## Article

# Research on Predictive Control Algorithm of Vehicle Turning Path Based on Monocular Vision 

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#### Abstract

To solve the issue that the monocular vision vehicle navigation system is limited by the field of vision acquired by the charge-coupled device camera and cannot acquire navigation turning path information throughout the turning process, decreasing the vehicle turning control accuracy, this paper proposed a turning control algorithm based on monocular vision vehicle turning path prediction. Firstly, the camera's distortion was adjusted. Secondly, the camera imaging model was built, and the turning path's position information was determined using the imaging position relationship. The vehicle motion model was built in accordance with the vehicle steering mode. Lastly, the cornering trajectory of a vehicle was estimated using the vehicle's front axle length and front-wheel adjustment data, determining the vehicle turning point and turn operations on the basis of the projected relationship between the vehicle turning track and the turning path position. The experimental results showed that the proposed algorithm can effectively measure the position parameters of the cornering path and complete vehicle cornering control. The maximum absolute error of intercept and slope in turn path position parameters were 0.2525 m and 0.014 m , respectively. The cornering control accuracy was 0.093 m and 0.085 m , which met the vehicle navigation cornering control requirements. At the same time, the research can provide theoretical reference for research on precise navigation control of other cornering vehicles and other path guidance modes.


Keywords: monocular vision; visual positioning; steering control; path prediction

## 1. Introduction

The development of artificial intelligence and digitization has laid the foundation for intelligent animal husbandry. To free the farmer from heavy labor, improve the quality of operation, and reduce the risk of zoonosis, intelligent animal husbandry has become an important trend of animal husbandry development and the only way to realize the scale and industrialization of animal husbandry [1,2]. Intelligent machines for patrol inspection and feeding of livestock have been designed for moving and independently completing a series of inspections and feedings during the production process of animal husbandry [3-5]. As an important part of intelligent animal husbandry, autonomous navigation and driving system provides important technical support. Because modern animal husbandry requires many functions, in order to make effective use of resources, the cultivation areas are divided into small areas for management, which is convenient for animal husbandry. The lanes in each cultivation area are very compact [2]. It is difficult to find a suitable turning point for each turn, and therefore the study of turn control algorithm is needed.

Machine vision technology has been widely used in automatic navigation and in the driving of vehicles operating in confined spaces because of its advantages in information collection and strong environmental adaptability [6]. Automatic turning control relies heavily on visual positioning algorithms and steering control algorithms. There are three types of visual sensors used for visual positioning: monocular vision positioning systems,
binocular vision positioning systems, and multi-ocular vision positioning systems [7]. In the literature [8-20], monocular vision technology is used to achieve spatial position acquisition, and the transformation relationship between the image coordinate system and vehicle coordinate system is obtained through geometric relations, resulting in position information acquisition. Various studies [21-33] have used binocular and multi-ocular stereo vision technology to acquire spatial location. When two or more cameras of different orientations are used, the image is matched, parallax is calculated, the target distance is obtained, and the target location is realized. The above research mainly focuses on the position measurement of points on the plane but cannot solve the position parameters of the line.

To regulate vehicle path tracking, researchers [34-45] have mostly used PID control and PID-based control algorithms such as PD, fuzzy PID, and nested PID. The control algorithm for paths, including the path of beeline and small bending curvature, is limited by the collection scope of the visual navigation system based on machine vision sensor; in the vehicle turning driving path, picture data loss occurs, it is unable to provide path location information, and then it turns to control.

GPS refers to the global positioning system. Its principle is to use navigation satellites to detect and locate the distance of objects on the ground. The application of this technology in agricultural production greatly improves the efficiency of agricultural production. However, the loss of GNSS signals due to extreme weather or obstruction limits its application in complex agricultural environments [46,47]. In order for high-precision positioning and autonomous navigation ability of agricultural machinery to be achieved, as well as for the operation efficiency of agricultural machinery to be improved, navigation is often carried out by the fusion of GPS and laser radar, gyroscope, depth camera, and other sensors [48-52]. However, as the accuracy increases, the cost will also increase.

In relation to the above problem, this paper proposes a vehicle based on the monocular vision path prediction control algorithm of turning, turning to the camera calibration at first, then using holes in the imaging model of image alignment parameters to solve the calculation of ground actual position parameters that are obtained through building the vehicle motion model to forecast the vehicle movement track of turning, as well as the turning point, implementing corresponding steering actions to realize off-line turning control of vehicles. The main contributions are as follows:

1. The turning path predictive control algorithm of animal husbandry machinery can provide theoretical reference for accurate navigation control under other path guidance modes.
2. The imaging model was built, and the turn path location parameters were calculated according to the camera imaging position relationship to determine the turn path location information.
3. On the basis of the vehicle motion model, structure size, and front wheel adjustment parameters, we predicted the vehicle turning trajectory, and the relative position of the vehicle trajectory was measured while the turning point was determined on the basis of the CCD camera.

The remainder of this paper is divided as follows: Section 2 contains the details of the turning path prediction algorithm. Section 3 details the simulation and experimental test results. Finally, Sections 4 and 5 state the discussion and conclusions of this study.

## 2. Materials and Methods

### 2.1. Camera Distortion Correction

The camera used was a THE TXY-616_720P industrial camera. The lens distortion of the camera was formed as a result of the camera's poor lens set and the curvature inaccuracy of the surface, severely limiting the positioning accuracy of THE CCD camera.

Lens distortion primarily involves radial and tangential distortion, with the mathematical models of tangential and radial distortion being as follows:

$$
\begin{gather*}
\left\{\begin{array}{c}
\Delta_{x r}=x\left(k_{1}\left(x^{2}+y^{2}\right)+k_{2}\left(x^{2}+y^{2}\right)^{2}\right) \\
\Delta_{y r}=y\left(k_{1}\left(x^{2}+y^{2}\right)+k_{2}\left(x^{2}+y^{2}\right)^{2}\right)
\end{array}\right.  \tag{1}\\
\left\{\begin{array}{c}
\Delta_{x d}=2 p_{1} x y+p_{2}\left(3 x^{2}+y^{2}\right) \\
\Delta_{y d}=2 p_{1}\left(3 y^{2}+x^{2}\right)+2 p_{2} x y
\end{array}\right. \tag{2}
\end{gather*}
$$

where $\Delta_{x r}$ and $\Delta_{x d}$ are transverse tangential and radial distortion variables, respectively; $\Delta_{y r}$ and $\Delta_{y d}$ are the tangential and radial distortions along the longitudinal direction, respectively; $p_{1}$ and $p_{2}$ are the tangential distortion coefficients; $k_{1}$ and $k_{2}$ are the radial distortion coefficients; $x$ and $y$ are the coordinate values of the pixel points in the pixel coordinate system with the optical center point as the center, the horizontal direction as the horizontal axis, and the vertical direction as the longitudinal axis. Let the undistorted coordinates be ( $X_{\text {correct }}, Y_{\text {correct }}$ ), and the actual coordinate value can be found through Formula (3):

$$
\left\{\begin{array}{l}
X_{\text {correct }}=x+\Delta_{x r}+\Delta_{x d}  \tag{3}\\
Y_{\text {correct }}=y+\Delta_{y r}+\Delta_{y d}
\end{array}\right.
$$

This work primarily used a camera calibrator and OpenCV to calculate and correct camera distortion parameters. First, we used the current CCD camera to collect and calibrate the black-and-white checkerboard image from different angles (the checkerboard had a side length of 30 mm ), and then we imported the collected image into the camera calibrator to calculate the camera distortion parameters, as shown in Table 1. To adjust the camera, we entered the parameters into the OpenCV camera distortion repair application. The effect is shown in Figure 1.

Table 1. Camera Calibration Parameters.

|  | Camera Calibration Parameters |
| :--- | :---: |
| k1 | -0.345128826247987 |
| k2 | 0.191077749729993 |
| p1 | $2.450580059939514 \times 10^{-4}$ |
| p2 | $-1.939065402701762 \times 10^{-4}$ |



Figure 1. Camera calibration diagram.

### 2.2. Camera Imaging Model Construction and Turning Path Location

### 2.2.1. Camera Imaging Model Construction

The vision autonomous navigation system is primarily based on a CCD camera to gather, identify, and fit path image information, then direct and control the vehicle on the basis of the relative location of the path and the vehicle. As shown in Figure 2, the CCD imaging concept can be simplified into an imaging model. In the figure, $h$ denotes the camera height, and $\theta$ is the camera acquisition angle. Point Z is the position of the
camera optical center (camera imaging), point $O$ is the intersection of the optical axis and the ground, OZ is the camera optical axis, plane ABCD is the image acquisition range of the CCD camera at the current position, and plane $A^{\prime} B^{\prime} C^{\prime} D^{\prime}$ is the $C C D$ imaging equivalent image plane; the plane is perpendicular to the optical axis $O Z$, and points $\mathrm{A}^{\prime}, \mathrm{B}^{\prime}, \mathrm{C}^{\prime}, \mathrm{D}^{\prime}$ are points $A, B, C, D$ corresponding to the imaging position of the image plane, respectively, taking point $O$ as the coordinate origin to establish the equivalent image plane coordinate system U-O-V and ground coordinate system X-O-Y.


Figure 2. Camera imaging model.

### 2.2.2. Position Measurements of Turning Path

The position relationship between the image plane and the ground can be obtained from the spatial position relationship between the image plane coordinate system and the ground coordinate system in the camera imaging model, as shown in Figure 3a,b. Point Q in Figure 3a corresponds to any point on the plane ABCD, its plane coordinate was ( $Q_{x}$, $Q_{y}$ ); $Q^{\prime}$ is a point on the image plane corresponding to point $Q$, and its coordinate was ( $Q_{u}$, $\left.Q_{v}\right)$. The corresponding relationship between pavement coordinate and image points can be established using the collinear principle of actual point $Q$, image point $Q^{\prime}$, and camera optical center point $Z$, as indicated in Formula (4).

$$
\left\{\begin{array}{l}
Q_{x}=\frac{Q_{u}\left(Q_{y}+h \tan \theta\right)}{Q_{v}+h \tan \theta}  \tag{4}\\
Q_{y}=\frac{Q_{v} h(\cos \theta+\sin \theta \tan \theta)}{h-Q_{v} \sin \theta}
\end{array}\right.
$$



Figure 3. Position relationship diagram. (a) Y-V relationship diagram. (b) X-U relationship diagram.
As shown in Figure 4, the known image resolution was $H \times W$. The position parameter slope and intercept of the turning path in the pixel coordinate system I-O-J were $\alpha_{p}$ and $\beta_{p}$, respectively. Then, using the relative position relationship between the pixel coordinate system and the image coordinate $\alpha_{t}$ and intercept $\beta_{t}$, we were able to calculate the slope of the position parameter of the turning path in the equivalent picture plane as follows:

$$
\left\{\begin{array}{l}
\alpha_{t}=-\alpha_{p}  \tag{5}\\
\beta_{t}=\left(\frac{H-\alpha_{p} W}{2}-\beta_{p}\right) \times L_{M N} / W
\end{array}\right.
$$



Figure 4. Coordinate system position relationship diagram.
In this formula, $L_{M N}$ represents the actual distance between the effective image plane and the actual ground intersection MN, as shown in Figure 3b. Formula (4) shows the slope of the actual position parameter of the turning path $\alpha$ and intercept $\beta$ slope of the position parameter in equivalent image plane $\alpha_{t}$ and intercept $\beta_{t}$ conversion formula, respectively, which is as follows:

$$
\left\{\begin{array}{l}
\alpha=\frac{\alpha_{t} h(\cos \theta+\sin \theta \tan \theta)}{h-\beta_{t} \sin \theta}  \tag{6}\\
\beta=\frac{\beta_{t} h(\cos \theta+\sin \theta \tan \theta)}{h-\beta_{t} \sin \theta}
\end{array}\right.
$$

### 2.3. Vehicle Movement Model and Turn Control Algorithm Design

### 2.3.1. Vehicle Motion Model Establishment

Vehicle steering is made up of three components: articulated steering, differential steering, and Ackerman steering. This paper's cornering control object is an agricultural vehicle. This vehicle's steering mode is Ackerman steering. Ackerman steering has been designed to solve the problem of different corners of left and right steering wheels caused by different steering radii of left and right steering wheels when the vehicle is steering. According to Ackermann steering geometry, the steering mechanism is designed so that when the vehicle is turning along a bend, the steering angle of the inner wheel is about 2-4 degrees greater than that of the outer wheel, and thus the circles of the four wheel paths meet approximately at the instantaneous steering center on the extension line of the rear axle, allowing the vehicle to turn smoothly. The schematic of its steering principle is shown in Figure 5a. Or represents the steering center of the vehicle, $\varphi_{1}$ represents the deflection angle of the outer wheel, $\varphi_{2}$ represents the deflection angle of the inner wheel, and F represents the center point of the front axle. R represents the midpoint of the rear axle, and $L$ represents the wheelbase.


Figure 5. Vehicle steering diagram. (a). Ackerman steering principal diagram. (b). Vehicle two-wheel steering diagram.

The vehicle motion model can be simplified into a two-wheel model, as shown in Figure 5b, where $\mathrm{X}-\mathrm{O}-\mathrm{Y}$ is the ground acquisition coordinate system for the camera and Or is the front-wheel angle of the vehicle. For $\delta$ instantaneous rotational axis, set the starting coordinates of point F and point R to $\left(r_{x}, r_{y}\right),\left(f_{x}, f_{y}\right)$ at front-wheel angle when $\Delta t$ period is
$\delta$. If the speed of the vehicle is $V$, then position coordinates of point F and point R after $\Delta t$ period are $\left(r_{x}(\Delta t), r_{y}(\Delta t)\right),\left(f_{x}(\Delta t), f_{y}(\Delta t)\right)$. The expression is shown in Formula (7), in which $\omega$ is the vehicle circumferential angular velocity, and $O r_{x}$ and $O r_{y}$ are the position coordinates of the vehicle rotation center Or, respectively. Their calculation formulas are shown in Formulas (8) and (9).

$$
\begin{gather*}
{\left[\begin{array}{l}
r_{x}(\Delta t) \\
r_{y}(\Delta t) \\
f_{x}(\Delta t) \\
f_{y}(\Delta t)
\end{array}\right]=\left[\begin{array}{cccc}
\cos \omega \Delta t & \sin \omega \Delta t & 0 & 0 \\
-\sin \omega \Delta t & \cos \omega \Delta t & 0 & 0 \\
0 & 0 & \cos \omega \Delta t & \sin \omega \Delta t \\
0 & 0 & -\sin \omega \Delta t & \cos \omega \Delta t
\end{array}\right] \times\left[\begin{array}{c}
r_{x}-O r_{x} \\
r_{y}-O r_{y} \\
f_{x}-O r_{x} \\
f_{y}-O r_{x}
\end{array}\right]+\left[\begin{array}{c}
O r_{x} \\
O r_{y} \\
O r_{x} \\
O r_{x}
\end{array}\right]}  \tag{7}\\
\omega=\frac{V \times \tan \delta}{L}  \tag{8}\\
{\left[\begin{array}{l}
O r_{x} \\
O r_{y}
\end{array}\right]=\left[\begin{array}{cc}
\tan \left(\tan ^{-1} \frac{f_{y}-r_{y}}{f_{x}-r_{x}}-\frac{\pi}{2}\right) & -1 \\
\tan \left(\tan ^{-1} \frac{f_{y}-r_{y}}{f_{x}-r_{x}}-\delta-\frac{\pi}{2}\right) & -1
\end{array}\right]^{-1} \times\left[\begin{array}{c}
\tan \left(\tan ^{-1} \frac{f_{y}-r_{y}}{f_{x}-r_{x}}-\frac{\pi}{2}\right) \times f_{x}-f_{y} \\
\tan \left(\tan ^{-1} \frac{f_{y}-r_{y}}{f_{x}-r_{x}}-\delta-\frac{\pi}{2}\right) \times r_{x}-r_{y}
\end{array}\right]} \tag{9}
\end{gather*}
$$

### 2.3.2. Design of Turning Control Algorithm

A turning control method based on vehicle turning path prediction was devised on the basis of the path acquisition features of a vehicle navigation system based on machine vision. First, the turning trajectory of the vehicle is anticipated on the basis of the vehicle structure size, front-wheel adjustment parameters, and driving speed. Then, the relative position of the vehicle trajectory in the waiting turning path is determined on the basis of the CCD camera and the steering point. When the vehicle reaches the steering point, the steering wheel angle is adjusted to control the vehicle in order to drive to the predetermined waiting turning path. The angle sensor [53] used to measure the front-wheel rotation angles corresponding to different steering wheel angles is shown in Table 2, and the relationship between steering wheel rotation and front-wheel deflection is fit using the MATLAB data fitting tool. The relationship is shown in Formula (10), and the diagram is shown in Figure 5.

$$
\begin{equation*}
y=\mathrm{a} x^{4}+\mathrm{b} x^{3}+\mathrm{c} x^{2}+\mathrm{d} x+\mathrm{e} \tag{10}
\end{equation*}
$$

where a is equal to 0.000004473 ; b is equal to 0.00001372 ; c is equal to 0.001587 ; d is equal to 0.04146 ; e is equal to $0.001829 ; y$ is the front-wheel angle, road; and $x$ is the steering wheel angle, road.

Table 2. Angle sensor parameters.

| Range | $\mathbf{3 6 0 ^ { \circ }}$ Measure, Sinusoidal Output | Repeatability | $\pm \mathbf{0 . 0 5 \%}$ |
| :---: | :---: | :---: | :---: |
| Linear range | $\pm 45^{\circ} \pm 30^{\circ} \pm 20^{\circ}$ | Sensitivity | $\approx 40 \mathrm{mV} / 1^{\circ}(\mathrm{Vin}=5 \mathrm{~V})$ |
| Working voltage | $\mathrm{DC} 5 \mathrm{~V}(\mathrm{DC} 3.8 \sim 8 \mathrm{~V})$ | Working current | $<20 \mathrm{~mA}$ |
| Output | $1 \sim 4 \mathrm{~V}$ | Rotation mode | Continuous rotation |
| Temperature range | $0 \sim 95 \% \mathrm{RH}$ | Service temperature | $-25^{\circ} \mathrm{C} \sim+75^{\circ} \mathrm{C}$ |

The test object's steering actuator is a stepping motor gear packing actuator. The structure is shown in Figure 6. It mainly uses a stepping motor as a power source to drive the steering wheel through a gear transmission to realize the front-wheel steering control. The working speed of the stepper motor was $5.3 \mathrm{rad} / \mathrm{s}$, and the gear ratio was $1: 4$, and therefore the adjustment speed of the steering wheel was $1.33 \mathrm{rad} / \mathrm{s}$. Simultaneously, the front and rear wheelbase 1 of the vehicle was measured to be 3.25 m , and the maximum deflection angles of the left and right front wheels were $35^{\circ}$ and $21^{\circ}$, respectively.


Figure 6. Fitting diagram of the relationship between the steering wheel and front-wheel angle.
On the basis of the constructed vehicle motion model, the steering actuator is shown in Figure 7, and the angular position relationship between the steering wheel and front wheel and vehicle structural parameters, taking a right turning as an example, we found that the front-wheel adjustment angle was $0^{\circ}-21^{\circ}$, and the vehicle driving speed was $1.8 \mathrm{~km} / \mathrm{h}$; $(\Delta t=0.01)$ calculates the turning path of the vehicle. The results are shown in Figure 8.


Figure 7. Steering actuator.


Figure 8. Vehicle turning trajectory prediction.
As shown in Figure 8, the front-wheel adjustment state of the vehicle during cornering is mainly divided into two stages, namely, the angle adjustment stage and the angle fixing
stage. $R R_{1}$ and $F_{1}$ stages are steering wheel adjustment stage, and $R_{1} R_{2}$ and $F_{1} F_{2}$ stages are steering wheel fixing stage, in which the vehicle makes a circular motion.

The best turning point of the vehicle is when the turning path is tangent to the driving track of the center point of the vehicle's front axle. Since the angle between the turning path and the current vehicle's driving track is about $90^{\circ}$, the tangent point between the turning path and vehicle's driving track is in the $\mathrm{F}_{1} \mathrm{~F}_{2}$ stage, i.e., steering wheel fixing stage.

We set the turning path expression to $y=\alpha x+\beta$, according to the positional relationship when the vehicle trajectory is tangent to the transverse road to be turned when the distance $s$ between the vehicle rotation circle center Or and the turning path in the $F_{1} F_{2}$ stage are equal to the current vehicle radius $r$, being the best turning point of the vehicle. The calculation formulas of $S$ and $R$ are as follows:

$$
\begin{gather*}
S=\frac{\left|\alpha O r_{x}-O r_{y}+\beta\right|}{\sqrt{\alpha^{2}+1}}  \tag{11}\\
R=\frac{L}{\sin \delta} \tag{12}
\end{gather*}
$$

where $O r_{x}$ and $O r_{y}$ are the coordinates of the circle center $O r$, and $L$ is the vehicle wheelbase, $\delta$ front-wheel angle.

## 3. Results

To verify the feasibility of the vehicle turning control algorithm proposed in this paper, on the basis of the path detection algorithm developed in the early stage, we integrated the steering control algorithm designed in this paper. The industrial computer used was a Lenovo laptop, CPU was i76700HQ, memory was 8 G , and the graphics card was GTX960M; VS2013 MFC software development program was used to write the vehicle turning control program. The program interface during the experiment is shown in Figure 9, showing that the camera interface was corrected for distortion, and the red guide line was the path navigation line. The program was mainly composed of three parts: vehicle status, serial port setting, and image source selection. The vehicle status includes four parameters: vehicle deflection angle, front wheel deviation, front wheel deflection angle, and adjustment angle, which were calculated by the data transmitted by the program and sensor. We build a test platform with agricultural vehicles as the test object. Sensors, cameras, laptops, and other equipment were placed and installed as shown in Figure 10a,b, and the industrial computer was placed in the passenger seat. In September 2019, the navigation test site of the North Campus factory of Shihezi University in Xinjiang was used for obtaining the turning path position and vehicle turning test.

As shown in Figure 11a, the CCD camera was installed in the test vehicle's front, $h$ was 1.2 m above the ground, and the angle between the optical axis and the ground plane $\theta$ was $30^{\circ}$. At the same time, the guidance line of the navigation test path, as shown in Figure 11b, was drawn in the test area, wherein the color of the guidance line was yellow, the width of the line was 2.5 cm , and the length of the longitudinal path in the early stage was 20 m . The length of the turning path was 10 m .

During the test, the vehicle was first guided to the early longitudinal path on the basis of the straight-line path tracking algorithm developed in the early stage. Setting Cartesian coordinates at the test site, we recorded the turning track when the vehicle was turning. We found the best turning point according to Formulas (8) and (9). The tangent of the best turning point fell on the established coordinate axis, and the actual measured value was obtained. Then, the path blocking method was adopted to conduct tests on turning paths A, B, and C according to the length of the front bridge. We recorded the vehicle steering motion at the beginning of the path location of turning parameters of CCD camera measurement (slope alpha and intercept beta) and compared it with the actual value; the vehicle turning control accuracy took place at the same time at the front axle length of 1.72 m , with the results being shown in Table 3. At the same time, we obtained the turning path measuring position and actual position parameters of a function, which are shown in

Figures 12 and 13. When the front axle length was 1.43 m , the experimental results were as they are shown in Table 4. At the same time, the one-time function of measured position parameters and actual position parameters of the turning path was obtained, as shown in Figures 14 and 15.


Figure 9. Program interface.

(a)

(b)

Figure 10. Equipment layout. (a) Fixed position of equipment. (b) Position of angle sensor.

(a)

(b)

Figure 11. Turning test chart. (a) Camera installation drawing. (b) Schematic of the test path.
Table 3. Test results.

| Lateral to Turn the Path | Measuring Position |  | Actual Location | Path Measurement Error | Steering <br> Accuracy |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\alpha$ | $\beta$ | $\alpha$ | $\beta$ | $\alpha$ | $\beta$ |
| A | 1.5469 | -6.641 | 1.5589 | -6.7395 | -0.012 | 0.0985 |
| B | 1.8290 | -8.130 | 1.8298 | -8.2493 | -0.008 | 0.1193 |
| The average | 2.356 | -8.7432 | 2.3600 | -8.8118 | -0.004 | 0.0686 |
| The maximum | 1.9106 | -7.8381 | 1.9162 | -7.9335 | -0.006 | 0.0955 |
|  | 2.356 | -8.7432 | 2.36 | -8.8118 | -0.004 | 0.1193 |



Figure 12. Measurement position of turning path (length of the front axle is 1.72 m ).


Figure 13. Actual position of turning path (length of the front axle is 1.72 m ).

Table 4. Test results.

| Lateral to Turn the Path | Measuring Position |  | Actual Location | Path Measurement Error | Steering <br> Accuracy |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\alpha$ | $\beta$ | $\alpha$ | $\beta$ | $\alpha$ | $\beta$ |  |
| A | 1.5165 | -6.353 | 1.5179 | -6.3795 | -0.014 | 0.0265 | 0.085 |
| B | 1.7982 | -7.980 | 1.8102 | -8.2325 | -0.012 | 0.2525 | 0.072 |
| C | 2.314 | -8.135 | 2.320 | -8.1785 | -0.006 | 0.0435 | 0.083 |
| The average | 1.8648 | -7.4893 | 1.8737 | -7.7175 | -0.011 | 0.1075 | 0.08 |
| The maximum | 2.285 | -8.135 | 2.293 | -8.356 | -0.014 | 0.2525 | 0.085 |



Figure 14. Measurement position of turning path (length of the front axle is 1.43 m ).


Figure 15. Actual position of turning path (length of the front axle is 1.43 m ).
According to the relationship between the measured position and the actual position of path $A$ for $\alpha$ and $\beta$, the errors were -0.012 and 0.0985 , respectively. Similarly, for path B $\alpha$ and $\beta$, the errors were -0.008 and 0.1193 , respectively. For path $C \alpha$ and $\beta$, the errors were -0.004 and 0.0686 , respectively. It is known that the greater the angle between the transverse path and the longitudinal path, the smaller the slope error, and the algorithm can meet the requirements of vehicle turning.

According to the relationship between the measured position and the actual position of path $A \alpha$ and $\beta$, the errors were -0.014 and 0.0265 , respectively. Similarly, for path $B$ $\alpha$ and $\beta$, the errors were -0.01 and 0.2525 respectively. For path $C \alpha$ and $\beta$, the errors were -0.008 and 0.0435 , respectively. It is known that the greater the angle between the transverse path and the longitudinal path, the smaller the slope error, and the algorithm can meet the requirements of vehicle turning.

Conclusions can be drawn from the above experimental results and data analysis when the length of the front axle was 1.72 m . We found from the tangent function expression of the turning path, as shown in Figures 12 and 13, that the turning path predictive control algorithm had good stability when the turning path was $90^{\circ}$ and $105^{\circ}$. When the turning path is greater than or equal to $120^{\circ}$, the turning path predictive control algorithm will
have obvious errors. Through the experiments, we found that if the turning path is greater than $120^{\circ}$, there will be errors. However, this will not affect the actual effect of turning, and therefore the error is within a reasonable range that the algorithm can control. When the front axle size was 1.43 m , we see from Figures 14 and 15 that when the turning path was greater than $120^{\circ}$, the error decreased, the turning control accuracy of the control algorithm was improved, and the stability of the control algorithm was better. As shown in Tables 3 and 4, for different front axle sizes, the turning control algorithm proposed in this paper was able to effectively measure the position parameters of the path to be turned and to turn the vehicle in order to identify the turning point. The maximum measurement errors of the slope of the position parameters of the path to be turned were 0.012 and 0.014 , respectively, and the maximum measurement errors of the intercept were 0.1193 and 0.2525 , respectively. The maximum accuracy of the turning control was 0.093 m and 0.085 m .

Because the testing ground is not an absolute plane, there will be a certain deviation in the position state of the CCD camera relative to the ground during vehicle driving, resulting in a certain error in the measurement of position parameters of the path to be turned. Due to impact of the stability of vehicle speed, there is a certain deviation between the predicted path and the actual paths. The above two errors lead to a certain deviation between the determined turning point and the ideal turning point, and therefore there is a certain error in the vehicle turning control, in which the path position measurement error and steering control error are within the effective range. Hence, it meets the turning control requirements for in-vehicle navigation operation.

## 4. Discussion

This paper studied the turning control algorithm based on path prediction and mainly discusses path position measurement, vehicle path prediction, and turning point determination based on monocular vision.
(1) Aiming at the navigation path position measurement problem, firstly, we corrected the camera's distortion. The imaging model was constructed, and the path position acquisition function was established according to the camera imaging position relationship in order to realize the measurement of path position parameters of working pavement.
(2) Aiming at the steering point prediction, firstly, we established the vehicle motion model according to the vehicle steering mode, predicted the vehicle turning trajectory according to the vehicle structure size and front-wheel adjustment parameters, and distinguished the steering point according to the path position parameters.

## 5. Conclusions

The test results show that the vehicle steering control algorithm based on monocular vision proposed in this paper can accurately identify steering points, optimize the field of vision block problem during monocular vision steering, and improve the accuracy of vehicle steering control. The steering control algorithm is mainly based on the steering of the front axle. According to the length of the front axle, the maximum measuring error of the position slope and position parameter of the turning path was found to be 0.014 , and the maximum measuring error of the intercept was 0.2525 m . The maximum accuracy of steering control was 0.093 m and 0.085 m , meeting the steering control requirements for in-vehicle navigation operation.

In the future, a comparison with some other experimental studies should be performed, and the following further research should also be conducted. The turning predictive control algorithm, based on machine vision, needs to be improved. On the basis of our test, we found that once there is strong light or deep shadows, the detection of navigation guides becomes unstable and the camera lens deflection is too large to cause bias. The integration of GPS and multi-sensor and deep learning is worth studying and applying at this step. In order to make the cornering prediction algorithm more adaptable, more factors should be considered, such as the influence of speed changes at a certain angle, tire pressure, etc., on
vehicle cornering. In future work, these factors can be further incorporated into the scope of algorithm optimization in order to improve the accuracy of cornering predictive control.

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