



# Article Assessment and Perspectives of Heat Transfer Fluids for CSP Applications

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**Abstract:** Different fluid compositions have been considered as heat transfer fluids (HTF) for concentrating solar power (CSP) applications. In linear focusing CSP systems synthetic oils are prevalently employed; more recently, the use of molten salt mixtures in linear focusing CSP systems has been proposed too. This paper presents a comparative assessment of thermal oils and five four nitrate/nitrite mixtures, among the ones mostly employed or proposed so far for CSP applications. The typical medium-size CSP plant (50 MWe) operating with synthetic oil as HTF and the "solar salt" as TES was considered as a benchmark. In the first part of the paper, physical properties and operation ranges of different HTFs are reviewed; corrosion and environmental issues are highlighted too. Besides an extensive review of HTFs based on data available from the open literature, the authors report their own obtained experimental data needed to thoroughly compare different solutions. In the second part of the paper, the impact of the different HTF options on the design and operation of CSP plants are analyzed from techno-economic perspectives.

**Keywords:** CSP; linear focusing concentrators; parabolic trough; linear Fresnel collectors; heat transfer fluids; molten salts; thermal oil; synthetic oil; mineral oil

# 1. Introduction

The current global energy transition from a fossil-based to a sustainable economy entails the assessment, use and combination of different energy sources and technologies. This pathway involves the short-term exploitation of mature and cost-competitive technologies; in the meantime, fast technology improvements are needed to bring other technologies to competitive readiness and costs. This is the case of Concentrating Solar Power (CSP): as power grid operators suffer by the steady increase of vRES (variable Renewable Energy Sources) driving grid to instability, the last decades showed a steady increase of the interest on CSP for utility-scale dispatchable power generation.

First utility-scale CSP experiences started in the USA in the 1980s. However, it is since 2008 that worldwide installed CSP capacity started to significantly grow, from <0.5 GWe to >5 GWe in less than 10 years [1]. This growth was mainly driven by the extensive diffusion of CSP in Spain and, after 2012, in the USA. More recently big projects are contributing to the further growth of CSP in other "sun belt" countries such as Morocco, South Africa, Emirates, China, Chile and India.



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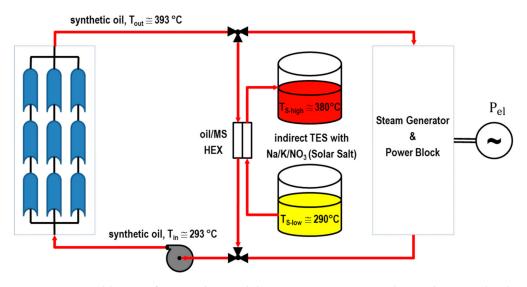
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**Copyright:** © 2021 by the authors. Licensee MDPI, Basel, Switzerland. This article is an open access article distributed under the terms and conditions of the Creative Commons Attribution (CC BY) license (https:// creativecommons.org/licenses/by/ 4.0/). Most CSP plants installed so far are based on Parabolic Trough (PT) collectors using Thermal Oil (TO) as Heat Transfer Fluid (HTF) up to about 393 °C; indirect Thermal Energy Storage (TES) is used with a molten salt mixture—namely the "solar salt", a mixture of NaNO<sub>3</sub> (60%wt.) and KNO<sub>3</sub> (40%wt.)—as heat storage medium. This PT-TO technology is highly modular, mature, reliable and cost-effective, enabling to obtain relevant capacity factors. However, power production costs remain an issue [2] and some improvements can be obtained with the changeover of the thermal fluid, with potential improvements on plant design, costs and performance, as discussed in the following sections.

Figure 1 shows a general scheme of a CSP plant with linear concentrators. This common layout makes use of parabolic trough collectors arranged in loops to heat up a thermal oil, typically a diphenyl oxide (DPO, e.g., Therminol-VP) up to about 390–400 °C. The thermal oil transfers the captured solar heat to the solar salt in an oil/molten salt heat exchanger (HEX); here, the solar salt is used as heat storage medium in a 2-tanks TES arrangement in the range from 290 °C (cold molten salt tank temperature) to 380 °C (hot molten salt tank temperature). The common "standard" size of CSP plants using parabolic troughs with thermal oil (PT-TO) is 50 MWe, although larger and smaller power stations (from 5 MWe to 200 MWe or more) have been built worldwide [2–5].



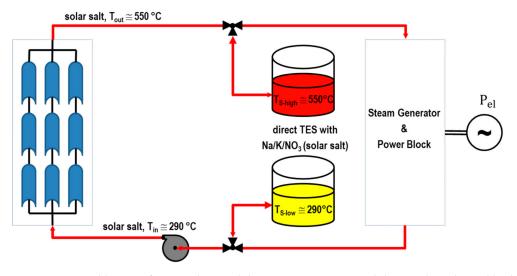
**Figure 1.** General layout of a CSP plant with linear concentrators, synthetic oil HTF and indirect TES system.

Some advanced large-scale stations use a different CSP technology based on a Solar Tower with Molten Salt (ST-MS), where the solar salt is also used as HTF in the solar receiver up to about 565 °C; in ST-MS systems a "direct" TES is applied, i.e., the same HTF is also used as heat storage medium.

Recent studies show that the use of solar salt as HTF in linear concentrators can combine the benefits of the modularity, maturity and low-cost of the linear CSP technology with several benefits introduced by the use of the molten salt as HTF [3,6–9].

Figure 2 shows a different CSP plant layout where the solar salt is used also as HTF in the solar field. In this case, thanks to the higher stability of the HTF, higher temperatures can be obtained, while the use of a "direct" TES system avoids the installation of an intermediate oil/MS heat exchanger.

Experimental test loops have been developed to validate concepts and components of the PT-MS technology using solar salt HTF [10–12]. In this contest, a 1 MWe PT-MS demo plant has been built in a desert area in Borg El Arab, Egypt, in the framework of the European project MATS—Multipurpose Applications by Thermodynamic Solar [13]. Another test loop based on a ternary molten salt mixture (KNO<sub>3</sub>, NaNO<sub>2</sub> and NaNO<sub>3</sub>, also called HITEC<sup>®</sup>) in linear Fresnel collectors was built in Spain in 2013 [14].



**Figure 2.** General layout of a CSP plant with linear concentrators and direct solar salt used both as HTF and TES medium.

Similar layout can be applied also when a different molten salt mixture is used as HTF in linear concentrators, depending on the heat load and end-users' requirements, for example for the supply of heat for industrial processes [15].

It is possible to estimate the operating and management costs as about 2% of the initial investment (see Sau et al. and Delise et al. [16,17]). Other aspects are difficult to be quantified, especially regarding how the handling of operating problems can be related to the properties of the used thermal fluids. As a rule, the employment of oils or low melting HTF mixtures facilitate filling and emptying operations, along with start-up procedures. Thermal oils are, in principle, more handy, but they pose severe problems concerning safety and dispersion in the environment (see Delise et al. [16]). All in all, as far as management issues are concerned, low melting mixtures are preferable as heat transfer and storage fluids.

This paper presents a comparative assessment between thermal oils (synthetic and mineral oils) and four molten nitrate/nitrite mixtures, considering the ones mostly employed or proposed so far for CSP applications. The methodology can be extended to other formulations such as more complex ternary or quaternary molten salt mixtures or silicone oils; however, the majority of new formulations still lack basic data (e.g., physical/chemical properties, corrosion and costs) to be thoroughly included in the comparison. It is also worth mentioning that nanoparticles added fluids have been recently proposed for the applications concerned, obtaining significant increases in both specific heat and thermal conductivity, due to the presence of small scale effects [18–20]. Anyway, these pioneering HTFs are not considered in this work, given the lack of information about their cost and stability.

In the first part of the paper, physical properties and operation ranges are reviewed, along with corrosion, safety and environmental issues. In the second part of the paper, the impact of the different HTF options on the design and operation of CSP plants is analyzed from techno-economic perspectives.

### 2. Material Properties

### 2.1. Physical Properties and Operation Range

### 2.1.1. Thermal Oils

Thermal oils can be classified into three main groups according to their origin and chemical composition: synthetic oils, mineral oils and "others". The last category includes promising thermal oils of different kind (i.e., non-hydrocarbon-based oils), such as silicon-based oils; this type of HTF is not addressed in this paper due to uncertainties on physical properties, costs and environment impact.

In the case of synthetic oils, the main components are obtained via chemical synthesis processes or using methods different from conventional refining, being these materials generally based on aromatic-like compounds. As representative examples of this category, three widely commercialized synthetic oils HTFs are considered in this paper [1], namely:

- Therminol 66, mainly consisting of terphenyl and partially hydrogenated therphenyls;
- Therminol SP (or Dowtherm HT) composed by benzene alkyl derivates;
- Therminol VP-1, an eutectic mixture of diphenyl oxide (DPO) and biphenyl.

Differently, mineral oils are derived as a by-product of refining processes. Therminol XP (White Mineral Oil) is considered in this work.

Table 1 reports the thermo-physical properties of the above mentioned four thermal oils, along with the operating temperature ranges recommended by manufacturers.

<b>ble 1.</b> Thermo-physical properties of Therminol 66, Therminol SP, Therminol VP-1 and Therminol XP.
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Thermi	nol 66 [21]			
Chemical composition (%wt)	Hydrogenated terphenyl/Polyphenyls/Terphenyl (81/14/5)			
Temperature range	$0 \div 345 ^{\circ}\text{C}$			
Density $\left \frac{kg}{m^3}\right $ vs. temperature [°C]	$\rho = 1020.6 - 0.6 \cdot T + 3.2 \cdot 10^{-4} \cdot T^2$			
Dynamic Viscosity [cP] vs. temperature [°C]	$\mu = 58,191 \cdot T^{-2.093}$			
Thermal conductivity $\left[\frac{W}{K*m}\right]$ vs. temperature $\left[^{\circ}C\right]$	$k = 0.1 - 3.3 \cdot 10^{-5} \cdot T - 1.5 \cdot 10^{-7} \cdot T^2$			
Heat capacity $\begin{bmatrix} J \\ \circ C * g \end{bmatrix}$ vs. temperature $\begin{bmatrix} \circ C \end{bmatrix}$	$cp = 1.5 + 3.3 \cdot 10^{-3} \cdot T - 8.9 \cdot 10^{-7} \cdot T^2$			
Maximum film temperature	375 °C			
Boiling point (1013 mbar)	359 °C			
Therminol SP/I	Dowtherm HT [22]			
Chemical composition	Benzene, C14-30-alkyl derivatives			
Temperature range	$-10 \div 315$ °C			
Density $\left \frac{\text{kg}}{\text{m}^3}\right $ vs. temperature [°C]	$\rho = 885.6 - 0.69 \cdot T + 1.9 \cdot 10^{-4} \cdot T^2 - 8.9 \cdot 10^{-7} \cdot T^3$			
Dynamic Viscosity [cP] vs. temperature [°C]	$\mu = 22,321 \cdot T^{-1.945}$			
Thermal conductivity $\left[\frac{W}{K*m}\right]$ vs. temperature $\left[^{\circ}C\right]$	$k = 0.1 - 1.1 \cdot 10^{-4} \cdot T - 1.5 \cdot 10^{-8} \cdot T^2 + 1.8 \cdot 10^{-11} \cdot T^3$			
Heat capacity $\begin{bmatrix} J \\ \circ C \ast g \end{bmatrix}$ vs. temperature $[\circ C]$	$cp = 1.8 + 3.6 \cdot 10^{-3} \cdot T - 4.9 \cdot 10^{-7} \cdot T^2 - 7.9 \cdot 10^{-10} \cdot T^3$			
Maximum film temperature	335 °C			
Boiling point (1013 mbar)	351 °C			
Thermin	ol VP-1 [23]			
Chemical composition	Eutectic mixture of diphenyl oxide (DPO) and biphenyl			
Temperature range	$-20 \div 400 ^{\circ}\text{C}$			
Density $\lfloor \frac{\text{kg}}{\text{m}^3} \rfloor$ vs. temperature [°C]	$\rho = 1083.25 - 0.90797 \cdot T + 0.00078116 \cdot T^2 - 2.367 \cdot 10^{-6} \cdot T^3$			
Dynamic Viscosity [cP] vs. temperature [°C]	$\mu = 171.5 \cdot T^{-1.155}$			
Thermal conductivity $\left[\frac{W}{K*m}\right]$ vs. temperature $\left[^{\circ}C\right]$	$\mathbf{k} = 0.1377 - 8.19 \cdot 10^{-5} \cdot \mathbf{T} - 1.922 * 10^{-7} \cdot \mathbf{T}^2 + 2.5 \cdot 10^{-11} \cdot \mathbf{T}^3$			
Heat capacity $\left[\frac{J}{\circ C \ast g}\right]$ vs. temperature $\left[\circ C\right]$	$cp = 1.498 + 0.002414 \cdot T - 5.9 \cdot 10^{-6} \cdot T^2 - 2.9 \cdot 10^{-8} \cdot T^3$			
Maximum film temperature	430 °C			
Boiling point (1013 mbar)	257 °C			
Therminol XP [24]				
Chemical composition	White Mineral Oil			
Temperature range	$-20 \div 315 ^{\circ}\text{C}$			
Density $\left\lfloor \frac{\text{kg}}{\text{m}^3} \right\rfloor$ vs. temperature [°C]	$\rho = 890.8 - 0.6 \cdot T + 4.2 \cdot 10^{-5} \cdot T^2 - 5.3 \cdot 10^{-7} \cdot T^3$			
Dynamic Viscosity [cP] vs. temperature [°C]	$\mu = 33,553 \cdot T^{-1.999}$			
Thermal conductivity $\left[\frac{W}{K*m}\right]$ vs. temperature $\left[^{\circ}C\right]$	$k = 0.1 - 6.3 \cdot 10^{-5} \cdot T - 1.2 \cdot 10^{-7} \cdot T^2 - 1.6 \cdot 10^{-12} \cdot T^3$			
Heat capacity $\left[\frac{kJ}{\circ C * kg}\right]$ vs. temperature $\left[\circ C\right]$	$cp = 1.7 + 5 \cdot 10^{-3} \cdot T - 2.8 \cdot 10^{-6} \cdot T^2 + 1.5 \cdot 10^{-10} \cdot T^3$			
Maximum film temperature	330 °C			
Boiling point (1013 mbar)	358 °C			

# 2.1.2. Molten Salts

As introduced in the previous section, the so-called "solar salt" is today the most used nitrate mixture as heat transfer fluid and storage material for CSP applications [25]. Actually, this fluid shows good thermo-physical properties, high thermal stability (up to 600 °C [17]), low cost and no toxicity. One significant shortcoming of the solar salt is related to its freezing point around 238 °C [25]. This feature leads to the necessity of applying specific anti-freezing systems along the HTF loop.

With the aim to mitigate the drawback of the relatively high freezing temperature of the solar salt, molten salt mixtures with lower melting temperatures (<150 °C) were proposed in the scientific literature [25]. Table 2 summarizes the thermo-physical properties of the molten salt mixtures here considered; the "solar salt" properties can be compared with those of three alternative molten salt mixtures.

The molten salt mixture commercially known as Hitec<sup>®</sup> (Table 2) is characterized by the presence of sodium nitrite and shows a relatively low freezing point together with good features concerning specific heat, density and viscosity. However, this material is toxic and less thermally stable than the solar salt under air [25,26].

Another interesting alternative is represented by the molten salt mixture commercially known as Hitec XL<sup>®</sup>, composed by calcium, potassium and sodium nitrates [16]. However, as shown in Table 2, the viscosity of the Hitec XL<sup>®</sup> mixture is much higher than the one of the solar salt, especially at temperatures approaching the freezing point, while its thermal stability in presence of air is similar to the one of Hitec<sup>®</sup> [25].

Finally, the addition of lithium nitrate significantly decreases the freezing point of the molten salt mixture while, at the same time, similar thermo-physical properties and stability of the solar salt can be obtained [25]. This finding makes the Li-containing mixture rather interesting. However, cost and availability of lithium can represent a bottleneck for its utilization in CSP plants with large-scale TES capacity. Among several proposed compositions, the most studied mixture with Na/K/Li nitrates [25] is the one reported in Table 2.

Table 2. Thermo-physical properties of the	e "solar salt", Hitec <sup>®</sup> , Hitec XL <sup>@</sup>	$\mathbb{B}$ and lithium containing ternary molten salt mixtures.
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	Solar Salt	
Chemical composition (%wt)	NaNO <sub>3</sub> /KNO <sub>3</sub> (60/40)	
Density $\left[\frac{\text{kg}}{\text{m}^3}\right]$ vs. temperature [°C]	$\rho=2090-0.63{\cdot}T$	[27]
Dynamic Viscosity [cP] vs. temperature $[^{\circ}C]$	$\mu = 71,645 \cdot T^{-1.763}$	(*)
Thermal conductivity $\left[\frac{W}{\circ C*m}\right]$ vs. temperature $\left[\circ C\right]$	$k = 0.3804 + 3.452 \cdot 10^{-4} \cdot T$	[28]
Heat capacity $\left[\frac{kJ}{\circ C * kg}\right]$ vs. temperature $\left[\circ C\right]$	$cp = 1.5404 + 3.0924 \cdot 10^{-5} \cdot T$	(*)
Thermal stability (max operation temperature)	600 °C	[16]
Liquidus temperature (initial solidification point)	238 °C	[29,30]
Hi	tec <sup>®</sup> (Na/K nitrate/nitrite)	
Chemical composition (%wt)	NaNO <sub>3</sub> /KNO <sub>3</sub> /NaNO <sub>2</sub> (7/53/40)	
Density $\left[\frac{\text{kg}}{\text{m}^3}\right]$ vs. temperature [°C]	$\rho=-0.9{\cdot}T+2269.4$	[31]
Dynamic Viscosity [cP] vs. temperature [°C]	$\mu = 146,452 \cdot T^{-1.903}$	(*)
Thermal conductivity $\left[\frac{W}{\circ C*m}\right]$ vs. temperature $\left[\circ C\right]$	$\mathbf{k}=0.5843\mp0.0006\!\cdot\!\mathbf{T}$	[26]
Heat capacity $\left[\frac{kJ}{C*kg}\right]$ vs. temperature $\left[^{\circ}C\right]$	$cp = 1.55 - 0.0001 \cdot T$	[26]
Thermal stability (max operation temperature)	450 under air; 530 $^\circ\mathrm{C}$ under inert gas	[26]
Liquidus temperature (initial solidification point)	141 °C	[32]

	Solar Salt	
Hi	tec XL <sup>®</sup> (Na/K/Ca nitrate)	
Chemical composition (%wt)	NaNO <sub>3</sub> /KNO <sub>3</sub> /Ca(NO <sub>3</sub> ) <sub>2</sub> (15/43/42)	
Density $\left[\frac{\text{kg}}{\text{m}^3}\right]$ vs. temperature [°C]	$\rho = 2240 - 0.827 \cdot T$	[33]
Dynamic Viscosity [cP] vs. temperature [°C]	$\mu = 509,611 \cdot T^{-2.072}$	(*)
Thermal conductivity $\left[\frac{W}{\circ C*m}\right]$	~0.519 (constant in the operative range)	[31]
Heat capacity $\left[\frac{kJ}{C*kg}\right]$ vs. temperature $\left[^{\circ}C\right]$	$cp = 1.542 - 0.000322 \cdot T$	(*)
Thermal stability (max operation temperature)	≤425 °C	[34]
Liquidus temperature (initial solidification point)	~125 °C	[35]
	Na/K/Li nitrate	
Chemical composition (%wt)	NaNO <sub>3</sub> /KNO <sub>3</sub> /LiNO <sub>3</sub> (18/45/37)	
Density $\left[\frac{\text{kg}}{\text{m}^3}\right]$ vs. temperature [°C]	$\rho=2051-0.6639{\cdot}T$	[31]
Dynamic Viscosity [cP] vs. temperature [°C]	$\mu = 58,725 \cdot T^{-1.69}$	(*)
Thermal conductivity $\left[\frac{W}{\circ C*m}\right]$	$k = 0.0005 \cdot T + 0.4$	[31]
Heat capacity $\left[\frac{kJ}{C \cdot kg}\right]$ vs. temperature $\left[^{\circ}C\right]$	$cp = 1.5395 + 0.0003 \cdot T$	(*)
Thermal stability (max operation temperature)	600 °C	[17]
Liquidus temperature (initial solidification point)	120 °C	[36]

Table 2. Cont.

\* data obtained by ENEA in the framework of the "Concentrating Solar Power" project under the "Electric System Research" Programme 2019–2021.

# 2.1.3. Comparison between HTFs' Thermo-Physical Properties

Figures 3–6 show a comparison between the thermo-physical properties of the HTFs considered in this work within their respective operating temperature ranges. Clearly, molten salts are characterized by higher densities, while thermal oils show higher specific heat, lower viscosity and lower thermal conductivity.

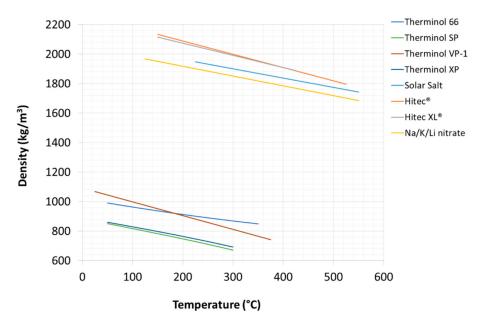


Figure 3. Comparison between the densities of different HTFs.

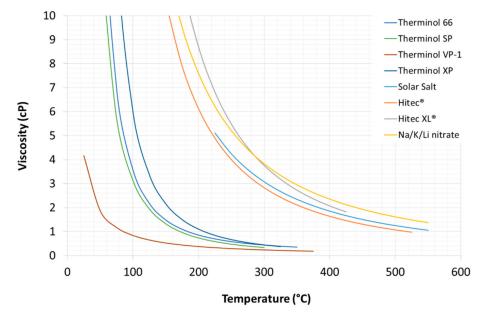


Figure 4. Comparison between the dynamic viscosity of different HTFs.

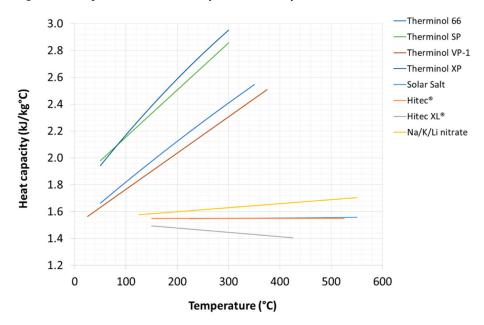


Figure 5. Comparison between the heat capacity of different HTFs.

Thermal oils are used both as heat transfer fluids and as storage materials only in CSP plant with TES based on solid fillers [37]; therefore, the volumetric energy density of thermal oil is less important than for molten nitrates/nitrites, which are also employed as sensible heat storage medium [25]. Figure 7 shows compares the four molten salt formulations, considering average values for density and heat capacity in their respective working temperature ranges.

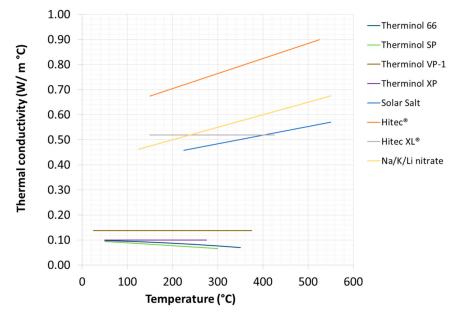
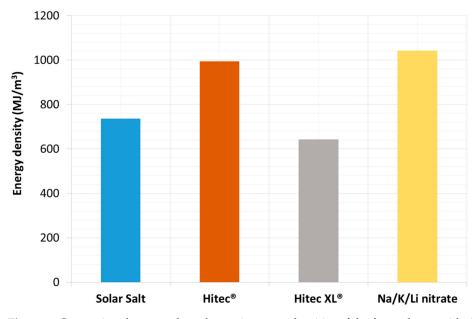


Figure 6. Comparison between the thermal conductivity of different HTFs.



**Figure 7.** Comparison between the volumetric energy densities of the four salts, considering sensible heat release and average values for densities and heat capacities in the respective operation ranges: 290–550 °C for the Solar Salt; 200–530 °C for Hitec<sup>®</sup>; 200–425 °C for Hitec XL<sup>®</sup>; 200–550 °C for Na/K/Li nitrate.

### 2.2. Material Corrosion Issues

Besides thermo-physical properties, a comprehensive assessment of different HTF solutions includes the assessment of corrosion issues towards steel components of the CSP plant such as pipes, valves, tanks and welding joints. Specifically, compared to thermal oils, molten nitrates show some drawbacks linked to the degradation of nitrates into nitrites, with possible oxygen generation, when the fluid is heated above 295 °C [38].

Studies carried out so far on steel corrosion with mixtures of molten nitrates are not adequately systematic. Furthermore, these tests sometime provide contradictory results; there are no validated/shared methods for carrying out the corrosion tests. Additionally, test conditions are not always representative of the real environment in CSP plants: for example, TES tanks in CSP plants are not perfectly airtight and the presence of air in the Table 3 summarizes the most significant results published in the scientific literature regarding the corrosion of different types of steel in contact with the most used molten nitrate salt binary and ternary mixtures.

Molten Salt Mixture	Steel	Corrosion Test	Duration [h]	Conditions	T [°C]	Ref.	Corrosion Rate (µm/Year)	Corrosion Products
Solar salt analytic	cal grade							
	X80 C-steel	static	250	Air	350	[39]	20	
			250	$N_2$	350	[39]	10	
			3000	$N_2$	350	[39]	5	
			3000	Air	350	[39]	5	
	316L	static	250	Air	530	[39]	60	
	0102	static	3000	Air	530	[39]	10	
		static	250	Air	550	[40]	70	Fe <sub>2</sub> O <sub>3</sub> , Fe <sub>3</sub> O <sub>4</sub>
			3000	Air	550	[40]	9	Fe <sub>2</sub> O <sub>3</sub> , Fe <sub>3</sub> O4
		static	200	Air	570	[41]	45	10301
		State	2000	Air	570	[41]	10	
								Fe <sub>2</sub> O <sub>3</sub> ,
	321H	static	250	Air	550	[40]	110	Fe <sub>3</sub> O <sub>4</sub>
			3000	Air	550	[40]	9	$Fe_2O_3$ , $Fe_3O_4$
	321SS	static	3000	Air	600	[42]	16	Fe <sub>2</sub> O <sub>3</sub> , Fe3O4
	347SS	static	3000	Air	600	[42]	16	Fe <sub>2</sub> O <sub>3</sub> , Fe <sub>3</sub> O <sub>4</sub>
Solar salt with 19		npurity 1%						
	X80		300	Air	350	[39]	20	
	C-steel							
			3000	Air	350	[39]	5	
	304	static	250	Air	530	[39]	40	
		dynamic	250	Air	530	[39]	60	
		static	2000	Air	530	[39]	20	
	316L	S/D	250	Air	530	[39]	15/40	
			2000	Air	530	[39]	10	
	600 Ni alloys	S/D	2000	Air	530	[39]	0/5	
	825 Ni		2000	A =	F20	[20]	0 /E	
	alloys	S/D	2000	Air	530	[39]	0/5	
Hitec <sup>®</sup>								
	X80 C-steel	static	250	Air	350	[39]	15	
	C SILLI		3000	Air	350	[39]	5	
	304	static	200	Air	530	[39]	110	
	504	Static	3000	АШ	550	[39]	10	
	316L	static	200	Air	530	[39]	40	
	310L	Static	3000	Air Air	530 530		40 5	
@			3000	Alf	550	[39]	3	
Hitec XL <sup>®</sup>	204	at a f	1500	A :	210	[40]	-	
	304	static	1500	Air *	310	[43]	7	
	304	static	1500	Air	310	[43]	7.6	
	316	static	1500	Air *	310	[43]	4.6	
	316	static	1500	Air	310	[43]	4.7	

 Table 3. Corrosion effects for the most used binary and ternary molten salt mixtures.

\* dry salt (200  $^{\circ}$ C for 12 h).

The most important parameters to investigate the corrosion phenomena are:

- type and purity of the salt mixture. In particular, the chloride content is considered critical if >0.1%wt., although in literature there are data up to 1% [44];
- corrosion test conditions: static or dynamic immersion, thermal cycling, temperature, nitrogen or air atmosphere, duration;
- oxides produced by corrosion;
- corrosion rate (CR), expressed as thickness change per year, according to the formula:

$$CR = \frac{\Delta M \cdot 8760}{\rho \, S \cdot \Delta t} \tag{1}$$

where  $\Delta M$  is difference between the final and the initial mass of the specimen; 8760 is the number of hours per year;  $\rho$  is the density of the steel sample; S is the surface area exposed to corrosion;  $\Delta t$  is the duration of the test (hours).

To date, a threshold value beyond which the compatibility performance of a steel is not considered optimal has not yet been defined, but it is possible to state that the thermomechanical properties of steel remain substantially unchanged with CR <  $25 \mu$ m/year [42].

As can be seen from the data reported in Table 3 the results obtained by different working groups (SNL in the USA, CEA in France, LNEG in Portugal, CIEMAT in Spain, Fraunhofer in Germany, etc.) and by ENEA [8] are quite consistent.

The first studies on corrosion resistance of steels in contact with mixtures of molten salts were presented by Bradshaw et al. [45] in the 1980s. Inconel steels 800, 304 and 316 were tested in static immersion and showed good performances: CR between 5–12  $\mu$ m/year at 600 °C was determined with a corrosion mechanism possibly caused by the dissolution of chromium (in the form of chromates) in the molten salt. In general, steels with a chromium content above 9% performed well. However, the authors highlighted that these tests are not representative of the working conditions of the material, which is subjected to additional stress due to thermal cycling.

In the European project OPTS [46] ENEA promoted the development of shared standard methods for corrosion tests. Currently, research is more oriented towards less expensive materials, capable of guaranteeing excellent performance such as, for example, ferritic steels, which are cheaper than austenitic steels and can be used in specific operating conditions. Kruzniga et al. [42] undertook an interesting test, proposing the study of the corrosion resistance of steels such as 321 (stabilized with titanium) and 347 (stabilized with niobium) in contact with the solar salt at 400, 500, 600 and 680 °C. Both showed an acceptable CR, with a value of 16  $\mu$ m/year at 600 °C; the 347 steel type showed overall more reliable because the 321 steel showed signs of spallation of the protective oxide.

Wang et al. [2] compared the corrosion resistance of some steels in static and dynamic immersion conditions, and verified the results obtained by applying an electrochemical method for the anodic oxidation of the specimens. Results showed that the steels treated with the solar salt binary mixture at 530 °C have a corrosion resistance that can be described by the sequence 304 < 316L < 600 < 825. The same sequence was also found for commercial solar salt containing chloride impurities that, however, showed a more marked corrosion effect, confirming the importance of the purity of the salt.

The same behavior was found for the HITEC<sup>®</sup> mixture that showed a more aggressive effect than the solar salt tested under the same operating conditions. Furthermore, under dynamic immersion conditions, the corrosion rate is higher than under static conditions. Finally, the X80 carbon steel proved to be suitable up to 350 °C with the solar salt mixture.

Gomes et al. [40] tested 316L and 321H steels under static immersion conditions in solar salt at 550 °C, near the critical temperature for the stability of molten nitrates (Table 2). The corrosion rates are certainly acceptable for both, equal to 9  $\mu$ m/year. However, the 316L steel showed better resistance to corrosion, attributed to the presence of the passivating mixed oxide FeCr<sub>2</sub>O<sub>4</sub>, stabilized by the presence of molybdenum: after 1000 h of immersion, 321H steel showed signs of dissolution of the protective oxide layer FeCr<sub>2</sub>O<sub>4</sub> (spallation) while the 316L remained stable.

Helmersonn et al. [41] studied 316L, 316LN, 316plus, 316Ti, 347H and 800H steels as an alternative to 347H steel so far often adopted as a material for high temperature storage tanks in commercial CSP plants. Corrosion rates after 200 h of immersion in the solar salt at 570 °C resulted in the sequence 800H > 347H > 316plus > 316Ti > 316L > 316LN; after 2000 h, all steels showed acceptable CR values around 20  $\mu$ m/year and can be considered good alternatives. However, 316Ti steel proved to be less resistant, giving the phenomenon of spallation of the protective oxide.

Finally, Grosu et al. [43] recently studied the corrosion resistance of 304 and 316 steels at 310  $^{\circ}$ C with the ternary mixture HITEC XL<sup>®</sup> used both as it is and after a drying step at 200  $^{\circ}$ C for 12 h. Corrosion rates after 1500 h are acceptable.

The most concrete experience under representative CSP operation conditions is represented by the tests carried out in the 10 MW Solar Two plant near Barstow, California. In this plant, A516-70 steel was used in the low temperature TES tank, while 304SS and 316SS steel were used as construction material for the high temperature TES tank and the receiver; the latter, however, showed visible corrosion stress over time, with the occurrence of cracking phenomena [47]. This effect was due to the precipitation of chromium carbide above 400–450 °C, and can be avoided using special alloys added with titanium or niobium, such as AISI 321 and 347 [42,48].

To summarize, from the results available in the open literature it can be concluded that less expensive carbon steels may be employed for molten salt mixtures operating at low temperatures (200–400 °C), while for higher temperatures (400–600 °C) special stainless steels or alloys containing nickel, which have a higher corrosion resistance, should be used [39].

## 2.3. Environment and Safety Issues

Toxicity and environmental compatibility are clearly major issues for large-scale utilization of heat transfer materials. Tables 4 and 5 report the safety indications for thermal oils and salts, respectively. In general, salt mixtures are safer than oils which are highly flammable, toxic and pollutant, even not considering the sub-products due to thermal degradation [1]. However, nitrites and lithium can also be harmful for the humans and the environment, and the thermal decomposition of nitrates and nitrites can lead to the formation of nitrogen oxides [25].

		Environmental —	Flammability		
HTF	HTF Toxicity Safety		Flash Point	Self-Ignition Point	Regulations
Therminol 66	n.a.	H411: Toxic to aquatic life with long-lasting effects	170 °C	374 °C	Listed as "Dangerous substances covered by the hazard categories" subject to the qualifying quantities in Seveso III **
Therminol SP	H304: may be fatal if swallowed or inhaled	H412: Harmful to aquatic life with long-lasting effects	177 °C	343 °C	Not listed as dangerous

Table 4. Toxicity, environmental compatibility and safety indications for thermal oils.

#### Flammability Environmental HTF Toxicity Regulations Self-Ignition Safety **Flash Point** Point Listed as H315: Causes skin "Dangerous irritation substances covered H332: Harmful if H410: Very toxic to by the hazard Therminol VP-1 110 °C 621 °C inhaled. aquatic life with categories" subject H335: May cause long lasting effects. to the qualifying respiratory quantities in irritation. Seveso III \*\* Not classified as dangerous for the Not listed as environment Therminol XP 199 °C 346 °C n.a. according to the dangerous EC Regulation No. 1272/2008.

\*\* Direttiva Seveso 2012/18/UE (Seveso III): this Directive lays down rules for the prevention of major accidents which involve dangerous substances, and the limitation of their consequences for human health and the environment, with a view to ensuring a high level of protection throughout the European Union in a consistent and effective manner.

Table 5. Toxicity, environmental compatibility and safety indications for molten salts.

HTF	Toxicity	Environmental Safety	Flammability	Regulations
Solar salt	H315: Causes skin irritation H319: Causes serious eye irritation H335: May cause respiratory irritation	Not classified as dangerous for the environment according to the criteria of EC Regulation No. 1272/2008.		
Hitec®	H315: Causes skin irritation H319: Causes serious eye irritation H335: May cause respiratory irritation H350: May cause cancer (oral) H373: May cause damage to organs (blood, heart, liver) through prolonged or repeated exposure (oral) H301: Toxic if swallowed.	H410 Very toxic to aquatic life with long lasting effects.	H272—May intensify fire; oxidizer	KNO3 Listed as "Dangerous substances covered by the hazard categories" subject to the
Hitec XL®	H319: Causes serious eve irritation the criteria of EC	dangerous for the environment according to	Seveso I	qualifying quantities in Seveso III **
Na/K/Li nitrate	H302: Harmful if swallowed H315: Causes skin irritation H319: Causes serious eye irritation H335: May cause respiratory irritation	Not classified as dangerous for the environment according to the criteria of EC Regulation No. 1272/2008.		-

\* referred to the more dangerous component; \*\* Direttiva Seveso 2012/18/UE (Seveso III): This Directive lays down rules for the prevention of major accidents which involve dangerous substances, and the limitation of their consequences for human health and the environment, with a view to ensuring a high level of protection throughout the European Union in a consistent and effective manner.

# Table 4. Cont.

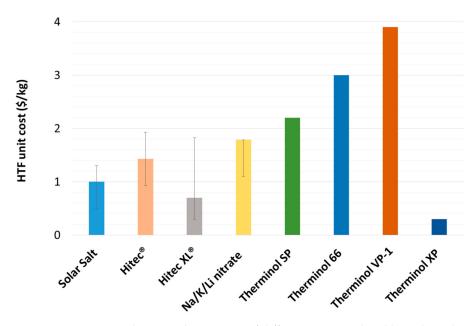
### 2.4. Material Costs

A comprehensive assessment and comparison of HTFs also includes market prices of the fluid to be purchased and of the respective construction materials to be used.

Table 6 reports recent cost ranges for the HTF here considered, resulting from available literature sources. As expected, there are significant oscillations in the reported values. In this work, the authors also considered the prices according to ENEA's experience obtained in several CSP research projects [49,50]. Figure 8 summarizes the data along with an indication of the literature data sources.

Table 6. Market price ranges for different HTFs.

HTF	Unit Cost (\$/kg)	Ref.
Solar Salt	0.5–1.3	[45,49–53]
Hitec <sup>®</sup>	0.9–1.9	[45,49–53]
Hitec XL <sup>®</sup>	0.7–1.8	[45,49–53]
Na/K/Li nitrate	1.1–1.8	[48–50,54]
Therminol SP (synthetic oil)	2.2	[48,54]
Therminol 66 (synthetic oil)	3.0	[48]
Therminol VP-1 (synthetic oil)	3.9	[54]
Therminol XP (mineral oil)	0.3	[54]



**Figure 8.** Comparison between the unit cost of different HTFs. Colored bars show the actual values based on ENEA's experience; error bar include the cost range from different sources.

Regarding construction materials, as discussed in a previous section special austenitic stainless steels are needed for applications above 400 °C [16,17]. From the literature, it can be stated that the market prices of these alloys (AISI 321 and 347) are similar to the ones of the other austenitic steels. Carbon steels seem suitable for operative temperatures lower than 400 °C. Table 7 resumes the costs obtained by a preliminary market research; considering averaged values, it is realistic to take 2.73 US\$/kg for stainless steels and 0.99 US\$/kg for carbon steels. However, it is noteworthy that these values can significantly change due to market fluctuations.

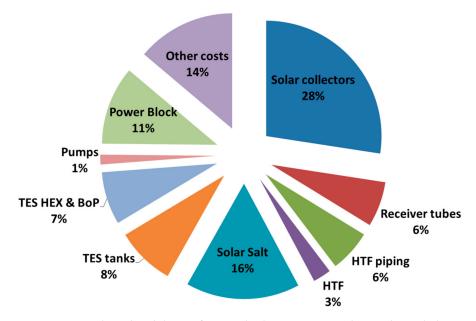
Construction Material	Unit Cost (\$/kg)	Ref.
AISI 321 plate	3.020	[55]
AISI 347 plate	1.750	[56]
SS 347	3.300	[57]
Stainless steel	2.000-5.000	[58,59]
A516 Gr 70 steel plate	0.650-0.765	[60,61]
Carbon steel	0.500	[62]
Stainless steel/carbon steel	1.700/0.970 (dollar/lb)	[59]

Table 7. Steels costs obtained by a preliminary market research.

# 3. Techno-Economic Analysis for CSP Applications

The choice of a specific HTF has different implications on the design of a CSP plant and its performances. In this section the impact of the use of a specific HTF is assessed considering techno-economic aspects at CSP system level, with a semi-quantitative approach.

Figure 9 shows a typical cost breakdown for a 50 MWe "standard" CSP plant with parabolic troughs, synthetic oil HTF (PT-TO) and 6 h thermal storage with the "Solar Salt", according to the general scheme in Figure 1.



**Figure 9.** Typical cost breakdown of a "standard" 50 MWe CSP plant with parabolic trough collectors, synthetic oil HTF and heat storage (6 h). Solar collectors include mirrors, steel structures, foundations, receivers, electronics and electrical equipment. TES tanks include insulation materials and foundations. TES HEX and BoP include the oil/molten salt heat exchanger, ancillary equipment and Balance of Plant (BoP). Other costs include additional general direct costs such as: site improvement, civil works and electrical/mechanical works.

Clearly, for a given power capacity (e.g., 50 MWe) the larger the capacity factor of the plant, the larger will be the size and cost impact of Solar Field and TES, while other costs such as Power Block, pumps and "TES HEX & BoP" remain nearly unaffected by the increased productivity.

In order to compare the different HTFs options at the CSP system level, the PT-TO layout (Figure 1) is taken as benchmark to estimate relative costs and productivity of other solutions.

As far as the plant performance is concerned, changing the thermal fluid has an impact on the operating temperatures of basic units such as the HTF loop, the TES and the steam Rankine cycle. On the one hand, the higher the maximum HTF temperature, the higher will be the Power Block efficiency. On the other hand, the higher the HTF operating temperature the higher will be the heat losses during the HTF circulation in the solar field.

Considering a 50 MWe CSP plant with overall reflective area of the solar field of 735,750 m<sup>2</sup>, 2.0 solar multiple and 35.6% Power Block efficiency [60], the average solar heat collected by the HTF will be  $Q_{\rm HTF} = 2.0 * (50 \text{ MWe})/0.356 = 280.9 \text{ MW}_{\rm th}$ ; this thermal output ( $Q_{\rm HTF}$ ) results from the difference between the effective solar radiation on solar receivers ( $Q_{\rm rec}$ ) minus the overall heat losses in the solar field ( $Q_{\rm loss}$ ):

$$Q_{\rm HTF} = Q_{\rm rec} - Q_{\rm loss} \tag{2}$$

 $Q_{loss}$  can be estimated as function of the average temperature of the HTF in the solar field, as reported elsewhere [13].

For the 735,750 m<sup>2</sup> solar field here examined with the HTF temperatures in the range of 293–393 °C (Figure 1) the estimated average heat loss is  $Q_{loss} = 32.2 \text{ MW}_{th}$  and the resulting effective solar radiation on solar receivers is (from Equation (2)) is  $Q_{rec} = Q_{HTF} + Q_{loss} = 313.1 \text{ MW}_{th}$ . This value mainly depends on the solar collectors' optical performances, i.e., geometrical/optical features, rather than on the used HTF. Therefore, Equation (2) can be used to evaluate the effectively collected solar heat when using different HTFs, fixing a constant value  $Q_{rec} = 313.1 \text{ MW}_{th}$  for the assumed plant size.

The overall effect in terms of power output can be determined by the following equation:

Power output = 
$$\eta_{t-el} \cdot (Q_{rec} - Q_{loss})$$
 (3)

where  $\eta_{t-el}$  is the efficiency of the Power Block (i.e., the steam Rankine cycle).

In terms of plant design and fixed costs, solar collectors, power block, pumps and other general direct costs are not expected to be significantly affected by a changeover of the HTF. Differently, major cost effects deal with the TES materials and components.

For equivalent heat storage capacity (hours or MWh thermal) the size of the TES system (i.e., volume of the TES tanks and total mass of molten salt) depends on the operative temperatures of the heat storage loop: the larger the temperature difference  $T_{S-high} - T_{S-low}$ , the smaller will be the required size (and costs) of the TES system according to the following relationship:

$$Q_{\text{TES}} = M_{\text{MS}} \cdot c_{\text{pMS}} \cdot \left( T_{\text{S-high}} - T_{\text{S-low}} \right)$$
(4)

where  $Q_{TES}$  is the specified heat storage capacity of the CSP plant,  $M_{MS}$  is the required total mass of molten salts to be loaded in the TES tanks, and  $c_{pMS}$  is the heat capacity of the molten salt used as heat storage medium.

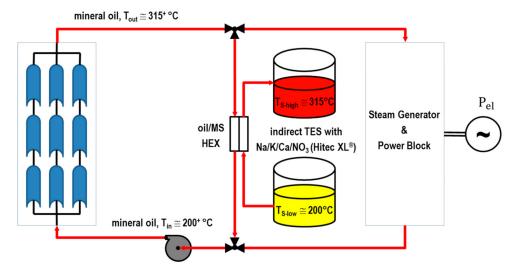
Consequently, for a given heat storage capacity ( $Q_{TES}$ ) the quantity of molten salt required can be determined by the following equation:

$$M_{\rm MS} = \frac{Q_{\rm TES}}{c_{\rm pMS} \cdot \left( T_{\rm S-high} - T_{\rm S-low} \right)}$$
(5)

while the volume of the TES tanks is proportional to  $M_{MS}$  divided by the density of the molten salt at  $T_{S-high}$ .

### 3.1. Different Layouts for Different HTFs

Figure 10 shows the general layout of a PT-TO CSP plant where the synthetic oil is replaced with a mineral oil (e.g., Therminol XP): since the molten salt TES system is still indirectly heated with the thermal oil, this changeover of the HTF does not imply a modification of the general layout of the CSP plant (compared to Figure 1). However, the mineral oil can be operated up to ~315 °C. Under such lower temperature range, for the TES system it is preferable to replace the typical binary solar salt mixture with the ternary Na/K/Ca/NO<sub>3</sub> one (Hitec XL<sup>®</sup>) characterized by lower cost (Figure 8) and wider



operation range at lower temperatures: it is possible to use the indirect two-tank TES system loaded with the Hitec XL<sup>®</sup> salt operating from  $T_{S-high} \cong 315 \text{ }^{\circ}\text{C}$  to  $T_{S-low} \cong 200 \text{ }^{\circ}\text{C}$ .

**Figure 10.** General layout of a CSP plant with linear concentrators, indirect TES and a mineral oil as HTF.

Under these conditions, compared to the benchmark case, lower steam temperature (and pressure) can be obtained, with lower Power Block efficiency, here assumed to be 32.1%, resulting in about 10% loss in power production. However, when the HTF temperature in the solar field range from 200 °C to 315 °C (average: 257.5 °C) evaluated heat losses in the 735,750 m<sup>2</sup> solar field will be around  $Q_{loss} = 15.4$  MW<sub>th</sub> and  $Q_{HTF} = Q_{rec} - Q_{loss} = 313.1 - 32.2 = 297.6$  MW<sub>th</sub>.

In conclusion, replacing the synthetic oil with a mineral oil will reduce the operative temperatures of the HTF loop, with the following effects:

- decreased Rankine cycle efficiency from 35.6% to 32.1%;
- reduced heat losses with an increase of the collected solar heat by about 6%;
- net effect on productivity, according to Equation (3), being about 4% reduction of power output.

The cost of the mineral oil HTF is marginal when compared to the synthetic oil (see Figure 8). The cost of the salt (Hitec  $XL^{(R)}$ ) to be loaded the TES system is slightly smaller than the one of the solar salt in the reference synthetic oil case, mainly thanks to the wider temperature difference ( $T_{S-high} - T_{S-low}$ ) and to the lower cost of the ternary Na/K/Ca/NO<sub>3</sub> mixture compared to the binary "solar salt" mixture (Figure 8). The volume of the TES tanks remains almost unaffected due to the relatively smaller energy density of the Hitec XL<sup>(R)</sup> molten salt (Figure 7).

In conclusion, replacing the traditional synthetic oil with a mineral oil HTF will not significantly change the CSP plant layout but the lowering of the operative temperatures will reduce both the overall power productivity ( $\sim -4\%$ ) and the fixed direct costs ( $\sim -4\%$ ).

Figure 2 shows the general layout of a PT-MS CSP plant with solar salt (binary Na/K/NO<sub>3</sub> mixture) and direct TES up to 550  $^{\circ}$ C.

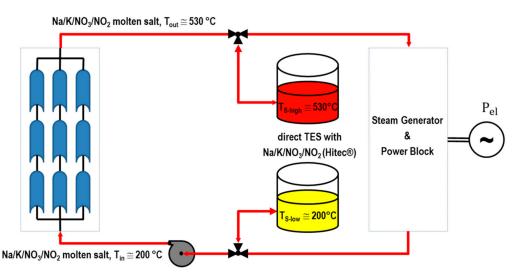
The increased maximum temperature of the HTF cycle from  $\sim$ 393 °C to 550 °C has a positive effect on the efficiency of the Rankine cycle; however, due to the higher heat losses, there isn't any expected relevant difference on plant productivity.

As far as the plant costs are concerned, the changeover of the HTF in the solar field will not significantly affect the costs for the solar collectors; the only relevant extra cost is the auxiliary heating system, by direct Joule effect, needed to pre-heat the solar receivers before flooding the solar field and to manage emergency draining operations [13]. Differently, solar receiver tubes and HTF piping materials for solar salt are more expensive than the ones used for thermal oils: a correction factor of 1.5 can be applied to take into account

the higher cost for piping materials. The additional cost due to auxiliary electrical heating of the piping should considered too. According to Figure 8 and Table 6, the cost for the HTF will be at least 1/3 the cost of the synthetic oil. However, the largest difference between solar salt and the synthetic oil (benchmark) is linked with the cost of the TES units: thanks to the wider operation range of the HTF, the TES size will be about 40% smaller, with significant gain in the cost of solar salt inventory and tanks; additionally, in the "direct" solar salt layout the intermediate oil/MS and ancillary instrumentation will not be required.

In conclusion, the changeover from the traditional PT-TO layout with synthetic oil to the direct solar salt PT-MS system results in minor effects on power productivity but significant benefits in terms of plant costs mainly due to the reduction of the TES units (about -15%).

Figure 11 shows the general layout of a PT-MS CSP plant with the ternary mixture with Na/K nitrate/nitrite (Hitec<sup>®</sup>) up to 530 °C and direct TES storage. In fact, although under air this fluid is unstable above 440 °C, its thermal stability limit can be increased up to 534 °C if it is maintained under inert (e.g., nitrogen) atmosphere [26]. Thanks to the lower freezing temperature, this salt can be operated in a wider temperature range, with the cold molten salt tank around 200 °C.



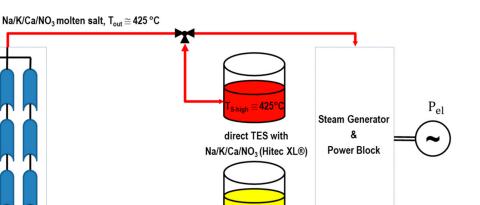
**Figure 11.** General layout of a CSP plant with linear concentrators and direct molten salt type Hitec<sup>®</sup> as HTF and TES material.

Thanks to the lower freezing temperature it possible to circulate the HTF at lower temperatures to reduce the heat losses. Overall there is a potential slight increase on plant productivity (~+3%) vs. the benchmark PT-TO case. If compared to the solar salt HTF, the Hitec<sup>®</sup> requires slightly smaller volumes (and costs) for the TES tanks given its higher energy density per unit volume (Figure 7), but the total cost for the salt is slightly higher due to the higher unit cost of the ternary nitrate/nitrite mixture.

It is noteworthy that the ternary mixture  $Na/K/NO_3/NO_2$  (Hitec<sup>®</sup>) also requires ancillary equipment to prevent air to enter the TES system.

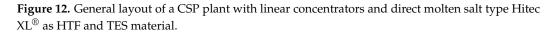
In conclusion, the use of the ternary mixture  $Na/K/NO_3/NO_2$  (Hitec<sup>®</sup>) results in similar performances than the solar salt. Heat losses are reduced but slightly more complex design of the TES system is required to avoid air contact with the molten salt.

Figure 12 shows the general layout of a PT-MS CSP where the ternary mixture Na/K/Ca/NO<sub>3</sub> (Hitec XL<sup>®</sup>) is used as HTF and heat storage material in a direct TES system.



 $\simeq 200$ 

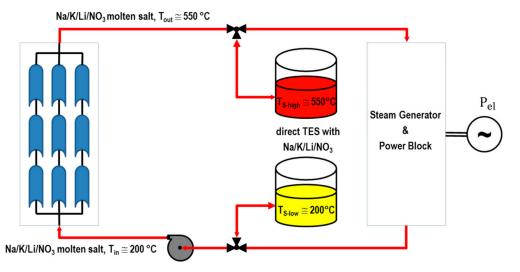
Na/K/Ca/NO<sub>3</sub> molten salt,  $T_{in} \cong 200 \,^\circ C$ 

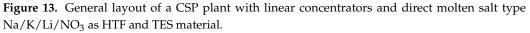


Thanks to the max operative temperature of the HTF around 425  $^{\circ}$ C, the efficiency of the thermal cycle is expected to be slightly higher than the reference one (i.e., PT-TO with synthetic oil up to ~393  $^{\circ}$ C); an overall slight increase of plant productivity is also expected when compared to other molten salt solutions, due to the lower heat losses.

Thanks to the lower price of this HTF, the major cost benefits derive from the overall lower cost of the salt. In total, this solution leads to about 12% savings in terms of CSP plant costs compared with the benchmark PT-TO case. It is also worth noting that, differently from the previous Hitec<sup>®</sup> case, the TES system can be maintained and operated under atmospheric air, without the necessity of nitrogen inertization.

Figure 13 shows the general layout of a PT-MS CSP when the ternary mixture  $Na/K/Li/NO_3$  is applied as HTF and heat storage material in a direct TES system.



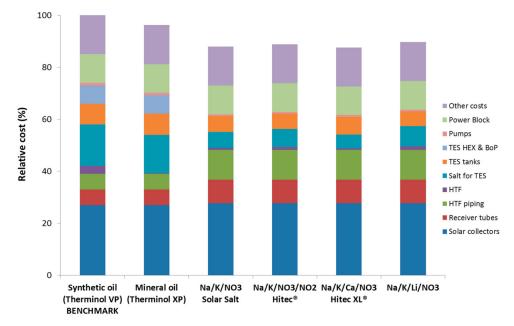


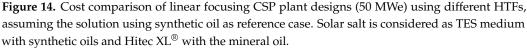
Due to the similar temperature range, performances and costs are expected to be similar to the Hitec<sup>®</sup> case.

### 3.2. Comparison between the HTFs

In the previous sections it is highlighted that the use of different HTFs has an impact on the design, performances and costs of linear-focusing CSP plants. As far as the performance is concerned, no major differences are expected ( $\leq \pm 5\%$ ) between different solutions in terms of overall productivity: the expected gain attainable by higher HTF temperatures, leading to higher Power Block efficiencies, is to some extent vanished when heat losses from the HTF loop are considered.

Differently, the use of different HTFs has a significant impact on CSP plant costs. Figure 14 shows the relative total and breakdown costs for the different CSP plant configurations here investigated. Solar collectors are not expected to sensibly change their cost by the changeover of the HTF; differently the cost of solar receiver tubes and piping will depend on the HTF due to the different temperatures and corrosion issues: a coefficient been applied to adapt the cost of typical carbon steel tubes to stainless steel alloy used with molten salts. Another cost item largely affected by the changeover of the HTF is the solar salt inventory needed as heat storage medium and the TES heat exchanger/BoP, both significantly reduced when the temperatures are increased and the indirect oil/molten salt TES system is converted to direct molten salt.





In general, the changeover of thermal oils to molten salts will increase the cost of the solar field (solar collectors, receiver tubes, HTF piping) but will significantly reduce the cost of the TES components (mainly salt quantity, tanks and heat exchangers). The net effect is a significant reduction of the fixed investment for the CSP plant.

Differences in Operation and Maintenance (O&M) costs are uneasy to estimate due to the different operation scenarios. Besides typical O&M cost factors of the two technologies (PT-TO and PT-MS) the PT-TO system is characterized by higher maintenance costs due to the degradation of the thermal oil, with consequent additional costs for periodical oil refilling, exhaust oil disposal and the replacement of receiver tubes due to hydrogen generation and permeation (through the steel tube) towards the vacuum annulus that reduces the performances of the receiver tubes [63].

All considered, the use of molten salts can lead to significant economic benefits compared to thermal oils (synthetic or mineral). In particular, solar salt and Hitec XL<sup>®</sup> resulted as the most advantageous HTFs, also considering that they present more favorable safety and environmental features compared the other thermal fluids.

This paper provides an extensive review of the properties of different heat transfer fluids (HTFs) proposed so far for CSP plants with linear concentrators and TES. Besides the physical/chemical properties, different HTFs are analyzed and compared in terms of material corrosion effects, environmental impact and costs. This review highlights the relevant environmental improvements that can be obtained when conventional synthetic oils are replaced with molten salts as HTFs. The extensive review of HTFs was based on data available from the open literature combined with experimental data obtained by the authors to thoroughly compare different solutions.

The different options have also been compared in terms of CSP plant design, costs and performances.

Replacing synthetic oils with mineral oils will not change the general layout of the CSP plant with indirect heat storage (TES) system. A loss of productivity (around 4%) is expected due to the reduction of the maximum operative temperatures (from ~393 °C to ~315 °C) but some savings in plant costs are also expected due to the lower cost of the mineral oil.

It is also highlighted that in CSP plants using (synthetic or mineral) oils as HTF, the ternary mixture Na/K/Ca/NO<sub>3</sub> (also known as Hitec XL<sup>®</sup>) can be used as heat storage medium in place of the typical binary "solar salt" mixture Na/K/NO<sub>3</sub>: thanks to the lower freezing temperatures and costs of the Ca-containing ternary mixture, the TES system will be more compact and the overall plant costs reduced.

Thermal oils (synthetic or mineral) and molten salt HTF have different application ranges in terms of maximum temperature of the thermal cycle: while mineral and synthetic oils can reach the abovementioned maximum temperatures of ~315 °C and ~393 °C, respectively, molten salts allow reaching maximum temperatures in the range from 400 °C to 550 °C or more. However, the easier management of CSP plants with thermal oils, especially in startup and shutdown operations, makes the use of thermal oils more attractive for the smaller scale CSP application range (~1 MWe or less).

When the thermal oil is replaced with molten salts, the general design of solar collectors (still representing the larger cost item of a CSP plant) remains unaltered, but the cost of the solar field increases to some extent, mainly due to the following reasons:

- need to apply an auxiliary (electrical) heating system with transformers to heat up the solar receiver tubes by direct Joule effect during plant start-up (flooding) and to manage emergency operations [13];
- electrical tracing and thicker thermal insulation of the HTF piping to prevent freezing of the molten salt;
- more expensive piping materials (stainless steel, e.g., ASTM 321H) should be used with molten salts, especially in piping sections where the higher temperatures (>400 °C) can be reached.

Differently, the use of molten salts as HTF leads to relevant cost reduction in the TES system design, mainly for the following reasons:

- avoidance of oil/MS heat exchangers and ancillary equipment related to the indirect TES system;
- use of smaller TES tanks with relevant savings in the procurement of the salt mixture and molten salt tanks.

The net effect is a significant reduction (>10%) of overall cost (CAPEX) for the CSP plant. It is worth observing that the above assessment was made assuming typical CSP plants with "medium size" heat storage capacity, i.e., in the range of 6–8 h [5,60]. However, CSP plants with larger TES size (up to 16 h or more) are expected to be developed in the future due to the increasing importance of "dispatchability" in the management of power grids with increasing penetration of vRES (variable Renewable Energy Sources) and technologies such as PV and wind [64]. The lower cost impact of the TES in CSP plants with molten salt HTF will make the molten salt technology even more convenient from economic perspectives.

This economic competitiveness can be further improved by developing innovative solutions/methods that will reduce the impact of the abovementioned factors that make the cost of the solar field more expensive.

Minor differences are expected ( $<\pm5\%$ ) in terms of overall power productivity of the CSP plant when the thermal oil is replaced with molten salts. Although the higher temperatures achievable with molten salts allow reaching higher efficiencies of the thermoelectric generation cycle, this gain is (to some extent) lost by the higher heat losses in the HTF loop occurring during solar collection periods and night circulation of the HTF in the solar field.

As far as operation and maintenance (O&M) costs are concerned, the thermal oil technology is expected to show higher maintenance costs due to the degradation of the thermal oil (i.e., additional costs for periodical oil refilling, exhaust oil disposal and the replacement of damaged receiver tubes due to hydrogen generation and permeation through the steel tube).

Finally, the following conclusions are drawn from the comparison of the different molten salt solutions considered in this paper:

- ternary mixtures such as Na/K/NO<sub>3</sub>/NO<sub>2</sub> (Hitec<sup>®</sup>) and Na/K/Li/NO<sub>3</sub> show similar performances in terms of operative temperature range, heat losses, productivity, compactness of the TES tanks and costs;
- the ternary mixture Na/K/NO<sub>3</sub>/NO<sub>2</sub> (Hitec<sup>®</sup>) requires a slightly more complex design of the TES system to avoid air contact with the molten salt in the TES tanks;
- the "solar salt" binary mixture (Na/K/NO<sub>3</sub>) and the ternary mixture Na/K/Ca/NO<sub>3</sub> (Hitec XL<sup>®</sup>) show lower costs than other HTFs;
- the ternary mixture Na/K/Ca/NO<sub>3</sub> (Hitec XL<sup>®</sup>) has lower max operative temperature (~425 °C) which reduces the efficiency of the thermal cycle but, thanks to the lower heat losses, net productivity is expected to be similar or even slightly higher than other PT-MS cases;
- Solar Salt and Hitec XL<sup>®</sup> show particularly favorable characteristics in terms of toxicity and environmental compatibility;
- technical issues with pumping and heat transfer may arise with Hitec XL<sup>®</sup> and Na/K/Li/NO<sub>3</sub> due to their high viscosity (Figure 4) when the temperature approaches the lower limit (here assumed 200 °C).

Regarding the salts' compatibility with construction materials, according to the available literature, the effective performances are more dependent on the working temperature than on the chemical composition of the salt mixture. As a rule, carbon steel can be considered for tanks and pipelines below 400  $^{\circ}$ C, while special stainless steels, such as AISI 321 and 347, should be used at higher temperatures.

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### Glossary

Definition	Nomenclature	Units
Specific heat	ср	kJ kg K W m K
Thermal conductivity	k	W m K
Dynamic viscosity	μ	Pas
Density	ρ	$\frac{\text{kg}}{\text{m}^3}$
Efficiency of the Power Block	$\eta_{t-el}$	% %
Molten salt Mass loaded in the TES	M <sub>MS</sub>	kg
Collected Solar heat in solar field	Q <sub>HTF</sub>	MW thermal
Solar radiation on solar receivers	Q <sub>rec</sub>	MW thermal
Heat losses in the solar field	Q <sub>loss</sub>	MW thermal
Heat storage capacity	Q <sub>TES</sub>	MW thermal (or hours)
Time	t	sec. or hours
Maximum temperature of the HTF loop and TES system	T <sub>S-high</sub>	°C
Minimum temperature of the HTF loop and TES system	T <sub>S-low</sub>	°C

# Abbreviations

Balance of Plant	BoP
Corrosion Rate	CR
Concentrating Solar Power	CSP
Carbon Steel	C-steel
Diphenyl Oxide	DPO
Heat Exchanger	HEX
Heat Transfer Fluid	HTF
Molten Salt	MS
Not available	n.a.
Operation and Maintenance	O&M
Parabolic Trough	PT
Parabolic Trough with Molten Salt HTF	PT-MS
Parabolic Trough with Thermal Oil HTF	PT-TO
Static and Dynamic (for corrosion tests)	S/D
Solar Tower with Molten Salt HTF	ST-MS
Thermal Energy Storage	TES
Thermal Oil	TO
variable Renewable Energy Source	vRES

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