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Aerodynamic Analysis of Variable Camber-Morphing Airfoils with Substantial Camber Deflections

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Abstract: In recent years, morphing wings have become not only a concept, but an aerodynamic solution for the aviation industry to take a step forward toward future technologies. However, continuously morphing airfoils became an interesting answer to provide green energy solutions. In this paper, the authors conducted experimental research on a continuously camber-morphing airfoil using the Particle Image Velocimetry (PIV) and Computational Fluid Dynamics (CFD) methods. The main objective of this work was to research a variety of morphing airfoils with different camber deflections. An average velocity distribution and turbulence distribution were compared and are discussed. The two-dimensional PIV results were compared to the CFD simulations to validate the numerical method's accuracy and obtain the aerodynamic coefficient's trends. A further comparison revealed that morphing airfoils have better aerodynamic performance than conventional airfoils for very low camber deflections and create substantial amounts of drag for significant camber deflections.

Keywords: morphing airfoil; micro UAV; wind turbines; tidal turbines; particle image velocimetry (PIV); computational fluid dynamics (CFD)

1. Introduction

Morphing wings are intelligent wing structures and their concept is far from new, but the development of smart materials in the 21st Century allowed researchers to excavate them from the dust of times. There are many camber-morphing ideas: some of them have superior aerodynamic improvements, and some have minor ones [1]; however, the fact is that camber-morphing is an aerodynamic improvement, just as nature has shown from the beginning. Observing the smooth and gentle way birds of pray sail through the air using their wing-morphing abilities has inspired researchers of all times to create visionary aerodynamic concepts [2–4]. The dynamic development of intelligent shape-changing materials in the 21st Century empowered the scientific world to excavate these concepts and adjust them to the economic needs of the current era of aviation [5–7].

Over the years, there have been numerous projects and research works conducted regarding camber-morphing airfoil concepts for commercial and micro UAV aircraft [1,8,9]. Nguyen et al. [10] presented the Variable Camber Continuous Trailing Edge Flap (VCCTEF) concept for commercial aircraft application, which has been developed by NASA for a few years. The conclusion demonstrated significant drag reduction via the application of the



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Copyright: © 2024 by the authors. Licensee MDPI, Basel, Switzerland. This article is an open access article distributed under the terms and conditions of the Creative Commons Attribution (CC BY) license (https:// creativecommons.org/licenses/by/ 4.0/). morphing airfoil concept and great potential for fuel and energy savings by the adaptive change of airfoil during flight. Ting et al. [11] further researched the VCCTEF concept. Using numerical simulations, they analysed the aerodynamic loads and hinge moments acting on the Generic Transport Model (GTM) morphing wing aircraft, finding that the increase in the hinge moment is proportional to the stiffness decrease. A consecutive step in developing the VCCTEF concept was an optimisation study conducted by a similar team of researchers as in the previously recalled work. They revealed that a three-segment morphing trailing edge has the potential to be the most optimal for drag reduction for transport aircraft [12]. Jo et al. [13] compared the NACA8410 and NACA2410 airfoils and then created wings with a variably camber-morphing along the span direction. Using CFD simulations, they discovered that the NACA8410 has a significantly higher lift force coefficient and a better pressure distribution in the examined cases than the NACA2410.

As for publications considering bio-inspired morphing wings at very low Reynolds numbers [14], they usually refer to micro Unmanned Aerial Vehicles (UAVs) (see Figure 1). Anoyoji et al. [15] studied owl-like airfoils for low Reynolds number flights of Micro Air Vehicles (MAVs). The NACA0012 airfoil was compared with a smooth owl-like airfoil for Re = 23,000 - 60,000. The authors found that a strong under-camberfor morphing airfoils causes a higher lift force coefficient compared to the NACA0012 airfoil. Another bio-inspired structure for MAVs was researched by Bardera et al. [16]. They conducted three-dimensional studies using the PIV and CFD methods for a stingray-like MAV. Three different morphing stages were examined, and it was concluded that semi-deformation demonstrated the best aerodynamic improvement. Tamai et al. [17] proposed a flexiblemembrane airfoil for an MAV. It could change the shape of the wing in flight to adapt to the current conditions like natural flyers do. They investigated the created membrane airfoil using PIV. Majid and Jo [14] investigated morphing wings at a low Reynolds number. They examined nine cases in CFD simulations, achieving an 18.7% increase in the lift-to-drag ratio for the morphing geometry compared to the conventional airfoil. Bardera et al. [6] established morphing geometries that could prevent flow from separation, significantly enhance the lift force coefficient, and reduce turbulence. A technical approach to UAV morphing wings was presented by Dhileep et al. [18]. They investigated a single corrugated variable camber concept and demonstrated that the morphing airfoils exhibited increased the performance compared to the traditional ones. The potential existence of the optimal deflection angle to provide the maximum lift-to-drag ratio for every angle of attack was another important conclusion of this work.



Figure 1. Example of UAV aircraft from Wrocław University of Science and Technology.

There are also publications considering extremely low Reynolds numbers. Zhu et al. [19] conducted research on a rectangular cylinder for Re = 200–400. They studied the flutter phenomenon of a canonical rectangular cylinder model. Their final conclusion was that studying the behaviour of the vortices is crucial to understanding the flutter phenomenon and that even simple low Reynolds number simulations contain important data that can be utilised for future research on higher Reynolds numbers. Another research work on low Reynolds number was conducted by Liu et al. [20]. They created a model of two cylinders in tandem and studied the fluid forces acting on the model for Re = 75–200. The authors demonstrated the reduction of vortex shedding, as well as the decrease of the fluid forces compared to the extensively researched single-cylinder model.

The airfoil morphing concept went beyond only aviation applications a long time ago. Currently, this is a promising solution raised in every field that uses airfoils or hydrofoils as more energy efficient and having higher aerodynamic performance. Fatiha et al. [21] presented a morphing hydrofoil for ship propeller blades. They established that the hydrofoil shape has a significant impact on hydrodynamic forces, and with morphing, the hydrofoil cavitation phenomenon can be controlled. Remaining in heavy load variations, there is one important subject to consider-wind gusts. For example, waves create hydrokinetic loads on tidal turbines, and strong wind gusts may create potentially dangerous aerodynamic loads on wind turbines. An example of such a treacherous condition is a downburst, in aviation considered very risky during the landing of aircraft and for UAVs. Frant et al. [22] created an aerodynamic state-of-the-art CFD method to model gusts, allowing modelling any speed and direction or time-varying direction of a gust. They conducted aerodynamic wind tunnel tests to validate the numerical model. The results showed that a 10 m/s gust can significantly change the aerodynamic forces and moments acting on the wing. Another numerical model was proposed by Zhou et al. [23]. The authors created a surrogate model for wind turbine wake prediction. The potential of this model enables predicting fluctuating wake structures' dynamics in hazardous terrain and extreme weather conditions.

Wind turbines are the main objective of current green energy solutions. The aim is to harvest more energy with the lowest possible costs. Thus, the number of wind turbines is growing drastically. Longer blades and higher pylons allow exploiting faster breezes at greater altitudes. This causes higher loads on turbine blades and generates potential construction difficulties. Passive adaptive blades have proven their ability to reduce loads on wind turbine blades in varying weather conditions. Changes in aerodynamic coefficients created by trailing-edge deflection present a linear relation to wind speed [24]. Experimental and numerical approaches to passive adaptive wind turbine blades were presented in Murray's work [25]. The research revealed a significant reduction of loads for adaptive flexible airfoils compared to rigid blades. Beyene and Peffley [26] conducted wind tests confirming that flexible blades decrease loads to even 10% compared to rigid ones. They concluded that flexible blades adapt better to different load conditions with zero additional costs for harvested energy. The innovative response to the growing problem of high loads on turbine blades could be morphing airfoils. They could also be utilised to support wind turbine rotor braking in a variety of situations. With the development of smart materials and the continuously appearing new technical approaches to morphing airfoils, these active flexible structures could adapt more efficiently to varying weather conditions or hazardous situations and improve efficiency with their tested load reduction and aerodynamic efficiency improvement [27].

Following green energy solutions, morphing airfoils have found their place in tidal turbine applications. Oceanic currents are an unexploited source of green energy with great potential and could seriously strengthen energy security. Hoerner et al. [28] experimented with hydrokinetic vertical-axis or cross-flow tidal turbine (CFTT) morphing blades. They concluded that the flexibility of the camber-morphing airfoil has a significant impact on the wake structure and allows less accidental wake changes over time. A passive–adaptive geometry of airfoils was presented by Castorrini et al. [29]. The authors conducted 2D and 3D FSI simulations of three small rotor geometries, also for fluctuating water flow

conditions. The study revealed the computational fluid–structure interaction to have great possibilities for modelling tidal turbine rotors. Present tidal turbines encounter several inconveniences such as current fluctuations caused by waves, large hydrokinetic loads' inconstancy, or interaction with turbine supports. In their theoretical study, Pisetta et al. [30] discussed morphing blades with such flexibility, for which the load increase from current fluctuations can be reduced by even 99% without affecting the amount of energy harvested.

Another interesting application of morphing airfoil structures is the automotive industry. Flexible structures can create additional passively developed forces for increased grip and enlarged stability in corners. Mishra et al. [31] conducted research on Flexi Wings for Formula 1 race car application. This concept can provide more precise air flow direction adjustment, which enhances manoeuvrability and optimises steering control. The interaction between components can significantly increase or decrease race car performance. Cravero et al. [32] conducted a numerical analysis of the aerodynamic interaction between the front wing and front wheel in a Formula 1 race car. The research revealed that the rotating wheel enhances the aerodynamic performance of the race car's front section and that swirling flow over the front wing lowers the pressure before the air intake, generating increased air intake efficiency. This phenomenon could be strengthened by the application of a morphing wing structure. Active aerodynamic morphing structures and their ability to change aerodynamic performance could enhance safety during the Brake-In-Turn manoeuvre. Broniszewski and Piechna [33] created an algorithm combining a CFD solver with a car dynamics' solver. The authors proposed an aerodynamic solution for in-turn braking enhancement by creating additional downforce on the rear wheels and enabling active prevention of control loss during corner braking.

Camber-morphing airfoils are a multidisciplinary subject with an emphasis on aviation. The recognition of the turbulence around morphing structures, as well as creating new camber-morphing airfoils and comparing them to conventional ones are crucial for creating optimal aerodynamic morphing airfoils for aviation [13] as energy solutions [34]. Thus, this paper investigates changes in the aerodynamic efficiency of the airfoils for representative angles of attack and presents the turbulence distribution at very low Reynolds numbers. The main objective of this work is to research a variety of morphing airfoils with different camber deflections. The majority of the literature focuses on very limited camber deflections, and the authors find high cambered morphing airfoils an interesting and insufficiently researched field of aerodynamics. The research was conducted in a two-dimensional way, as for the PIV method, as for CFD simulations. Experiments were performed in a hydrodynamic tunnel using PIV for six airfoil models with various morphing stages. The CFD simulations were obtained using the *OpenFOAM* software and utilising the k ω -SST turbulence model. The CFD environment was a representation of an experiment conducted in an aerodynamic water tunnel. The lift force coefficients, drag coefficients, and lift-to-drag ratio characteristics are presented.

2. Materials and Methods

2.1. Camber-Morphing Airfoils

The literature presents camber-morphing airfoils in various ways. The deformation could be applied to the leading edge section [35], trailing edge section [36], or both sections of an airfoil [37,38]. The most popular is a continuous camber-morphing trailing edge, as it replaces conventional flaps and is undisturbed by the slit geometry. It is also most suitable for wind or water turbine applications. The authors created an airfoil with 75% camber morphing. The base airfoil was the NACA24012, and a 25% mean aerodynamic chord (MAC) from the leading edge was fixed. Figure 2 presents the morphing airfoil geometries: The Airfoil 1 geometry corresponds to the NACA24012 airfoil. Airfoils 2–6 have variable continuous camber deflections. Airfoil 1 has an initial angle of attack of 0°. Camber-morphing airfoils have the trailing edge moved downwards, and the same as Airfoil 1's position of the leading edge section, they have different initial angles of attack, as presented in Table 1. The geometries were prepared to cover camber deflections from

non-cambered (Airfoil 1—NACA24012 airfoil) to fully cambered (Airfoil 6). Between these two airfoils, an additional four were created to cover the whole range of deflections.



Figure 2. Geometries of tested airfoils.

Table 1. Airfoil dimensions.

Parameter	Airfoil 1	Airfoil 2	Airfoil 3	Airfoil 4	Airfoil 5	Airfoil 6
Chord (c), mm	250	249	246	240	231	221
Initial angle of attack, $^\circ$	0	7	13	19	26	32

Three-dimensional models were created from the prepared two-dimensional airfoil geometries. The models were cut into pieces suitable for a 3D printer. Then, the geometries were printed from polylactic acid (PLA) material using an additive manufacturing method. As this method of printing is imperfect and leaves the surface of the obtained geometry rough, the printed models were first glued from pieces into full models, then coated with plastic primer and aerosol filler. Next, the surface was abraded and polished, then finally coated with paint (Figures 3 and 4).

The experiments in the water tunnel and CFD simulations were performed for every airfoil geometry with a setting angle of 0°. The angle of attack and setting angle coincide only for Airfoil 1 due to the bending of the airfoil and moving the trailing edge downwards, which causes an increase in the initial angle of attack for the morphed airfoils.



Figure 3. Process of preparing models for painting. From left: abrading, glued models.

2.2. Water Tunnel Experiments

The majority of published articles correspond only to numerical calculations [39]. To fully understand the processes in the wake structure behind the airfoil, it is crucial to perform experiments with the ability to observe emerging turbulence [40]. Hydrodynamic tunnels appear to be the best solution for observing turbulence as water has a density about 800-times greater and a dynamic viscosity 50-times higher than air in a standard

AIRFOIL 1 AIRFOIL 2 AIRFOIL 3 **AIRFOIL 4** AIRFOIL 6 **AIRFOIL 5**

atmosphere. Therefore, water has more potential for studying turbulence as it is easier to achieve flow visualisation in a denser environment than in an air aerodynamic tunnel,

where the visualisation medium is quickly dispersed [41,42].

Figure 4. Presentation of completed models for experiments in the water tunnel.

2.2.1. Experimental Setup

The experiments were performed in the Rolling Hills Research Corporation hydrodynamic tunnel model 2436 with closed fluid circulation (Figure 5). The test section is a rectangular shape of 1830 mm long, 610 mm wide, and 915 mm deep.



Figure 5. The Rolling Hills Research Corporation hydrodynamic tunnel.



The model was mounted on a three-dimensional support system, enabling active movement of the tested geometry during the experiment. The tunnel is equipped with a thermometer and velocity meter for the fluid flow. The measurement of these two values, in cooperation with computer software, allows one to calculate in time Reynolds number. The water flow is provided by an inverter and controlled in computer software in the range from 0 to 280 mm/s [43,44].

The experiments were performed for flow velocity v = 0.0762 m/s (which corresponds to 3 in) and kinematic viscosity $v = 8.71 \times 10^{-7}$ m²/s. Calculated from the below-presented equation, the Reynolds number for the representative Airfoil 1 was Re = 21,900.

$$Re = c * v / v, \tag{1}$$

The test section was equipped with two linear lasers synchronised to reduce to minimum shade region around the tested geometry (Figure 6). The high-speed camera was positioned in front of the tested geometry (Figure 7).



Figure 6. Schematic of the water tunnel experiment station.



Figure 7. Presentation of test section with PIV equipment.

2.2.2. Particle Image Velocimetry

Particle Image Velocimetry (PIV) is widely used in the analysis of fluid dynamics [45] and fluid–structure interactions [46]. It is a non-intrusive optical method used for the experimental calculation of fluid in a region of interest. Neutrally buoyant particle seeding is highlighted by the linear laser in the area of analysis (Figure 8).



Figure 8. The image taken during the measurements in the water tunnel.

Images of the illuminated particle patterns are captured by a high-speed camera. Then, the calculations are made, taking into consideration the following two images, and the velocity filed distribution is recorded into one matrix. Further, the mean value of the velocity is calculated based on the following 500 matrices. Taking into consideration the image acquisition frequency (frame per second parameter of the high-speed camera), each of the PIV results represents the mean values of the velocity measured over time for 0.5 s. In our study, we used the high-speed camera with the 250 frames per second setting. The final output of the PIV process is a velocity field representation, as presented in Section 3. This visualisation shows the fluid velocities for all tested airfoils.

The results of the Particle Image Velocimetry (PIV) method depend significantly on the interrogation window size (number of pixels taken in cross-correlation calculation). Smaller interrogation windows yield higher spatial resolution, allowing for the detection of finer flow structures. This is especially important in turbulent flows, in the region of analysis close to the airfoil trailing edge. However, smaller windows also mean fewer particles per window, which affects the accuracy and reliability of the velocity measurements. The size of the interrogation window affects the number of particle image pairs available for cross-correlation analysis within each window. A larger window contains more particle pairs, which improves the statistical reliability of the velocity measurement. However, if the window is too large, it may encompass regions of flow with significantly different velocities, leading to decreased accuracy due to averaging over disparate flow regions.

2.3. Computational Fluid Dynamics Simulations

2.3.1. Mathematical Model

In this study, flow simulations were modelled by solving combinations of steady-state incompressible Navier–Stokes equations given by:

$$(\mathbf{u} \cdot \nabla)\mathbf{u} = -\nabla(p/\rho) + \nu \nabla^2 \mathbf{u},$$
 (2)

and the continuity equation is stated as:

$$\nabla \cdot \mathbf{u} = 0 \tag{3}$$

where the fluid velocity vector was deemed as $\mathbf{u} = (u_x, u_y)$, the fluid density is ρ , pressure is stated as p, and the kinematic viscosity is labelled as v. The gravity term is omitted in Equation (2) in order to avoid streaming flow interference [47]. Equations (2) and (3) underwent discretisation through the Finite-Volume Method (FVM) approach. The numerical simulations were computed using *OpenFOAM* (https://www.openfoam.com/), a C++ open-source toolbox designed primarily for the development of customised numerical solvers [48–50]. The simulations conducted in this study employed the semi-implicit method for pressure linked equations (SIMPLE) solver algorithm [51–53]. The chosen turbulence model was the *k*- ω shear stress transport (SST), which is widely recognised as an industry standard [54–56] and combines the strengths of both the Wilcox *k*- ω model and the *k*- ϵ model, utilising a blending function. The convective form of the equations for the steady-state *k*- ω SST model is expressed as follows [57]:

$$\nabla \cdot (\mathbf{u}k) = \frac{P}{\rho} - \beta^* \omega k + \nabla \cdot [(\nu + \sigma_k \nu_t) \nabla k]$$
(4)

$$\nabla \cdot (\mathbf{u}\omega) = \frac{\gamma}{\mu_t} P - \beta \omega^2 + \nabla \cdot \left[(\nu + \sigma_\omega \nu_t) \nabla \omega \right] + 2(1 - F_1) \frac{\sigma_{\omega 2}}{\omega} \nabla k \nabla \omega$$
(5)

where ρ is the fluid density, k is the turbulent kinetic energy, ω is the specific dissipation rate, **u** is the fluid velocity vector, μ_t is the turbulent eddy viscosity, ν is the kinematic viscosity, ν_t is the turbulent kinematic viscosity, σ_k is the turbulent Prandtl number for k, σ_{ω} is the turbulent Prandtl number for ω , P is the production term, and F_1 is the blending function. Variables such as β , β^* , γ , σ_k , σ_{ω} , and $\sigma_{\omega 2}$ are constants.

2.3.2. Aerodynamic Coefficients

The lift, drag, and momentum coefficients are defined as follows:

$$C_L = \frac{F_L}{A_{ref} \cdot p_d} \tag{6}$$

$$C_D = \frac{F_D}{A_{ref} \cdot p_d} \tag{7}$$

$$C_M = \frac{M}{A_{ref} \cdot l_{ref} \cdot p_d} \tag{8}$$

$$p_d = \frac{\rho_{ref} \cdot U_{mag}^2}{2} \tag{9}$$

where F_L and F_D are forces perpendicular to and parallel with the direction of the flow, M is the moment acting around the axis of rotation, A_{ref} corresponds to the total wing surface area, and l_{ref} in this case is the chord length. Dynamic pressure p_d is defined by the reference density ρ_{ref} and the velocity magnitude U_{mag} . For incompressible cases, Equations (6)–(8) are solved using the kinematic pressure $p_k = p_d / \rho$ [58,59].

2.3.3. Discretisation Schemes

The discretisation schemes are discussed based on the *OpenFOAM* source code. The summary and accuracies [60–62] of the selected schemes are provided in Table 2. The *steadyState* time scheme sets temporal derivative contributions to zero. *Gauss linear* is the default gradient scheme that was used. The *Gauss* entry specifies the standard finite-volume discretisation with Gaussian integration, which requires the interpolation of the values from the cell centres to the face centres. The interpolation scheme is then given by the *linear* interpolation or central differencing. The discretisation of the velocity gradient was overridden with the *cellLimited* scheme, which improves boundedness and stability. The divergence schemes contain both advection terms and other terms, which are often diffusive in nature. The non-advective terms used the *Gauss* integration with *linearUpwind* or *upwind*

interpolation. The *bounded* variant of the Gauss scheme helps maintain boundedness and promotes better convergence. The surface normal gradient is evaluated at a cell face.

Schemes	Terms	Scheme Type	Accuracy
Time	default	steadyState	time derivative set to 0
Gradient	default ∇ u	Gauss linear cellLimited Gauss linear 1	2nd order, unbounded 2nd-order, cell-limited version
Divergence	$ \begin{array}{l} \text{default} \\ \nabla \cdot (\phi \mathbf{u}) \\ \nabla \cdot (\phi k) \\ \nabla \cdot (\phi \omega) \\ \nabla \cdot \left((\nu_{\text{Eff}} \cdot \text{dev}^2(\mathbf{T}(\nabla \mathbf{u}))) \right) \end{array} $	none; bounded Gauss linearUpwind grad(U) bounded Gauss upwind bounded Gauss upwind Gauss linear	specified for each term 2nd-order, bounded 1st-order, bounded 1st-order, bounded 2nd-order, unbounded
Laplacian	default	Gauss linear corrected	2nd-order, unbounded
Interpolation	default	linear	2nd-order, unbounded
Surface normal gradient	default	corrected	2nd-order

Table 2. Summary of discretisation schemes used in the study.

The calculation is 2nd-order accurate, if the vector connecting the cell centres is orthogonal to the face (*orthogonal* scheme) or an explicit non-orthogonal correction has been added (*corrected* scheme), which is recommended for maximum non-orthogonality $\leq 75^{\circ}$. As meshes in this study had a maximum non-orthogonality of 35° , this exact term was used. The *Gauss* scheme is the only choice of Laplacian discretisation. In all cases, the *linear* scheme was used for interpolation of the diffusivity. The cases used the same array of surface normal gradient schemes based on the maximum non-orthogonality in the mesh, as described above. The interpolation schemes contain terms that are interpolations of values typically from cell centres to face centres. There are numerous interpolation schemes in *OpenFOAM*, but the *linear* interpolation is the most frequently used [60–62]. While the majority of schemes were 2nd-order accurate, $\nabla \cdot (\phi k)$ and $\nabla \cdot (\phi \omega)$ used 1st-order schemes in order to provide better stability of the simulations, which was recommended due to their diffusive nature.

2.3.4. Geometry for the CFD Case Study

The geometries analysed in the study are shown in Figure 2. Airfoil chords varied between c = 221 and 250 mm and were differentiated by camber lines. More detailed information on the airfoil geometries can be found in Table 1.

2.3.5. Computational Domain

The determination of computational domain dimensions conventionally relies on the geometric characteristics of the subject of investigation. In the context of airfoils and wings, this parameter is typically defined by the mean aerodynamic chord (MAC) or simply the chord length, denoted as *c*. The exact size of the computational domain needed to accurately capture the physics varies depending on the problem. However, there are general guidelines for determining the domain dimensions. For 2D subsonic aerodynamic simulations, it is recommended that the domain length should be *5c* at minimum or ideally $\geq 100c$ [63–65]. Nonetheless, this is a broadly generalised guideline for external aerodynamics, with Reynolds numbers reflecting real-world flow conditions. In addition to these general prerequisites, feasibility and computing power must also be taken into account. Under appropriately chosen boundary conditions, the determination of the computational domain size, in terms of external aero- or hydrodynamics, focuses on one primary consideration: the magnitude of error resulting from the domain boundaries positioned too closely to the studied object [66].

As the CFD method served merely as a supportive research tool in this preliminary study of morphing airfoils and considering that the flow conditions were characterised by low Reynolds numbers, very low flow velocity, and water as the medium, which did not impose real external aerodynamic demands, we opted for smaller domain dimensions of *30c*. This decision represents an acceptable trade-off, balancing potential errors as highlighted on NASA's website [66] and in other studies [67]. There is a substantial number of airfoil studies that propose similar domain dimensions, as exemplified in [68,69].

The airfoils were meshed with the use of the mesh utility named snappyHexMesh, which comes with the *OpenFOAM* package. This tool produces hex-dominant meshes from triangulated geometries and adds layers during the concluding stage of the meshing process. As the primary focus of the study was centred on Particle Image Velocimetry (PIV) experiments, a decision was made to prioritise cell refinement in the near-airfoil region to deliver high-resolution results, which would serve as a reliable basis for comparison between the images extracted from the PIV results and CFD simulations. The development of the numerical mesh was divided into two parts: the computational domain and the circular shape containing the airfoil. The computational domain mesh had an opening where the numerical mesh of the airfoil was subsequently inserted and stitched together. This solution will allow the easy rotation of the airfoil in future wide-rangenumerical simulations. These methods are widely known and often utilised by researchers, as exemplified in [69,70].

Figure 9 illustrates the conceptual shape of the computational domain with marked distances and dimensions.



Figure 9. Computational domain and boundary conditions for the airfoil simulations.

The airfoil chord length is denoted as *c* and is provided in Table 1. The size of the domain was specified as *10c* for the front, top, and bottom of the airfoil and *20c* behind the airfoil. This resulted in dimensions of *30c* along the *x*-axis and *20c* along the *y*-axis. A smaller dimension was selected for the *y*-axis due to the consistent inflow direction across the cases. Specifically, the angle of attack will be manipulated by rotating the inner cell zone containing the airfoil. Consequently, the predominant flow direction remains along the *x*-axis until it reaches the airfoil. The dimensions in the *y*-axis direction were tailored to accommodate flow deflection and turbulent effects induced by the airfoil. Bearing this

consideration in mind, it was found that employing 20c dimensions downstream of the airfoil adequately captured the resulting flow pattern.

The *z*-axis dimension is absent in this study as the cases were two-dimensional (2D). The centre point of the mesh, (x, y) = (0, 0), was specified as the airfoil's aerodynamic centre. The rotating cell zone had a diameter of 2*c* to accommodate the airfoil and proper cell refinement zones around it. The angle of attack could be obtained by rotating the inner cell zone, which contained the airfoils. The numerical grid was regularly refined in the area around the airfoils and in their slipstream. Additionally, the region near the airfoils also underwent refinement. Each cell level step had a blending function of seven cells in between the cell levels. In the arbitrary mesh interface (AMI) region, the meshes had one refinement function of level 5 at a distance of 0.05 m from the airfoil geometry. The airfoil silhouette was represented by a surface refinement cell level of 6, which translates to a cell edge length of 1.56×10^{-4} m.

The length of the edges of the base cells was 0.01 m, and the cell refinement in the domain was performed with an expansion factor $E = \frac{\delta_t}{\delta_b}$, where δ_t is the ratio of the cell height at the top of the grid block to the cell height at the bottom of the grid block δ_b . For the lower grid block, $E_b = 4$, and for the upper grid block $E_t = \frac{1}{4}$. The developed 2D numerical grids comprised around 3,370,000 cells. To represent the viscous sublayer, n = 25 layers with an inflation factor of $\delta = 1.2$ were generated. The thickness of the first layer was $y = 7.862 \times 10^{-7}$ m, ensuring a dimensionless wall distance y + < 1 for all geometries (Figure 10). Key features of the numerical grids are presented in Figure 11.



Figure 10. Distribution of *y*+ around tested airfoil. Example for Airfoil 1.

2.3.6. Flow Characteristics and Boundary Conditions

Numerical studies simulated the 2D hydrodynamics around the airfoil geometries in a steady state. The flow was deemed as subsonic with very low Mach or Reynolds numbers. The material settings were the same as is the case for water at 26.3 °C. The material was a single-phase, non-reacting, and incompressible Newtonian fluid. The flow characteristics can be found in Table 3.

Label	Quantity
Туре	Steady-state 2D hydrodynamics
Fluid	Newtonian, single-phase, incompressible
Material	Water at 26.3 °C
Water density	$ ho = 996.784 \ \frac{\text{kg}}{\text{m}^3}$
Kinematic viscosity of fluid	$\nu = 8.71 \times 10^{-7} \frac{\text{m}^2}{\text{s}}$
Reynolds number	Re = 19,400–21,900
Streamwise far-field flow speed	$U = 0.0762 \frac{m}{s}$
Characteristic length (airfoil chord)	c = 0.231 - 0.25 m



Figure 11. A detailed look at critical sections of the numerical grid—examples of Airfoil 1 grid: (a) computational domain view, (b) cell levels 0, 2, and 3 close-up, (c) cell level 3 and rotating cell region (AMI) close-up, (d) rotating cell region (AMI) and airfoil wall surface cell level 6 close-up, (e) front section of airfoil close-up, (f) leading-edge close-up, and (g) layers in boundary layer on leading edge close-up.

Equations (11)–(13) [71] were used to calculate the initial values of k and ω . Tables 4 and 5 represent the boundary conditions for the steady-state simulations. The velocity components

14 of 22

 u_x , u_y are calculated from the velocity magnitude **u** and angle of attack α . The equations for k, ω , and ω_{moll} were taken from the *OpenFOAM* user guide [71] and are presented below:

$$(u_x, u_y) = (\mathbf{u}\cos\alpha, \mathbf{u}\sin\alpha) \tag{10}$$

$$k = \frac{3}{2} (\mathbf{u}I)^2 \tag{11}$$

$$\omega = \frac{k^{0.5}}{C_{\mu}^{0.25} \cdot L}$$
(12)

$$v_{wall} = \frac{6\nu}{\beta_1 \nu^2} \tag{13}$$

where *I* is the turbulence intensity, **u** is the velocity magnitude inside the domain, C_{μ} is a constant of 0.09, *L* is the reference length scale, *v* is the kinematic viscosity, β_1 is a constant of 0.075, and *y* is the wall normal distance.

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Table 4. Boundary conditions for velocity, pressure, and kinematic viscosity.

BC/Field	u, <u>m</u>	$p_{\prime} \frac{m^2}{s^2}$	v_t , $\frac{\mathrm{m}^2}{\mathrm{s}}$
Inlet	$u_x = 0.0762$	$\frac{\partial p}{\partial n} = 0$	solved by k- ω model
Outlet	$\frac{\partial \mathbf{u}}{\partial n} = 0$ and $u_x = 0.0762$	p=0	solved by k- ω model
Top/bottom	cyclic	cyclic	empty
Left/right	empty	empty	cyclic
AMI domain/airfoil	cyclic	cyclic	cyclic
Airfoil	$\mathbf{u} = 0$	$\frac{\partial p}{\partial n} = 0$	$v_{t_{wall}}$ function

Table 5. Boundary conditions for turbulent kinetic energy and turbulence specific dissipation rate.

BC/Field	$k, \frac{m^2}{s^2}$	ω , $\frac{1}{s}$
Inlet	$rac{\partial k}{\partial n}=0$ and $k=8.71 imes 10^{-7}$	$rac{\partial \omega}{\partial n}=0$ and $\omega=0.0077$
Outlet	$\frac{\partial k}{\partial n} = 8.71 \times 10^{-7}$	$\frac{\partial \omega}{\partial n} = 0$ and $\omega = 0.0077$
Left/right	empty	empty
Top/bottom	cyclic	cyclic
AMI domain/airfoil	cyclic	cyclic
Airfoil	k_{wall} function	$\omega_{\scriptscriptstyle wall}$ function

The convergence criterion for steady-state simulations utilised residual control for velocities $u_x|u_y \le 1 \times 10^{-6}$, kinematic pressure $p \le 1 \times 10^{-6}$, turbulent kinetic energy, and the specific dissipation rate $k|omega \le 1 \times 10^{-6}$. This meant that the simulations automatically stopped when all residuals reached the specified threshold. If these criteria were not satisfied, a backup convergence criterion was set at a total number of iterations n = 35,000.

3. Results and Discussion

In this section, the results of the obtained experiments and simulations are discussed. As explained in Section 2.3.5, the main focus of the study was centred on the PIV experimental approach. CFD simulations were performed to create an aerodynamic coefficient characteristics' trend. For that reason, the medium remained water and the boundary conditions, such as velocity, remained unchanged. The validation of the numerical model was based on the velocity distribution. The obtained CFD results were of good quality and good enough to provide sufficient aerodynamic coefficient numerical results, especially for the preliminary research this work presents. Nevertheless, the authors present some insight into the differences between experimental and computational velocity distribution results.

In the figures (Figures 12–17), the PIV and CFD results are presented. The shapes of the airfoils were inserted into PIV results to clarify the actual airfoil shape in every

condition, as the shaded region in PIV varies and is not always identical to the tested geometry, especially in the area in front of the leading edge.

Figure 12 presents the velocity distribution for the NACA24012 airfoil, in this article, named Airfoil 1. As one can see, the velocity distribution is in the same range for both graphic results, and the laminar flow from the CFD method is less turbulent. Even for low Reynolds numbers and non-morphed airfoils, the wake structure behind the airfoil is turbulent, which is clearly visible in the discussed figure on the PIV result. The CFD results are numerical simulations taking ideal conditions and simplifying the environmental models, as the PIV method is an experimental method conducted in a real water tunnel. This simple comparison confirms the demand for an experimental approach to the turbulence study.



Figure 12. Velocity distribution result from PIV (left) and CFD (right) for Airfoil 1.

In Figures 13 and 14, the CFD results, there are two areas of increased velocity, one above the airfoil and a second behind the trailing edge. The PIV results show a similar distribution of velocity, but in the CFD results, the velocity in this area is higher. The turbulence is comparable, taking into consideration that the PIV results are average values from 500 camera frames, and the CFD result is presented for specific iterations. Despite the clearly visible differences between the presented results, the velocity distribution and turbulence distribution are comparable, which enables the validation of the CFD results.

In Figure 15, again, there is an area of increased velocity above the airfoil that spreads widely behind the airfoil geometry. This occurs for both the CFD and PIV results, and only the values of the maximum velocity differ. The wake structure is moved downwards in the CFD result, due to the increased velocity area behind the airfoil. Behind the trailing edge, in the PIV result, there is an increase in the velocity, and for the CFD results, this increase has a circular shape. As the PIV results' values were averaged and show only the velocity distribution and not the eddy circulations, the actual flow is wrapping around the trailing edge, as shown in the CFD result.



Figure 13. Velocity distribution result from PIV (left) and CFD (right) for Airfoil 2.



Figure 14. Velocity distribution result from PIV (left) and CFD (right) for Airfoil 3.

Figure 15. Velocity distribution result from PIV (left) and CFD (right) for Airfoil 4.

For Airfoil 5's graphic results (Figure 16), they are similar to the previous areas of increased velocity. One is placed above the airfoil and other behind the trailing edge of the airfoil. The increased velocity area behind the trailing edge is longer and wider and the wake structure is greater compared to Airfoil 4's results. On the upper side of the airfoil in both results, there is an area of increased velocity, suggesting the indication of a large eddy in this area. The upper side of the wake structure in the CFD results is slightly moved downwards compared to the PIV result.

Figure 16. Velocity distribution result from PIV (left) and CFD (right) for Airfoil 5.

The fully morphed airfoil's geometry results are presented in Figure 17. The flow is fully separated, and the areas of increased velocity above the airfoil and behind the trailing edge present the maximum region. In the CFD result, the wake structure is significantly moved downwards for this specific iteration, as it fluctuates upwards and downwards in the real condition. In the middle of the airfoil's upper surface, there is a clearly visible eddy in both graphic results.

Figure 17. Velocity distribution result from PIV (left) and CFD (right) for Airfoil 6.

Relating to all above-mentioned graphic results, the velocity distribution under the airfoils is highly comparable and adequate for both research methods. The wake structure has similar eddy structures for both conducted research methods. The CFD results are characterised by slightly higher maximum velocity values, but their distribution over the presented region is adequate. The wake structure is raising proportionally to the camber deflection of the airfoil. The two mean regions of increased velocity are recognised, one above the rear side of the airfoil and the other one behind the trailing edge. These regions are also rising with camber deflection. The boundary layer separation occurs at the same point for every airfoil geometry for both experiment methods. The CFD results are similar to the PIV results in the eddy distribution, though the fluctuation of the wake structure in the CFD results causes the wake structure to descend. This occurs more intensively with the increase in the camber deflection.

In the PIV results, one can notice small eddies behind the airfoil for every airfoil geometry. These structures do not appear in the CFD results, as CFD creates smoother results. During experiments, many eddies were visible in the turbulence region behind the airfoil. The CFD results are not as clearly visible due to the simplifications, by which the computational solutions are affected.

The differences between the PIV and CFD results described above are a representation of the demand for an experimental approach to flow research. Nevertheless, the similarities of both methods are clearly visible and precisely described to ensure the validation of the numerical model. The CFD numerical results were obtained and are presented in Table 6. A graphic representation of the results is presented in Figure 18.

Geometry	Lift Force Coefficient, Cl, -	Drag Coefficient, Cd, -	Pitching Moment Coefficient, Cm, -	Lift-to-Drag Ratio, L/D, -
Airfoil 1	0.0186	0.0296	0.0047	0.63
Airfoil 2	1.0193	0.2632	0.2270	3.87
Airfoil 3	1.3994	0.6259	0.2917	2.24
Airfoil 4	2.2312	1.2223	0.4082	1.83
Airfoil 5	2.4449	2.0139	0.5768	1.21
Airfoil 6	1.9709	2.5618	0.7049	0.77

Table 6. Computational fluid dynamic simulations' results.

Figure 18a shows the lift force coefficient characteristics for all geometries. The increase in the lift force coefficient occurs for Airfoils 1 to 5, then it slightly descends for Airfoil 6.

The drag coefficient (Figure 18b) increases with the increase in camber deflection for all geometries. As demonstrated above, the wake structure behind the airfoil rises with camber deflection, and greater turbulence implies higher drag.

Figure 18. Characteristics of (**a**) lift force coefficient, (**b**) drag coefficient, (**c**) lift-to-drag ratio, and (**d**) pitching moment coefficient for each geometry for setting angle 0° .

The lift-to-drag ratio is a very useful value; it is the relation between the lift force coefficient and drag coefficient. It reveals the maximum amount of aerodynamic energy an airfoil is capable of generating. For an aircraft, it would be the distance the aircraft is capable of flying from 1000 m in standard atmosphere ideal conditions. Camber-morphing airfoils with slight camber deflection are usually characterised by better aerodynamic characteristics. Figure 18c shows that the highest lift-to-drag ratio appears for Airfoil 2, a slightly cambered one, as the generated drag is still low and even marginal camber deflection generates high values of lift. Behind Airfoil 2, this characteristic drops significantly due to a lower difference in the increase in the lift force coefficient than for Airfoil 2 and an increase in the drag coefficient. For Airfoils 3 to 6, the lift-to-drag ratio descends linearly as the turbulence region and drag enhance drastically.

Every camber deflection provokes the pitching moment to increase. Aircraft pilots experience this phenomenon when the flaps are extended during every approach to landing. The pitching moment coefficient value for every geometry is presented in Figure 18d. The pitching moment increases nearly linearly with the camber deflection of an airfoil.

4. Conclusions

In this paper, the authors presented and discuss substantial camber-morphing airfoil geometry experiments. The research was conducted with two methods: experimental in the water tunnel using the PIV method for velocity distribution representation and computational using CFD software for obtaining the coefficients and graphic velocity distribution representation.

This work presents preliminary research with the main objective to study a variety of camber-morphing airfoils with a wide range of camber deflections.

The CFD and PIV velocity distribution results exhibited significant similarities, which enables the validation of the numerical model. The aerodynamic coefficients were consistent with the predictions' trends. It can be stated that Airfoil 2 had the best aerodynamic performance. Most research pursues creating the optimal airfoil with the highest lift force coefficient and lowest drag coefficient. It was demonstrated that camber-morphing airfoils can create drastic amounts of drag, retaining smooth and continuous surfaces. This could be a solution for micro UAV aircraft to reduce landing speed, without the necessity of installing wing mechanisation, which disturbs the airflow over the wing surface. Highly cambered airfoils can significantly reduce the landing distance, as they create massive amounts of drag, maintaining high lift.

However, airfoils with higher camber deflections can be utilised in various environments. This could be advantageous for wind turbines or tidal turbine engineers struggling with load reduction in hazardous atmospheric conditions. As highly cambered morphing airfoils create substantial drag, they could be utilised for rotor braking.

The CFD results were comparable, but they were not identical to the experimental results. As the simulations were performed in the exact environment as the experiments in the water tunnel, the simulations provided accurate and comparable results, but the differences were visible. This is a confirmation of the persisting need for an experimental approach to wake structures and turbulence research in the era of numerical simulations.

The next step of this preliminary research will be conducting experiments and numerical simulations on a wider range of angles of attack. The authors would like to obtain the full aerodynamic characteristics of the presented camber-morphing airfoils and to perform calculations correlating high amounts of produced drag with loads acting on turbine rotors and aircraft wings.

Substantial camber-morphing airfoils may be implemented in various applications from diverse fields. Though their applications are limited due to the limited abilities of current mechanical features, there remains demand for conducting aerodynamic research and for verifying different aerodynamic solutions for future use. The authors pushed the morphing airfoil concept a step further, creating a geometry that produces drag instead of decreasing it while maintaining a continuous surface. It is a solution with great potential and perspectives for further investigation.

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Abbreviations

The following abbreviations are used in this manuscript:

2D	two-dimensional
3D	three-dimensional
AMI	arbitrary mesh interface
CFD	Computational Fluid Dynamics
FSI	fluid-structure interaction
FVM	Finite-Volume Method
L/D	lift-to-drag ratio
MAC	mean aerodynamic chord
NACA	National Advisory Committee for Aeronautics
NASA	National Aeronautics and Space Administration
OpenFOAM	Open source field operation and manipulation
PIV	Particle Image Velocimetry
RANS	Reynolds-averaged NavierStokes
SIMPLE	semi-implicit method for pressure-linked equations
SST	shear stress transport
UAV	Unmanned Aerial Vehicle

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