



An Improved Capacitive Sensor for Detecting the Micro-Clearance of Spherical Joints

Wen Wang ¹, Wenjun Qiu ¹, He Yang ^{1,*}, Haimei Wu ¹, Guang Shi ¹, Zhanfeng Chen ¹, Keqing Lu ¹, Kui Xiang ² and Bingfeng Ju ^{2,3}

- ¹ School of Mechanical Engineering, Hangzhou Dianzi University, Hangzhou 310018, China; wangwn@hdu.edu.cn (W.W.); qwjhdu@163.com (W.Q.); hellen0810@foxmail.com (H.W.); shiguang@hdu.edu.cn (G.S.); czf@hdu.edu.cn (Z.C.); lkq@hdu.edu.cn (K.L.)
- ² School of Mechanical Engineering, Zhejiang University, Hangzhou 310027, China;
- snowboy2001102@163.com (K.X.); mbfju@zju.edu.cn (B.J.)
- ³ State Key Lab of Fluid Power & Mechatronic Systems, Zhejiang University, Hangzhou 310027, China
- * Correspondence: yanghe@hdu.edu.cn; Tel.: +86-571-8691-9155

Received: 8 May 2019; Accepted: 14 June 2019; Published: 14 June 2019



Abstract: Due to the flexible and compact structures, spherical joints are widely used in parallel manipulators and industrial robots. Real-time detection of the clearance between the ball and the socket in spherical joints is beneficial to compensate motion errors of mechanical systems and improve their transmission accuracy. This work proposes an improved capacitive sensor for detecting the micro-clearance of spherical joints. First, the structure of the capacitive sensor is proposed. Then, the mathematical model for the differential capacitance of the sensor and the eccentric micro-displacement of the ball is deduced. Finally, the capacitance values of the capacitive sensor are simulated with Ansoft Maxwell. The simulated values of the differential capacitances at different eccentric displacements agree well with the theoretical ones, indicating the feasibility of the proposed detection method. In addition, the simulated results show that the proposed capacitive sensor could effectively reduce the capacitive fringe effect, improving the measurement accuracy.

Keywords: spherical joint; clearance measurement; capacitive sensor; eccentric displacement

1. Introduction

Due to its compact structure and flexible motion, precision spherical joints have been widely used in parallel manipulators and industrial robots [1]. In the traditional applications, spherical joints are usually assumed to be ideal and the clearance between the ball and the socket in spherical joints is ignored to simplify the dynamic model of the multibody mechanical systems. However, the existence of joint clearance is inevitable, due to the errors caused by fabrication and assembly of the mechanical components. It not only increases the vibration, noise and wear rate, but also brings undesired effects on the motion accuracy and dynamic behavior of the mechanism [2–10]. In addition, high-precision detection of joint clearance is a prerequisite for analyzing and compensating the motion error caused by the joint clearance. Therefore, real-time detection of the micro-clearance in spherical joints is essential to reveal and improve the real dynamic performance of the mechanical systems.

Many approaches have been proposed for the clearance or displacement measurement, e.g., magnetic sensor [11], inductive sensor [12], capacitive sensor [13], etc. In the case of the magnetic sensor, the displacement is obtained by measuring the variations of the magnetic flux [14]. However, the drawback of the magnetic sensor includes nonlinearity, remanence and temperature dependence [15]. Another method is to deploy the inductive sensors, which have a dual-coil structure, i.e., transmitter and receiver coils. The target displacement is calculated by measuring the voltage variation of the



receiver coil induced by the eddy current from the transmitter coil [16–19]. Compared with the inductive sensor, the capacitive sensor has the advantages of good performance and high accuracy. Ahn [20] proposed a cylindrical capacitive sensor to measure the radial motion of rotating machinery. The radial motion is obtained by detecting the total radial capacitance. Hu et al. [21] proposed a capacitive sensor to capture the clearance of a spherical joint. The capacitive sensor includes a spherical plate and six small arc plates. A point capacitance model is deduced to calculate the clearance from the measured capacitance values and validated by the simulation using Maxwell software. However, there is an average deviation of 10%–13% between the theoretical and simulated capacitance values. Wang et al. [22] proposed a spherical differential capacitive sensor to detect the clearance of a precision spherical joint. The spherical differential capacitive sensor includes eight spherical plates and a ball. The clearance of spherical joints is obtained by measuring the differential capacitance of eight capacitors. However, the capacitive fringe effect, caused by the divergence of the electric field line at the corners of the capacitive plates, could produce additional capacitance and thus lead to the measurement errors [23,24].

To reduce the capacitive fringe effect, this work further proposes an improved capacitive sensor to measure the micro-clearance of a spherical joint. First, the structural design and working principle is presented in Section 2. Then, the mathematical model of the proposed detection method is deduced in Section 3. Finally, the capacitive fringe effect and the sensor performance are simulated using Ansoft Maxwell 16.0 in Section 4 and the theoretical and simulated results are further discussed in Section 5.

2. Detecting Method

2.1. Structural Design

The proposed capacitive sensor consists of a ball and six spherical plates (Figure 1). The former is the excitation plate (CP_e) while the latter are sensing plates (CP_{si} , i = 1, 2, ..., 6). The six sensing plates have the same thickness and area. They are concentrically distributed around the ball. As a result, the sphere center of six spherical plates coincides with the rotational center of the ball. As shown in Figure 1b, the coordinate system OXYZ is defined in the sensor with the origin O at the rotational center of the ball, the X-axis is chosen along the centerline of plate CP_{s1} and the Y-axis is defined along the centerline of the plate CP_{s2} . In this work, the proposed sensing plates have a structure of the spherical cap (Figure 2). The central angle subtended from the apex of the cap to the edge of the cap for the plates $CP_{s1} \sim CP_{s4}$ is θ_0 , while that for the plates CP_{s5} and CP_{s6} is θ_1 and θ_2 , due to the motion requirement of the output rod of spherical joints. To achieve the insulation between the sensing plates and the excitation plate, a dielectric material with lubricating and abrasion-resistant properties is deposited on the inner surface of the sensing plates.



Figure 1. Structural model of the proposed capacitive sensor (**a**) structural assembly (**b**) distribution of the capacitive plates.



Figure 2. Sketch of the sensing plates (a) $CP_{s1} \sim CP_{s4}$ (b) CP_{s5} and CP_{s6} .

2.2. Measuring Principle

In the proposed capacitive sensor, the ball of the spherical joint is employed as a common excitation plate. Each sensing plate (CP_{si} , i = 1, 2, ..., 6) and the common excitation plate (CP_e) produce a capacitor (C_i , i = 1, 2, ..., 6). Ideally, the clearance between the ball and six sensing plates is identical, and thus the capacitance value of the capacitor is equal to each other. Once the ball has an eccentric displacement, the center of the ball deviates from the sphere center of six sensing plates. As a result, the clearance between the ball and six sensing plates undergoes different variations, and the capacitance value of each capacitor alters correspondingly. Thus, the eccentric displacement of the ball in the socket could be obtained by detecting the variation of the capacitance values of six capacitors.

To detect the eccentric displacement of the ball, three capacitor pairs (C_x , C_y and C_z) are established for the measurement of the displacement components along three orthogonal directions (*X*-axis, *Y*-axis and *Z*-axis), respectively. The eccentric displacement (δ_x) of the ball along the *X*-axis can be calculated from the differential capacitance (ΔC_x) of the capacitor pair (C_x) along the *X*-axis, which consists of the capacitor C_1 and the capacitor C_3 . Similarly, the eccentric displacement (δ_y) of the ball along the *Y*-axis, which includes the capacitor C_2 and the capacitor C_4 . In addition, the eccentric displacement (δ_z) of the ball along the *Z*-axis can be calculated from the differential capacitance (ΔC_z) of the capacitor pair (C_z) along the *Z*-axis, which comprises of the capacitor C_5 and the capacitor C_6 . Thus, the eccentric displacement components (δ_x , δ_y and δ_z) of the ball can be expressed by the following equations:

$$\delta_x = f_x \Delta C_x = f_x (C_1 - C_3) \tag{1}$$

$$\delta_y = f_y \Delta C_y = f_y (C_2 - C_4) \tag{2}$$

$$\delta_z = f_z \Delta C_z = f_z (C_5 - C_6) \tag{3}$$

where f_x , f_y and f_z are the mathematical functions for the eccentric displacement component and the corresponding differential capacitance along the X-axis, Y-axis and Z-axis, respectively.

3. Mathematical Model

To simplify the calculation of the capacitance value of each capacitor, two assumptions are made as follows: (1) the area element dA of each spherical-plate capacitor is assumed to be the parallel-plate capacitor with a uniform gap; (2) the fringe effect of the capacitive plate is neglected. Thus, the capacitance value of each capacitor can be given by the following equation:

$$C_i = \varepsilon \iint_{A_i} \frac{1}{d} dA \ (i = 1, 2, \dots, 6)$$
(4)

where is ε is the permittivity of the dielectric material, *d* is the clearance between the sensing plate and the ball, A_i is the area of the sensing plate CP_{si} , i = 1, 2, ..., 6.

3.1. Clearance between the Sensing Platees and the Ball

Figure 3 presents the schematic model for calculating the eccentric displacement of the ball. The surfaces of the sensing plates and the ball are assumed to be ideal spheres. The sphere center of sensing plates coincides with the origin *O* of the coordinate system *OXYZ*, and the center of the ball is denoted by the point *O'*. δ is the eccentric displacement from the center *O'* of the ball to the sphere center *O* of sensing plates. The point *M* locates on the surface of the ball. θ is the angle between the line *OM* and the *Z*-axis. φ is the angle between the line *OM* and the *X*-axis. *N* is the intersection point of the extension of the line *O'M* and the inner surface of sensing plates. *R* is the inner radius of the sensing plates and *r* is the radius of the ball. Thus, the clearance *d* between the ball and the sensing plates at the point *M* is denoted by the distance from point *M* to point *N*. It can be expressed by the following equation:

$$d = R - \delta_x \sin \theta \cos \varphi - \delta_y \sin \theta \sin \varphi - \delta_z \cos \theta - r \tag{5}$$



Figure 3. Sketch of the clearance between the ball and the sensing plates.

3.2. Dependence of Differential Capacitance on Eccentric Displacement

At the initial position, the ball has no eccentric displacements relative to the spherical sensing plates, the null clearance between the ball and the sensing plate is denoted by $d_0 = R - r$. To facilitate the mathematical deduction, four dimensionless parameters are set: $h = d/d_0$, $\lambda_x = \delta_x/d_0$, $\lambda_y = \delta_y/d_0$, $\lambda_z = \delta_z/d_0$. Then, Equation (5) can be rewritten into dimensionless form:

$$h = 1 - \lambda_x \sin\theta \cos\varphi - \lambda_y \sin\theta \sin\varphi - \lambda_z \cos\theta \tag{6}$$

Let $\lambda = \lambda_x \sin \theta \cos \varphi + \lambda_y \sin \theta \sin \varphi + \lambda_z \cos \theta$, we have $\lambda < 1$ since the eccentric displacement of the ball is smaller than the null clearance. Thus, 1/h can be expanded into Maclaurin series as follows:

$$\frac{1}{h} = \frac{1}{1-\lambda} = 1 + \lambda + \lambda^2 + \lambda^3 + \cdots$$
(7)

Substituting Equations (6) and (7) into Equation (4), the capacitance value of the capacitor can be expressed by:

$$C_i = \frac{\varepsilon}{d_0} \iint_{A_i} G(\theta, \varphi) dA \ (i = 1, 2, \dots, 6)$$
(8)

where $G(\theta, \varphi) = \sum_{n=0}^{\infty} (\lambda_x \sin \theta \cos \varphi + \lambda_y \sin \theta \sin \varphi + \lambda_z \cos \theta)^n$, $dA = R^2 \sin \theta d\theta d\varphi$.

The eccentric displacement of the ball along Z-axis can be obtained by calculating the capacitance difference ΔC_z , which can be obtained from capacitance value of capacitors CP_{s5} and CP_{s6} . Note that the central angles subtended from the apex of the cap to two edges of the cap for CP_{s5} and CP_{s6} are θ_1

and θ_2 . To calculate the integral in Equation (8), the integral interval of the variables θ and φ can be expressed as follows: $0 \le \varphi \le 2\pi$ and $\theta_1 \le \theta \le \theta_2$ for CP_{S5} , $0 \le \varphi \le 2\pi$ and $\pi - \theta_2 \le \theta \le \pi - \theta_1$ for CP_{s6} . Thus, the capacitance value of capacitors C_5 and C_6 can be given by:

$$C_5 = \frac{\varepsilon}{d_0} \int_{\theta_1}^{\theta_2} \int_0^{2\pi} G(\theta, \varphi) R^2 \sin \theta d\theta d\varphi$$
(9)

$$C_6 = \frac{\varepsilon}{d_0} \int_{\pi-\theta_2}^{\pi-\theta_1} \int_0^{2\pi} G(\theta, \varphi) R^2 \sin\theta d\theta d\varphi$$
(10)

Since the rotational angle of spherical joints in most practical applications is in the range of $-15^{\circ}-15^{\circ}$ [25], let $\theta_1 = 35^{\circ}$ and $\theta_2 = 45^{\circ}$. Then, Equations (9) and (10) can be rewritten by neglecting the terms of higher order beyond the fifth order:

$$C_5 = \frac{\varepsilon R^2}{d_0} F_5(\lambda_x, \lambda_y, \lambda_z)$$
(11)

$$C_6 = \frac{\varepsilon R^2}{d_0} F_6(\lambda_x, \lambda_y, \lambda_z)$$
(12)

where

$$F_{5}(\lambda_{x},\lambda_{y},\lambda_{z}) = 0.8390 + 0.5372\lambda_{z} + \dots \\ 0.3453\lambda_{z}^{2} + 0.2469(\lambda_{x}^{2} + \lambda_{y}^{2}) + \dots \\ 0.2227\lambda_{z}^{3} + 0.4718\lambda_{z}(\lambda_{x}^{2} + \lambda_{y}^{2}) + \dots \\ 0.1441\lambda_{z}^{4} + 0.1097(\lambda_{x}^{2} + \lambda_{y}^{2})^{2} + 0.6034\lambda_{z}^{2}(\lambda_{x}^{2} + \lambda_{y}^{2}) + \dots \\ 0.0936\lambda_{z}^{5} + 0.3478\lambda_{z}(\lambda_{x}^{2} + \lambda_{y}^{2})^{2} + 0.6454\lambda_{z}^{3}(\lambda_{x}^{2} + \lambda_{y}^{2})$$
(13)

$$F_{6}(\lambda_{x},\lambda_{y},\lambda_{z}) = 0.8390 - 0.5372\lambda_{z} + \dots \\ 0.3453\lambda_{z}^{2} + 0.2469(\lambda_{x}^{2} + \lambda_{y}^{2}) - \dots \\ 0.2229\lambda_{z}^{3} - 0.4718\lambda_{z}(\lambda_{x}^{2} + \lambda_{y}^{2}) + \dots \\ 0.1441\lambda_{z}^{4} + 0.1097(\lambda_{x}^{2} + \lambda_{y}^{2})^{2} + 0.6034\lambda_{z}^{2}(\lambda_{x}^{2} + \lambda_{y}^{2}) - \dots \\ 0.0936\lambda_{z}^{5} - 0.3478\lambda_{z}(\lambda_{x}^{2} + \lambda_{y}^{2})^{2} - 0.6454\lambda_{z}^{3}(\lambda_{x}^{2} + \lambda_{y}^{2})$$
(14)

Using Equations (11)–(14), we obtain the differential capacitance ΔC_z along the Z-axis,

$$\Delta C_z = C_5 - C_6 = \frac{\varepsilon R^2}{d_0} \Delta F_z \tag{15}$$

where

Δ

$$F_{z} = 1.0744\lambda_{z} + 0.4454\lambda_{z}^{3} + 0.5426\lambda_{z}(\lambda_{x}^{2} + \lambda_{y}^{2}) + \dots$$

$$0.1872\lambda_{z}^{5} + 0.6956\lambda_{z}(\lambda_{x}^{2} + \lambda_{y}^{2})^{2} + 1.2908\lambda_{z}^{3}(\lambda_{x}^{2} + \lambda_{y}^{2})$$
(16)

Similarly, the differential capacitance ΔC_x and ΔC_y along the *X* and *Y* axes can be obtained. To avoid the structural interference between six sensing plates, the central angle θ for $CP_{s1} \sim CP_{s4}$ ranges from 0° to 30°. Then, the differential capacitance ΔC_x and ΔC_y along the *X* and *Y* axes can be given by,

$$\Delta C_x = C_1 - C_3 = \frac{\varepsilon R^2}{d_0} \Delta F_x \tag{17}$$

$$\Delta C_y = C_2 - C_4 = \frac{\varepsilon R^2}{d_0} \Delta F_y \tag{18}$$

Δ

where

$$F_{x} = 1.5708\lambda_{x} + 1.3745\lambda_{x}^{3} + 0.2945\lambda_{x}(\lambda_{y}^{2} + \lambda_{z}^{2}) + \dots$$

$$1.2108\lambda_{x}^{5} + 0.06136\lambda_{x}(\lambda_{y}^{2} + \lambda_{z}^{2})^{2} + 0.8181\lambda_{x}^{3}(\lambda_{y}^{2} + \lambda_{z}^{2})$$
(19)

$$\Delta F_y = 1.5708\lambda_y + 1.3745\lambda_y^3 + 0.2945\lambda_y (\lambda_x^2 + \lambda_z^2) + \dots 1.2108\lambda_y^5 + 0.06136\lambda_y (\lambda_x^2 + \lambda_z^2)^2 + 0.8181\lambda_y^3 (\lambda_x^2 + \lambda_z^2)$$
(20)

It can be observed from Equations (15)–(20) that the differential structure could eliminate the even-order terms of the eccentric displacement of the ball. This could improve the sensitivity of the capacitive sensor and reduce the nonlinear error.

According to Equations (15)–(20), the high-order terms of the differential capacitance have the coupling of the displacements along three orthogonal directions (i.e., *X*-axis, *Y*-axis and *Z*-axis) if the ball has an eccentric displacement along three orthogonal directions. This may lead to the nonlinear error of the calculated capacitance. To illustrate the effect of high-order terms, the maximum nonlinear errors of differential capacitances are examined. E_x , E_y and E_z denote the maximum nonlinear errors caused by the high-order terms for ΔC_x , ΔC_y and ΔC_z , respectively. They can be rewritten as follows:

$$E_{x} = E_{y} = max(\frac{\Delta F_{x} - 1.5708\lambda_{x}}{1.5708\lambda_{x}}) = max(\frac{\Delta F_{y} - 1.5708\lambda_{y}}{1.5708\lambda_{y}})$$

$$E_{z} = max(\frac{\Delta F_{z} - 1.0744\lambda_{z}}{1.0744\lambda_{x}})$$
(21)

Figure 4 presents the dependence of the maximum nonlinear error on the eccentricity of the ball. The eccentricity of the ball ρ is defined by $\rho = \sqrt{\lambda_x^2 + \lambda_y^2 + \lambda_z^2}$. As shown in Figure 4, the maximum nonlinear errors caused by the high-order terms exhibit a quasi-exponential increase with the rising eccentricity of the ball. The maximum nonlinear errors are of 0.88% and 3.62% at the eccentricity $\rho = 0.1$ and $\rho = 0.2$, respectively. Thus, the effect of the high-order terms on the differential capacitance can be neglected, provided that the eccentricity of the ball is small. As such, the dependence of the differential capacitances (ΔC_x , ΔC_y and ΔC_z) on the eccentric displacements (δ_x , δ_y and δ_z) can be described by linear functions,

$$\delta_x = \frac{d_0^2}{1.5708\epsilon R^2} \Delta C_x$$

$$\delta_y = \frac{d_0^2}{1.5708\epsilon R^2} \Delta C_y$$

$$\delta_z = \frac{d_0^2}{1.0744\epsilon R^2} \Delta C_z$$
(22)



Figure 4. Dependence of the maximum nonlinear errors on the eccentricity of the ball.

3.3. The Capacitive Fringe Effect Analysis

The capacitive plates of sensors have a limited size in practical applications, the electric field lines diverge at the edges of the capacitive plates and the charge density is higher at the edges or tips, resulting in non-uniform distribution of electric charge. This phenomenon is referred as the capacitive fringe effect. It may increase the capacitance value of a capacitor and thus have a great impact on the detecting accuracy of capacitive sensors. Thus, the capacitance value of a capacitor can be expressed as follows,

$$C = C_T + C_E \tag{23}$$

where *C* is the actual capacitance value, C_T is the theoretical capacitance value obtained from Equation (4), C_E is the additional capacitance value caused by capacitive fringe effect. Then, the capacitance error η caused by the fringe effect is defined as follows,

$$\eta = \frac{C - C_T}{C_T} \tag{24}$$

4. Simulation Setup

In this work, the capacitance values of the capacitive sensor are simulated using ANSOFT Maxwell. Two simulations were carried out. The effect of the plate structure on the capacitive fringe effect was examined first. Then, the relation between eccentric displacement of the ball and differential capacitance of the sensor was investigated in detail.

To explore the effect of the plate structure on the capacitive fringe effect, two spherical capacitive plates are examined. The spherical-cap capacitive plate proposed in this work is shown in Figure 5a while the spherically-trapezoid capacitive plate proposed in [22] is presented in Figure 5b. The central angle of the spherical-cap capacitive plate is of 30°. The spherical radius of the inner surface for both capacitive plates is of 25 mm and the effective area of two plates is equal to each other. The material of the capacitive plates is copper and the dielectric material is air. The size of simulation domain is four times that of the model to fully reflect the effect of the fringe effect on the capacitance.



Figure 5. Simulation model of spherical capacitive plate (**a**) SCE: spherical-cap plate (**b**) STE: spherically-trapezoid plate.

To explore the relation between the differential capacitance and the eccentric displacement, a simulation model of the proposed capacitive sensor is built using ANSOFT Maxwell (Figure 6). The spherical radius and thickness of the capacitive plates is of 25 mm and 0.5 mm, respectively. The radius of the ball is of 24.8 mm. The null clearance between the ball and the sensing plates is of 0.2 mm. The central angle subtended by the spherical cap for $CE_{s1} \sim CE_{s4}$ ranges from 0° to 30° while that for CE_{s5} and CE_{s6} is in the range of 35°–45°. Copper is adopted as the material of the excitation and

sensing plates, and air is applied as the dielectric material. In the simulation, the eccentric displacement of the ball varies from -40 to $40 \mu m$, with a step of 5 μm .



Figure 6. Simulation model of the proposed capacitive sensor.

5. Experimental Setup

In this work, an experimental investigation is carried out to validate the feasibility of the proposed method. The experimental setup mainly consists of one-dimensional (1D) precision positioning stage, two-dimensional (2D) positioning stage, 1D precision lifting stage, plate assembly, fixed platform and high-precision LCR meter (Figure 7a). The plate assembly includes six plates, a ball and two plate supports (Figure 7b). The 1D precision positioning stages are attached to plates, while the 2D positioning stage and 1D precision lifting stage are connected to the ball. The material of the plate and the plate supports is copper and aluminum, respectively. The surfaces of the plate supports are hard oxidized to ensure the insulation between the plates and the plate supports.



Figure 7. Experimental setup (a) Photograph of the test rig, (b) photograph of plate assembly.

The concentric arrangement of six capacitive plates is a prerequisite for the detection of the eccentric displacement. In order to make six plates mounted concentrically, a standard steel ball is used as a reference. It has a radius of 25 mm, which equals to the inner radius of the plates. Initially, the standard ball is placed on the plate CP_{S6} . Then, the plates $CP_{S1} \sim CP_{S4}$ are attached to the standard steel ball by adjusting four one-dimensional (1D) precision positioning stages. Finally, the plate CP_{S5} is adjusted to fit the steel ball. A multimeter with its working mode set to the ohmic file is used to detect the contact between the plate and the standard ball. Once six plates are attached to the standard ball, they are concentric about the center of the ball and the position of each plate is recorded.

Another steel ball with a radius of 24.8 mm is used to achieve eccentric displacement. The relative position between the ball and capacitive plates is adjusted by a 2D precision positioning stage and a 1D precision lifting stage. Initially, the ball is positioned at the concentric center of six plates. Then, the ball moves along the direction of X = Y = Z direction, with a step of 10 µm. A high-precision LCR meter (GWINSTEK LCR-8101G) is used to detect the capacitance between the sensing plates and the ball.

6. Results and Discussion

6.1. Effect of Plate Structure on the Capacitive Fringe Effect

The plate structure has a vital impact on the capacitive fringe effect. In this section, two spherical plates are examined, one is the spherical-cap plate, the other is the spherically-trapezoid plate. The capacitance errors caused by the capacitive fringe effect are investigated in detail at different null clearance and plate thickness.

Figure 8 presents the dependence of the capacitance errors on the null clearance between the ball and the sensing plates. The clearance varies from 0.2 to 2 mm, with a step of 0.2 mm. The thickness of the sensing plates is of 2 mm. Two observations can be made. First, for both plates, the capacitance errors caused by the capacitive fringe effect exhibit a remarkable increase with the rising null clearance. In other words, the capacitive fringe effect could be reduced by reducing the null clearance. Second, at the same null clearance, the capacitance error for spherical-cap plate is smaller than that for spherically-trapezoid plate. This indicates that the spherical-cap plate proposed in this work could reduce the capacitive fringe effect, compared with the spherically-trapezoid plate mentioned in [22].



Figure 8. Effect of null clearance on the capacitance errors caused by capacitive fringe effect.

Figure 9 shows the dependence of the capacitance errors on the plate thickness. The plate thickness varies from 0.5 to 5 mm, with a step of 0.5 mm. The null clearance is fixed at 1 mm. In case of spherical-cap plate, the capacitance error caused by the capacitive fringe effect goes up from 25% to 34% as the plate thickness increases from 0.5 mm to 5 mm. This suggests that the capacitive fringe effect can be reduced by using thin plates. In case of spherically-trapezoid plate, the capacitance error exhibits a larger upward tendency, rising from 33% to 52%, with the increase of the plate thickness. In addition, the capacitance error for spherically-trapezoid plate is larger than that for spherical-cap plate. These indicate that the capacitive fringe effect for spherically-trapezoid plate is more serious and more sensitive to plate thickness. In other words, the spherical-cap plate proposed in this work is beneficial to the reduction of the capacitive fringe effect.





Figure 9. Effect of plate thickness on the capacitance errors caused by capacitive fringe effect.

Figure 10 presents the magnitude distribution of the electric field strength along the edge of the capacitive plates. The theoretical and simulated values are further fitted with a ninth-order polynomial for a better comparison. γ is the dimensionless distance normalized by the perimeter of the sensing plate. The clearance between the ball and the sensing plate is of 1.2 mm and the plate thickness is of 2 mm. In case of the spherically-trapezoid plate, the electrons concentrate at the tip of the capacitive plate, resulting an enhanced divergence of the electric field line at the tip. Thus, the electric field strength at the tips A, B, C and D is much larger than that in other places and the capacitive fringe effect becomes more serious. In case of the spherical-cap plate, the electric field strength is relatively uniform along the edge of the capacitive plate. Thus, the proposed capacitive sensor in this work could reduce the capacitive fringe effect and improve the measurement accuracy.



Figure 10. Magnitude distribution of electric field strength along the edge of the capacitive plates.

6.2. Characteristics of the Capacitive Sensor

To explore the characteristics of the proposed capacitive sensor, the differential capacitances at different eccentric displacements are investigated in detail. Typical eccentric displacements are examined, that is, displacement along the coordinate axis ($\delta_y = \delta_z = 0$, $\delta_x = \delta_y = 0$), displacement in the coordinate plane ($\delta_x = \delta_z$, $\delta_x = \delta_y$) and displacement in the three-dimensional space ($\delta_x = \delta_y = \delta_z$).

Figures 11 and 12 present the dependence of the differential capacitances ΔC_x , ΔC_y and ΔC_z on the eccentric displacements along the *X* and *Z* axes, respectively. The eccentric displacement δ_x or δ_z varies from -40 to 40 µm, with a step of 5 µm. The maximal eccentricity of the ball is 0.2. Three observations can be made. First, the simulated values of differential capacitances agree well with

their theoretical counterparts, exhibiting a similar variation tendency. This indicates the feasibility and effectivity of the proposed method. Second, if the eccentric displacement is along the X-axis, the variation of ΔC_y and ΔC_z exhibits a different trend with that of ΔC_x . As δ_x rises from -40 to 40 µm, ΔC_x goes up linearly from -9 to 9 pF while ΔC_y and ΔC_z remain to zero. This indicates the decoupled effect of the displacement along the three directions, which is consistent with Equations (1)–(3). Third, if the eccentric displacement is along the Z-axis, ΔC_x and ΔC_y remain unchanged, while ΔC_z exhibits a linear relationship with δ_z .



Figure 11. Dependence of the differential capacitance on the eccentric displacement along the *X*-axis (a) ΔC_x , (b) ΔC_y and (c) ΔC_z .



Figure 12. Dependence of the differential capacitance on the eccentric displacement along the *Z*-axis (a) ΔC_x , (b) ΔC_y and (c) ΔC_z .

Figures 13 and 14 show the dependence of the differential capacitances ΔC_x , ΔC_y and ΔC_z on the eccentric displacement along the X = Z and the X = Y directions, respectively. The eccentric displacements $\delta_x = \delta_z$ or $\delta_x = \delta_y$ vary from -40 to 40 µm, with a step of 5µm. The eccentricity of the ball is from 0 to 0.283. It can be seen that the simulated values of the differential capacitance exhibit a

good agreement with the theoretical ones, except a small deviation over $\pm(30-40) \mu m$. In the case of the eccentric displacement along the X = Z direction, ΔC_x and ΔC_z rise linearly with the increase of the eccentric displacement, while ΔC_y remains unchanged over the range of $-40 \mu m \le \delta_x = \delta_z \le 40 \mu m$. In the case of the eccentric displacement along the X = Y direction, ΔC_x and ΔC_y exhibit a linear rising trend with the eccentric displacement.



Figure 13. Dependence of the differential capacitance on the eccentric displacement along the X = Z direction (**a**) ΔC_x , (**b**) ΔC_y and (**c**) ΔC_z .



Figure 14. Dependence of the differential capacitance on the eccentric displacement along the X = Y direction (**a**) ΔC_x , (**b**) ΔC_y and (**c**) ΔC_z .

Figure 15 presents the dependence of the differential capacitances ΔC_x , ΔC_y and ΔC_z on the eccentric displacement along the X = Y = Z direction. The eccentric displacements $\delta_x = \delta_y = \delta_z$ vary from -40 to 40 µm. The eccentricity of the ball is less than 0.346. Several observations can be made. Firstly, the variation of the simulated values of the differential capacitances exhibit a similar trend with that of the theoretical ones. Secondly, as the eccentric displacement rises from -40 to 40 µm, ΔC_x ,

 ΔC_y and ΔC_z increase linearly. Note that the increasing magnitude of ΔC_z is less than that of ΔC_x and ΔC_y , suggesting that the sensitivity of capacitor pair C_z is less than that of C_x and C_y . Thirdly, the experimental values of the differential capacitances ΔC_x , ΔC_y and ΔC_z present a similar variation tendency with the simulated and theoretical counterparts. This indicates the feasibility of the proposed detection method. Note that the experimental values in ΔC_x exhibit an averaged relative derivation of 10.91% from simulated values and of 10.90% from theoretical values. This can be ascribed to the manufacturing and installation errors of the plates and the ball.



Figure 15. Dependence of the differential capacitance on the eccentric displacement along the X = Y = Z direction. (a) ΔC_x , (b) ΔC_y , and (c) ΔC_z .

In summary, the theoretical and simulated analysis of the differential capacitances is presented to explore the characteristics of the proposed capacitive sensor. The simulated values of the differential capacitances agree well with their theoretical counterparts, exhibits a linear relation with the eccentric displacement of the ball. This suggests the feasibility of the proposed detection method. More importantly, the spherical-cap plates proposed in this work could greatly reduce the nonlinear error caused by the capacitive fringe effect. This is also reflected by the good agreement between the theoretical and simulated values of the differential capacitances at different eccentric displacements. Besides, the simulated results show that a thin capacitive plate and a small null clearance could contribute to the reduction of the capacitive fringe effect. Note that a common method to avoid the fringing effect is to deploy guard rings. However, this requires the insulation between the guard ring and the capacitive plates and the minimized distance between them. In real applications, these requirements cannot be achieved easily and the effect of the guard ring could be discounted. Compared with our previously proposed sensor with guard ring [22], the present sensor could reduce the fringing effect and achieve a better linearity. This could improve the measurement accuracy of the sensor.

The sensor proposed in this work consists of a ball and six sensing plates. To achieve the insulation between the capacitive plates, a real joint should have a solid dielectric to separate the plates and it must also be the bearing surface for the ball. This can be done by depositing the dielectric on the surfaces of the sensing plates and the ball. Note that when the dielectric on their surfaces becomes worn, an air gap between the sensing plates and the ball could be widened. In addition, if the dielectric material is totally worn, the sensing plate would contact with the ball and short-circuit will be actuated. In other words, the wear of the dielectric material can be detected. It should be noted that the air gap between the ball and the sensing plates mainly results from the fabrication errors, assembly errors and the requirement of maintaining the relative motion. The eccentric displacement of the ball in the socket will inevitably occur during the operation of spherical joints. For instance, variation in force and torque caused by the motion of the robotic arm and shock transmission could lead to the eccentric displacement of the ball. As a result, there is a variation in the air gap between the ball and each sensing plate. Thus, the air is used as dielectric material in the simulation model.

This work presents a preliminary experimental validation. The similar variation tendency between the experimental and theoretical differential capacitances suggests the feasibility of the proposed sensor. However, there are small deviations between the experimental and theoretical results, which may result from the machining and assembly errors of the plates and the ball. Further studies are still required to improve the accuracy and linearity of the real capacitive sensor. Accurate fabrication and careful examination of the capacitive plates are required before conducting the experiments. Moreover, the machining errors in the ball (e.g., deviation from a sphere) and in the plates (e.g., shape, thickness) should be calibrated. Thus, extensive studies are still required to explore the calibration methods for the machining and assembly errors and to improve the accuracy of a real capacitive sensor.

7. Conclusions

This work proposes an improved capacitive sensor for detecting the eccentric displacement of spherical joints. The structure of the capacitive sensor is proposed and the mathematical model of the proposed detection method is deduced. The simulation with Ansoft Maxwell and an experimental investigation are carried out to verify the feasibility of the proposed method. Main conclusions can be made, as follows.

- 1. The proposed capacitive sensor consists of a ball and six spherical-cap plates. The ball is used as a common excitation plate while six spherical-cap plates are deployed as sensing plates. Each sensing plate and the excitation plate produce a capacitor and every two capacitors with a symmetric distribution form a capacitor pair. The eccentric displacement of the ball can be obtained by detecting the differential capacitances of three capacitor pairs.
- 2. The mathematical model of the eccentric displacements and the differential capacitances is derived. The nonlinear errors caused by the high-order terms is of 0.88% and 3.62% at the eccentricity of the ball $\rho = 0.1$ and $\rho = 0.2$, respectively. Thus, the relation between the differential capacitances (ΔC_x , ΔC_y and ΔC_z) and the eccentric displacements (δ_x , δ_y and δ_z) can be described by linear function, provided that the eccentricity of the ball is less than 0.2.
- 3. The capacitance errors caused by the capacitive fringe effect are examined. The capacitive fringe effect could be reduced by reducing the null clearance and plate thickness. In addition, the capacitance error for spherical-cap plate is smaller than that for spherically-trapezoid plate. This indicates that the spherical-cap plate proposed in this work could contribute to the reduction of the capacitive fringe effect, in comparison with the spherically-trapezoid plate.
- 4. The simulated and experimental values of the differential capacitance agree well with the theoretical counterparts, exhibiting a linear relation between the eccentric displacement and the differential capacitance. This indicates the feasibility and effectivity of the proposed capacitive sensor.

Author Contributions: Conceptualization, W.W.; Methodology, W.W., W.Q. and H.W.; Numerical simulation, W.Q., H.W. and K.X.; Data Analysis, W.Q., G.S. and K.L.; Writing—Original Draft Preparation, W.Q. and H.Y.; Writing—Review & Editing, H.Y., Z.C., and W.W.; Supervision, W.W. and B.J.; Funding Acquisition, W.W. and B.J. All authors regularly discussed the progress during the entire work.

Funding: This research was supported by the Zhejiang Provincial Natural Science Foundation of China under Grant No. LZ16E050001, and the National Natural Science Foundation of China under Grant Nos. U1709206 and 51275465.

Conflicts of Interest: The authors declare no conflict of interest.

References

- Robertson, A.P.; Slocum, A.H. Measurement and characterization of precision spherical joints. *Precis. Eng.* 2006, 30, 1–12. [CrossRef]
- Tian, Q.; Flores, P.; Lankarani, H.M. A comprehensive survey of the analytical, numerical and experimental methodologies for dynamics of multibody mechanical systems with clearance or imperfect joints. *Mech. Mach. Theory* 2018, 122, 1–57. [CrossRef]
- 3. Erkaya, S. Clearance-induced vibration responses of mechanical systems: Computational and experimental investigations. *J. Braz. Soc. Mech. Sci. Eng.* **2018**, *40*, 90. [CrossRef]
- 4. Flores, P.; Ambrosio, J.; Claro, J.C.; Lankarani, H.M. Dynamics of multibody systems with spherical clearance joints. *J. Comput. Nonlinear Dyn.* **2006**, *1*, 240–247. [CrossRef]
- 5. Erkaya, S. Effects of Joint Clearance on the Motion Accuracy of Robotic Manipulators. *J. Mech. Eng.* **2018**, *64*, 82–94.
- 6. Erkaya, S. Experimental investigation of flexible connection and clearance joint effects on the vibration responses of mechanisms. *Mech. Mach. Theory* **2018**, *121*, 515–529. [CrossRef]
- 7. Flores, P.; Lankarani, H.M. Spatial rigid-multibody systems with lubricated spherical clearance joints: Modeling and simulation. *Nonlinear Dyn.* **2010**, *60*, 99–114. [CrossRef]
- 8. Chen, X.L.; Gao, W.H.; Deng, Y.; Wang, Q. Chaotic characteristic analysis of spatial parallel mechanism with clearance in spherical joint. *Nonlinear Dyn.* **2018**, *94*, 2625–2642. [CrossRef]
- Varedi-Koulaei, S.M.; Daniali, H.M.; Farajtabar, M.; Fathi, B.; Shafiee-Ashtiani, M. Reducing the undesirable effects of joint clearance on the behavior of the planar 3-RRR parallel manipulators. *Nonlinear Dyn.* 2016, 86, 1007–1022. [CrossRef]
- 10. Muvengei, O.; Kihiu, J.; Ikua, B. Dynamic analysis of planar rigid-body mechanical systems with two-clearance revolute joints. *Nonlinear Dyn.* **2013**, *73*, 259–273. [CrossRef]
- 11. George, B.; Tan, Z.; Nihtianov, S. Advances in capacitive, eddy current, and magnetic displacement sensors and corresponding interfaces. *IEEE Trans. Ind. Electron.* **2017**, *64*, 9595–9607. [CrossRef]
- 12. Han, Y.; Zhong, C.; Zhu, X.; Zhe, J. Online monitoring of dynamic tip clearance of turbine blades in high temperature environments. *Meas. Sci. Technol.* **2018**, *29*, 045102. [CrossRef]
- 13. Lawson, C.P.; Ivey, P.C. Tubomachinery blade vibration amplitude measurement through tip timing with capacitance tip clearance probes. *Sens. Actuators A Phys.* **2005**, *118*, 14–24. [CrossRef]
- 14. Zhang, J.K.; Wang, R.B.; Deng, Z.Y.; Kang, Y.H. A displacement sensing method based on alternating current magnetic flux measurement. *Meas. Sci. Technol.* **2018**, *29*, 085010. [CrossRef]
- 15. Ripka, P.; Janosek, M. Advances in magnetic field sensors. IEEE Sens. J. 2010, 10, 1108–1116. [CrossRef]
- 16. Podhraški, M.; Trontelj, J. A differential monolithically integrated inductive linear displacement measurement microsystem. *Sensors* **2016**, *16*, 384. [CrossRef]
- 17. Tseng, V.F.; Xie, H. Resonant inductive coupling-based piston position sensing mechanism for large vertical displacement micromirrors. *J. Microelectromec. Syst.* **2016**, *25*, 207–216. [CrossRef]
- 18. Nabavi, M.R.; Nihtianov, S.N. Design strategies for eddy-current displacement sensor systems: Review and recommendations. *IEEE Sens. J.* 2012, *12*, 3346–3355. [CrossRef]
- 19. Yamaguchi, T.; Ueda, M. An active sensor for monitoring bearing wear by means of an eddy current displacement sensor. *Meas. Sci. Technol.* **2007**, *18*, 311–317. [CrossRef]
- 20. Ahn, H. A cylindrical capacitive sensor (CCS) for both radial and axial motion measurements. *Meas. Sci. Technol.* **2006**, *17*, 2027–2034. [CrossRef]
- 21. Hu, P.H.; Lu, Y.C.; Chen, S.Y.; Hu, Y.; Zhu, L.Q. Measurement method of rotation angle and clearance in intelligent spherical hinge. *Meas. Sci. Technol.* **2018**, *29*, 64012. [CrossRef]
- 22. Wang, W.; Yang, H.; Zhang, M.; Chen, Z.F.; Shi, G.; Lu, K.Q.; Xiang, K.; Ju, B.F. A novel method for the micro-clearance measurement of a precision spherical joint based on a spherical differential capacitive sensor. *Sensors* **2018**, *18*, 3366. [CrossRef]
- 23. Wang, W.; Wen, Y.H.; Yu, J.P.; Chen, Z. Impact of fringe effect on measuring accuracy of planar capacitive sensors. *Sens. Lett.* **2011**, *9*, 1458–1461. [CrossRef]

- 24. Wang, W.; Yang, H.; Zhang, M.; Chen, Z.F.; Shi, G.; Lu, K.Q.; Xiang, K.; Ju, B.F. A novel approach for detecting rotational angles of a precision spherical joint based on a capacitive sensor. *Micromachines* **2019**, *10*, 280. [CrossRef] [PubMed]
- 25. Merlet, J.-P. Parallel Robots; Springer: Dordrecht, The Netherlands, 2002.



© 2019 by the authors. Licensee MDPI, Basel, Switzerland. This article is an open access article distributed under the terms and conditions of the Creative Commons Attribution (CC BY) license (http://creativecommons.org/licenses/by/4.0/).