



# Article Numerical Investigation on Wellbore Temperature Prediction during the CO<sub>2</sub> Fracturing in Horizontal Wells

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**Abstract:** A novel model is established to predict the temperature field in the horizontal wellbore during  $CO_2$  fracturing. The pressure work and viscous dissipation are considered, and the transient energy, mass and momentum equations as well as the  $CO_2$  physical properties are solved fully coupled. The model passes the convergence test and is verified through a comparison using the COMSOL software. Then, a sensitivity analysis is performed to study the effects of the treating parameters. Results illustrate that the relationship between the injection rate and the stable bottom-hole temperature (hereinafter referred to as BHT) is non-monotonic, which is different from the hydraulic fracturing. The existence of the horizontal section will increase the BHT at 2 m<sup>3</sup>/min condition but reduce the BHT at 10 m<sup>3</sup>/min condition. The problem of high wellbore friction can be alleviated through tube size enhancement, and the ultimate injection rate allowed increased from 2.7 m<sup>3</sup>/min to 29.6 m<sup>3</sup>/min when the tube diameter increased from 50.3 mm to 100.3 mm. Additionally, the open-hole completion method of the horizontal section can increase the BHT to 2.7 °C but reduce the near formation temperature to 24.5 °C compared with the casing completion method.

Keywords: CO<sub>2</sub>; non-isothermal flow; fracturing; horizontal wellbore; heat transfer; numerical model

# 1. Introduction

Shale gas is one of the most important unconventional oil and gas resources [1,2]. Slickwater fracturing and horizontal well drilling are the basic technologies for the development of the shale gas reservoirs [3]. However, the usage of the water-based fracturing fluid can generate formation damage and consume a large amount of water resources.  $CO_2$ fracturing can simultaneously avoid the above disadvantages and achieve the effects of carbon sequestration (CCS) [4,5] and gas recovery enhancement (EGR) [6–9], which is a potential alternative technology to the slick-water fracturing. Unlike the water-based fracturing fluid, the physical properties of  $CO_2$  vary significantly with temperature and pressure [10,11], which can affect the sand-carrying capacity and the performance of the stimulation. Additionally, it is impractical to modify the  $CO_2$  temperature during the fracturing process based on the temperature monitor data because of the expense and time consumed. Therefore, accurately predicting the temperature and pressure of the horizontal well  $CO_2$  fracturing process is necessary in terms of both the environmental and economic aspects.

The study of the wellbore temperature field can be traced back to 1937 [12]. After a long-term development, the wellbore temperature prediction methods can be divided into three categories. The first category is the quasi-steady numerical method represented by the Ramey's model [13], which assumes that the fluid and wellbore zones are steady-state heat transfer, and the formation zone is transient heat transfer. The quasi-steady numerical



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**Copyright:** © 2021 by the authors. Licensee MDPI, Basel, Switzerland. This article is an open access article distributed under the terms and conditions of the Creative Commons Attribution (CC BY) license (https:// creativecommons.org/licenses/by/ 4.0/). method is applicable to production and injection (flooding process) wells, and many researchers have extended the model to various wellbore conditions [14–27]. However, it is not suitable for the fracturing process [28]. The second category is the transient numerical method represented by the Eickmeier's model [29], which assumes that all the calculation zones are transient heat transfer. Considering the transient heat transfer condition, this kind of method is widely used in the temperature prediction of the hydraulic fracturing and acidification treatments [30–33], in which the injection rates are typically high. The third category is the analytic method. The assumptions of the steady [34] and transient [35–41] heat transfers are both used in the analytic method, it requires further simplified conditions to obtain the solution, which restricts its applications. The three kinds of wellbore temperature prediction methods discussed above are all applied to the CO<sub>2</sub> injection process.

As for the quasi-steady numerical method, Lu and Connell [42] used the quasi-steady heat transfer equation and steady-state mass and momentum equations to simulate the non-isothermal flow of a vertical well during the injection of a multicomponent mixture containing CO<sub>2</sub>. The physical properties of the mixture were calculated by the Pen– Robinson Equations of State (hereinafter referred to as EoS). Wang [43] combined the Span–Wagner model [44], Fenghour model [45], and Vesovic model [46] with the quasisteady temperature equation and the steady mass and momentum equations to analyze the wellbore temperature and pressure distribution in a vertical well during supercritical  $CO_2$  drilling. Dou [47] applied a similar method to simulate the wellbore temperature field of the CO<sub>2</sub> flooding process and compared it with the measured data to verify the accuracy of the model. Li [48] expanded the application of Wang and Dou's work to CO<sub>2</sub> sequestration and CO<sub>2</sub> fracturing. Sun [49] applied the quasi-steady numerical method to the CO<sub>2</sub> injection heat recovery process of a geothermal U-pipe system and simulated the temperature and pressure distribution of CO<sub>2</sub> in a closed cycle system.

Although the quasi-steady numerical method performed well in low injection rate (typically  $< 0.1 \text{ m}^3/\text{min}$ ) conditions such as CO<sub>2</sub> flooding, CO<sub>2</sub> sequestration, and CO<sub>2</sub> heat recovery, it is not suitable for a short time and fast injection processes such as fracturing owing to the adaptation of the steady heat transfer assumption in the fluid energy equation. As stated above, the transient numerical method can effectively solve this problem. Pan [50] developed a vertical wellbore simulator following the assumption of the transient nonisothermal flow of  $CO_2$  brine mixture and using the drift flux model (DFM) to describe momentum conservation. Then, Pan [51,52] combined their wellbore model with the multiphase flow to analyze the non-isothermal flow of the mixture in the formation. In their wellbore heat transfer model, the fluid heat transfer equation is transient. By contrast, the heat transfer of the wellbore zone is characterized by the comprehensive heat transfer coefficient, which is derived on the basis of the steady-state heat transfer assumption. Guo [53,54] and Gong [55,56] considered the influence of the frictional heat and the Joule-Thomson effect based on the Eickmeier's model to establish a wellbore temperature field model for CO<sub>2</sub> fracturing in a vertical well, in which the CO<sub>2</sub> physical property model and the transient continuity and momentum equation are coupled.

Compared with the numerical method, research scarcely focuses on the analytic method. Singhe [57] considered the Joule–Thomson effect on the basis of Hagoort's model [40] to build an analytical wellbore temperature model for CO<sub>2</sub> injection wells and verified it through the data of the Ketzin area in Germany; however, the analytical solution is based on the quasi-steady heat transfer assumption, which is also not suitable for the fracturing process. Additionally, some researchers [58,59] used 2D turbulence models such as the k- $\varepsilon$  model to investigate the flow and heat transfer characteristics of the CO<sub>2</sub> injection process in vertical wells. However, considering the computation and the convergence requirements, utilizing it is difficult in the simulation of the horizontal wells. In general, although the above literature brought important achievements for the wellbore temperature prediction of CO<sub>2</sub> injection process, it has focused on the vertical

wells. Moreover, an efficient wellbore temperature prediction model remains lacking for horizontal wells, which are extensively used in unconventional oil and gas reservoirs.

In this article, a transient fully coupled model of wellbore temperature for  $CO_2$  fracturing in horizontal wells is established, which combines the 1D non-isothermal flow in the axial direction with the 1D transient heat transfer in the radial direction. The effects of the pressure work and viscous dissipation are considered in the energy equation, and the transient mass and momentum equations as well as the  $CO_2$  physical property models are solved simultaneously. The 1D + 1D model is verified through a convergence test and a comparison using the COMSOL software. Finally, the parameter analysis is performed to provide guidance for the treatment of the  $CO_2$  fracturing in horizontal wells.

# 2. Theory

# 2.1. Physical Model

Figure 1 illustrates the physical model. The wellbore is divided into vertical, curved, and horizontal sections. In the vertical and curved sections, the wellbore structure includes the tubing, annulus, casing, and the cement sheath. In the horizontal section, as the openhole completion method is considered here, the wellbore structure only includes tubing and annulus.



Figure 1. Physical model.

#### 2.2. Basic Assumption

The horizontal well CO<sub>2</sub> fracturing has three characteristics. First, the aspect ratio of the wellbore of oil and gas wells is usually very large  $(1 \times 10^3 \text{ to } 1 \times 10^4)$  given the enormous gap between the length of the axial direction and the width of the radial direction. Second, the injection rate is typically above 3 m<sup>3</sup>/min, which results in a high Reynolds number (Re > 1 × 10<sup>6</sup>) and a turbulent flow during fracturing. Last but not least, the L-shaped structure of the horizontal wellbore cannot be simplified through the method of axisymmetric modeling as the vertical well in 2D or 3D space. All of the above necessitate the meshes in the 2D, or the 3D Computational Fluid Dynamics (hereinafter referred to as CFD) model becomes a huge number to obtain stability and convergence. An example is the *k*- $\varepsilon$  model. The dimensionless wall distance function *y*+ of the first boundary layer grids should be 30–200 [58,60], and the dimensionless wall distance function can be calculated

as  $y_{+} = v\Delta y/\mu$ , where *y* is the thickness of the first boundary layer (please notice that parameters that are not further explained in the main text hereinafter are listed at the Nomenclature section). For CO<sub>2</sub> fracturing, this process requires the thickness of the first boundary layer to be less than 0.001 m (as shown in Figure 2).



Figure 2. Grids of the 2D model.

To solve the above problems, considering the following two points: (1) the radial dimension is pretty small compared with the axial dimension, (2) the effect of the axial heat conduction on the wellbore temperature can be ignored [53,54], the following assumptions are used:

- 1. The CO<sub>2</sub> zone is considered a one-dimensional axial non-isothermal flow;
- 2. The wellbore and formation zones are considered a one-dimensional radial transient heat transfer;
- 3. The heterogeneity and anisotropy of the formation are ignored;
- 4. At the initial moment, the wellbore is full of fluid, and the fluid is stationary.

# 2.3. Mathematical Model

The calculation zone includes the  $CO_2$ , wellbore, and formation zones. The  $CO_2$  zone is considered a non-isothermal flow, and the continuity equation, momentum equation, as well as the energy equation must be solved simultaneously. The wellbore zone is considered a fluid natural convective heat transfer, and the formation zone is considered a solid heat transfer process, for which the energy equation must be solved.

Continuity equation of the CO<sub>2</sub> zone:

$$v\frac{\partial\rho_{\rm f}}{\partial z} + \rho_{\rm f}\frac{\partial v}{\partial z} + \frac{\partial\rho_{\rm f}}{\partial t} = 0 \tag{1}$$

Momentum equation of the CO<sub>2</sub> zone:

$$\frac{\partial p}{\partial z} = \rho_{\rm f}g\sin\theta - \frac{1}{2}f_{\rm D}\frac{\rho_{\rm f}v^2}{d_{\rm ti}} - \rho_{\rm f}\frac{\partial v}{\partial t} \tag{2}$$

Energy equation of the CO<sub>2</sub> zone:

$$A\rho_{\rm f}C_{\rm p_f}\frac{\partial T_{\rm f}}{\partial t} + A\rho_{\rm f}C_{\rm p_f}v\frac{\partial T_{\rm f}}{\partial z} = Q_{\rm wall} + Q_{\rm fric} + Q_{\rm pres}.$$
(3)

where  $f_D$  is the friction coefficient. The fitting formula ( $R^2 > 0.999$ ) [61] of the CO<sub>2</sub> fracturing wellbore friction plate [62] is used here to obtain the expression of  $f_D$ .

$$P_{\rm f} = 0.156Q_{\rm inj}^2 + 1.639Q_{\rm inj} - 0.228 \quad d_{\rm ti} = 50.3 \text{ mm}$$

$$P_{\rm f} = 0.037Q_{\rm inj}^2 + 0.777Q_{\rm inj} - 0.169 \quad d_{\rm ti} = 62 \text{ mm}$$

$$P_{\rm f} = 0.017Q_{\rm inj}^2 + 0.268Q_{\rm inj} - 0.041 \quad d_{\rm ti} = 76 \text{ mm} \quad '$$

$$P_{\rm f} = 0.003Q_{\rm inj}^2 + 0.089Q_{\rm inj} - 0.021 \quad d_{\rm ti} = 100.3 \text{ mm} \quad '$$
(4)

$$f_{\rm D} = \frac{2P_{\rm f} d_{\rm ti} / 100}{\rho v^2}.$$
 (5)

 $Q_{\text{wall}}$ ,  $Q_{\text{fric}}$  and  $Q_{\text{pres}}$  are the heat flows of the unit axial length generated by the heat transfer, viscous dissipation, and pressure work, respectively,  $W/(m^2 \cdot K)$ . They are defined as follows:

$$Q_{\text{wall}} = \pi d_{\text{ti}} h_{\text{t}} (T_{\text{f}} - T_{\text{t}}), \tag{6}$$

$$Q_{\rm fric} = f_{\rm D} v \cdot \rho_{\rm f} A v^2 / (2d_{\rm ti}),\tag{7}$$

$$Q_{\text{pres}} = A\alpha_{\text{p}}T_{\text{f}}(\partial p/\partial t + v \cdot \partial p/\partial z).$$
(8)

where  $h_t$  is the heat transfer coefficient of the forced heat convection. The Gnielinski correlation, which is valid for  $0.5 \le Pr \le 2000$  and  $3000 \le Re \le 5 \times 10^6$ , is applied here to calculate  $h_t$  [63]:

$$h_{\rm t} = \frac{\lambda_{\rm f}(f_{\rm D}/8)({\rm Re} - 1000){\rm Pr}}{2r_{\rm ti} \left[1 + 12.7(f_{\rm D}/8)^{1/2} {\binom{2/3}{\rm Pr} - 1}\right]}.$$
(9)

Pr is the Prandtl number, defined as  $Pr = C_p \mu / \lambda$ . Unless otherwise specified, the heat flow mentioned in the following refers to the heat flow of the unit axial length.

#### 2.3.1. Energy Equation of the Wellbore Zone

The wellbore zone includes the tubing, annulus, casing, and the cement sheath. Except for the annulus, the remaining regions are all solid, and only the heat conduction requires consideration. For the annulus, the internal fluid is in a state of natural convection, which is characterized here by the natural convection heat transfer coefficient. The overall governing equation of the wellbore zone is as follows:

$$\frac{1}{r}\frac{\partial}{\partial r}\left(\lambda r\frac{\partial T}{\partial r}\right) = \rho C_{\rm p}\frac{\partial T}{\partial t}.$$
(10)

In the formula,  $\rho$ ,  $C_p$  and  $\lambda$  take the values of different materials according to different spatial positions, and the temperature at the interface of different zones is continuous. The specific form of the governing equation of each part is different, and the wellbore structure in the horizontal section is also different from the vertical and curved sections. The governing equations for each part of the wellbore are given below.

#### Vertical Section (Curved Section)

Figure 3 presents the radial grids of the vertical section (curved section), in which the white point with blue edge indicates the location of the various radius of the tubing, casing, cement sheath, and formation, while the red point with black edge indicates the location of the calculation temperature.



Figure 3. Radial grids of the vertical (curved) section.

The energy equation of the tubing:

$$\frac{2r_{\rm to}h_{\rm an}(T_{\rm an}-T_{\rm t})}{r_{\rm to}^2 - r_{\rm ti}^2} - \frac{2r_{\rm ti}h_{\rm t}(T_{\rm t}-T_{\rm f})}{r_{\rm to}^2 - r_{\rm ti}^2} = \rho_{\rm t}C_{\rm p_t}\frac{\partial T_{\rm t}}{\partial t}.$$
(11)

The energy equation of the annulus:

$$\frac{2r_{\rm ci}h_{\rm an}(T_{\rm c}-T_{\rm an})}{r_{\rm ci}^2 - r_{\rm to}^2} - \frac{2r_{\rm to}h_{\rm an}(T_{\rm an}-T_{\rm t})}{r_{\rm ci}^2 - r_{\rm to}^2} = \rho_{\rm an}C_{\rm p_{\rm an}}\frac{\partial T_{\rm an}}{\partial t}.$$
(12)

The energy equation of the casing:

$$\frac{4r_{\rm co}\lambda_{\rm c,h}(T_{\rm h}-T_{\rm c})}{(r_{\rm co}^2-r_{\rm ci}^2)(r_{\rm h}-r_{\rm ci})} - \frac{2r_{\rm ci}h_{\rm an}(T_{\rm c}-T_{\rm an})}{r_{\rm co}^2-r_{\rm ci}^2} = \rho_{\rm c}C_{\rm p_{\rm c}}\frac{\partial T_{\rm c}}{\partial t}.$$
(13)

The energy equation of the cement sheath:

$$\frac{4r_{\rm h}\lambda_{\rm h,r}(T_{\rm r_1} - T_{\rm h})}{(r_{\rm h}^2 - r_{\rm co}^2)(r_{\rm r_1} - r_{\rm co})} - \frac{4r_{\rm co}\lambda_{\rm c,h}(T_{\rm h} - T_{\rm c})}{(r_{\rm h}^2 - r_{\rm co}^2)(r_{\rm h} - r_{\rm ci})} = \rho_{\rm h}C_{\rm p_{\rm h}}\frac{\partial T_{\rm h}}{\partial t}.$$
(14)

where  $h_{an}$  is the natural convection heat transfer coefficient in annulus, W/(m<sup>2</sup>.°C), defined as

$$h_{\rm an} = \frac{0.049 (\rm{GrPr})^{1/3} \rm{Pr}^{0.074} \lambda_{\rm an}}{r_{\rm to} \ln(r_{\rm ci}/r_{\rm to})}.$$
 (15)

Gr is the Grashof number of the annulus fluid, defined as

$$Gr = \frac{(r_{ci} - r_{to})^3 g \rho_{an}^2 \beta_{an} (T_f - T_c)}{\mu_{an}^2}.$$
 (16)

where  $\lambda_{c,h}$  and  $\lambda_{h,r}$  are the composite thermal conductivity, which can have the following definition based on Fourier law:

$$\lambda_{c,h} = \frac{\lambda_c \lambda_h \ln\left(\frac{r_{co} + r_h}{r_{co} + r_{ci}}\right)}{\lambda_c \ln\left(\frac{r_{co} + r_h}{2r_{co}}\right) + \lambda_h \ln\left(\frac{2r_{co}}{r_{co} + r_{ci}}\right)},\tag{17}$$

$$\lambda_{\rm h,r} = \frac{\lambda_{\rm h} \lambda_{\rm r} \ln\left(\frac{r_{\rm h} + r_{\rm r_1}}{r_{\rm h} + r_{\rm co}}\right)}{\lambda_{\rm h} \ln\left(\frac{r_{\rm h} + r_{\rm r_1}}{2r_{\rm h}}\right) + \lambda_{\rm r} \ln\left(\frac{2r_{\rm h}}{r_{\rm h} + r_{\rm co}}\right)}.$$
(18)

# Horizontal Section

Figure 4 illustrates the radial grids of the horizontal section. The implications of the two kinds of points are the same as in Figure 3.



Figure 4. Radial grids of the horizontal section.

The wellbore in the horizontal section only includes the tubing and the annulus. The energy governing equation of the tubing is the same as the vertical (curved) section. The annulus energy governing equation is as follows:

$$\frac{2r_{\rm h}h_{\rm an}(T_{\rm r_1} - T_{\rm an})}{r_{\rm h}^2 - r_{\rm to}^2} - \frac{2r_{\rm ti}h_{\rm an}(T_{\rm an} - T_{\rm t})}{r_{\rm h}^2 - r_{\rm to}^2} = \rho_{\rm an}C_{\rm p_{\rm an}}\frac{\partial T_{\rm an}}{\partial t}.$$
(19)

# 2.3.2. Energy Equation of the Formation Zone

No flow occurs in the formation. Accordingly, the energy governing equation in the formation can be simplified to the heat conduction equation:

$$\frac{4r_{\mathbf{r}_{i}}\lambda_{\mathbf{r}}(T_{\mathbf{r}_{i+1}}-T_{\mathbf{r}_{i}})}{\left(r_{\mathbf{r}_{i}}^{2}-r_{\mathbf{r}_{i-1}}^{2}\right)\left(r_{\mathbf{r}_{i+1}}-r_{\mathbf{r}_{i-1}}\right)} - \frac{4r_{\mathbf{r}_{i-1}}\lambda_{\mathbf{r}}(T_{\mathbf{r}_{i}}-T_{\mathbf{r}_{i-1}})}{\left(r_{\mathbf{r}_{i}}^{2}-r_{\mathbf{r}_{i-1}}^{2}\right)\left(r_{\mathbf{r}_{i}}-r_{\mathbf{r}_{i-2}}\right)} = \rho_{\mathbf{r}}C_{\mathbf{p}_{\mathbf{r}}}\frac{\partial T_{\mathbf{r}_{i}}}{\partial t}.$$
(20)

Owing to the difference between the radial grids of the vertical (curved) and horizontal sections, the equations at the boundary of the formation and the wellbore are different.

The first layer grids of the vertical (curved) section:

$$\frac{4r_{r_1}\lambda_r(T_{r_2,j}-T_{r_1,j})}{(r_{r_1}^2-r_h^2)(r_{r_2}-r_h)} - \frac{4r_h\lambda_{h,r}(T_{r_1,j}-T_{h,j})}{(r_{r_1}^2-r_h^2)(r_{r_1}-r_{co})} = \rho_r C_{p_r} \frac{\partial T_{r_1}}{\partial t}.$$
(21)

The first layer grids of the horizontal section:

$$\frac{4r_{r_{1}}\lambda_{r}(T_{r_{2},j}-T_{r_{1},j})}{(r_{r_{1}}^{2}-r_{h}^{2})(r_{r_{2}}-r_{h})} - \frac{2r_{h}h_{an,j}(T_{r_{1},j}-T_{an,j})}{r_{r_{1}}^{2}-r_{h}^{2}} = \rho_{r}C_{p_{r}}\frac{\partial T_{r_{1}}}{\partial t}.$$
(22)

#### 2.4. Initial and Boundary Conditions

#### 2.4.1. Initial Conditions

The initial wellbore temperature is calculated from the original geothermal gradient. As for the pressure, considering that the wellbore is assumed to be full of  $CO_2$ , the initial wellbore hydrostatic pressure can be obtained through an iterative method combined with the  $CO_2$  density model. The governing equations for the initial conditions are as follows:

$$\begin{cases} T|_{t=0} = T_{\text{surf}} + g_{\text{G}} \cdot z\\ p|_{t=0} = \int_0^z \rho(T, p) g dz \end{cases}$$
(23)

# 2.4.2. Boundary Conditions

For the flow field in the  $CO_2$  zone, a given injection rate at the wellhead is used as the velocity boundary, and a pressure boundary is required at the bottom of the well. The current study uses the fracture closure pressure. For the temperature field of the  $CO_2$ zone, the injection temperature at the wellhead is given as the inlet boundary, and the bottom-hole of the well is insulated. However, the thermal convection is considered at the bottom-hole. The governing equations for the boundary conditions are as follows:

$$\begin{cases} v|_{z=0} = Q_{\text{inj}} / (\pi r_{\text{ti}}^2) \\ p|_{z=z_{\text{bh}}} = p_{\text{w}} \\ T|_{z=0} = T_{\text{inj}} \\ \frac{\partial T}{\partial z}|_{z=z_{\text{bh}}} = 0 \end{cases}$$

$$(24)$$

#### 2.5. Solving Method

In the 1D + 1D transient fully coupled model, the continuity, momentum, and energy equations are coupled together through the interaction among pressure, temperature, and  $CO_2$  physical properties to form a transient nonlinear system, which requires an iteration solving method. The Span–Wagner EoS [44], Fenghour model [45] and Vesovic model [46] are used here as the physical properties models, which are proven to be highly accurate [10,11]. Two iteration loops are set up during the solution process: the pressure-speed and temperature iterations. Figure 5 shows the model solution flowchart.



Figure 5. Physical model.

# 3. Validation

# 3.1. Test of the Convergence

This paper used the iterative method to solve the velocity, pressure, and temperature. To test the convergence of the numerical algorithm, the changes in the calculation results must be analyzed during the iteration. Given that the velocity-pressure iteration is nested in the temperature iteration, the convergence test was performed on the temperature calculation results. Table 1 details the parameters of the benchmark case.

#### Table 1. Parameters of the basic computation case.

Parameters	Values	Parameters	Values
Depth	1600 m	Surface temperature	20 °C
Injection temperature	0 °C	Closure pressure (hereinafter referred to as BHP)	32 MPa
Injection rate	6 m <sup>3</sup> /min	Geothermal gradient	0.03 °C/m
Tubing inner diameter	76 mm	Injecting time	120 min
Tubing outer diameter	89 mm	Tubing (Casing) heat capacity	460 J/(kg·°C)
Casing inner diameter	157.8 mm	Annulus capacity	4180 J/(kg·°C)
Casing outer diameter	177.8 mm	Cement capacity	880 J/(kg·°C)
Cement sheath diameter	237.8 mm	Formation capacity	833 J/(kg·°C)
Tubing (Casing) density	7800 kg/m <sup>3</sup>	Tubing (Casing) heat conductivity	$53 \text{ W/(m} \cdot ^{\circ}\text{C})$
Annulus density	$1000 \text{ kg/m}^3$	Annulus heat conductivity	0.557 W/(m·°C)
Cement density	$2000 \text{ kg/m}^3$	Cement heat conductivity	$0.627 \text{ W}/(\text{m} \cdot ^{\circ}\text{C})$
Formation density	$2505.53 \text{ kg/m}^3$	Formation heat conductivity	$2.5 \text{ W}/(\text{m}\cdot^{\circ}\text{C})$
Curved section length	200 m	Horizontal section	600 m
Completion method of the vertical (curved) section	Casing completion	Completion method of the horizontal section	Open-hole completion

The convergence condition of the temperature calculation is defined as

$$\|T_{\rm f}^{\rm k} - T_{\rm f}^{\rm k-1}\|_2 \le 0.01 \tag{25}$$

where  $||X||_2$  represents the 2-norm of vector *X*. Figure 6a presents the  $||T_f^k - T_f^{k-1}||_2$  with various iteration steps and the number of iterations used to reach the convergence condition at different time steps.

Figure 6b indicates that as the number of iteration steps increased, the error continued to decrease; as the time step increased, the number of iteration steps used to achieve convergence condition also continued to decrease. Thus, the numerical solution method used in this paper met the requirements of convergence.

#### 3.2. Comparison of the Non-Isothermal Flow Simulation in a Horizontal Tube with COMSOL

Currently, measured data of  $CO_2$  fracturing are lacking in horizontal wells. Accordingly, a comparison method with commercial software was used to verify the calculation results of the 1D + 1D transient fully coupled model in the horizontal section. COMSOL is a multi-physics coupled finite element software. The non-isothermal tube flow module can simulate thermal convection and heat conduction in the tube. However, in COMSOL, the wellbore wall layer (tubing, annulus, casing, cement sheath, etc.) notably adopts the steady-state heat transfer assumption and does not consider the natural convection in the annulus [64].

The calculation parameters were based on Table 1. The vertical section was ignored, and the horizontal section was extended to 1000 m. Simultaneously, the initial values were set as the parameters of the depth of 1600 m, implying that the initial temperature was 68 °C, and the initial pressure was 32 MPa, based on Table 1. As the horizontal wellbore was completed with the open-hole completion method, the casing and cement sheath were removed. The natural convection in the annulus was also ignored in the 1D + 1D fully coupled model for comparative analysis.



**Figure 6.** Test of the convergence; left: (a)  $||T_f^k - T_f^{k-1}||_2$  with various iteration steps; right: (b) Number of iterations with various time steps.

Figure 7a shows the variation of the BHT with time. Apparently, after ignoring the natural convection in the annulus, the results of the 1D + 1D transient fully coupled model were close to the results of COMSOL. However, at the early stage, the change rate of BHT calculated by COMSOL was faster than that of the 1D + 1D transient fully coupled model because of the steady-state wall heat transfer assumption used in COMSOL. The latter ignored the temperature stabilization process of each wall surface of the wellbore and underestimated the CO<sub>2</sub> temperature. However, due to the large injection rate and the neglection of the natural convection, the effect of the wall heat transfer on the CO<sub>2</sub> temperature was small; the temperature difference between the two was only 0.2 °C after 120 min. Considering that the treatment of wall heat transfer was the main difference between COMSOL and the 1D + 1D transient fully coupled model, another comparison

was performed in which the wall heat transfer term was removed both from COMSOL and the 1D + 1D transient fully coupled model. As shown in Figure 7b, the calculation results of the two were highly consistent after neglecting the effect of wall heat transfer, and the deviation did not exceed 0.03 °C. This finding validated the model accuracy in the horizontal section.



**Figure 7.** Comparison with commercial software; left: (**a**) Comparison of BHT; right: (**b**) Comparison of temperature profiles.

#### 4. Results and Discussions

This section presents an analysis of the effects of the treating parameters using the 1D + 1D fully coupled model. Table 1 details the parameters of the benchmark case.

#### 4.1. Analysis of the Injection Rate

The cases were considered when the injection rates were  $2 \text{ m}^3/\text{min}$ ,  $6 \text{ m}^3/\text{min}$ , and  $10 \text{ m}^3/\text{min}$ , based on the benchmark case.

# 4.1.1. Analysis of the Time Domain

Figure 8 shows the variation of the BHT with time under various injection rates. When the injection time was less than 50 min, an increase in the injection rate speeded up the decrease in the temperature. This finding was due to the change in the convective heat transfer efficiency with the CO<sub>2</sub> velocity. Figure 9 shows the average CO<sub>2</sub> velocity under different injection rates. As the size of the tubing was unchanged, the velocity was basically linear with the injection rate. However, faster convective heat transfer will also hasten the stability of the CO<sub>2</sub> temperature. Using 0.03 °C/min as the criterion of the temperature stability, Figure 10 shows that the temperatures of the cases of 2 m<sup>3</sup>/min, 4 m<sup>3</sup>/min, and 6 m<sup>3</sup>/min did not reach the stable state after injection for 120 min. The stable temperature of CO<sub>2</sub> was related to the heat transfer mechanism determined by the combination of the pressure work, viscous dissipation, and wall heat transfer. The analysis of space domain will illustrate this process.

Figure 11 shows the variation of the wellhead pressure (hereinafter referred to as WHP) with time under various injection rates. Given that no flow occurred at the initial time, the initial WHP was a constant with the value of 17.5 MPa. When fracturing started,  $CO_2$  flowed into the wellbore and generated friction. To keep the BHP constant, the WHP increased. The enhancement was determined by the friction, which increased significantly as the injection rate increased. Figure 12 shows that the wellbore friction reached 67.7 MPa at 10 m<sup>3</sup>/min. In addition, Figure 11 shows that the pressure increased to a maximum within a short period of time, and then the pressure decreased slightly. This phenomenon was caused by the variation of the  $CO_2$  density, which was affected by the  $CO_2$  temperature. As shown in Figure 13, after 120 min, the  $CO_2$  density at the bottom-hole increased by 185 kg/m<sup>3</sup> compared with that after 1 min.



Figure 8. Variation of BHT with time under various Q<sub>inj</sub>.



Figure 9. Mean velocities.



Figure 10. Bottom-hole temperature change rates.



**Figure 11.** Variation of the WHP with time under various  $Q_{inj}$ .



Figure 12. Wellbore friction.



**Figure 13.** CO<sub>2</sub> density profiles.

4.1.2. Analysis of the Space Domain

Figure 14 shows the temperature profiles of  $CO_2$  with various injection rates after 120 min. With the increase in the measured depth, the axial direction and the wellbore structure changed. This was divided into three sections: vertical, curving, and horizontal sections. In the vertical section, the temperature of  $CO_2$  decreased first and then increased with the increase in the injection rate. This phenomenon was caused by the heating effect of the viscous dissipation. As shown in Figure 15a, the heat flow generated by the viscous dissipation was only 120 W/m at 2 m<sup>3</sup>/min but reached 4796 W/m at 10 m<sup>3</sup>/min. In the curved section, the well inclination angle slowly changed from 0° to 90°, and the gradient of the hydrostatic pressure gradually disappeared. Thus, the total pressure drop in the curved section was significantly increased, which resulted in an increase in the cooling effect of pressure work. As shown in Figure 15b, from the bottom of the vertical section to

 $9 \text{ m}^3/\text{min}: -2228 \text{ W/m}$  $-3773 \,\text{W/m}$ Injection time: 120 min 0  $2~{\rm m}^3/{\rm min}$ 200 $6 \text{ m}^3/\text{min}$  $10~{\rm m}^3/{\rm min}$ 400 600 · 800 Measured depth (m) 10001200 1400 V section  $\uparrow$ 1600 C section ↓ 1800H section  $\downarrow$ 2000 2200 $2400 \cdot$  $\frac{1}{5}$ 10 15 2025 30 35Ó Temperature profiles (°C)

the bottom of the curving section, the variations of the heat flow of the pressure work were as follows:

$$2 \text{ m}^{3}/\text{min}: 196 \text{ W/m} \xrightarrow{\text{Curvedsection}} -116 \text{ W/m}$$
  

$$6 \text{ m}^{3}/\text{min}: -317 \text{ W/m} \xrightarrow{\text{Curvedsection}} -1202 \text{ W/m}$$
(26)  

$$9 \text{ m}^{3}/\text{min}: -2228 \text{ W/m} \xrightarrow{\text{Curvedsection}} -2772 \text{ W/m}$$

**Figure 14.** CO<sub>2</sub> temperature profiles with various *Q*<sub>inj</sub>.



Figure 15. Cont.



**Figure 15.** Heat flow profiles with various  $Q_{inj}$ ; left: (**a**) Profiles of  $Q_{fric}$ ; middle: (**b**) Profiles of  $Q_{pres}$ ; right: (**c**) Profiles of  $Q_{wall}$ .

Therefore, in the curved section, the cooling effect of pressure work was obviously enhanced, leading to a decrease in the temperature gradients of all three cases. In the horizontal section, the absolute value of the  $Q_{\text{pres}}$  under the injection rate of 10 m<sup>3</sup>/min continued to increase, whereas the  $Q_{\text{pres}}$  under the injection rate of 2 m<sup>3</sup>/min did not change much. Simultaneously, as shown in Figure 15c, compared with the vertical and curved sections, the  $Q_{\text{wall}}$  in the horizontal section increased significantly, which was due to the open-hole completion method. Section 4.4 discusses the effects of horizontal completions. Generally, with the combined effects of the pressure work and wall heat transfer, the temperature gradient in the horizontal section was higher than that of the vertical section under the 2 m<sup>3</sup>/min case but lower than that of the vertical section under the 10 m<sup>3</sup>/min case. The 6 m<sup>3</sup>/min case fell in between the other two cases. Therefore, compared with the vertical section, the existence of the horizontal section increased the BHT under a low injection rate but reduced the BHT under a high injection rate.

Figure 16 shows the pressure profiles of  $CO_2$  with various injection rates after 120 min.  $CO_2$  pressure was determined by the hydrostatic pressure and wellbore friction. In the vertical section, when the injection rate was 2 m<sup>3</sup>/min, the hydrostatic column pressure was higher than the wellbore friction, resulting in a positive pressure gradient. Additionally, the cases of 6 m<sup>3</sup>/min and 10 m<sup>3</sup>/min were contrary to the 2 m<sup>3</sup>/min case. In the horizontal section, as the hydrostatic pressure no longer increased, the  $CO_2$  pressure decreased along the axial direction under any injection rates. The curved section was the transition between the vertical and horizontal sections, during which the hydrostatic column pressure gradient slowly decreased to zero.



**Figure 16.** CO<sub>2</sub> pressure profiles with various *Q*<sub>inj</sub>.

#### 4.2. Analysis of the Injection Temperature

The cases where the injection temperatures were -20 °C, 0 °C and 20 °C were considered on the bases of the benchmark case.

# 4.2.1. Analysis of the Time Domain

Figure 17 shows the variation of the BHT with time under various injection temperatures. Evidently, the increase in the injection temperature effectively enhanced the BHT. When the injection temperature increased to 20 °C, the BHT maintained above the supercritical temperature during the entire fracturing process. Figure 18 shows the variation of the WHP with time under various injection temperatures. As the lower temperatures increased the  $CO_2$  density, which enhanced the hydrostatic pressure, the WHP of the -20 °C case was 1.8 MPa lower than that of the 20 °C case.

#### 4.2.2. Analysis of the Space Domain

Figure 19 shows the temperature profiles of CO<sub>2</sub> with various injection temperatures after 120 min. When the injection temperature increased from -20 °C to 20 °C, the temperature of CO<sub>2</sub> increased significantly. However, the temperature gradient of CO<sub>2</sub> decreased continuously, especially in the horizontal section. Figure 20 indicates that when the temperature increased from -20 °C to 20 °C, the temperature difference between the wellhead and the bottom-hole decreased by 5.2 °C. This phenomenon was caused by the effects

of the injection temperature on  $Q_{\text{pres}}$  and  $Q_{\text{wall}}$ . As shown in Figure 21a,b, respectively, the heating effect of  $Q_{\text{wall}}$  reduced, whereas the cooling effect of  $Q_{\text{pres}}$  increased with the enhancement of the injection temperature. Therefore, with the increase in the injecting temperature, the CO<sub>2</sub> temperature gradient decreased.

Figure 22 illustrates the pressure profiles of  $CO_2$  with various injection temperatures after 120 min. As stated above, the hydrostatic pressure increased with the reduction of the temperature. Therefore, the lower the temperature, the higher the pressure in the vertical section. However, given that the hydrostatic pressure gradient disappeared, the pressure profiles of the three cases in the horizontal section were slightly different.



**Figure 17.** Variation of BHT with time under various  $T_{inj}$ .



**Figure 18.** Variation of WHP with time under various  $T_{inj}$ .



**Figure 19.**  $CO_2$  temperature profiles with various  $T_{inj}$ .



**Figure 20.** Difference between BHT and wellhead temperature (hereinafter referred to as WHT) with various  $T_{inj}$ .

# 4.3. Analysis of the Tube Inner Diameters

The cases where the tube inner diameters were 62, 76, and 100.3 mm were considered on the bases of the benchmark case. Table 2 shows the specific size of the tube.

Table 2. Size of t	he tube.
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Inner Diameter	Outer Diameter	Outer Diameter
Mm	mm	in
62	73	$2\frac{7}{8}''$
76	88.9	$3\frac{1}{2}''$
100.3	114.3	$4\frac{1}{2}''$



**Figure 21.** Heat flow profiles with various  $T_{inj}$ ; left: (a) Profiles of  $Q_{pres}$ ; right: (b) Profiles of  $Q_{wall}$ .



**Figure 22.** CO<sub>2</sub> pressure profiles with various  $T_{inj}$ .

# 4.3.1. Analysis of the Time Domain

Figure 23 shows the variation of the BHT with time under various tube inner diameters. The smaller the inner diameter of the tube, the faster the cooling rate. However, the CO<sub>2</sub> temperature of the 62-mm case was 11.8 °C higher than that of the 100.3-mm case after 120 min. This phenomenon was caused by the variation of the convective heat transfer efficiency and  $Q_{\text{fric}}$ . As shown in Figure 24, the flow velocity enhanced obviously with the decrease in the tube inner diameter, which resulted in a higher convective heat transfer effectiveness. However, the extra friction from the higher flow velocity  $Q_{\text{fric}}$  increased by 3489 W/m, which made the lowest temperature higher in the 62-mm case. Figure 25 shows the variation of the WHP with time under various tube inner diameters. Its effects on WHP were similar to that of the injection rate. When the inner diameter decreased from 100.3 mm to 62 mm, the wellhead pressure increased by 79.6 MPa.



**Figure 23.** Variation of BHT with time under various  $d_{ti}$ .



**Figure 24.** Mean v and mean  $Q_{\text{fric}}$  under various  $d_{\text{ti}}$ .



**Figure 25.** Variation of WHP with time under various  $d_{ti}$ .

# 4.3.2. Analysis of the Space Domain

Figure 26 shows the temperature profiles of  $CO_2$  with various tube inner diameters after 120 min. In the vertical section, when the inner diameter increased, the effect of viscous dissipation decreased and the  $CO_2$  temperature decreased. However, in the horizontal section, the temperature gradient of the 62-mm case decreased significantly. Although the viscous dissipation increased with the increase in the wellbore friction, the cooling effect of the pressure work also became increasingly obvious. The combined effect of the two factors determined the temperature gradient of  $CO_2$ . Figure 27a provides the sum of  $Q_{\rm fric}$  and  $Q_{\rm pres}$  under various tube inner diameters after 120 min, indicating that the smaller the inner diameter, the greater the heat flow gradient, especially in the horizontal section. The heat flow reduction rate was very fast in the 62-mm case: at a depth of 2300 m, the cooling effect of the pressure work exceeded the warming effect of the viscous dissipation, which created a negative sum of the two heat flows. Although the total heat flow remained positive throughout the wellbore due to the effect of the wall heat transfer (as shown in Figure 27b), the temperature gradient was anticipated to be negative along the wellbore if the horizontal section continued to extend in the 62-mm case.



**Figure 26.**  $CO_2$  temperature profiles with various  $d_{ti}$ .



**Figure 27.** Heat flow profiles with various  $d_{ti}$ ; left: (**a**) Profiles of  $Q_{pres} + Q_{fric}$ ; right: (**b**) Profiles of  $Q_{pres} + Q_{fric} + Q_{wall}$ .

Figure 28 indicates the pressure profiles of  $CO_2$  with various tube inner diameters after 120 min. In the vertical section, the hydrostatic pressure of the 100.3-mm case was greater than the wellbore friction, and the pressure gradient was positive; the wellbore friction of the 76- and 62-mm cases was greater than the hydrostatic pressure, and the pressure gradient was negative. In the horizontal section, as the hydrostatic pressure gradient was 0, the pressure gradients of the three cases were controlled by the wellbore friction, which made them all negative. The simulation results showed that the wellbore friction could be reduced significantly by increasing the inner diameter. Therefore, by increasing the inner diameter, the maximum of the  $CO_2$  injection rate could be increased, which is beneficial for improving the sand-carrying capacity of the  $CO_2$  fracturing fluid. At present, the newly developed 3000 type fracturing pump can stably provide 140 MPa pumping pressure [65]. Under the parameters of the benchmark case, using the wellhead pressure of 140 MPa as the ultimate pressure, the ultimate injection rate corresponding to different inner diameters

was obtained as shown in Figure 29. Apparently, when the inner diameter was enlarged from 50.3 mm to 100.3 mm, the ultimate injection rate was increased from  $2.7 \text{ m}^3/\text{min}$  to 29.6 m<sup>3</sup>/min.



**Figure 28.** CO<sub>2</sub> pressure profiles with various  $d_{ti}$ .



**Figure 29.** Ultimate injection rates with various  $d_{ti}$ .

#### 4.4. Analysis of the Completion Method of the Horizontal Section

The cases where the completion methods of the horizontal section were casing completion and open-hole completion were considered on the bases of the benchmark case. The main difference between the two methods was that casing and cement sheath existed between the annulus and the formation in the casing completion, which were otherwise absent in the open-hole completion.

# 4.4.1. Analysis of the Time Domain

Figure 30 shows the variation of the BHT with time under various completion methods. The BHT of the open-hole completion in the horizontal section was 2.7 °C higher than that of the casing completion after 120 min. The variation of the annulus natural convection explained this phenomenon. In the open-hole completion case, the radius of the annulus

was much larger than that of the casing case, which enhanced the efficiency of the natural convection. As shown in Figure 31a, compared with the casing completion case, the average annulus natural convection heat transfer coefficient of open-hole completion was  $30 \text{ W/(m}^{2.\circ}\text{C})$  higher. The enhancement of the natural convection in the annulus improved the wall heat transfer efficiency. Figure 31b indicates that the average  $Q_{wall}$  with open-hole completion method was 313 W/m higher than that of the casing completion method. Figure 32 shows the variation of the WHP with time under various completion methods. Apparently, the completion method had little effect on the WHP. This finding is further discussed in the analysis of the space domain.



Figure 30. Variation of BHT with time under various completion methods of the H section.



Figure 31. Cont.



**Figure 31.** Effects of the completion method of the horizontal section on mean  $h_{an}$  and mean  $Q_{wall}$ ; left: (a) Mean  $h_{an}$ ; right: (b) Mean  $Q_{wall}$ .



Figure 32. Variation of WHP with time under various completion methods of the H section.

4.4.2. Analysis of the Space Domain

Figure 33 shows the temperature profiles of CO<sub>2</sub> with various completion methods after 120 min. In the vertical and curved sections, the temperature profiles coincided, and the difference between the two cases only appeared after entering the horizontal section. As discussed above, the difference in the  $Q_{wall}$  was the main cause of the temperature variation. Figure 34 provides the  $Q_{wall}$  profiles with different horizontal section completion methods, which indicated that  $Q_{wall}$  was the same in the vertical and curved sections, whereas a jump ( $\Delta Q_{wall} = 817 \text{ W/m}$ ) occurred at the boundary between the curved and horizontal section, which increased the CO<sub>2</sub> temperature gradient in the horizontal section in the open-hole completion method. Therefore, open-hole completion for the horizontal section was beneficial to increase the BHT.



Figure 33. CO<sub>2</sub> temperature profiles with various completion methods.



Figure 34. Profiles with various completion methods.

Figure 35 shows the variation of the  $CO_2$  pressure profiles with time under various completion methods after 120 min. Firstly, as the completion method did not change the flow characteristics, the wellbore friction was the same at different wellbore sections. Secondly, the variation of  $CO_2$  density occurred at the horizontal section, in which the hydrostatic pressure gradient was 0. Therefore, the completion method had little effect on  $CO_2$  pressure in both the time and space domains.

In addition, the usage of the open-hole completion method also significantly increased the cooling effect of  $CO_2$  on the formation. As shown in Figure 36a,b, after injecting 120 min, the formation temperature near the wellbore of the open-hole completion case was lower by 24.5 °C than in the casing completion case.



Figure 35.  $CO_2$  pressure profiles with various completion methods.



**Figure 36.** Radial temperature distribution with various well completion methods after 120 min; left: (a) Casing completion; right: (b) Open-hole completion.

# 5. Conclusions

A 1D + 1D numerical model was established to investigate the horizontal wellbore temperature field in  $CO_2$  fracturing, which can provide an inexpensive and efficient way to investigate and guide the  $CO_2$  fracturing design. The convergence test and the comparison with the COMSOL software were performed to verify the model. The following conclusions were drawn on the bases of the assumptions and conditions of this work:

- 1. The higher the injection rate, the faster the reduction of temperature and the shorter the time for the BHT to reach stability. The relationship between the stable BHT and the injection rates is determined by the heat transfer mechanisms, which are non-monotonic. Under the parameters of the benchmark case, after 120 min, the corresponding injection rates were 2 m<sup>3</sup>/min, 10 m<sup>3</sup>/min, and 6 m<sup>3</sup>/min, according to the sequence of BHT from large to small.
- 2. In the horizontal section, the cooling effect of the pressure work and the heating effect of the wall heat transfer are enhanced. Under the combined effect of the two, the temperature gradient in the horizontal section was higher than that in the vertical section when the injection rate was  $2 \text{ m}^3/\text{min}$ . The opposite was the case where the injection rate was  $10 \text{ m}^3/\text{min}$ . Therefore, compared with the vertical section, the existence of the horizontal section will increase the BHT when the injection rate is low and reduce the BHT when the injection rate is high.
- 3. Wellbore friction increases exponentially with the increase in the injection rate. Enhancing the tube diameter can effectively reduce the wellbore friction, thus extra attention to casing fracturing is necessary. Based on the benchmark case and taking WHP at 140 MPa as the ultimate pressure, the ultimate injection rate increased from 2.7 m<sup>3</sup>/min to 29.6 m<sup>3</sup>/min when the tube diameter increased from 50.3 mm to 100.3 mm.
- 4. The open-hole completion method in the horizontal section enlarges the annulus space more than the casing completion method, which improves the natural convection heat transfer efficiency of the annulus. Based on the benchmark case, the BHT of the open-hole completion in the horizontal section was 2.7 °C higher than that of the casing completion after 120 min, whereas its effects on pressure were negligible. In addition, the open-hole completion in the horizontal section will significantly enhance the cooling effect of CO<sub>2</sub> on the formation near the wellbore. After 120 min, the formation temperature of the bottom-hole with the open-hole method was 24.5 °C lower than that in the casing completion method.

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# Nomenclature

- A cross sectional area of tubing,  $m^2$
- BHT bottom-hole temperature, °C
- BHP bottom-hole pressure, Pa
- $C_{\rm p}$  isobaric heat capacity, J/(kg·°C)
- $C_{pc}$  isobaric heat capacity of casing, J/(kg·°C)
- $C_{pf}$  isobaric heat capacity of CO<sub>2</sub>, J/(kg·°C)
- $C_{\text{pt}}$  isobaric heat capacity of tubing, J/(kg·°C)
- $C_{\text{pan}}$  isobaric heat capacity of annulus fluid, J/(kg·°C)
- $C_{\rm ph}$  isobaric heat capacity of cement, J/(kg·°C)
- $C_{pr}$  isobaric heat capacity of rock, J/(kg·°C)
- $d_{\rm ti}$  tubing internal diameter, m
- $f_{\rm D}$  coefficient of wellbore friction
- g gravity acceleration,  $m/s^2$
- $g_{\rm G}$  geothermal gradient, °C/m
- Gr Grasse number
- $h_{an}$  heat transfer coefficient of natural convection, W/(m<sup>2</sup>·°C)
- $h_{\rm t}$  heat transfer coefficient of forced convection, W/(m<sup>2</sup>·°C)
- *p* CO<sub>2</sub> pressure, Pa
- $p_{\rm w}$  bottom-hole pressure, Pa
- $P_{\rm f}$  wellbore friction of CO<sub>2</sub> fracturing, MPa/100 m
- *r* radial distance, m
- *r*<sub>ti</sub> tubing internal radius, m
- *r*to tubing external radius, m
- $r_{\rm ci}$  casing internal radius, m
- *r*<sub>co</sub> casing external radius, m
- $r_{\rm h}$  cement sheath radius, m
- $r_{\rm r,i}$  formation radius of unit i, m
- t time, s
- *T* temperature,  $^{\circ}$ C
- v CO<sub>2</sub> velocity, m/s
- $T_{inj}$  injection temperature, °C
- $T_{\text{ini}}$  initial temperature, °C
- $T_{\rm f}$  CO<sub>2</sub> temperature, °C
- $T_{\rm t}$  tubing temperature, °C
- $T_{an}$  annulus temperature, °C
- $T_{\rm c}$  casing temperature, °C
- $T_{\rm h}$  cement sheath temperature, °C
- $T_{r,i}$  formation temperature of unit i, °C
- WHP wellhead pressure, Pa
- *z* measured depth, m
- $z_{bh}$  measured depth of the bottom-hole, m
- $\alpha_{\rm p}$  coefficient of CO<sub>2</sub> thermal expansion, 1/°C, take  $-1/\rho_f(\partial \rho/\partial T_f)$  here
- $\beta_{an}$  coefficient of annulus thermal expansion, 1/°C, take 2.5 × 10<sup>-4</sup> here
- $\theta$  deviation angle
- $\rho$  density, kg/m<sup>3</sup>
- $\rho_{\rm f}$  density of CO<sub>2</sub>, kg/m<sup>3</sup>
- $\rho_{\rm t}$  density of tubing, kg/m<sup>3</sup>
- $\rho_{an}$  density of annulus fluid, kg/m<sup>3</sup>
- $\rho_{\rm c}$  density of casing, kg/m<sup>3</sup>
- $\rho_{\rm h}$  density of cement, kg/m<sup>3</sup>
- $\rho_{\rm r}$  density of rock, kg/m<sup>3</sup>
- $u_{\rm f}$  viscosity of CO<sub>2</sub>, Pa·s
- $u_{an}$  viscosity of annulus, Pa·s
- $\lambda$  thermal conductivity, W/(m·°C)
- $\lambda_{\rm f}$  thermal conductivity of CO<sub>2</sub>, W/(m·°C)
- $\varepsilon_{an}$  thermal conductivity of annulus fluid, W/(m.°C)
- $\lambda_{\rm c}$  thermal conductivity of casing, W/(m·°C)
- $\lambda_{\rm h}$  thermal conductivity of cement, W/(m·°C)
- $\lambda_{\rm r}$  thermal conductivity of rock, W/(m·°C)

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