



Article Velocity and Singularity Analysis of a 5-DOF (3T2R) Parallel-Serial (Hybrid) Manipulator

Pavel Laryushkin ¹, Anton Antonov ², Alexey Fomin ², *⁰ and Terence Essomba ³

- ¹ Department of Fundamentals of Machine Design, Bauman Moscow State Technical University (BMSTU), Moscow 105005, Russia; pav.and.lar@gmail.com
- ² Mechanisms Theory and Machines Structure Laboratory, Mechanical Engineering Research Institute of the Russian Academy of Sciences (IMASH RAN), Moscow 101000, Russia; antonov.av@imash.ru
- ³ Department of Mechanical Engineering, National Central University, 300, Taoyuan 32001, Taiwan; tessomba@cc.ncu.edu.tw
- * Correspondence: alexey-nvkz@mail.ru

Abstract: This article presents the velocity and singularity analysis for a five-degree-of-freedom (5-DOF) parallel-serial manipulator. The hybrid structure of the manipulator combines a tripod-like parallel part and a serial part, represented as two carriages moving in perpendicular directions. This manipulator provides its end-effector with a 3T2R motion pattern, which includes three independent translations and two independent rotations. First, the study briefly discusses the manipulator design and the results of the position analysis. These results form the basis for the subsequent velocity and singularity analysis, performed by screw theory. The screw coordinates of the unit twists are written for each manipulator joint, and then through the reciprocal screw approach, the actuation and constraint wrenches of the manipulator are obtained by simple inspection. Based on these twists and wrenches, the paper forms the velocity equation and shows an example of the inverse velocity analysis for a given end-effector trajectory. The same example is solved by numerical differentiation to verify the proposed approach. Next, the paper investigates singular configurations by analyzing the wrench system of the manipulator and presents several conditions for serial and parallel singularities. Each condition has both a symbolic representation, given by an equation for screw coordinates of certain wrenches, and a visual representation, which shows the manipulator in a singular configuration.

Keywords: 5-DOF manipulator; parallel-serial (hybrid) manipulator; three translations and two rotations motion pattern (3T2R); inverse kinematics; velocity analysis; singularity analysis; MATLAB simulation

1. Introduction

Hybrid mechanisms and manipulators represent mechanical systems composed by stacking kinematic chains with a parallel or serial structure [1]. These systems have advantages of both types of kinematic chains, such as an extended workspace [2] and the possibility to bypass or avoid singular configurations [3].

The current research considers hybrid mechanisms in which the serial kinematic chain is stacked on the parallel one. The examples of such systems are five-degrees-of-freedom (5-DOF) CaHyMan, composed of 3-DOF parallel and 2-DOF serial chains [4]; a 5-DOF machine tool, which combines a 2-DOF parallel part and a 3-DOF serial part [5]; a 5-DOF polishing machine with a 3T2R motion pattern, which includes a 3-DOF parallel mechanism for vertical motion and XY rotations and a 2-DOF serial mechanism for XY positioning [6].

Many researchers have studied the kinematics of the hybrid robots, including their position and velocity analysis. For example, these issues were considered by Tanev [7] for two serially connected tripod mechanisms; Moosavian et al. [8] for a planar robot with a serial PUMA-type robotic arm attached to its output link; Zheng et al. [9] for two tripods, designed such that the smaller one is inside the larger one; Nazari et al. [10] for a tripod with cylindrical and revolute joints and a serial robotic arm mounted on the tripod moving



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Copyright: © 2022 by the authors. Licensee MDPI, Basel, Switzerland. This article is an open access article distributed under the terms and conditions of the Creative Commons Attribution (CC BY) license (https:// creativecommons.org/licenses/by/ 4.0/). plate; Kucuk and Gungor [11] for a Stewart platform placed on a SCARA-type robot; Zhang et al. [12] for a 5-DOF Exection robot, which consists of a tripod and a 2-DOF serial wrist mechanism; Xu et al. [13] for a polishing machine based on a translational Delta-like robot with linear drives and a separate serial module with three rotational DOFs; Gim et al. [14] for a humanoid robotic leg composed of two planar closed loops, each followed by a link with passive joints and connected to each other and the moving plate through an actuated revolute joint; Milutinović et al. [15] for a Tricept robot based on a tripod with translational DOFs equipped with a 2-DOF wrist; My and Hoan [16] for a serial robotic arm with two intermediate parallel loops.

The singularity analysis of hybrid robots has also been considered by several researchers. Nasseri et al. [17] studied singularities of a 6-DOF micromanipulator for intraocular surgery based on two serially connected 2-DOF planar mechanisms followed by a tool gripper with one translational and one rotational DOF. Simas and Di Gregorio [18] analyzed workspace and singularities of a 4-DOF single-loop robot with an additional "rotation amplifier". Rakhodaei et al. [19] examined singularity-free path planning of a robot consisting of a serially connected hexapod and tripod.

During the singularity analysis of hybrid robots, one can often consider its parallel part only, because, in general, parallel singularities have more negative effects on the mechanism performance than the serial ones. For example, Amine et al. [20] carried out a singularity analysis of the Exection robot ignoring its wrist part. Ma et al. [21] used another approach and investigated singularities of a robotic leg of a quadruped walking manipulator by substituting the original hybrid mechanism with a tripod-like parallel equivalent.

The current article aims at velocity and singularity analysis of a recently proposed 5-DOF hybrid manipulator [22], which includes parallel and serial parts and provides its output link with a 3T2R motion pattern. Manipulator design and its workspace, thoroughly discussed in [22], allow it to find applications in processing machine components with complex shapes, as well as mechanical elements, which have a longitudinal dimension greater than the transverse one.

The paper has the following structure. Section 2 describes the manipulator design and explains its mobility. Section 3 briefly discusses the position analysis—a preliminary step before considering the instantaneous kinematics. Section 4 continues this study and presents velocity analysis using screw theory. This section also provides a numerical example of solving the inverse velocity problem for a specified end-effector trajectory. Section 5 analyzes singular configurations of the manipulator and visualizes several examples. Sections 6 and 7 discuss and summarize the results and mention directions for future research.

2. Manipulator Design

Let us consider the manipulator design. Figure 1 shows its kinematic scheme (Figure 1a) and CAD model (Figure 1b). The manipulator consists of a tripod-like parallel part and a serial part, represented by two carriages moving orthogonally. Figure 1a has the following notations: 1—fixed link (base); 2 and 4—driving links; 3—spider of the universal (Cardan) joint; 5—rod; 6—platform (output link of the parallel part); 7 and 8—rigidly connected links that form a carriage moving along the longitudinal dimension of platform 6; 9—carriage that moves along the transversal dimension of platform 6; 10—end-effector (output link of the serial part and the entire manipulator) mounted rigidly on carriage 9.

Each kinematic chain (leg) of the parallel part has a linear drive that provides displacement q_1 , q_2 , or q_3 . The chain with the universal joint restricts the platform rotation about the axis perpendicular to the spider plane and two translations in any direction orthogonal to the axis of the chain linear drive. Two other chains with spherical joints do not constrain the platform motion. Thus, the parallel part provides platform 6 with one translational and two rotational DOFs. In the serial part, linear drives displace links 7–8 and 9 for q_4 and q_5 , respectively—the serial part provides end-effector 10 with two translational DOFs in a platform plane relative to platform 6.



Figure 1. 5-DOF (3T2R) parallel-serial (hybrid) manipulator: (a) structural scheme; (b) CAD model.

In summary, we can specify five independent motions of end-effector 10: three translations and two rotations (3T2R motion pattern), where the parallel part provides one translation and two rotations (1T2R motion pattern) and the serial part adds two translations (2T motion pattern). One should remember that the 3T2R motion pattern can degenerate in singular configurations, which is studied later in this paper.

3. Position Analysis

Though the paper aims at the velocity and singularity study, the position analysis is a necessary preliminary step that provides relations between the coordinates of the links. We addressed this topic in detail in [22], and in the current section, we omit the math and briefly mention the key results essential for the subsequent analysis.

Let us first introduce the following notations (Figure 2):

- OXYZ is a stationary reference frame attached to the base arbitrarily.
- $SX_SY_SZ_S$ is a reference frame attached to the end-effector such that axis Z_S is directed along the tool (unit vector $\hat{\mathbf{n}}$) and the remaining axes (X_S and Y_S) have an arbitrary direction; vector \mathbf{p}_S and rotation matrix \mathbf{R}_S define the position and orientation of $SX_SY_SZ_S$ relative to OXYZ.
- $\mathbf{q} = \begin{bmatrix} q_1 & \dots & q_5 \end{bmatrix}^T$ are the actuated coordinates according to the previous section; all these coordinates are measured about the axes defined by unit vectors $\hat{\mathbf{s}}_1 \dots \hat{\mathbf{s}}_5$ (in *OXYZ*).
- α and β are the angles in the universal joint measured about its axes defined by unit vectors û₁ and û₂ (in OXYZ).

Parameters \mathbf{p}_S and $\hat{\mathbf{n}}$ (or \mathbf{R}_S) describe the end-effector configuration and can define the posture of the entire 5-DOF system: we can express coordinates of all the links as functions of these parameters (solve the inverse kinematics problem). We can also use other sets of parameters to define the manipulator configuration, for example, taking \mathbf{q} and solving the direct kinematics problem. The choice of \mathbf{p}_S and $\hat{\mathbf{n}}$ is natural and does not lead to cumbersome calculations [22]. Thus, we assume we are given \mathbf{p}_S and $\hat{\mathbf{n}}$, so we can solve the inverse kinematics by the following steps (see [22] for details):

1. As the end-effector connects with the platform by two prismatic joints, vector $\hat{\mathbf{n}}$ uniquely defines the orientation of the latter. The platform orientation, on the other hand, depends only on two angles α and β in the universal joint. This condition allows us to express $\hat{\mathbf{n}}$ as a function of α and β and find these angles from the corresponding equations.

- 2. Having found α and β , we can write a vector loop equation for \mathbf{p}_S . This vector depends only on parameters q_1 , q_4 , and q_5 , which we find from the obtained equation.
- 3. Having found q_1 , we know the platform configuration relative to OXYZ. Hence, we know the coordinates of platform spherical joints A_2 and A_3 relative to the same frame. We can also write coordinates of spherical joints C_2 and C_3 as functions of q_2 and q_3 . Coordinates of the spherical joints in each kinematic chain are connected by a known and constant distance between the joints. This allows us to form corresponding equations and find parameters q_2 and q_3 .



Figure 2. Kinematic analysis of the considered manipulator.

Thus, given the end-effector configuration (\mathbf{p}_S and $\hat{\mathbf{n}}$), we determined all the parameters necessary to describe coordinates of any manipulator link. In subsequent velocity and singularity analysis, we assume we know all the coordinates involved.

4. Velocity Analysis

Velocity analysis plays a crucial role in manipulator design and control. The major aim of this analysis is to find relations between the velocities of the links, particularly between the end-effector velocity and the drive speeds. There are two prevailing approaches to solve the problem. The first one consists in differentiating constraint equations, obtained during position analysis, with respect to time [23] (p. 153). Though the approach is straightforward, it can lead to cumbersome relations when analyzing complex mechanical systems. An alternative method applies screw theory and considers twists associated with manipulator joints [24]. This technique is more elegant, and it also reveals the physical meaning of the relations obtained, which is important for singularity analysis.

In this paper, we use the second approach mentioned above. Studies [25–27] provide comprehensive information about the screw theory, and works [28–31] show its application for velocity analysis. We encourage a reader novel to the subject of screw theory to have a look at these studies.

4.1. Theory

To find a relation between end-effector twist **V** and drive speeds $\dot{\mathbf{q}} = \begin{bmatrix} \dot{q}_1 & \cdots & \dot{q}_5 \end{bmatrix}^1$, we should consider (unit) twists of the manipulator joints. Figure 3 shows all these twists: blue arrows designate zero-pitch twists, which correspond to rotations, and red arrows designate infinite-pitch twists, which correspond to translations.



Figure 3. Joint (unit) twists of the manipulator. Blue arrows designate zero-pitch twists; red arrows designate infinite-pitch twists.

End-effector twist $\mathbf{V} = \begin{bmatrix} \boldsymbol{\omega}^T & \boldsymbol{\upsilon}_S^T \end{bmatrix}^T$ consists of point *S* velocity vector $\boldsymbol{\upsilon}_S$ and angular velocity vector $\boldsymbol{\omega}$; both vectors are written relative to the *OXYZ* frame. Therefore, it is natural to select point *S* as a reference point and write screw coordinates of the joint twists relative to a reference frame, centered at point *S* with axes parallel to the axes of *OXYZ*.

First, let us find screw coordinates of infinite-pitch twists $\xi_{q1} \dots \xi_{q5}$ (Figure 3), which correspond to the actuated joints. The axis of each twist $\xi_{q1} \dots \xi_{q5}$ is collinear with unit vector $\hat{s}_1 \dots \hat{s}_5$, so we can write:

$$\boldsymbol{\xi}_{q1} = \begin{bmatrix} \mathbf{0} \\ \hat{\mathbf{s}}_1 \end{bmatrix}, \dots, \boldsymbol{\xi}_{q5} = \begin{bmatrix} \mathbf{0} \\ \hat{\mathbf{s}}_5 \end{bmatrix}.$$
(1)

Note that vectors $\hat{\mathbf{s}}_1$, $\hat{\mathbf{s}}_2$, and $\hat{\mathbf{s}}_3$ are defined by the manipulator design and have known and constant components; vectors $\hat{\mathbf{s}}_4$ and $\hat{\mathbf{s}}_5$ depend on the manipulator configuration and can be obtained through the preceding position analysis.

Zero-pitch twists are ξ_{C1u1} and ξ_{C1u2} , which correspond to rotations in the universal joint (Figure 3); ξ_{Akx} , ξ_{Aky} , ξ_{Akz} , and ξ_{Ckx} , ξ_{Cky} , ξ_{Ckz} , k = 2, 3, which correspond to rotations in the spherical joints. The axes of the first two twists are collinear with the axes of the universal joint, and these twists have the following coordinates:

$$\boldsymbol{\xi}_{C1u1} = \begin{bmatrix} \hat{\mathbf{u}}_1 \\ \boldsymbol{\rho}_{SC1} \times \hat{\mathbf{u}}_1 \end{bmatrix}, \boldsymbol{\xi}_{C1u2} = \begin{bmatrix} \hat{\mathbf{u}}_2 \\ \boldsymbol{\rho}_{SC1} \times \hat{\mathbf{u}}_2 \end{bmatrix}, \quad (2)$$

where vector $\mathbf{\rho}_{SC1}$ defines position of point C_1 relative to point S; we know this vector, as well as $\hat{\mathbf{u}}_1$ and $\hat{\mathbf{u}}_2$, from the position analysis. Moreover, vector $\hat{\mathbf{u}}_1$ has constant components in the considered reference frame.

Axes of the twists corresponding to the spherical joints can be directed arbitrarily. Therefore, for any manipulator configuration, we can assume these axes are always parallel to the coordinate axes. We obtain:

$$\begin{aligned} \boldsymbol{\xi}_{Akx} &= \begin{bmatrix} \begin{bmatrix} 1 & 0 & 0 \end{bmatrix}^{\mathrm{T}} \\ \boldsymbol{\rho}_{SAk} \times \begin{bmatrix} 1 & 0 & 0 \end{bmatrix}^{\mathrm{T}} \end{bmatrix}, \boldsymbol{\xi}_{Ckx} = \begin{bmatrix} \begin{bmatrix} 1 & 0 & 0 \end{bmatrix}^{\mathrm{T}} \\ \boldsymbol{\rho}_{SCk} \times \begin{bmatrix} 1 & 0 & 0 \end{bmatrix}^{\mathrm{T}} \end{bmatrix}, \\ \boldsymbol{\xi}_{Aky} &= \begin{bmatrix} \begin{bmatrix} 0 & 1 & 0 \end{bmatrix}^{\mathrm{T}} \\ \boldsymbol{\rho}_{SAk} \times \begin{bmatrix} 0 & 1 & 0 \end{bmatrix}^{\mathrm{T}} \end{bmatrix}, \\ \boldsymbol{\xi}_{Cky} &= \begin{bmatrix} \begin{bmatrix} 0 & 1 & 0 \end{bmatrix}^{\mathrm{T}} \\ \boldsymbol{\rho}_{SCk} \times \begin{bmatrix} 0 & 1 & 0 \end{bmatrix}^{\mathrm{T}} \end{bmatrix}, \\ \boldsymbol{\xi}_{Akz} &= \begin{bmatrix} \begin{bmatrix} 0 & 0 & 1 \end{bmatrix}^{\mathrm{T}} \\ \boldsymbol{\rho}_{SAk} \times \begin{bmatrix} 0 & 0 & 1 \end{bmatrix}^{\mathrm{T}} \end{bmatrix}, \\ \boldsymbol{\xi}_{Ckz} &= \begin{bmatrix} \begin{bmatrix} 0 & 0 & 1 \end{bmatrix}^{\mathrm{T}} \\ \boldsymbol{\rho}_{SCk} \times \begin{bmatrix} 0 & 0 & 1 \end{bmatrix}^{\mathrm{T}} \end{bmatrix}, \end{aligned}$$
(3)

where vectors ρ_{SAk} and ρ_{SCk} define the position of points A_k and C_k , k = 2, 3, relative to point *S*; we know these vectors from the position analysis.

Next, consider three kinematic chains between the end-effector and the base formed by three legs of the parallel manipulator and the platform serial part, which is identical for each chain. We can group the twists of each chain in matrix T_i , i = 1...3:

$$\mathbf{T}_{1} = \begin{bmatrix} \xi_{q1} & \xi_{C1u1} & \xi_{C1u2} & \xi_{q4} & \xi_{q5} \end{bmatrix},$$

$$\mathbf{T}_{2} = \begin{bmatrix} \xi_{q2} & \xi_{C2x} & \xi_{C2y} & \xi_{C2z} & \xi_{A2x} & \xi_{A2y} & \xi_{A2z} & \xi_{q4} & \xi_{q5} \end{bmatrix},$$

$$\mathbf{T}_{3} = \begin{bmatrix} \xi_{q3} & \xi_{C3x} & \xi_{C3y} & \xi_{C3z} & \xi_{A3x} & \xi_{A3y} & \xi_{A3z} & \xi_{q4} & \xi_{q5} \end{bmatrix}.$$
(4)

We can now write the following velocity equation for each **T**_{*i*}:

$$\mathbf{T}_{i} \begin{bmatrix} q_{i} \\ \dot{\Theta}_{i} \\ \dot{q}_{4} \\ \dot{q}_{5} \end{bmatrix} = \mathbf{V}, \tag{5}$$

where vector θ_i includes speeds in all passive (unactuated) joints in the *i*-th leg.

Usually, we are not interested in vector θ_i . To exclude this vector from the velocity Equation (5), we apply an algorithm based on the reciprocal screw approach [24]. Consider leg 1 first and let ζ_{1i} be a wrench reciprocal to all twists of \mathbf{T}_1 except for ξ_{qi} :

$$\xi \circ \zeta_{1j} = 0, \xi \in \mathbf{T}_1 \mid \xi \neq \xi_{qj}, \ j = 1, 4, 5,$$
 (6)

where " \circ " is a reciprocal product of two screws [25] (p. 24).

By taking the reciprocal product of both sides of relation (5) with ζ_{1i} , we obtain:

$$(\boldsymbol{\xi}_{qj} \circ \boldsymbol{\zeta}_{1j}) \dot{\boldsymbol{q}}_{j} = \mathbf{V} \circ \boldsymbol{\zeta}_{1j}.$$
 (7)

Equation (7) includes unknown wrench ζ_{1j} . To find screw coordinates of this wrench, let us first introduce a new wrench ζ_c reciprocal to all twists of **T**₁:

$$\boldsymbol{\xi} \circ \boldsymbol{\zeta}_c = \boldsymbol{0}, \quad \forall \boldsymbol{\xi} \in \mathbf{T}_1. \tag{8}$$

Wrench ζ_c represents a constraint imposed on the end-effector by leg 1. In any nonsingular configuration, rank(**T**₁) = 5; therefore, we can determine unique wrench ζ_c (up to a nonzero multiplier) from (8). By observing the leg topology, we see that the end-effector cannot rotate around the axis perpendicular to the axes of the universal joint. Hence, ζ_c is an infinite-pitch wrench with the following screw coordinates:

$$\boldsymbol{\zeta}_{c} = \begin{bmatrix} \mathbf{0} \\ \hat{\mathbf{u}}_{1} \times \hat{\mathbf{u}}_{2} \end{bmatrix}.$$
(9)

As shown in [32], wrench ζ_{1j} can now be found uniquely (up to a nonzero multiplier) using Equation (6) together with the following condition:

$$\zeta_{1i}^{\mathrm{T}}\zeta_{c} = 0. \tag{10}$$

Equations (6) and (10) allow us to compute numerically screw coordinates of each wrench ζ_{1j} . We can also determine these wrenches geometrically. As ξ_{C1u1} and ξ_{C1u2} are zero-pitch twists and ζ_c is an infinite-pitch wrench, these screws describe only rotational freedoms and constraints of the end-effector. Therefore, conditions (6) and (10) imply that any wrench ζ_{1j} should refer only to translational constraints, i.e., all three ζ_{1j} are zero-pitch wrenches, whose axes pass through point C_1 . Now, let us consider wrench ζ_{11} for a moment. A zero-pitch wrench is reciprocal to an infinite-pitch twist if their axes are orthogonal [25] (p. 25); therefore, to satisfy condition (6), the axis of ζ_{11} must be orthogonal to the axes of ξ_{q4} and ξ_{q5} . We can apply the same logic to ζ_{14} and ζ_{15} and calculate coordinates of all these wrenches as follows:

$$\begin{aligned} \boldsymbol{\zeta}_{11} &= \begin{bmatrix} \hat{\mathbf{s}}_{4} \times \hat{\mathbf{s}}_{5} \\ \boldsymbol{\rho}_{SC1} \times (\hat{\mathbf{s}}_{4} \times \hat{\mathbf{s}}_{5}) \end{bmatrix}, \\ \boldsymbol{\zeta}_{14} &= \begin{bmatrix} \hat{\mathbf{s}}_{1} \times \hat{\mathbf{s}}_{5} \\ \boldsymbol{\rho}_{SC1} \times (\hat{\mathbf{s}}_{1} \times \hat{\mathbf{s}}_{5}) \end{bmatrix}, \\ \boldsymbol{\zeta}_{15} &= \begin{bmatrix} \hat{\mathbf{s}}_{1} \times \hat{\mathbf{s}}_{4} \\ \boldsymbol{\rho}_{SC1} \times (\hat{\mathbf{s}}_{1} \times \hat{\mathbf{s}}_{4}) \end{bmatrix}. \end{aligned}$$
(11)

Let us now consider leg k, k = 2, 3, and rewrite Equation (5):

$$\mathbf{T}_{k}^{\prime} \begin{bmatrix} \dot{q}_{k} \\ \dot{\boldsymbol{\Theta}}_{k} \end{bmatrix} = \mathbf{V}_{P}, \tag{12}$$

where

$$\mathbf{T}'_{k} = \begin{bmatrix} \xi_{qk} & \xi_{Ckx} & \xi_{Cky} & \xi_{Ckz} & \xi_{Akx} & \xi_{Aky} & \xi_{Akz} \end{bmatrix},$$
$$\mathbf{V}_{P} = \mathbf{V} - (\xi_{q4}\dot{q}_{4} + \xi_{a5}\dot{q}_{5}).$$

Using the same approach as before, we can define the wrench ζ_k reciprocal to all twists of \mathbf{T}'_k except for ξ_{qk} and obtain the following relation:

$$\left(\boldsymbol{\xi}_{qk}\circ\boldsymbol{\zeta}_{k}\right)\dot{\boldsymbol{q}}_{k}=\mathbf{V}_{P}\circ\boldsymbol{\zeta}_{k}.$$
 (13)

In any nonsingular configuration, $\operatorname{rank}(\mathbf{T}'_k) = 6$, and there always exists a linear dependence between the twists of the leg spherical joints. Therefore, we can uniquely determine wrench ζ_k (up to a nonzero multiplier) and calculate its screw coordinates with ease. It is also clear that ζ_k should be a zero-pitch wrench, whose axis passes through points A_k and C_k , i.e.,

$$\boldsymbol{\zeta}_{k} = \begin{bmatrix} \hat{\mathbf{w}}_{k} \\ \boldsymbol{\rho}_{SCk} \times \hat{\mathbf{w}}_{k} \end{bmatrix}, \tag{14}$$

where $\hat{\mathbf{w}}_k$ is a unit vector parallel to line $A_k C_k$ (Figure 2); we know this vector from the position analysis.

To visualize the results obtained so far, Figure 4 shows all the wrenches calculated above for a general manipulator configuration.



Figure 4. Wrenches of the manipulator.

Finally, using relations (7) and (13), we exclude all θ_i from Equation (5) and rewrite it as follows (the right side of the equation below overloads the reciprocal product notation " \circ ," but we hope the meaning is clear from the context):

$$\begin{bmatrix} \xi_{q1} \circ \zeta_{11} & 0 & 0 & 0 & 0 \\ 0 & \xi_{q2} \circ \zeta_{2} & 0 & \xi_{q4} \circ \zeta_{2} & \xi_{q5} \circ \zeta_{2} \\ 0 & 0 & \xi_{q3} \circ \zeta_{3} & \xi_{q4} \circ \zeta_{3} & \xi_{q5} \circ \zeta_{3} \\ 0 & 0 & 0 & \xi_{q4} \circ \zeta_{14} & 0 \\ 0 & 0 & 0 & 0 & \xi_{q5} \circ \zeta_{15} \\ 0 & 0 & 0 & 0 & 0 \end{bmatrix} \dot{\mathbf{q}} = \begin{bmatrix} \zeta_{11}^{T} \\ \zeta_{2}^{T} \\ \zeta_{3}^{T} \\ \zeta_{14}^{T} \\ \zeta_{15}^{T} \\ \zeta_{C}^{T} \end{bmatrix} \circ \mathbf{V}.$$
(15)

As the screw coordinates of all twists and wrenches can be calculated from the formulae devised throughout this section, Equation (15) allows us to solve a forward or inverse velocity problem and analyze singular configurations of the manipulator.

We should mention one more thing about Equation (15). Suppose we are given endeffector trajectory $\mathbf{p}_{S}(t)$ and $\hat{\mathbf{n}}(t)$ and have to calculate actuator speeds $\dot{\mathbf{q}}(t)$, where *t* is the time. The right side of Equation (15) includes end-effector twist $\mathbf{V} = \begin{bmatrix} \boldsymbol{\omega}^{\mathrm{T}} & \boldsymbol{\upsilon}_{S}^{\mathrm{T}} \end{bmatrix}^{\mathrm{T}}$. Linear velocity $\boldsymbol{\upsilon}_{S}$ is a time derivative of vector \mathbf{p}_{S} , i.e., $\boldsymbol{\upsilon}_{S} = \dot{\mathbf{p}}_{S}$, and we can set it explicitly. To obtain angular velocity $\boldsymbol{\omega}$, we can first write the following expression [33] (p. 20):

$$\hat{\mathbf{n}} = \boldsymbol{\omega} \times \hat{\mathbf{n}} = -\Lambda(\hat{\mathbf{n}})\boldsymbol{\omega},\tag{16}$$

where $\Lambda(\hat{\mathbf{n}})$ is a skew-symmetric matrix representation of vector $\hat{\mathbf{n}}$; $\hat{\mathbf{n}}$ is a time derivative of vector $\hat{\mathbf{n}}$, which we can also determine explicitly.

Expression (16) represents a system of three linear equations with respect to unknown components of $\boldsymbol{\omega}$. In addition, 3 × 3 skew-symmetric matrices have rank two [33] (p. 22); therefore, we cannot use (16) to find unique $\boldsymbol{\omega}$. To obtain the third independent equation, we can equate the last rows of both sides of (15), concerning (9):

$$(\hat{\mathbf{u}}_1 \times \hat{\mathbf{u}}_2)^{\mathrm{T}} \boldsymbol{\omega} = 0.$$
(17)

Any two equations of (16) with Equation (17) represent a system of three (independent) linear equations, which we can use to calculate ω .

4.2. Numerical Example

Let us consider an example of velocity analysis when we have to find actuator speeds given the end-effector motion. We examine the manipulator with the same geometrical parameters as in [22], and we suppose the end-effector follows a spiral-helical trajectory also studied earlier in [22]:

$$\mathbf{p}_{S}(t) = \begin{bmatrix} \frac{30\varphi(t)}{2\pi}\cos\varphi(t)\\ \frac{30\varphi(t)}{2\pi}\sin\varphi(t)\\ 240 - \frac{30\varphi(t)}{2\pi} \end{bmatrix} \text{mm, } \mathbf{\hat{n}}(t) = \begin{bmatrix} \sin\frac{\pi}{12}\cos\varphi(t)\\ \sin\frac{\pi}{12}\sin\varphi(t)\\ \cos\frac{\pi}{12} \end{bmatrix}, \quad (18)$$
$$\varphi(t) = 6\pi t/15 \text{ rad, } t = (0, 15) \text{ s.}$$

Trajectory (18) corresponds to three revolutions of point *S* about the *Z* axis of the *OXYZ* frame (Figure 5). Both radial and axial pitches of the curve equal 30 mm. At the same time, the end-effector forms an angle of 15° with the *Z* axis during the motion. We consider this (arbitrarily selected) trajectory to verify the developed techniques for a general case, when the end-effector varies all its coordinates during the motion.



Figure 5. Spiral-helical trajectory for velocity analysis. Green arrows show end-effector orientation.

We simulated the proposed algorithm using the MATLAB package, and Figure 6 shows the computed actuator speeds (red lines). We also calculated these speeds by a numerical differentiation (green dots) of actuated coordinates (blue lines), obtained from the preceding inverse kinematics. Both results coincide completely, which verifies the suggested method.



Figure 6. Results of velocity analysis: blue lines— \mathbf{q} ; red lines— \mathbf{q} according to the proposed algorithm; green dots— \mathbf{q} by numerical differentiation of \mathbf{q} .

5. Singularity Analysis

Singularity analysis aims to detect manipulator configurations where it loses or attains DOFs or even changes its motion type. Such configurations should be avoided for proper manipulator performance, and various authors have proposed different approaches to classify and determine singularities [34–37].

In this section, we use the results obtained in the previous section to discuss two common types of singularities: serial singularities when the end-effector loses a DOF and parallel singularities when it gains an additional (uncontrolled) DOF [38]. Constraint singularities [39] are not possible for this manipulator, since only one leg imposes constraints on the moving platform.

5.1. Serial Singularities

The condition for this type of singularity is that the matrix on the left side of Equation (15) becomes rank-deficient [34]. This 6×5 matrix has an upper triangular structure with the last row full of zeros; hence, it will lose its rank if at least one element on the main diagonal equals zero.

Let us first consider the first, fourth, and fifth rows of the matrix. Given Equations (1) and (11), we can write the following conditions:

$$\begin{aligned} \boldsymbol{\xi}_{q1} \circ \boldsymbol{\zeta}_{11} &= \hat{\mathbf{s}}_{1}^{\mathrm{T}} (\hat{\mathbf{s}}_{4} \times \hat{\mathbf{s}}_{5}) = 0, \\ \boldsymbol{\xi}_{q4} \circ \boldsymbol{\zeta}_{14} &= \hat{\mathbf{s}}_{4}^{\mathrm{T}} (\hat{\mathbf{s}}_{1} \times \hat{\mathbf{s}}_{5}) = 0, \\ \boldsymbol{\xi}_{q5} \circ \boldsymbol{\zeta}_{15} &= \hat{\mathbf{s}}_{5}^{\mathrm{T}} (\hat{\mathbf{s}}_{1} \times \hat{\mathbf{s}}_{4}) = 0. \end{aligned}$$
(19)

Any condition from (19) will be satisfied if a parallelepiped formed by vectors \hat{s}_1 , \hat{s}_4 , and \hat{s}_5 degenerates. As, by the manipulator design, vector \hat{s}_1 has a constant direction and vectors \hat{s}_4 and \hat{s}_5 are orthogonal to each other, this situation is only possible if the platform is tilted such that \hat{s}_1 becomes parallel to a plane spanned by \hat{s}_4 and \hat{s}_5 (Figure 7a). The end-effector loses its ability to translate along direction $\hat{s}_4 \times \hat{s}_5$, i.e., perpendicular to the plane mentioned above.



Figure 7. Serial singularities: (a) parallelepiped formed by vectors \hat{s}_1 , \hat{s}_4 , and \hat{s}_5 degenerates; (b) \hat{w}_2 is othogonal to \hat{s}_2 .

Next, we consider one of the two remaining conditions concerning Equation (14):

$$\begin{aligned} \boldsymbol{\xi}_{q2} \circ \boldsymbol{\zeta}_2 &= \hat{\mathbf{w}}_2^{\mathrm{T}} \hat{\mathbf{s}}_2 = 0, \\ \boldsymbol{\xi}_{q3} \circ \boldsymbol{\zeta}_3 &= \hat{\mathbf{w}}_3^{\mathrm{T}} \hat{\mathbf{s}}_3 = 0. \end{aligned}$$
(20)

Conditions (20) are satisfied if \hat{w}_2 or \hat{w}_3 is perpendicular to \hat{s}_2 or \hat{s}_3 , respectively. Figure 7b shows such a case for leg 2. In this case, the platform loses one DOF because there arises a zero-pitch wrench similar to ζ_2 , which constrains the platform translation parallel to vector \hat{w}_2 . Hence, the end-effector will lose one DOF too. If we want to find its remaining DOFs, we should first determine two twists, which define platform motion, and then augment them with twists ξ_{q4} and ξ_{q5} . The wrench system reciprocal to these four twists will define two constraints on the end-effector motion.

5.2. Parallel Singularities

These singularities occur when the matrix on the right side of Equation (15) loses its rank [34], i.e., when wrenches that form this matrix are linearly dependent. We can also treat this condition as follows: the end-effector will gain an uncontrolled motion ($\mathbf{V} \neq 0$) when all its drives are stopped ($\dot{\mathbf{q}} = 0$).

Let us first focus on the 3-DOF parallel part of the manipulator (we explain the reasons later) and assume that its actuated pairs, which correspond to q_i , i = 1...3, are stopped. In this case, leg 1 provides the platform with two rotational DOFs, which come from the universal joint, and therefore imposes four constraints (three translational and one rotational). We can represent these constraints by four wrenches: three (linearly independent) zero-pitch wrenches with axes passing through point C_1 and one infinite pitch wrench ζ_c . Legs 2 and 3 impose constraints described by zero-pitch wrenches ζ_2 and ζ_3 , respectively. As all translational movements of the platform are prevented by three zero-pitch wrenches mentioned above, ζ_2 and ζ_3 should only constrain two rotational DOFs not blocked by ζ_c . In other words, if we calculate screw coordinates of ζ_2 and ζ_3 with C_1 being the reference point, the moment parts (last three components) of these wrenches along with the vector defining the axis of ζ_c should be linearly independent and span a three-dimensional constraint space. In the most general case, this condition can be violated when the three vectors mentioned above lie in the same plane, i.e.,

$$((\boldsymbol{\rho}_{C12} \times \hat{\mathbf{w}}_2) \times (\boldsymbol{\rho}_{C13} \times \hat{\mathbf{w}}_3))^{\mathrm{T}} (\hat{\mathbf{u}}_1 \times \hat{\mathbf{u}}_2) = 0,$$
(21)

where ρ_{C12} and ρ_{C13} are vectors from point C_1 to any point on a line spanned by vector $\hat{\mathbf{w}}_2$ or $\hat{\mathbf{w}}_3$, respectively.

The platform will attain an uncontrolled rotation about the axis, which passes through point C_1 orthogonally to the plane mentioned above. We can mention several particular cases:

- 1. $\hat{\mathbf{w}}_2$ and $\hat{\mathbf{w}}_3$ are collinear (Figure 8a): ζ_2 and ζ_3 are linearly dependent, and their moment parts are collinear.
- 2. $\hat{\mathbf{w}}_2$ (or $\hat{\mathbf{w}}_3$) is on a line passing through point C_1 (Figure 8b): ζ_2 (or ζ_3) has a zero moment part.
- 3. $\hat{\mathbf{w}}_2$ and $\hat{\mathbf{w}}_3$ lie in a plane passing through point C_1 (Figure 8c): the moment parts of ζ_2 and ζ_3 are collinear.
- 4. $\hat{\mathbf{w}}_2$ (or $\hat{\mathbf{w}}_3$) lie in the spider plane, spanned by vectors $\hat{\mathbf{u}}_1$ and $\hat{\mathbf{u}}_2$ (Figure 8d): the moment part of ζ_2 (or ζ_3) is collinear with the axis of ζ_c .
- 5. $\hat{\mathbf{w}}_2$ and $\hat{\mathbf{w}}_3$ are parallel and lie in a plane parallel to the spider plane (Figure 8e): the moment parts of ζ_2 and ζ_3 and the axis of ζ_c lie in a common plane.

The above cases describe possible parallel singularities of the 3-DOF parallel manipulator. In cases 2 and 4, the condition can hold for both \hat{w}_2 and \hat{w}_3 simultaneously: the platform will attain two uncontrolled rotational DOFs about any axes, which pass through point C_1 and lie in the spider plane.



Figure 8. Parallel singularities: (a) $\hat{\mathbf{w}}_2$ and $\hat{\mathbf{w}}_3$ are collinear; (b) $\hat{\mathbf{w}}_2$ passes through C_1 ; (c) $\hat{\mathbf{w}}_2$ and $\hat{\mathbf{w}}_3$ lie in a plane passing through C_1 ; (d) $\hat{\mathbf{w}}_2$ lie in the spider plane, spanned by $\hat{\mathbf{u}}_1$ and $\hat{\mathbf{u}}_2$; (e) $\hat{\mathbf{w}}_2$ and $\hat{\mathbf{w}}_3$ are parallel and lie in a plane parallel to the spider plane.

Actually, the parallel singularities mentioned above correspond not only to the 3-DOF parallel part but for the entire manipulator. One can argue that we did not consider wrenches ζ_{11} , ζ_{14} , and ζ_{15} , presented in (15), but introduced three other zero-pitch wrenches in point C_1 . The reason for doing this is that translational constraints imposed by leg 1 are determined by the leg topology and do not depend on the platform orientation. Technically, in a nonsingular configuration, ζ_{11} , ζ_{14} , and ζ_{15} can also be used to analyze parallel singularities. On the other hand, if ζ_{11} , ζ_{14} , and ζ_{15} are linearly dependent, condition (19) will be satisfied, and the matrix on the left side of Equation (15) will lose its rank. Such a configuration, however, is not a parallel singularity, and there will be no uncontrolled instantaneous motion of the platform.

6. Discussion

Screw theory allows us to address both velocity and singularity analyses of the hybrid spatial manipulator by simple expressions, obtained mainly by inspection and with no complicated numerical calculations. For comparison, we could find matrices in the velocity Equation (15) by differentiating the kinematic constraint equations [22], but this approach would require some tricky manipulations with the vector loop expressions, which are unnecessary in the presented procedure. Moreover, in contrast to the formal numerical procedures, the applied method reveals the geometrical nature of singular configurations, which could be difficult to interpret if we directly computed and analyzed the matrix determinants in Equation (15). This geometrical insight is useful for better understanding of the manipulator performance.

During the velocity analysis, we obtained the relation that connects drive speeds $\dot{\mathbf{q}}$ and end-effector twist **V**. Some designing tasks may also require speeds in unactuated joints, i.e., vectors $\dot{\theta}_i$ in Equation (5). We can always include these speeds in our study by selecting wrenches that are not reciprocal to the twist in the passive joint that we are interested in. This approach allows us to compute speed in any joint of the manipulator.

During the singularity analysis, we established several conditions that correspond to serial and parallel singularities. In practice, we should check these conditions when planning motion trajectories to guarantee proper manipulator performance. Note that we can also avoid some singular configurations by suitable mechanical design. For example, singularities given in Figures 7a and 8a,e will most likely be beyond the manipulator workspace because of the joint constraints. Nevertheless, it is always desirable to analyze closeness to singularities [40] because the manipulator can lose its stiffness and accuracy and may require additional efforts from its actuators when it is nearby a singular posture.

7. Conclusions

This article has expanded the previous research performed on the 5-DOF 3T2R parallelserial (hybrid) manipulator. After a brief discussion of the manipulator design and its position analysis, the paper has presented an approach for velocity and singularity analysis based on the screw theory.

First, we considered the unit twists of each manipulator joint and obtained closedform expressions for actuation and constraint wrenches using the reciprocal screw method. Having determined screw coordinates of these twists and wrenches, we formed a system of velocity equations, which relates the end-effector twist to the drive speeds. To demonstrate the proposed techniques, we performed the inverse velocity analysis for a predefined spiral-helical trajectory. We found the values of the drive speeds and verified the results by comparing them to the values obtained through the numerical differentiation.

Next, we investigated serial and parallel singular configurations of the manipulator by analyzing wrench systems obtained during the velocity analysis. We established several conditions for both types of singularities and provided corresponding mathematical relations between the screw coordinates of the wrenches. We also considered two examples of the serial singularities and five examples of the parallel ones and gave a visual representation for each case. The material presented in the current study is essential for manipulator control and motion planning. Let us also mention several directions for future research. The first one, based on the performed velocity analysis, includes a dynamic study of the manipulator that will require the determination of accelerations of its links and composing equations of motions. The second possible direction is an optimal design problem: we can use the results of the singularity analysis to identify manipulator geometrical parameters that provide the required accuracy and stiffness and guarantee that the manipulator does not attain any singular configuration within its workspace. The results obtained in this paper can also be extended to other hybrid manipulators.

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