rotating-bending

## 1. Introduction

The basic principle of rotating-bending fatigue (RBF) tests is that a bending moment is exerted on a specimen at a critical location as the specimen is rotated about its longitudinal axis, resulting in a single fully reversed stress cycle for each rotation ( $R = -1$ ). The fatigue life can be expressed as the number of cycles ( $n_f$ ) of a particular stress amplitude ( $S_a$ ) that the specimen can withstand before undergoing complete fracture [1–5]. Although RBF tests are mostly used for stress-based fatigue studies, they are generally not stress-controlled. Instead, a constant force amplitude is used in most tests: weights are suspended from the specimen to exert the bending moment. In many instruments designed for such testing, the bending moment is applied as a constant deflection amplitude [1,6–8]. In both cases, however, the stress conditions during the fatigue test differ from the initial stress conditions due to cyclic changes in the specimen such as softening, hardening, or cracking. An increase in the ultimate strength after cyclic loading prior to crack nucleation is reported in [9].

Thus, such cyclic changes are generally overlooked in RBF tests. Their use is typically limited to deriving  $S$ - $n$  curves for stress-based total-fatigue-life evaluations. This type of fatigue testing and the effects of the loading mode are described in further detail elsewhere [1–3]. The standardized rotating-bar bending-fatigue testing is described in the ISO 1143 [4] and JIS Z 2274 [5], which are procedures that generally yield the fatigue life ( $n_f$  for a particular  $S_a$ ), and can be used to construct  $S$ - $n$  diagrams. However, information about the crack behavior, which is another important parameter for determining the fracture toughness of a material, cannot be obtained using these methods.

The ASTM E647 Standard Test Method for Measurement of Fatigue Crack Growth Rates is commonly used to determine the crack growth rate by measuring the crack using optical techniques or

equivalent methods [10]. However, continuous crack measurements during a rotating-bending test are not feasible due to the testing configuration. Usually, a test must be interrupted several times to take measurements, whether to extract replicas of the surface from which the arc length of the initial crack can be measured [11–13], or to record the angle of the crack on the surface with an angular micrometer [14], or estimate the crack depth based on the potential drop technique [15–18]. Potential drop technique has been also used for real-time measurements in axial loading configuration [19] or pure bending [20]. Additionally, the surface crack length has been monitored by optical measurement, while the shape, size, and depth of the crack were inferred by an area-compliance method [21]. In other fatigue configurations it is possible to use digital image correlation (DIC) to analyze the crack growth rates and crack tip displacement fields, taking images every certain number of cycles during the test [22,23]. Similar accuracy between DIC and potential drop methods has been reported in [24]. Also, fatigue crack growth has been described in diagrams of kinetic fatigue failure developed with the parameter of the energy dissipation of deformation [25] and has shown good results in components from old structures [26].

In this study, an approach to further extend RBF tests to be applicable for estimating the crack growth rate ( $da/dn$ ) by continuous force sensing under constant displacement conditions is presented.

## 2. Materials and Methods

### 2.1. Calculations

During constant displacement testing, the load required to maintain a constant deflection decreases progressively from the onset of fatigue cracking until overload fracture. This load alteration is proportional to crack growth as follows.

The deflection in a specific geometry depends on the elastic modulus of the material being tested and the load [27]. The equation for the deflection at the free end of a cantilever beam is as follows:

$$\delta = PL^3/3EI, \quad (1)$$

where  $P$  is the applied load,  $L$  is the distance between the load and the pivot point that produces the bending moment,  $E$  is the elastic modulus, and  $I$  is the moment of inertia based on the specimen's geometry. The value of  $I$  for a round shaft is calculated as a function of its diameter as follows:

$$I = \pi d^4/64, \quad (2)$$

where  $d$  is the diameter of the shaft. Thus, the load required to produce a certain deflection for a specific geometry depends on the material's elastic modulus. Moreover, the proposed method is independent of test stress or steel strength.

The nominal stress for a bending shaft can be calculated as follows:

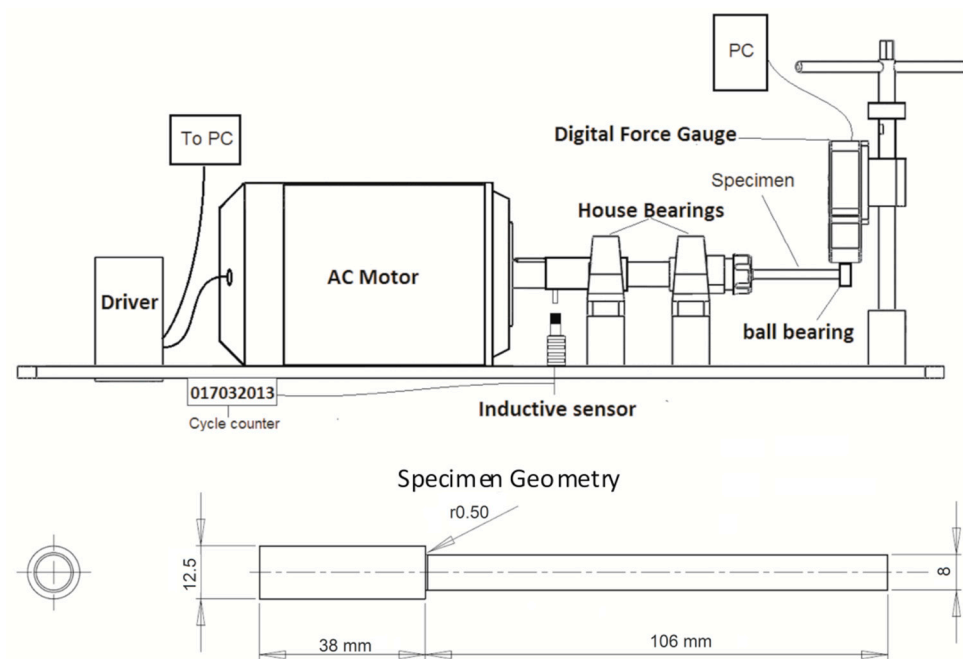
$$\sigma_{nom} = M(y)/I = 32M/\pi d^3, \quad (3)$$

where  $M(y)$  is the bending moment at the surface of the specimen.

### 2.2. Test Conditions

The cantilever RBF testing machine and specimen geometry used in this study are shown schematically in Figure 1. The custom-built machine was a constant-displacement-type instrument in which the free end of the rotating specimen is deflected using the shaft of a digital force gauge.

The gauge is lowered by turning the hand wheel to cause a fixed deflection of the specimen. The load was continuously recorded at a sampling rate of 10 Hz and the rotation of specimen was 20 Hz. An inductive sensor and a digital cycle counter were employed to count the number of fatigue cycles over the course of the test.



**Figure 1.** Schematic view of the cantilever rotating-bending fatigue (RBF) testing machine and specimen with a sharp notch.

#### Specimen Geometry and Heat Treatment

Each specimen was 144 mm long; 38 mm of this length was used for clamping with a collet chuck (type ER-25). The bending arm length from the point of loading to the shoulder fillet was 106 mm. Specimens were designed with a sharp notch in order to promote cracking from the entire circumference towards the center and form a circular pattern; the notch root radius was 0.5 mm, corresponding to a theoretical stress concentration factor ( $K_t$ ) for this geometry of 2.2 according to [28].

The chemical composition of the AISI/SAE 5160 steel used is shown in Table 1. The samples were austenitized at 900 °C followed by water quenching and tempering at different temperatures—250 °C, 350 °C, 450 °C, and 550 °C—to attain different strength levels. The fatigue life at different tempering temperatures is shown in Table 2.

**Table 1.** Chemical composition of AISI 5160 steel.

Element	wt %
C	0.580
Si	0.260
Mn	0.885
P	0.017
S	0.016
Cr	0.780
Ni	0.010
Sr	0.013
Ti	0.003
Fe	Balance

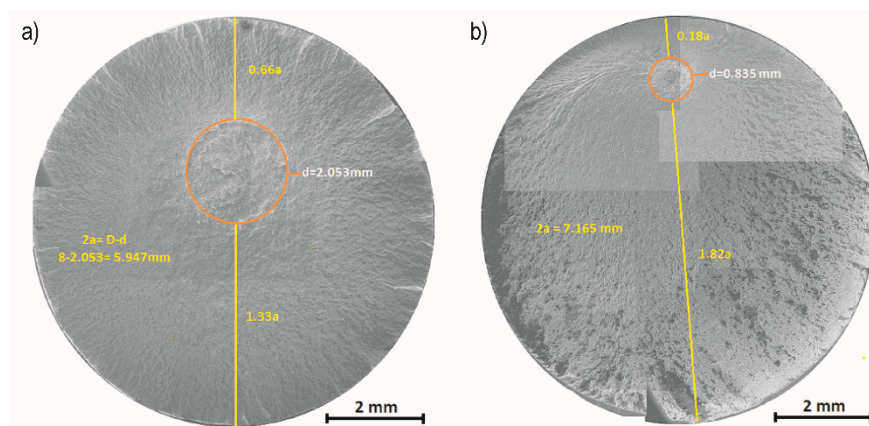
**Table 2.** Fatigue life measured at various stress levels with different tempering temperatures.

Stress Amplitude (MPa)	Load (N)	Deflection (mm)	Cycles to Failure at Different Tempering Temperatures			
			250 °C	350 °C	450 °C	550 °C
900	205	1.862	$3.04 \times 10^4$	$2.21 \times 10^4$	$3.10 \times 10^4$	$2.75 \times 10^4$
750	171	1.546	-	-	-	$1.30 \times 10^5$
600	141	1.278	$1.12 \times 10^5$	$2.16 \times 10^5$	$2.41 \times 10^5$	$4.19 \times 10^5$
580	133	1.202	-	-	-	runout
500	116	1.057	-	$5.13 \times 10^5$	$1.16 \times 10^6$	-
490	112	1.012	-	-	runout	-
475	109	0.985	-	runout	-	-
400	92	0.838	$4.69 \times 10^5$	runout	-	-
325	75	0.678	$8.75 \times 10^5$	-	-	-
290	67	0.606	runout	-	-	-

### 2.3. Crack Measurements and FEM

A series of crack measurements was performed physically on the fractured test specimens and the ratio between the initial load and the final load prior to fracture ( $P_f/P_0$ ) was correlated with the crack length. This approach provided information about the crack length at the point of fracture.

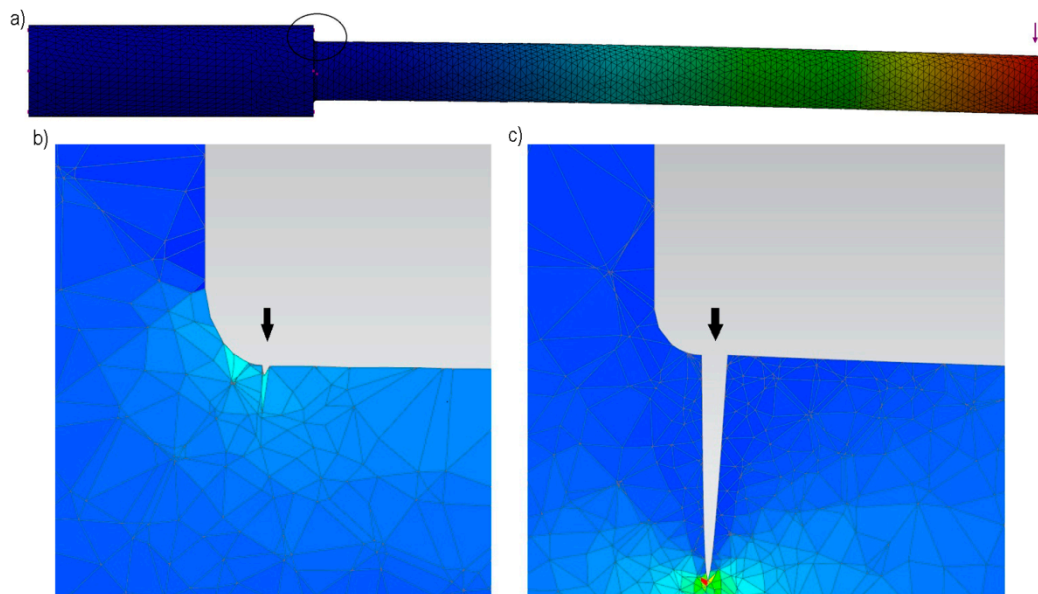
Figure 2 shows scanning electron microscope (SEM) images of typical samples. The orange circles delineate the well-defined fracture areas occurring around the sharp notch and the yellow lines indicate the path along which the two radial cracks would have extended from the notch root surface towards the final fracture area, passing through the center of the sample and the center of the fracture. In this way, the critical length of the radial cracks,  $2a$ , was estimated by subtracting the diameter of the final fracture area ( $D_{ff}$ ) from the initial specimen diameter ( $D_0$ ). Thus, the fracture area is related to the average extension of the fatigue crack at fracture. The diameter ratios ( $D_{ff}/D_0$  and  $D_n/D_0$ ) were obtained for each fracture analyzed.



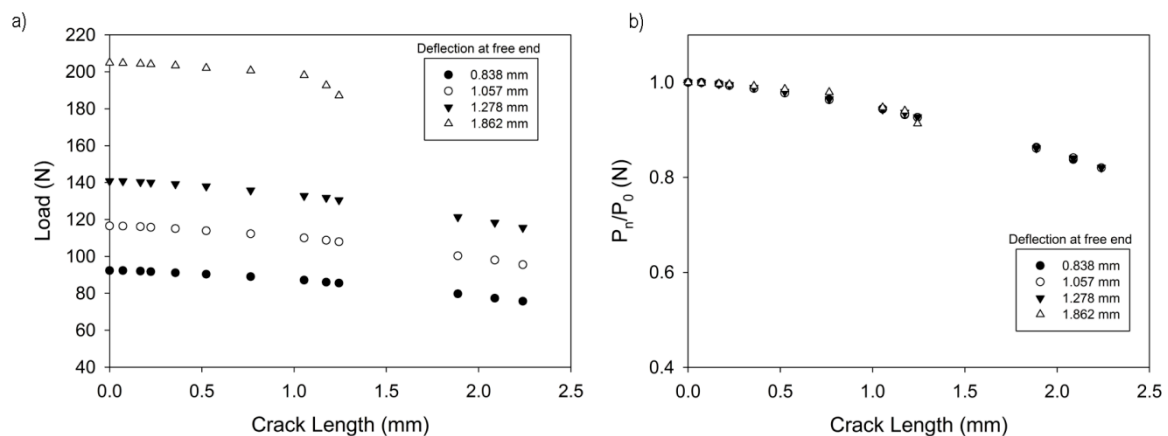
**Figure 2.** SEM micrographs of fractures depicting the final overloaded fracture area (circle) and the paths of two ideal cracks (lines). (a) High nominal stress; (b) low nominal stress.

Finite element modeling (FEM) in ANSYS Workbench was used to obtain the load ratio,  $P_n/P_0$ , for several  $D_n/D_0$  ratios ranging from 0.3 to 1. The analysis was made with 67,795 nodes and 37,664 elements. The geometry was constrained from the grip area and a force was imposed at the free end. A static analysis was carried out in the original geometry to assess the deflection at the free end produced by different loads using an elastic modulus of 207 GPa. A crack at the notch root was introduced in a 2D model along the diameter with the crack tip towards the center (see Figure 3). The analysis was repeated using different crack sizes (depths) and constraining the part to the original deflection obtained for the uncracked specimen for each load without the crack. The reduced loads

required to produce the same deflections for different crack sizes were determined. The ratios  $P_n/P_0$ , for different  $D_n/D_0$  are displayed in Figure 4.



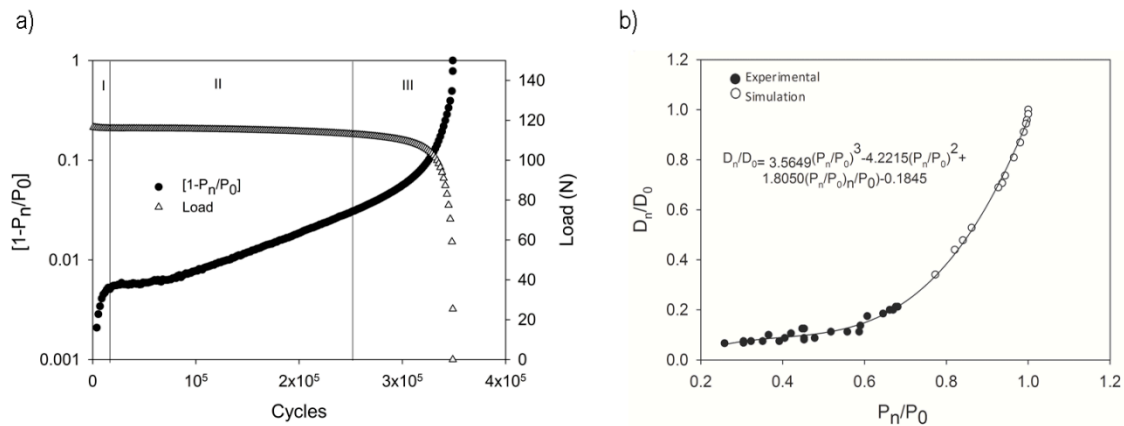
**Figure 3.** (a) Geometry used in FEM, showing the location of applied load; (b) small crack at notch root; (c) larger crack at the notch root.



**Figure 4.** (a) Required load to produce certain deflection for different crack lengths determined by FEM; (b) load ratio  $P_n/P_0$  at certain deflection.

### 3. Results and Discussion

A typical example of the raw data for the force versus the number of cycles during constant-displacement RBF testing is presented in Figure 5a. The load signal was smoothed using a cubic spline algorithm to reduce the noise produced by the variations in the crack length depending on the angular position, in order to obtain an average load. The applied load at the beginning of each test remained constant then decreased and continued to decrease until the specimen failed at a certain load.



**Figure 5.** Load and percentage of load change during testing for a deflection of 1.057 mm (a) and the correlation between the load ratio and diameter ratio (cracking) (b).

The load is a direct indicator of the crack nucleation and growth. The load change, which is expressed as a percent ( $[1 - P_n/P_0] \times 100$ ), is plotted as a function of the cycles to identify the different stages of fatigue. In this example, the initial load remained almost constant until the nucleation of the first microscopic fatigue cracks around the notch-root surface of the specimen (stage I), as indicated by a slight decrease in the percent load change after about 18,000 cycles. After this point, the cracks grew at a relatively slow rate towards the specimen core (stage II); this reduces the net cross section of the specimen, thus reducing the amount of force required to sustain the same deflection. Between cycles 25,000 and 32,000 (approximately), the force reduction was more significant. During this stage, the net cross section of the specimen was reduced so much that significant force relaxation occurred and the overloaded material could no longer withstand the applied stress; this promotes a faster (unstable) cracking (stage III) than in the previous stage. Finally, as the fatigue cracks reached a critical size, after about 35,000 fatigue cycles, the specimen fractured completely.

In both the experimental tests and the simulation, the  $P_n/P_0$  ratio was equal to 1 at the beginning of the test; at this time, no cracks were present, so  $D_n/D_0 = 1$ . Figure 5b shows a third-degree correlation fit between the load ratio and diameter ratio, as measured experimentally and predicted by FEM.

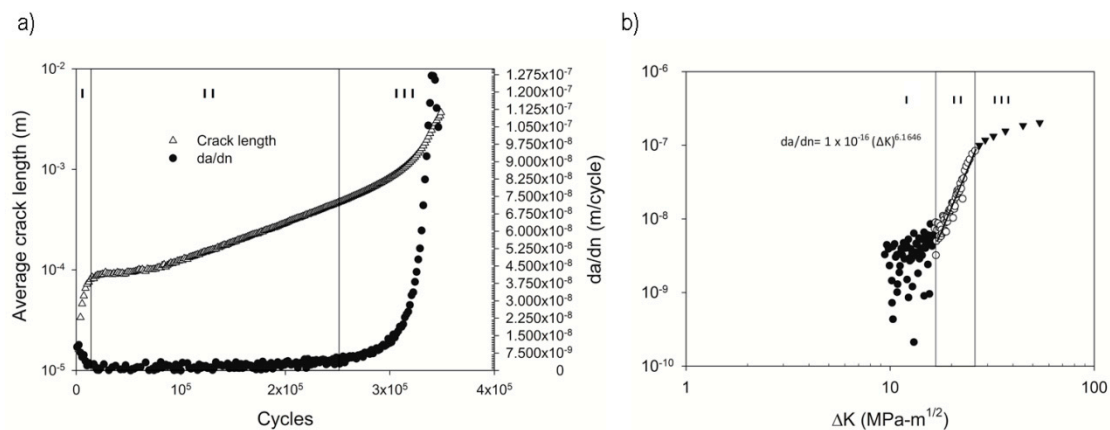
The obtained correlation was then applied to the measured load data to estimate the crack growth during each test. In Figure 6a, the average crack length (m) and the value of  $da/dn$  (m/cycle) are shown as functions of the cycle number. It can be seen in Figure 5a that the load ratio is an indicator of the crack growth. In addition, the range of the stress intensity factor was calculated as  $\Delta K = \Delta S \sqrt{\pi a Y}$  using only the  $S_{max}$  value as described in [15,29], where  $Y$  is a configuration factor used for a circumferential crack in opening mode I for a long rod, which is calculated as follows:

$$Y = \frac{1.12 + \alpha(1.3\alpha - 0.88)}{1 - 0.92\alpha}, \quad (4)$$

where  $\alpha$  is the normalized crack size, defined as the length of a single crack length over the radius.

The material constants  $C$  and the exponent  $m$  in the expression  $da/dn = C(\Delta K)^m$ , can be obtained from the log-log plot of  $\Delta K$  versus  $da/dn$ , see Figure 6b.





**Figure 6.** Average crack length and crack growth rate behavior as a function of the cycle number (a) and the stress intensity factor range against  $da/dn$  (b).

#### 4. Conclusions

In this paper, a methodology for continuous crack growth estimation based on the rotating bending in a constant deflection system was presented. The measurements were based on the characteristic circle fractures observed in sharp-notched specimens. The fractures were correlated with the fracture load to obtain a relationship between the instantaneous load that must be applied to maintain a constant displacement and the average crack size. A third-degree model was obtained for this testing configuration for steel samples tempered to different strength levels. This correlation is dependent on a material's elastic modulus rather than its yield strength. By this methodology, the stages of fatigue could be clearly distinguished and analyzed and it is possible to obtain the stress intensity factor and the material constants under these conditions.

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**Conflicts of Interest:** The authors declare no conflict of interest.

#### List of Symbols

- $\alpha$  = Normalized crack size
- $a$  = Average crack length
- $C$  = Material constant
- $D$  = Diameter of the gripped or loaded end of the specimen
- $D_0$  = Nominal diameter of the specimen at which the stress is the greatest
- $D_{ff}$  = Diameter of the final fracture
- $D_n$  = Diameter of the remaining cross section after  $n$  cycles
- $\Delta K$  = Stress intensity factor range
- $\Delta S$  = Stress range
- $d$  = Diameter of the shaft
- $da/dn$  = Average crack growth rate
- $\delta$  = Deflection
- $E$  = Elastic modulus
- FEM = Finite element modeling
- $I$  = Moment of inertia
- $K$  = Stress intensity factor
- $K_t$  = Stress concentrator

$L$  = Distance between the load and the pivot point

$M$  = Moment

$M(y)$  = Bending moment at the surface of the specimen

$m$  = Material Constant

$n$  = Number of cycles

$n_f$  = Number of cycles to failure

$P$  = Load (Force)

$P_0$  = Initial load

$P_n$  = Load at  $n$  cycles

$P_{OL}$  = Load prior to overloading

$P_f$  = Fracture load

$r$  = Radius at the end of the test section

$R$  = Ratio for cyclic loading (min/max)

RBF = Rotating bending fatigue

$S$  = Stress

$S_a$  = Stress amplitude

SEM = Scanning electron microscope

$\sigma_{nom}$  = Nominal stress

$Y$  = Configuration factor

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