





A Decentralized, Hybrid Photovoltaic-Solid Oxide Fuel Cell System for Application to a Commercial Building

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Abstract: New energy solutions are needed to decrease the currently high electricity costs from conventional electricity-only central power plants in Cyprus. A promising solution is a decentralized, hybrid photovoltaic-solid oxide fuel cell (PV-SOFC) system. In this study a decentralized, hybrid PV-SOFC system is investigated as a solution for useful energy supply to a commercial building (small hotel). An actual load profile and solar/weather data are fed to the system model to determine the thermoeconomic characteristics of the proposed system. The maximum power outputs for the PV and SOFC subsystems are 70 and 152 kWe, respectively. The average net electrical and total efficiencies for the SOFC subsystem are 0.303 and 0.700, respectively. Maximum net electrical and total efficiencies reach up to 0.375 and 0.756, respectively. The lifecycle cost for the system is 1.24 million USD, with a unit cost of electricity at 0.1057 USD/kWh. In comparison to the conventional case, the unit cost of electricity is about 50% lower, while the reduction in CO₂ emissions is about 36%. The proposed system is capable of power and heat generation at a lower cost, owing to the recent progress in both PV and fuel cell technologies, namely longer lifetime and lower specific cost.

Keywords: hybrid system; decentralized system; combined-heat-and-power; solid oxide fuel cells; photovoltaic; thermoeconomic modeling

1. Introduction

Efforts to increase energy efficiency have intensified over recent years due to the rapid increase of fossil fuel prices and also the need to decrease harmful emissions to the atmosphere [1]. Cogeneration allows the combination of various technologies to improve the fuel efficiency of electricity-only power plants or combined-heat-and-power (CHP) systems [2]. Earlier systems have included combined cycle power plants at large scale (10–100 MWe). In the resulting systems there exists a high level of complexity due to the increased number of parameters. Therefore, there is a need to apply advanced methodologies able to determine optimum solutions. However, this procedure becomes rigorous in case a number of parameters (e.g., thermodynamic, economic and environmental criteria) must be included [3]. Fuel cell technology has been proposed at the kW to the MW scale in a number of proposed systems. In lower temperature proton exchange membrane fuel cells (PEMFCs), CHP systems have been primarily applied at the kW scale, for smaller residential applications, where low-grade heat (recovered from the fuel cell exhaust) is usually adequate to cover residential load profiles, such as space heating and domestic hot water preparation [4,5]. These systems are sometimes operated jointly with vapor compression heat pumps to boost heat generation and/or to provide space cooling [6]. Such systems have been proposed for single-family households and in some cases for multi-unit

residential applications [7,8]. Coupling of SOFCs with absorption chiller-heater units has also been proposed for larger scale, decentralized applications such as commercial buildings [9]. Although the resulting operational configurations have led to high system efficiencies (80% to 90%), their complexity resulted in high capital cost, which often dominates lifecycle cost.

Recent progress in SOFC technology includes important advances, such as higher lifetime, lower capital cost, higher electrical efficiency and simpler fuel processing requirements (in the case of natural gas-fueled systems—as compared to PEMFC technology) [10,11]. For small-scale residential applications, SOFC-based, natural gas-fueled micro-CHP systems have been proposed through thermoeconomic modeling and optimization techniques and improved operational strategies [12,13]. The application of effective optimization techniques, such as decomposition strategies, have been applied in some cases for the design/synthesis optimization of such systems [7,8]. For large-scale applications, the possibility of combining SOFC technology with heat engines, such as gas turbine cycles, has been thoroughly investigated since the early 2000s. In natural gas-fueled hybrid systems, where high temperature SOFC stacks have been integrated with gas turbine cycles, effort was placed on the increase of system efficiency to lower fuel consumption [14]. Due to the complexity of the proposed systems, the design/synthesis options are usually evaluated with advanced optimization techniques [15,16].

More recent research effort has focused on the possibility of combining fuel cell technology with renewable energy sources (RES). The combination of RES with fuel cell technology is a more environmentally friendly solution than decentralized hybrid photovoltaic (PV)-gas turbine systems, because in the latter case emissions are generated on-site, i.e., near the serviced buildings [17,18]. The deployment of PV units continues to increase because of significant cost reductions in addition to supportive policies, such as net-metering [19]. In such systems, excess generation of electricity from RES, e.g., via solar PV panels or wind turbines, can be converted to hydrogen through an electrolyzer unit [20], stored in a hydrogen storage tank, and then reconverted to electricity when renewable energy is unavailable [21]. The design of such systems for variable load has proven difficult and in most cases the proposed systems have considered grid-connected operation to allow import/export of electricity, while in other cases a constant load operation was followed [22,23]. A combination of RES with natural gas (or biogas)-fueled fuel cell units could allow a rapid deployment of these hybrid systems [24]. Currently the application of hybrid PV-SOFC systems seems more attractive for commercial buildings as the load demand closely matches the solar energy availability. The use of dynamic or quasi-steady state modeling is usually required to model the system as realistically as possible [25,26].

The objective of this research study is the thermoeconomic modeling of a decentralized, hybrid PV-SOFC system for application to a commercial building. The PV subsystem, the fuel cell stack, and the steam methane reformer (SMR) reactor components are modeled in detail to allow a realistic representation of their operation at both design and off-design conditions. In addition, a significant shortcoming of previous studies on hybrid RES-fuel cell systems is the fact that, in most cases, actual load profiles have not been considered. The omission of an actual load profile prohibits the extraction of realistic outcomes on the actual viability of such systems. The current study considers both solar/weather data and an actual load profile for a commercial building for the whole year. This approach leads to a more accurate determination of the thermoeconomic characteristics of the proposed system, allowing a direct comparison to conventional useful energy generation. The fuel processor (pre-reformer) is of the SMR type, since it is more efficient than other technologies (e.g., partial oxidation), allowing more efficient natural gas conversion to hydrogen [11]. The current research study investigates the economic competitiveness of the proposed system in comparison to conventional or alternative power generation. Four different cases are investigated and compared, namely: (A) Central power grid connection (conventional), (B) central power grid connection assisted with PV arrays, (C) non-grid connected SOFC system and (D) decentralized hybrid PV-SOFC system (proposed system). The outcomes of the research work are expected to reveal the possibility of combining and utilizing two highly advantageous technologies, i.e., PVs and solid oxide fuel cells, with an analysis beyond

theoretical predictions. This is done with a detailed thermoeconomic modeling of the components, and further on with their overall integration in the system model. Moreover, through the development of a cost model, a complete thermoeconomic analysis is facilitated to lay out the characteristics of the proposed hybrid system.

2. System Configuration

The proposed system, shown in Figure 1, was designed to fully fulfill an actual load profile for a commercial building. It includes a natural gas-fueled SOFC subsystem and a solar PV subsystem. The system also includes DC/AC inverters to convert the DC current generated by the PV and the SOFC subsystems to AC electricity prior to distribution to the buildings. In the SOFC subsystem, natural gas (NG) is compressed in the fuel compressor and sulfur is removed with the desulfurizer. The NG is preheated through heat exchanger (HEx) HEx1 before entering the SMR. The endothermic process in the SMR requires external heating, which is generated by a catalytic combustor. The synthesis gas (syngas) at the SMR exit is fed to the fuel cell anode. Air drawn from the atmosphere is filtered and blown to HEx3 for preheating and then fed to the fuel cell cathode. The fuel cell reaction in the SOFC stack generates electricity and also a hot mixture at the fuel cell exit. The hot exhaust mixture is fed to the combustor, along with natural gas from the natural gas supply and air. The flue gas exiting the SMR is used to provide heat for the four heat exchangers (HEx1-HEx4). HEx2 is used to generate steam for the SMR. HEx4 is used to provide low-grade heat externally, i.e., to heat water from recovered heat and supply it to the hot water storage tank. Through the hot water storage tank, hot water is provided to the buildings. At the exit of HEx4, the exhaust flue gas is released to the atmosphere after separation of water through a water separator.



Figure 1. Schematic representation of the proposed hybrid photovoltaic-solid oxide fuel cell (PV-SOFC) system.

The main assumptions for the current study are the following:

- 1. The proposed system operates in complete autonomy, i.e., it is not connected to a central power grid (no import/export of electricity).
- 2. The maximum PV power output is set equal to the minimum electric load in the load profile to ensure no power is wasted. In turn, this value is used to size the SOFC subsystem. The system model is modeled in such a way to ensure that the proposed system is capable of completely covering the building load profile at all times, throughout the year.
- 3. Heat losses are considered in the three main components of the SOFC subsystem, namely: The SMR reactor, the SOFC stack, and the catalytic combustor. Additionally, pressure losses are considered in every component of the SOFC subsystem.
- 4. Additional heating (space heating and domestic hot water) is provided through natural gas-fired boilers, while space cooling is provided through electric vapor-compression heat pumps. This equipment is already available in the buildings and therefore its associated capital cost is not considered in the thermoeconomic modeling for this study.
- 5. The hourly solar and ambient temperature data used in the simulation of the PV subsystem are based on a Typical Meteorological Year—TMY2 for Nicosia, Cyprus [27].
- 6. The consumption data system is applied for a small hotel with load profile data extracted from [28]. The load profile includes the following loads (all in an electrical energy basis): Fans, interior equipment, interior lights, space cooling, space heating, and domestic hot water. The load profile is shown in Figure 2.



Figure 2. The load profile includes the following electrical loads (graph color in parenthesis): Fans (black), interior equipment (blue), interior lights (red), space cooling (green), space heating (purple), and domestic hot water (brown).

3. System Modeling

The modeling of the components of the proposed hybrid system was based on first principles to accurately represent the coupling and operation of the system as realistically as possible. After modeling each component, the components were coupled together to form the SOFC subsystem. Subsequently, simulation of the PV subsystem generates PV data for the simulation of the overall system model. Additionally, a cost model was developed for the economic analysis of the proposed

system. It includes all necessary cost functions and inputs needed for the calculation of capital costs, fuel cost, lifecycle cost and unit cost of electricity. The modeling of the system was developed with the software Engineering Equation Solver (EES)—Professional version. Hourly simulation data were generated for a complete year, i.e., 8760 hourly segments.

3.1. Photovoltaic Subsystem

The PV subsystem was based on the Hay-Davies-Klucher-Reindl (HDKR) modeling methodology [29], i.e., the total incident solar radiation on a tilted surface is calculated with a consideration of both the ground-reflected and the beam effects:

$$I_T = (I_b + I_d \cdot A_i) \cdot R_b + I_d \cdot (1 - A_i) \cdot \left(\frac{1 + \cos\beta}{2}\right) \cdot \left[1 + f \cdot \sin^3\left(\frac{\beta}{2}\right)\right] + I \cdot \rho_g \cdot \left(\frac{1 - \cos\beta}{2}\right).$$
(1)

In the PV array, the temperature was calculated with the relation (the effect of wind speed is considered negligible):

$$\frac{T_c - T_{amb}}{T_{NOCT} - T_{amb,NOCT}} = \frac{I_T}{I_{T,refer}} \cdot \left(1 - \frac{\eta_{ref}}{0.9}\right).$$
(2)

The array's maximum power point efficiency is:

$$\eta_{mp} = \eta_{refer} \cdot \left(1 + \mu_{mp} \cdot (T_c - T_{amb,NOCT})\right). \tag{3}$$

The PV array's electricity output is:

$$P_{pv} = A_{pv,array} \cdot I_T \cdot \eta_{mp}. \tag{4}$$

3.2. SOFC Subsystem

The SOFC subsystem includes the fuel processing subsystem with the fuel pre-reformer (SMR reactor), four heat exchangers, SOFC stack and actuators. For the configuration shown in Figure 1, the inputs are given in Table 1. The fuel utilization factor was set at 0.92, and fuel cell temperature was set at 750 °C [10]. The temperature of fuel at the fuel preheater exit, the temperature of the reformate at the SMR reactor exit/anode inlet, and the temperature of the flue gas exiting the catalytic combustor were set at 450, 650 and 1005 °C, respectively [11]. HEx4 flue gas exit temperature was set at 55 °C because it must be at 25 °C above the dew point of the combustion product gases [10]. The steam-to-carbon ratio was set at 2.5, which although it is a relatively low value, the SOFC can treat CO as fuel [30], and therefore CO content does not need to be significantly reduced prior to anode inlet.

Table 1. System input parameters of the solid oxide fuel cell (SOFC) subsystem.

	Parameter Description	Value
U_f	Fuel utilization factor	0.92
A_{fc}	Fuel cell effective cross-sectional area	144 cm ²
n _{cells}	Total number of cells in fuel cell stacks	12,000
T_{fc}	Fuel cell operating temperature	750 °C
\tilde{T}_6	Fuel preheater exit temperature	450 °C
T_8	SMR reactor reformate exit temperature	650 °C
T_{13}	Cathode inlet temperature	650 °C
T_{17}	Combustor exit temperature	1005 °C
T_{22}	HEx4 flue gas exit temperature	55 °C
T_{25}	Water pump 1 inlet temperature	40 °C
T_{28}	Hot water storage tank return temperature	40 °C
T_{29}	Hot water storage tank supply temperature	65 °C
SC	Steam-to-carbon ratio	2.5

3.2.1. SMR Reactor

An SMR reactor configuration was assumed for the pre-reformer. The SMR reactor model is based on chemical equilibrium [10–12]. Two chemical reactions were included: SMR reaction (methane-steam), and water gas shift (WGS) reaction (carbon monoxide-steam) [31]. Since the SMR reaction is endothermic, heat must be supplied by an external source [32] (in this case from the catalytic combustor).

For the SMR reaction, i.e., $CH_4+H_2O \Rightarrow CO_2+3H_2$, the overall change in Gibbs free energy is:

$$\Delta G_{smr} = -1 \cdot g_{CH_4} - 1 \cdot g_{H_2O} + 1 \cdot g_{CO} + 3 \cdot g_{H_2'}$$
(5)

$$\arg 1 = \left(\frac{-\Delta G_{smr}}{R \cdot T_{ref,out}}\right). \tag{6}$$

The equilibrium constant at the given temperature for the SMR reaction is:

$$K_{smr} = \exp(\arg 1). \tag{7}$$

For the WGS reaction, $CO + H_2O \rightleftharpoons CO_2 + H_2$, the overall change in Gibbs free energy is:

$$\Delta G_{wgs} = -1 \cdot g_{\rm CO} - 1 \cdot g_{\rm H_2O} + 1 \cdot g_{\rm CO_2} + 1 \cdot g_{\rm H_2\prime}$$
(8)

$$\arg 2 = \left(\frac{-\Delta G_{wgs}}{R \cdot T_{ref,out}}\right). \tag{9}$$

The equilibrium constant at the given temperature for the WGS reaction is:

$$K_{wgs} = \exp(\arg 2). \tag{10}$$

The molar flow output is defined as:

$$\dot{n}_{ref,out} = \dot{n}_{ref,in,CH_4} + \dot{n}_{ref,in,H_2O} + 2 \cdot X_{smr}.$$
(11)

The equilibrium constants for the aforementioned reactions are [33]:

$$K_{smr} \cdot y_{ref,out,CH_4} \cdot y_{ref,out,H_2O} = y_{ref,out,CO} \cdot y_{ref,out,H_2}^3 \cdot \left(\frac{p_{ref,out}}{p_{amb}}\right)^2,$$
(12)

$$K_{wgs} \cdot y_{ref,out,CO} \cdot y_{ref,out,H_2O} = y_{ref,out,CO_2} \cdot y_{ref,out,H_2}.$$
(13)

A molar flow rate balance for each species can be applied at the reformer inlet and outlet:

$$\dot{n}_{ref,out,CH_4} = \dot{n}_{ref,in,CH_4} - X_{smr}.$$
(14)

$$\dot{n}_{ref,out,H_2O} = \dot{n}_{ref,in,H_2O} - X_{smr} - X_{wgs}.$$
 (15)

$$\dot{n}_{ref,out,CO} = X_{smr} - X_{wgs}.$$
(16)

$$\dot{n}_{ref,out,H_2} = 3 \cdot X_{smr} + X_{wgs}.$$
(17)

$$\dot{n}_{ref,out,CO_2} = X_{wgs}.$$
(18)

The flue gas temperature at exit is calculated through an energy balance in the reformer:

$$\dot{Q}_{heat,smr} + \dot{E}_{in,smr} = \dot{E}_{out,smr} + \dot{Q}_{loss,smr}.$$
(19)

3.2.2. SOFC Stack

The SOFC stack model includes both the fuel cell reaction and direct internal reforming processes. For the latter, the reforming process takes place at the surface of the catalysts (anode), where hydrogen gas is mixed with steam before entering the anode [34]. Internal reforming is identical to the SMR reactor modeling equations, described in Section 3.2.1.

For the fuel cell reaction, the open circuit voltage is modeled as follows [10–12]:

$$E_{ocv} = E_{ocv,0} + \frac{R \cdot T_{fc}}{2 \cdot F} \cdot \ln\left(\frac{\frac{y_{ano,H_2} \cdot p_{fc}}{p_{amb}} \cdot \sqrt{\frac{y_{cat,O_2} \cdot p_{fc}}{p_{amb}}}}{\frac{y_{ano,H_2O} \cdot p_{fc}}{p_{amb}}}\right).$$
(20)

The reversible voltage is:

$$E_{ocv,0} = \frac{-\Delta g^o{}_f}{2 \cdot F}.$$
(21)

The Gibbs free energy is:

$$\Delta g^o{}_f = 1 \cdot g_{\rm H_2O} - 0.5 \cdot g_{\rm O_2} - 1 \cdot g_{\rm H_2}. \tag{22}$$

The activation losses are based on the Butler–Volmer equation, defined for the anode and cathode, respectively, to determine the current density:

$$i = i_{0,ano} \cdot \left(\exp\left(\alpha \cdot \frac{n_e \cdot F}{R \cdot T_{fc}} \cdot V_{act,ano}\right) - \exp\left(-(1-\alpha) \cdot \frac{n_e \cdot F}{R \cdot T_{fc}} \cdot V_{act,ano}\right) \right),$$
(23)

$$i = i_{0,cat} \cdot \left(\exp\left(\alpha \cdot \frac{n_e \cdot F}{R \cdot T_{fc}} \cdot V_{act,cat}\right) - \exp\left(-(1-\alpha) \cdot \frac{n_e \cdot F}{R \cdot T_{fc}} \cdot V_{act,cat}\right) \right),$$
(24)

where $i_{0,ano}$ and $i_{0,cat}$ are the exchange current densities for the anode and cathode, respectively:

$$i_{0,ano} = \gamma_{ano} \cdot \left(\frac{y_{ano,H_2} \cdot p_{fc}}{p_{amb}}\right) \cdot \left(\frac{y_{ano,H_2O} \cdot p_{fc}}{p_{amb}}\right)^{-0.5} \cdot \exp\left(-\frac{E_{act,ano}}{R \cdot T_{fc}}\right),\tag{25}$$

$$i_{0,cat} = \gamma_{cat} \cdot \left(\frac{y_{cat,O_2} \cdot p_{fc}}{p_{amb}}\right)^{0.25} \cdot \exp\left(-\frac{E_{act,cat}}{R \cdot T_{fc}}\right).$$
(26)

The activation overvoltage is determined as the sum of anode and cathode losses:

$$V_{act} = V_{act,ano} + V_{act,cat}.$$
(27)

Concentration losses are the gradual losses due to the reactant depletion in the catalyst layer, and they are defined as the difference between the Nernst potential at the catalyst layer and the bulk flow at both anode and cathode [35]. The limiting current densities for hydrogen and oxygen species are defined as follows, respectively:

$$i_{L,H_2} = 2 \cdot F \cdot C_{H_2,0} \cdot h_{m,H_2},$$
(28)

$$i_{L,O_2} = 4 \cdot F \cdot C_{O_2,0} \cdot h_{m,O_2},$$
(29)

where $C_{H_2,0}$ and $C_{O_2,0}$ are the concentration of species for hydrogen and oxygen, respectively.

The concentration losses are [35]:

$$V_{conc} = -\frac{R \cdot T_{fc}}{2 \cdot F} \cdot ln\left(\left(1 - \frac{i}{i_{L,H_2}}\right) \cdot \left(1 - \frac{i}{i_{L,O_2}}\right)^{0.5}\right).$$
(30)

The Ohmic losses are defined as the product of current density and Ohmic resistance:

$$V_{ohm} = i \cdot R_i. \tag{31}$$

Based on the above definitions, the cell voltage can be defined as follows [34]:

$$V_{cell} = E_{ocv} - V_{act} - V_{conc} - V_{ohm}.$$
(32)

Fuel cell stack voltage, current, and power are defined as follows, respectively:

$$V_{fc} = V_{cell} \cdot n_{cells},\tag{33}$$

$$I_{fc} = i \cdot A_{fc}, \tag{34}$$

$$P_{sofc} = V_{fc} \cdot I_{fc}. \tag{35}$$

The molar flow rate of oxygen at the inlet of the cathode can be calculated through an energy balance:

$$Q_{in,fc} = Q_{out,fc} + Q_{loss,fc} + P_{sofc}$$
(36)

3.2.3. Auxiliary Components

The auxiliary components are the actuators (air blower, fuel compressors and two water pumps), the catalytic combustor and the four heat exchangers. The actuators were modeled using fundamental equations, while the catalytic combustor model was based on an energy balance of products and reactants. The modeling of the heat exchangers was based on the Logarithmic Mean Temperature Difference (LMTD) method.

3.3. Overall System

The proposed hybrid system includes two prime movers for the generation of electrical energy. Additionally, in the case of the SOFC subsystem, heat is generated and recovered for external use in the buildings to satisfy the heating loads. Therefore, an algorithm must be included in the code of the system model to relate fuel cell power output, PV power output and power demand. Additionally, since the system is non-grid connected, it must be ensured that no excess power is generated from the PV subsystem.

$$If(P_{load} > P_{pvs}) then$$

$$P_{fcs} = P_{load} - P_{pvs}$$

$$P_{pv,exc} = 0$$
Else
$$If(P_{load} < P_{pvs}) then$$

$$P_{fcs} = 0$$

$$P_{pv,exc} = P_{pvs} - P_{load}$$
Else
$$P_{fcs} = 0$$

$$P_{pv,exc} = 0$$
EndIf
EndIf

The inverter power losses for the PV subsystem are calculated as follows:

$$\dot{P}_{loss,inv,pv} = \dot{P}_{pv} \cdot (1 - \eta_{inv,pv}), \tag{38}$$

$$\dot{P}_{pvs} = \dot{P}_{pv} - \dot{P}_{loss,inv,pv}.$$
(39)

Similarly, for the SOFC subsystem:

$$\dot{P}_{loss,inv,sofc} = \dot{P}_{sofc} \cdot \left(1 - \eta_{inv,sofc}\right). \tag{40}$$

The net electrical power output for the SOFC subsystem is defined as follows:

$$\dot{P}_{fcs} = \dot{P}_{sofc} - \dot{P}_{loss,inv,sofc} - \dot{P}_{ab} - \dot{P}_{comp} - \dot{P}_{pump1} - \dot{P}_{pump2}.$$
(41)

The net electrical efficiency for the SOFC subsystem can be based on the lower heating value (LHV) or the higher heating value (HHV), respectively [36]:

$$\eta_{el,net,LHV} = \frac{\dot{P}_{fcs}}{\dot{E}_{fuel,LHV}},\tag{42}$$

$$\eta_{el,net,HHV} = \frac{P_{fcs}}{\dot{E}_{fuel,HHV}}.$$
(43)

The thermal efficiency is the ratio of recovered heat rate actually used to cover the building heating loads (fully or partly) to the chemical energy rate of the fuel consumed by the SOFC subsystem:

$$\eta_{th} = \frac{Q_{th}}{\dot{E}_{fuel,LHV}} \tag{44}$$

The total SOFC subsystem efficiency is the sum of SOFC subsystem net electrical efficiency and thermal efficiency:

$$\eta_{fcs} = \eta_{el,net,LHV} + \eta_{th}.$$
(45)

The thermal-to-electric ratio is the ratio of recovered heat rate to net electrical power output:

$$TER = \frac{Q_{th}}{\dot{P}_{fcs}}.$$
(46)

When the recovered heat from the SOFC subsystem is inadequate to cover the heating loads, additional heat must be generated externally:

$$P_{heat,net} = \max(0, (P_{heat} + P_{dhw}) - P_{th}).$$

$$\tag{47}$$

The total load profile electrical energy requirement is the sum of electricity required to operate the fans, the interior lights, the interior equipment, the space cooling, and supplementary heating:

$$P_{load} = P_{fan} + P_{light} + P_{equip} + P_{cool} + P_{heat,net}.$$
(48)

3.4. Cost Model

A cost model was developed to determine the economic performance of the proposed hybrid system, based on the methodology found in [37]. The modeling equations are shown in Table 2, while the values of the constant parameters are given in Table 3. The specific cost of the PV array was set at 2.00 USD/W, which is based on approximate values given in [38]. The specific cost of the SOFC subsystem and the power subsystem were approximated from values given in [39], and they were set at 2.00 and 1.00 USD/W, respectively. The specific cost of the power subsystem included the two DC/AC inverters and the power conditioning components. The cost of fuel (i.e., natural gas) was set at 7.19 USD/MMBTU, which is the current cost in the European Union (EU) [40]. The lifetime was set at 20 years for the system (i.e., PV arrays and power subsystem) [19,37] and 5 years for the

SOFC subsystem [39], with a fuel cell operation factor set at 0.50, since the fuel cell is operated for approximately 50% of the time. The cost of the hot water storage tank was based on an approximation from values given in [41]. The values for the remaining parameters were taken from [37].

	Variable Description (Unit)	Model Equation
C _{fc}	Cost of SOFC subsystem (USD)	$N_{lt,fc,adj} = N_{lt,fc} / z_{fc} n_{re} = N_{lt} / N_{lt,fc,adj}$ $C_{fc} = n_{re} \cdot c_{fcs} \cdot \dot{P}_{fc,max}$
c_{fuel}	Cost of fuel in the first year (USD)	$c_{fuel} = c_{\$MMBtu} \cdot (3.6/3.41)[\$/GJ] \cdot \left 1 \times 10^{.9} \frac{\$/J}{\$/GJ} \right $
E_{py}	Annual fuel consumption (J)	$E_{py} = E_{fuel,in,yr} \cdot \left 3600 \frac{J}{W \cdot h} \right $
$c_{fy} \\ C_{pv}$	Annual cost of fuel (USD/year) Cost of PV arrays (USD)	$c_{fy} = E_{py} \cdot c_{fuel}$ $C_{pv} = c_{pvs} \cdot P_{pv,max}$
C_{inv}	Cost of power subsystem (USD)	$C_{inv} = c_{invs} \cdot \left(\dot{P}_{pv,max} + \dot{P}_{fc,max} \right)$
C _{sys} C _{down}	Total cost of system (USD) Down payment (USD)	$C_{sys} = C_{pv} + C_{fc} + C_{inv} + C_{hwst}$ $C_{down} = (1 - f_{loan}) \cdot C_{sys}$
AP_n	Capital recovery factor (-)	$AP_n = \frac{r_n}{1 - (1 + r_n)^{-N_{LT}}}$ $r_1 = r_{mL} - i r_2 = r_{mL} r_3 = \frac{r_2 - r_1}{0.01 + r_1} r_4 = \frac{r_{mL} - r_e}{1 + r_1}$
PA_n	Uniform series present worth factor (-)	$PA_n = (AP_n)^{-1}$
FP_n	Compound amount factor (-)	$FP_n = (1+r_n)^{-N_{LT}}$
PF_n	Present worth factor (-)	$PF_n = (FP_n)^{-1}$
C _{loan}	Cost of the loan (USD)	$C_{loan} = \frac{AP_1}{AP_2} \cdot f_{loan} \cdot C_{sys}$
D _{loan}	Tax deduction on the loan (USD)	$D_{loan} = t \cdot f_{loan} \cdot C_{sys} \left(\frac{AP_1}{AP_2} - \frac{AP_1 - r_1}{(1 + r_1) \cdot AP_3} \right)$
C_{twc}	Total worth of capital (USD)	$C_{twc} = C_{down} + C_{loan} - D_{loan}$
D_{dep}	Linear depreciation of capital (USD)	$D_{dep} = t \cdot PA_2 \cdot (C_{sys}/N_{LT})$
D _{cred}	Tax credit (USD)	$D_{cred} = t_{cred} \cdot C_{sys}$
D_{salv}	Salvage worth (USD) Tax paid on property (USD)	$D_{salv} = f_{salv} \cdot C_{sys} \cdot PF_2 \cdot (1 - t_{salv})$
C _{omi}	Operation, maintenance and insurance cost (USD)	$C_{omi} = f_{omi} \cdot C_{sys} \cdot PA_2 \cdot (1-t)$
C_{tcf}	Total cost of fuel (USD)	$C_{tcf} = c_{fy} \cdot \left(\frac{1-t}{AP_4}\right)$
LCC	Life cycle cost (USD)	$LCC = C_{twc} + C_{prop} + C_{omi} + C_{tcf} - (D_{dep} + D_{cred} + D_{salv})$
c _{el}	Unit cost of electricity (USD/kWh)	$P_{cs,life} = N_{lt} \cdot P_{load,yr}$ $c_{el} = LCC/P_{cs,life}$

 Table 2. Cost modeling equations for the proposed hybrid system.

Table 3. Parameters held constant in	1 the	cost model.
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	Parameter Description	Value
Cpvs	Specific cost of PV arrays	2.00 USD/W
cfcs	Specific cost of SOFC subsystem	2.00 USD/W
Cinvs	Specific cost of power subsystem	1.00 USD/W
c _{\$MMBtu}	Cost of fuel (natural gas)	7.19 USD/MMBTU
N _{lt}	System lifetime	20 years
$N_{lt,fc}$	SOFC subsystem lifetime	5 years
z_{fc}	Fuel cell operation factor	0.50
C_{hwst}	Cost of hot water storage tank	5000 USD
r _e	Real fuel price escalation rate	0.10
i	Inflation rate	0.01
r_m	Market discount rate	0.06
r_{mL}	Market loan rate	0.05
f _{loan}	Fraction of the capital cost paid through a loan	0.80
t	Incremental income tax	0.40
t _{cred}	Tax credit	0.02
fsalv	Salvage fraction	0.10
t_{salv}	Salvage tax	0.20
fprop	Property fraction	0.50
tprop	Property tax	0.25
f_{omi}	Operation and maintenance fraction	0.01

4. Results and Discussion

In this section the system model is validated with available literature data. Then, the performance of the proposed hybrid system is presented in detail. Finally, the proposed hybrid system is compared with conventional and alternative system configurations to analyze and investigate its competitiveness in regard to key thermoeconomic parameters.

4.1. Validation

For the validation of the SOFC stack, relevant literature data from [42] were used. As shown in Figure 3, the literature data compare well against the simulation data generated by the system model, with only a small deviation in the results. The PV subsystem was validated in a previous publication by some of the authors [43].



Figure 3. Validation of the modeled SOFC stack against literature data from [42].

4.2. Performance Characteristics of the Proposed Hybrid System

The proposed hybrid PV-SOFC system was sized in accordance with the requirements of the assumptions defined in Section 2. Based on these assumptions, the PV maximum power output is 70 kWe, while the SOFC maximum power output at full-load (i.e., design conditions) is 152 kWe. The average annual net electrical efficiency of the SOFC subsystem is 0.303, while total efficiency is 0.700. Maximum net electrical and total efficiencies can reach up to 0.375 and 0.756, respectively. In terms of annual useful energy generation, the electricity output (actual electricity delivered to the buildings) of the PV and SOFC subsystems is 135.9 and 451.2 MWh, respectively. The SOFC subsystem also provides 694.5 MWh of heating through heat recovery of the flue gas exhaust by the SOFC subsystem. This amount can almost completely cover the heating needs of the buildings, with only 7.2 MWh needed to be generated in addition. The current density at design conditions is 1228 A/m². A summary of the performance characteristics of the proposed system is given in Table 4. For an illustration of the performance of the system, Table 5 includes the values for the thermophysical parameters of the system at full load conditions for the SOFC subsystem.

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	Parameter Description	Value
\dot{P}_{pv}	PV maximum power output	70 kWe
P _{sofc}	SOFC maximum power output	152 kWe
$\eta_{el,net,LHV}$	Net electrical efficiency of SOFC subsystem	0.200 (full-load) 0.303 (average) 0.375 (maximum)
η_{fcs}	Total efficiency of SOFC subsystem	0.659 (minimum) 0.700 (average) 0.756 (maximum)
TER	Thermal-to-electric ratio	0.8 (minimum) 1.3 (average) 2.5 (maximum)
P_{pvs}	Annual electricity output of PV subsystem	135.9 MWh
P_{fcs}	Annual electricity output of SOFC subsystem	451.2 MWh
\dot{P}_{th}	Annual heat recovery from SOFC subsystem	694.5 MWh
P _{heat,net}	Annual additional heat generation	7.2 MWh
Pload	Annual electricity load requirement	587.1 MWh
E_{fuel}	Annual fuel consumption	1610 MWh
i _{fc,des}	Current density at design conditions	1228 A/m^2
$\eta_{inv,pv}$	Inverter efficiency of PV subsystem	0.961 (average)
η _{inv.sofc}	Inverter efficiency of SOFC subsystem	0.970 (average)
U_1	Overall heat transfer coefficient of HEx1	$90 \text{ W/m}^2 \cdot \text{K}$
U_2	Overall heat transfer coefficient of HEx2	$292 \text{ W/m}^2 \cdot \text{K}$
U_3	Overall heat transfer coefficient of HEx3	$4960 \text{ W/m}^2 \cdot \text{K}$
U_4	Overall heat transfer coefficient of HEx4	$8396 \text{ W/m}^2 \cdot \text{K}$

Table 4. Performance characteristics of the proposed hybrid PV-SOFC system.

Table 5. Values for the thermophysical parameters of the proposed PV-SOFC hybrid system at full loadconditions for the SOFC subsystem.

Node	<i>n</i> (kg/s)	p (Pa)	T (K)	y_{CH_4}	$y_{\rm CO}$	$y_{\rm CO_2}$	$y_{{ m H}_2}$	y_{H_2O}	y_{N_2}	y_{O_2}
1	0.0008	130,000	298	0.000	0.000	0.000	0.000	0.000	0.000	0.000
2	0.0008	138,081	303	0.000	0.000	0.000	0.000	0.000	0.000	0.000
3	0.0003	119,800	303	0.000	0.000	0.000	0.000	0.000	0.000	0.000
4	0.0005	138,081	303	0.000	0.000	0.000	0.000	0.000	0.000	0.000
5	0.0005	128,081	303	0.000	0.000	0.000	0.000	0.000	0.000	0.000
6	0.0005	126,800	723	0.000	0.000	0.000	0.000	0.000	0.000	0.000
7	0.0016	126,800	723	0.286	0.000	0.000	0.000	0.714	0.000	0.000
8	0.0016	121,800	923	0.040	0.080	0.077	0.546	0.257	0.000	0.000
9	0.0294	101,325	298	0.000	0.000	0.000	0.000	0.000	0.000	0.000
10	0.0294	101,325	298	0.000	0.000	0.000	0.000	0.000	0.000	0.000
11	0.0294	124,040	320	0.000	0.000	0.000	0.000	0.000	0.000	0.000
12	0.0294	122,800	923	0.000	0.000	0.000	0.000	0.000	0.000	0.000
13	0.0294	122,800	923	0.000	0.000	0.000	0.000	0.000	0.000	0.000
14	0.0000	119,800	923	0.000	0.000	0.000	0.000	0.000	0.000	0.000
15	0.0017	119,800	1023	0.002	0.039	0.142	0.055	0.763	0.000	0.000
16	0.0288	119,800	1023	0.000	0.000	0.000	0.000	0.000	0.805	0.195
17	0.0308	117,800	1278	0.000	0.000	0.020	0.000	0.066	0.753	0.160
18	0.0308	115,300	1192	0.000	0.000	0.020	0.000	0.066	0.753	0.160
19	0.0308	114,147	1183	0.000	0.000	0.020	0.000	0.066	0.753	0.160
20	0.0308	113,006	1121	0.000	0.000	0.020	0.000	0.066	0.753	0.160
21	0.0308	111,875	586	0.000	0.000	0.020	0.000	0.066	0.753	0.160
22	0.0308	110,757	328	0.000	0.000	0.020	0.000	0.066	0.753	0.160
23	0.0287	110,757	328	0.000	0.000	0.022	0.000	0.000	0.807	0.171
24	0.0020	110,757	328	0.000	0.000	0.000	0.000	0.000	0.000	0.000
25	0.0012	101,325	313	0.000	0.000	0.000	0.000	0.000	0.000	0.000
26	0.0012	128,081	313	0.000	0.000	0.000	0.000	0.000	0.000	0.000
27	0.0012	126,800	723	0.000	0.000	0.000	0.000	0.000	0.000	0.000
28	0.1743	110,000	313	0.000	0.000	0.000	0.000	0.000	0.000	0.000
29	0.1743	108,900	338	0.000	0.000	0.000	0.000	0.000	0.000	0.000
30	0.1743	120,000	338	0.000	0.000	0.000	0.000	0.000	0.000	0.000

4.3.1. System Cost Analysis

A cost analysis of the proposed hybrid system is given in Table 6. In terms of capital cost, the highest cost is allocated for the purchase of the SOFC subsystem at 607,540 USD, while the cost for the PV arrays is 140,132 USD. The cost of the power subsystem is also significant at 221,951 USD, which means that it constitutes about $\frac{1}{4}$ of the total system cost. The total cost of fuel for the operation of the system during its lifetime is estimated at 891,735 USD. Although natural gas prices are constantly fluctuating, it is not expected that this cost estimation will be significantly altered in the near future for the EU market, based on a statistical analysis of the prices for the last 10 years [40]. The lifecycle cost for the system is 1,241,369 USD, with a unit cost of electricity at 0.1057 USD/kWh.

	Output Parameter Description	Value
c _{fy}	Annual cost of fuel	43,995 USD/year
C_{pv}	Cost of PV arrays	140,132 USD
\dot{C}_{fc}	Cost of SOFC subsystem	607,540 USD
C_{inv}	Cost of power subsystem	221,951 USD
C_{sys}	Total cost of the system	974,623 USD
C_{down}	Down payment	194,925 USD
C _{loan}	Cost of the loan	714,977 USD
D _{loan}	Tax deduction on the loan	236,951 USD
C_{twc}	Total worth of capital	672,950 USD
D_{dep}	Depreciation of capital	242,919 USD
D _{cred}	Tax credit	19,492 USD
D_{salv}	Salvage worth	206,877 USD
C_{prop}	Tax paid on property	73,097 USD
Ċomi	Cost of operation, maintenance and insurance	72,876 USD
C_{tcf}	Total cost of fuel	891,735 USD
LĆĆ	Lifecycle cost	1,241,369 USD
c _{el}	Unit cost of electricity	0.1057 USD/kWh

Table 6. Cost analysis of the proposed hybrid PV-SOFC system.

4.3.2. Case Study: Comparison with Conventional and Other System Configurations

For a quantitative investigation of the possible merits of the proposed hybrid PV-SOFC system, four different case configurations were analyzed, in terms of thermoeconomic performance: (A) Power supply from a central power grid (conventional case), (B) power supply from a central power grid assisted with an on-site PV system, (C) power (and heat) supply from a decentralized SOFC system, and (D) power (and heat) supply from the proposed hybrid PV-SOFC system. A schematic representation of the four cases is given in Figure 4. The four cases can be compared in terms of two parameters: Unit cost of electricity and CO_2 emissions. The results from this comparison are shown in Figures 5 and 6, respectively. As observed, the proposed system outperforms all other configurations, in terms of both the unit cost of electricity and CO_2 emissions. In particular, in comparison to case A, the unit cost of electricity is about 50% lower (0.2128 vs. 0.1057 USD/kWh), while the reduction in CO_2 emissions is about 36% (673 vs. 428 g(CO_2)/kWh).

In comparison to cases B and C, the additional capital cost for purchasing the SOFC subsystem and the PV subsystem, respectively, is well reasoned by the reduction in fuel consumption, hence on the unit cost of electricity (0.1700 USD/kWh (case B) and 0.1265 USD/kWh (case C)). Similarly, in terms of CO₂ emissions, the proposed system manages to significantly reduce emissions. For case B, power generation remains heavily dependent on inefficient central power grid supply, and therefore CO₂ emission generation remains high. For case C, CO₂ emissions are even higher than case B, because power (and heat) generation is completely dependent on the SOFC system. On an annual basis, the fuel consumption is 154,530 kg of natural gas for case C, compared to a reduced consumption of 115,848 kg for the proposed system in case D. In terms of lifecycle cost, for case C this is 1,468,209 USD, i.e., 226,840 USD higher than the equivalent cost for the proposed system in case D.



Figure 4. Schematic representation of the four considered cases: (**A**) Power supply from a central power grid (conventional case), (**B**) power supply from a central power grid assisted with an on-site PV system, (**C**) power (and heat) supply from a decentralized SOFC system, and (**D**) power (and heat) supply from the proposed hybrid PV-SOFC system.



Figure 5. Comparison of the four cases under study in terms of the unit cost of electricity (in USD/kWh).



Figure 6. Comparison of the four cases under study in terms of CO_2 emissions (in g (CO_2)/kWh).

5. Conclusions

In this study a decentralized, hybrid PV-SOFC system is proposed for the fulfillment of a load profile for a commercial building (small hotel) in Cyprus. The system components are modeled in detail to allow a realistic simulation of the operation of the system. An actual load profile and solar/weather data are fed to the system model to determine the thermoeconomic characteristics of the proposed system. The system is sized based on the requirements of the load profile, with maximum power outputs for the PV and SOFC subsystems at 70 and 152 kWe, respectively. The system operates efficiently throughout the whole year for a transient load profile. The average net electrical and total efficiencies for the SOFC subsystem are 0.303 and 0.700, respectively. Maximum net electrical and total efficiencies reach up to 0.375 and 0.756, respectively. The total contribution of the two subsystems on a yearly basis for the fulfillment of the load profile is at 135.9 and 451.2 MWh for the PV and the SOFC subsystems, respectively. Application of the proposed hybrid system is favored over conventional power generation with electricity-only central power stations for technical and economic reasons. The proposed system can operate more efficiently in terms of net electrical efficiency (especially at part-load operation over a heat engine-based power generator), and, more importantly, it can take advantage of the heat recovery capability of the SOFC subsystem. Additionally, fuel consumption is reduced significantly, primarily because of the integration of the PV subsystem, and also due to the elimination of transmission and distribution losses.

The cost analysis of the proposed system shows that in terms of capital cost, the highest cost is for the purchase of the SOFC subsystem (607,540 USD), while the cost for the PV arrays is 140,132 USD. The cost of the power subsystem, which is usually underestimated, is also significant at 221,951 USD. The total cost of fuel for the operation of the system during its lifetime is estimated at 891,735 USD. Although natural gas prices are constantly fluctuating, it is not expected that this cost estimation will be significantly altered in the near future for the EU market, based on a statistical review of the prices for the last 10 years [33]. The lifecycle cost for the system is 1,241,369 USD, with a unit cost of electricity at 0.1057 USD/kWh. The proposed system outperforms conventional and other system configurations, in terms of both the unit cost of electricity and CO₂ emissions. In comparison to the conventional case,

the unit cost of electricity is about 50% lower (0.2128 vs. 0.1057 USD/kWh), while the reduction in CO_2 emissions is about 36% (673 vs. 428 g(CO_2)/kWh). The additional capital cost for purchasing the PV and the SOFC subsystems is well reasoned by the reduction in fuel consumption, hence on the unit cost of electricity. Similarly, in terms of CO_2 emissions, the proposed system manages to significantly reduce emissions, because power generation is independent of the inefficient central power grid supply. Additionally, the integration of the PV subsystem allows a significant reduction in power generation from the SOFC subsystem during solar energy availability.

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Nomenclature and Units

Α	Activation area (m ²)
A_i	Anisotropy index (-)
С	Concentration of species (kmol m^{-3})
Eocv	Open circuit voltage (V)
Ė	Energy rate (W)
f	Solar fraction (-)
F	Faraday's constant (Coulomb mol^{-1})
8	Specific Gibbs free energy (J kmol ⁻¹)
h_m	Average diffusivity (m s $^{-1}$)
i	Current density (A m^{-2})
Ι	Hourly irradiation (MJ m^{-2}), Current (A)
Ib	Beam radiation (MJ m^{-2})
I_d	Diffuse radiation (MJ m^{-2})
I_T	Total incident solar radiation (MJ m^{-2})
Κ	Reaction equilibrium constant (-)
'n	Molar flow rate (kmol s^{-1})
n _{cells}	Number of cells (-)
n _e	Number of electrons transferred per H ₂ molecule reacted (-)
р	Pressure (bar, Pa)
Р	Energy (kWh, MWh)
P	Power (W)
Ż	Heat rate (W)
R	Universal gas constant (J mol $^{-1}$ K $^{-1}$)
R_b	Ratio of beam radiation (-)
R_i	Ohmic resistance (Ωm^2)
SC	Steam-to-carbon ratio (-)
Т	Temperature (°C, K)
TER	Thermal-to-electric ratio (-)
U_f	Fuel utilization factor (-)
V	Voltage (V)
X	Conversion molar flow rate (kmol s^{-1})
у	Mole fraction (-)
Greek syn	nbols
α	Charge transfer coefficient (-)
β	PV tilt angle (degrees)
γ	Activity coefficient (A m ⁻²)
Δg_f^0	Gibbs free energy (J kmol $^{-1}$)
ΔĠ	Overall change in Gibbs free energy (J kmol $^{-1}$)

Greek syr	nbols
η	Efficiency (-)
μ_{mp}	Maximum power point efficiency temperature coefficient (-)
ρ_g	Ground reflectance (-)
Subscript	s/Superscripts
0	Theoretical (ideal) value
ab	Air blower
act	Activation
amb	Ambient conditions
ano	Fuel cell anode
array	Array
С	PV array
cat	Fuel cell cathode
cell	Cell
сотр	Fuel compressor
conc	Concentration
cool	Space cooling
dhw	Domestic hot water
el	Electrical
едиір	Interior equipment
exc	Excess
fan	Fans
fc	Fuel cell
fcs	Fuel cell subsystem
fuel	Fuel
heat	Heat
HHV	Higher heating value
in	Inlet flow
inv	Inverter
L	Limiting
LHV	Lower heating value
light	Interior lights
load	Load
loss	Loss
mn	Maximum point
net	Net value
NOCT	Nominal operating cell temperature
ohm	Ohmic
out	Exit flow
numn	Water pump
p ump 1171	Photovoltaic
p 0 11715	Photovoltaic subsystem
ref	Reformer
refer	Reference state
smr	Steam methane reformer
sofc	Solid ovide fuel cell
th	Recovered heat from fuel cell
71105	Water gas shift
Abbrevia	tions
CHP	Combined-beat-and-power
FFS	Engineering Equation Solver
FU	Furonean Union
нокъ	Hav-Davies-Klucher-Poindl
TIDKK	ray-Davies-Niucher-Nemui

Abbreviations

HEx	Heat exchanger
HHV	Higher heating value
LHV	Lower heating value
LMTD	Logarithmic Mean Temperature Difference
PEMFC	Proton exchange membrane fuel cell
PV	Photovoltaic
RES	Renewable energy sources
SMR	Steam methane reformer
SOFC	Solid oxide fuel cell
WGS	Water gas shift

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