



Article Anti-Windup Load Frequency Controller Design for Multi-Area Power System with Generation Rate Constraint

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Abstract: To deal with the problem of generation rate constraint (GRC) during load frequency control (LFC) design for a multi-area interconnected power system, this paper proposes an anti-windup controller design method. Firstly, an H_{∞} dynamic controller is designed to obtain robust performance of the closed-loop control system in the absence of the GRC. Then, an anti-windup compensator (AWC) is formulated to restrict the magnitude and rate of the control input (namely power increment) in the prescribed ranges so that the operation of generation unit does not exceed the physical constraints. Finally, the anti-windup LFC is tested on the multi-area interconnected power systems, and the simulation results illustrate the effectiveness of the proposed LFC design method with GRC.

Keywords: load frequency control (LFC); generation rate constraint (GRC); anti-windup control; robust controller

1. Introduction

In a multi-area interconnected power system, it is important for the system's operation to keep the active power balance and regulate the tie-line power at the scheduled value. Load frequency control (LFC) plays several key roles in the active power control of the interconnected power system [1,2], such as counteracting the load fluctuation, stabilizing the system frequency, regulating the tie-line power, and narrowing the area control error (ACE). Thus, the LFC is vital for the security and stability of power system.

On the issue of LFC design for a power system, an amount of work has been done in recent years. The conventional LFC usually adopts the proportional-integral (PI)-type controller because it has simpler structure and fewer tuning parameters. However, this kind of controller has shortcomings in terms of coping with the operating point change and the load disturbance, since they are designed on nominal operating points with fixed parameters. In order to obtain better performance of the PI-type LFC, the parameter optimization methods of the PI-type controller are proposed in [3–5]. To enhance the robustness and reliability of the control system, some fuzzy-logic-based LFC methods are introduced in [6–9]. In addition, some advanced control technologies are utilized to improve LFC performance, such as sliding mode methods [10–12], optimal or suboptimal feedback control methods [13–16], and robust control methods [17–19]. Considering the delay in the open communication network, the authors in [20–22] analyze the influence of time delay on the LFC and present the relevant controller design methods. To guarantee compliance with the control performance standards (CPS) of North American Electric Reliability Council (NERC) and reduce

wear and tear of generators, a decentralized model predictive control method is used to deal with the LFC problem [23]. For accommodating unexpected load change and faults, the supervisory control strategies in [24,25] are proposed to solve the load and frequency set-point problem. It is well known that due to physical limitations, generation units have inherent generation rate constraints (GRC), such as ramp rate constraints and upper-lower bound constraints. If GRC is not considered adequately in LFC design, the controller will not yield excellent performances, and even the closed-loop system stability may be destroyed under disturbances [26–28]. In the aforementioned research work, some studies make tentative consideration on the GRC problem. In [1,7,8,12], GRC is considered in the simulation, but neglected in the controller design. Therefore, the validity of these methods to deal with GRC lacks theoretical support. Towards LFC design with GRC, the extended integral control method in [26], the biased PI dual mode control method in [27], the Type-2 fuzzy approach in [28], and the anti-GRC PI-type controller in [29,30] are adopted to deal with the GRC problem. Unfortunately, the strict mathematical proof in the above methods is still absent.

Focusing on the LFC design with GRC, this paper proposes an anti-windup LFC design method for the multi-area interconnected power system. The designed LFC consists of a robust H_{∞} controller and an anti-windup compensator (AWC). The former is used to guarantee the stability and robustness of the closed-loop system without constraints, and the latter takes charge of restricting the rate and magnitude of control input in the prescribed ranges to make the operation of generation unit meet the GRC requirement. For verifying the proposed method, several multi-area interconnected power systems are employed for testing. The comparative simulation results show that the performances of the LCF are improved by the design method of this paper.

2. Load Frequency Control Model

The large interconnected power system is usually partitioned into several areas for management and control. Generally, for reducing the difficulty in the LFC design, each area in the LFC model is simplified to be an equivalent generator with a turbine and a governor shown in Figure 1. The dynamics of the generator, the turbine and the governor are described by three first-order inertial processes, respectively. In addition, since the generation unit has the physical operation limitations, the LFC model includes the GRC, namely the ramp rate and the upper-lower bound constraints of the generation units. As one knows, the GRC may generate adverse impact on the LFC performances if the GRC is not considered sufficiently in LFC design.



Figure 1. Load frequency control (LFC) diagram of Area *i*. ACE: area control error; GRC: generation rate constraint.

The GRC of LFC model shown in Figure 1 includes the magnitude and rate saturation of the states. This kind of state saturation nonlinearity causes much difficulty in controller design.

Remark 1. For a real power system, we know the fact that the generation unit will operate in the linear region (without touching the saturation bounds), if the power increment is limited in the magnitude and rate ranges appropriately. In other words, the generation unit can meet its GRC when the proper rate and magnitude constraints are imposed on the control signal of the LFC. Based on

the above fact, the LFC design with GRC can be solved through dealing with the problem on the controller synthesis subject to the magnitude and rate saturation of the control. Assuming that the deigned controller makes the generation unit operate in the linear region, the nonlinear GRC of the LFC model can be removed, and thus the original LFC model can be modified into a new one as shown in Figure 2.



Figure 2. Modified LFC diagram of Area *i*.

According to the LFC dynamic model diagram shown in Figure 2, the state-space LFC model can be written as:

$$\begin{split} \Delta f_i &= \frac{1}{H_i} \left(\Delta P_{ti} - \Delta P_{Li} - \Delta P_{tie_i} - D_i \Delta f_i \right) \\ \Delta \dot{P}_{ti} &= -\frac{1}{T_{ti}} \left(\Delta P_{ti} - \Delta P_{gi} \right) \\ \Delta \dot{P}_{gi} &= -\frac{1}{T_{gi}} \left(\Delta P_{gi} - \Delta P_{ci} + \frac{1}{R_i} \Delta f_i \right) \\ \Delta \dot{P}_{tie_i} &= \gamma_i \Delta f_i - \eta_i \\ \dot{I}_{ACE_i} &= ACE_i = \beta_i \Delta f_i + \Delta P_{tie_i} \\ y_{i1} &= \Delta f_i \\ y_{i2} &= ACE_i = \beta_i \Delta f_i + \Delta P_{tie_i} \\ y_{i3} &= I_{ACE_i} \end{split}$$
(1)

where $\gamma_i = \sum_{j=1, j \neq i}^{N} T_{ij}$; $\eta_i = \sum_{j=1, j \neq i}^{N} T_{ij} \Delta f_j$; f_i denotes the system frequency; P_{ti} denotes the turbine power; P_{gi} denotes the governor valve; P_{ci} denotes the governor power setpoint; P_{Li} denotes the load demand; P_{tie_i} denotes the net tie-line power; ACE_i denotes the area control error; I_{ACE_i} denotes the integral of ACE_i ; Δ denotes the deviation from normal value; β_i denotes the frequency bias coefficient;

 R_i denotes the deviation from normal value; p_i denotes the frequency bias coefficient; R_i denotes the droop coefficient; T_{gi} denotes the governor time constant; T_{ti} denotes the turbine time constant; H_i denotes the area aggregate inertia constant; D_i denotes the area load damp constant; and T_{ij} denotes the tie-line synchronizing coefficient.

Usually, we focus on the frequency deviation, the ACE and the control energy cost when evaluating the LFC performances. Thus, the controlled variables z for the H_{∞} control design are selected as follows:

$$\begin{cases} z_{i1} = \Delta f_i \\ z_{i2} = I_{ACE_i} \\ z_{i3} = \Delta P_{ci} \end{cases}$$
(2)

For convenience, by defining the state variables $x = [\Delta f_i, \Delta P_{ti}, \Delta P_{gi}, \Delta P_{tie_i}, I_{ACE_i}]^T \in \mathcal{R}^5$, the control variable $u = \Delta P_{ci} \in \mathcal{R}^1$, the output variables $y = [y_{i1}, y_{i2}, y_{i3}]^T \in \mathcal{R}^3$, the controlled variables $z = [z_{i1}, z_{i2}, z_{i3}]^T \in \mathcal{R}^3$, and the disturbance variables $d = [\Delta P_{Li}, \eta_i]^T \in \mathcal{R}^2$, the state-space model \mathcal{P} consisting of Equations (1) and (2) can be rewritten as:

$$\mathcal{P}: \begin{cases} \dot{x} = Ax + B_{u}u + B_{d}d \\ y = C_{y}x + D_{yu}u + D_{yd}d \\ z = C_{z}x + D_{zu}u + D_{zd}d \end{cases}$$
(3)

where:

$$A = \begin{bmatrix} -\frac{D_i}{H_i} & \frac{1}{H_i} & 0 & -\frac{1}{H_i} & 0\\ 0 & -\frac{1}{T_{ti}} & -\frac{1}{T_{ti}} & 0 & 0\\ -\frac{1}{R_i T_{gi}} & 0 & -\frac{1}{T_{gi}} & 0 & 0\\ \gamma_i & 0 & 0 & 1 & 0 \end{bmatrix}; B_u = \begin{bmatrix} 0\\ 0\\ \frac{1}{T_{gi}}\\ 0\\ 0 \end{bmatrix}; B_d = \begin{bmatrix} -\frac{1}{H_i} & 0\\ 0 & 0\\ 0 & 0\\ 0 & -1\\ 0 & 0 \end{bmatrix}$$
$$C_y = \begin{bmatrix} 1 & 0 & 0 & 0 & 0\\ \beta_i & 0 & 0 & 1 & 0\\ 0 & 0 & 0 & 0 & 1 \end{bmatrix}; D_{yu} = \begin{bmatrix} 0\\ 0\\ 0\\ 0 \end{bmatrix}; D_{yd} = \begin{bmatrix} 0 & 0\\ 0 & 0\\ 0 & 0 \end{bmatrix}$$
$$C_z = \begin{bmatrix} 1 & 0 & 0 & 0 & 0\\ 0 & 0 & 0 & 1\\ 0 & 0 & 0 & 0 \end{bmatrix}; D_{zu} = \begin{bmatrix} 0\\ 0\\ 1\\ 1 \end{bmatrix}; D_{zd} = \begin{bmatrix} 0 & 0\\ 0 & 0\\ 0 & 0 \end{bmatrix}$$

The control input with rate and magnitude saturations shown in Figure 2 can be defined as:

$$u = sat_{\mathbf{m}}(\varphi) = \begin{cases} \overline{m}, & \varphi > \overline{m} \\ \varphi, & \underline{m} \le \varphi \le \overline{m} \\ \underline{m}, & \varphi < \underline{m} \\ \overline{r}, & \mu > \overline{r} \\ \mu, & \overline{r} \le \mu \le \overline{r} \\ \underline{r}, & \mu < \underline{r} \end{cases}$$
(4)

where $sat(\cdot)$ denotes saturation function, $[\underline{m}, \overline{m}]$ and $[\underline{r}, \overline{r}]$ denote the magnitude bound and the rate bound, respectively.

3. Anti-Windup Load Frequency Controller Design

In this section, we design the LFC to ensure that the control input never exceeds the magnitude limit and the rate limit to meet the GRC. Based on the LFC model \mathcal{P} , the anti-windup schemes [31,32] are employed to synthesize the LFC in the following subsections.

3.1. Original H_{∞} Controller Design

According to the anti-windup scheme, a robust H_{∞} controller is designed on the basis of the LFC model \mathcal{P} in absence of the control input saturation in advance. Assuming that the system (A, B_u, C_y) is controllable and observable, we can design an H_{∞} dynamic controller \vec{C} with the following form:

$$\bar{\mathcal{C}}: \begin{cases} \dot{x}_{\bar{c}} = A_{\bar{c}}x_{\bar{c}} + B_{\bar{c}}u_{\bar{c}} \\ y_{\bar{c}} = C_{\bar{c}}x_{\bar{c}} + D_{\bar{c}}u_{\bar{c}} \end{cases}$$
(5)

where $x_{\bar{c}} \in \mathcal{R}^5$ are the state variables of the controller; $u_{\bar{c}} \in \mathcal{R}^3$ are the input variables of the controller (the measured variables of \mathcal{P} : $u_{\bar{c}} = y$); $y_{\bar{c}} \in \mathcal{R}^1$ is the output variable of the controller (the control input variable of \mathcal{P} : $y_{\bar{c}} = u$); and $A_{\bar{c}}$, $B_{\bar{c}}$, $C_{\bar{c}}$, $D_{\bar{c}}$ are the constant matrices with appropriate dimension.

Since the robust H_{∞} design method is well-known, we do not intend to repeat them. If the detailed introduction of the method is needed, one can refer to the literatures [33,34]. In this paper, we use the MATLAB/Robust Linear Matrix Inequality (LMI) Control Box [35] to solve the robust controller \bar{C} directly. Here, it is assumed that the closed-loop system consisting of \mathcal{P} and

 \bar{C} is well posed and gains the prescribed H_{∞} performance without consideration of the control input saturations.

3.2. Anti-Windup Compensator Design

To tackle the magnitude and rate saturations of the control input, we borrow the anti-windup control scheme [32] shown in Figure 3.



Figure 3. Structure of an anti-windup control scheme. AWC: anti-windup compensator.

In the anti-windup control approach, it is needed to compute the first-order derivative of the controller output $y_{\bar{c}}$. Here, the differentiator *s* is replaced by a linear filter $\frac{s}{1+\tau s}$ with a sufficiently small constant τ , considering that the controller output may be not strictly proper. The modified controller consists of the original controller \bar{C} and the filter $\frac{s}{1+\tau s}$ can be expressed as:

$$C: \begin{cases} \dot{x}_{c} = A_{c}x_{c} + B_{c}u_{c} \\ y_{c} = C_{c}x_{c} + D_{c}u_{c} \end{cases}$$
(6)

where $x_c = [x_{\bar{c}}^T, x_f]^T \in \mathcal{R}^6$ are the modified controller states; $x_f \in \mathcal{R}^1$ are the filter states; $u_c = u_{\bar{c}} \in \mathcal{R}^1$ is the modified controller input; $y_c = [y_{\bar{c}}, y_{\bar{c},d}]^T \in \mathcal{R}^2$ are the modified controller outputs; $y_{\bar{c},d} \in \mathcal{R}^1$ denote approximate derivatives of $y_{\bar{c}}$; and the parameter matrices of C are:

$$A_{c} = \begin{bmatrix} A_{\bar{c}} & 0\\ \frac{C_{\bar{c}}}{\tau} & \frac{-1}{\tau} \end{bmatrix}, B_{c} = \begin{bmatrix} B_{\bar{c}}\\ \frac{D_{\bar{c}}}{\tau} \end{bmatrix}, C_{c} = \begin{bmatrix} C_{\bar{c}} & 0\\ \frac{C_{\bar{c}}}{\tau} & \frac{-1}{\tau} \end{bmatrix}, D_{c} = \begin{bmatrix} D_{\bar{c}}\\ \frac{D_{\bar{c}}}{\tau} \end{bmatrix}$$

In Figure 3, the AWC is designed to cope with the controller limits. The AWC is formulated as follows:

$$\mathcal{AWC}: \begin{cases} \dot{x}_{aw} = Ax_{aw} + B_{u}(u - y_{\bar{c}}) \\ y_{aw} = C_{y}x_{aw} + D_{yu}(u - y_{\bar{c}}) \\ z_{aw} = C_{z}x_{aw} + D_{zu}(u - y_{\bar{c}}) \\ v = K_{aw} \begin{bmatrix} x_{aw} \\ \varphi_{aw} \end{bmatrix} \end{cases}$$
(7)

where $x_{aw} \in \mathcal{R}^5$ are the AWC states; $y_{aw} \in \mathcal{R}^3$ are the AWC output; $(u - y_{\bar{c}})$ and $\varphi_{aw} = (\varphi - y_{\bar{c}})$ serve as the AWC input; $z_{aw} \in \mathcal{R}^3$ are the AWC controlled variables; $v \in \mathcal{R}^1$ is the stabilizing signal which needs to be designed; and $K_{aw} \in \mathcal{R}^{1 \times 6}$ is the gain matrix.

The plant Equation (3), the control input limitation Equation (4), the modified Controller Equation (6), and the AWC Equation (7) are interconnected by the following relationship:

$$u_{\rm c} = y - y_{\rm aw}, \quad \mu = y_{\bar{\rm c},\rm d} + v, \quad \varphi_{\rm aw} = \varphi - y_{\bar{\rm c}}$$
(8)

From the interconnection diagram shown in Figure 3, by defining the coordinate $(x_{\ell}, x_{c}, x_{aw}, \varphi_{aw}) = (x - x_{aw}, x_{c}, x_{aw}, \varphi - y_{\bar{c}})$, after some derivations, we can obtain the equivalent expression of the whole closed-loop system as follows:

$$\begin{cases} \dot{x}_{\ell} = Ax_{\ell} + B_{u}y_{\bar{c}} + B_{d}d \\ y_{\ell} = C_{y}x_{\ell} + D_{yu}y_{\bar{c}} + D_{yd}d \\ z_{\ell} = C_{z}x_{\ell} + + D_{zu}y_{\bar{c}} + D_{zd}d \\ \dot{x}_{c} = A_{c}x_{c} + B_{c}y_{\ell} \\ y_{c} = C_{c}x_{c} + D_{c}y_{\ell} \end{cases}$$

$$\begin{cases} \dot{x}_{aw} = Ax_{aw} + B_{u} \left[sat_{m}(\varphi_{aw} + y_{\bar{c}}) - y_{\bar{c}}\right] \\ \dot{\varphi}_{aw} = sat_{r} \left(K_{aw} \left[\begin{array}{c} x_{aw} \\ \varphi_{aw} \end{array} \right] + y_{\bar{c},d} \right) - y_{\bar{c},d} \\ z_{aw} = C_{z}x_{aw} + D_{zu} \left[sat_{m}(\varphi_{aw} + y_{\bar{c}}) - y_{\bar{c}}\right] \end{cases}$$
(9b)

where $y_{\ell} = y - y_{aw}$; $z_{aw} = z - z_{\ell}$ denotes the mismatch between the desirable performance output z of the modified closed-loop system Equations (3), (4) and (6) and the actual performance output z_{ℓ} of the anti-windup closed-loop system Equations (3), (4), (6) and (7).

Theorem 1. Given the anti-windup closed-loop system Equations (3), (4), (6) and (7), if $x_{aw}(0) = 0$ and $\varphi(0) = y_{\bar{c}}(0)$, then the control input *u* of the plant never exceeds the magnitude and rate saturation bounds. Moreover, if the K_{aw} selection guarantees the asymptotic stability of the subsystem Equation (9b), then the following conclusions hold [31,32,36]:

- Given any response of the modified closed-loop system Equations (3), (4) and (6) such that $y_{\bar{c}} = sat_{m}(y_{\bar{c}})$ and $y_{\bar{c},d} = sat_{r}(y_{\bar{c},d})$ for all *t*, then $z_{\ell} = z$ for all *t*, namely, the response of the anti-windup closed-loop system coincides with the response of the modified closed-loop system;
- The origin of the anti-windup closed-loop system is asymptotically stable.

Remark 2. (1) Under the initial conditions: $x_{aw}(0) = 0$ and $\varphi(0) = y_{\bar{c}}(0)$, obviously, the control input *u* meets the magnitude and the rate constraints since they are prescribed by two saturation functions; (2) If the gain matrix K_{aw} keeps the subsystem Equation (9b) stable under the foregoing initial conditions, we know that the variables $x_{aw} = 0$, $\varphi_{aw} = 0$, and v = 0, thus $z_{aw} = C_z x_{aw} + D_{zu} [sat_m(\varphi_{aw} + y_{\bar{c}}) - y_{\bar{c}}] = D_{zu} [sat_m(y_{\bar{c}}) - y_{\bar{c}}] = 0$ with the given assumption $y_{\bar{c}} = sat_m(y_{\bar{c}})$ for all t, so $z_{\ell} = z$ for all t is obtained based on the definition $z_{aw} = z - z_{\ell}$. (3) Given that K_{aw} guarantees the asymptotic stability of subsystem Equation (9b), it can be known that $x_{aw} \to 0$, $\varphi_{aw} \to 0$, $v \to 0$, $y_{\bar{c}} \to sat_m(y_{\bar{c}})$, and $y_{\bar{c},d} \to sat_r(y_{\bar{c},d})$ from Equation (9b), then the magnitude and rate saturations of the control input are ignored, and the differentiator $s/(1 + \tau s)$ offsets the integrator 1/s in the control loop shown Figure 3, thus the asymptotic stability of the anti-windup closed-loop system is guaranteed by the original robust H_{∞} controller \bar{c} .

The proof of the above theorem is omitted in this paper, since it has been presented in the literature [31,32,36] in detail. According to the theorem, the key step for synthesizing the anti-windup controller is to design the gain matrix K_{aw} to keep the subsystem Equation (9b) stable. In terms of the recipe in [32], the gain matrix K_{aw} is selected to stabilize the following dynamic model:

$$\begin{bmatrix} \dot{x}_{aw} \\ \dot{\phi}_{aw} \end{bmatrix} = \left(\begin{bmatrix} A & B_{u} \\ 0 & 0 \end{bmatrix} + \begin{bmatrix} 0 \\ I \end{bmatrix} K_{aw} \right) \begin{bmatrix} x_{aw} \\ \phi_{aw} \end{bmatrix}$$
(10)

Here, the LQR method can be used to obtain K_{aw} . Obviously, K_{aw} stabilizing the dynamic model Equation (10) implies the asymptotic stability of Equation (9b), when y_c , $y_{c,d}$ and v are sufficiently small (not to cause the saturation nonlinearity).

Remark 3. From the control diagram shown in Figure 3, we can see that the AWC is inserted between the robust H_{∞} controller \bar{c} and the plant \mathcal{P} , thus the closed-loop system has a typical structure of the cascade control system. To obtain satisfied performances of the closed-loop system, it is required that the response time of the AWC in inner loop should be much shorter than that of the H_{∞} controller \bar{c} in outer loop. Thus, when solving the gain matrix K_{aw} based on the dynamic model Equation (10), we need to take the above requirement into account.

4. Case Study

To test the proposed LFC design method, firstly a typical two-area interconnected power system shown in Figure 4 is selected to make simulations. In the test system, each area is represented by a equivalent generation unit with a turbine and a governor. For simplicity, it is assumed that the two areas are identical, and the corresponding parameters are as [29]: $T_{g1} = T_{g2} = 0.08$ (s), $T_{t1} = T_{t2} = 0.3$ (s), $H_1 = H_2 = 0.1667$ (pu.s), $D_1 = D_2 = 0.0083$ (pu/Hz), $T_{12} = T_{21} = \gamma_1 =$ $\gamma_2 = 0.545$ (pu/Hz), $R_1 = R_2 = 2.4$ (Hz/pu), $\beta_1 = \beta_2 = 0.425$ (pu/Hz), rate constraint (pu/s): [-0.0017, 0.0017], magnitude constraint (pu): [-0.1, 0.1]. Then, based on the above parameters, the two-area system shown in Figure 4 is modified into a single-area system and two three-area systems in the following simulations.

In this section, the proposed method for LFC design is compared with the methods proposed by Tan [29] and Anwar [30] for three scenarios with different load disturbances.

4.1. Scenario 1: Simulations on Single-Area System

In Scenario 1, we set the same load disturbances in the two areas of the system shown in Figure 4. By this way, each area can be treated as a single-area system, since the two areas have the same structure and parameters.



Figure 4. Diagram of a two-area interconnected power system.

Towards the LFC model of the test system, by the proposed method, we design the anti-windup LFC consisting of a robust H_{∞} dynamic controller, a linear filter, a magnitude and rate saturation loop, and an AWC as follows:

The robust H_{∞} dynamic controller:

$$\bar{\mathcal{C}}': \begin{cases} \dot{x}_{\bar{c}} = A'_{\bar{c}}x_{\bar{c}} + B'_{\bar{c}}u_{\bar{c}} \\ y_{\bar{c}} = C'_{\bar{c}}x_{\bar{c}} + D'_{\bar{c}}u_{\bar{c}} \end{cases}$$
(11)

where:

$$A_{\bar{c}}' = \begin{bmatrix} -9.94374 & -2.32119 & 0.620889 & 4.096447 & 312.625 \\ 21.43163 & -15.2773 & -0.43206 & 12.18535 & 1229.041 \\ 117.987 & -167.214 & -881.295 & -93.2623 & -113.066 \\ 2211.94 & -3412.12 & -1198.68 & -15702.6 & 189.4382 \\ 759871.3 & -734389 & -395045 & 84813.64 & -157822 \end{bmatrix}$$

$$B_{\bar{c}}' = \begin{bmatrix} 146.091 & 153.625 & 1.605 \\ 894.647 & 414.317 & -405.444 \\ 8964.850 & -4553.067 & -240549.496 \\ 641800.483 & -1031816.318 & 31109.807 \\ 54307506.731 & 30731255.138 & 1104996.282 \end{bmatrix}$$

$$C_{\bar{c}}' = \begin{bmatrix} -0.027 & 0.024 & -0.006 & -0.041 & -3.200 \end{bmatrix}$$

$$D_{\bar{c}}' = \begin{bmatrix} -1.134 & -1.560 & -0.006 \end{bmatrix}$$

The linear filter (approximate differentiator):

$$s/(1+\tau s) = s/(1+0.01s)$$
(12)

The magnitude and rate bounds of the control input:

$$[\underline{m}, \overline{m}] = [-0.1, 0.1], \ [\underline{r}, \overline{r}] = [-0.0017, 0.0017]$$
(13)

The AWC:

where $K'_{aw} = [0.908 \ 3.929 \ 1.367 \ -3.777 \ 3.162 \ 6.646]$.

Based on the designed anti-windup LFC Equations (11)–(14), we make two tests: one is for step load decrease $\Delta P_{L1} = \Delta P_{L2} = -0.015$ (pu), the other is for step load increase $\Delta P_{L1} = \Delta P_{L2} =$ 0.01 (pu). The system responses to the load decrease and increase are shown in Figures 5 and 6, respectively. Considering that Area 1 and 2 have the same responses, we only illustrate the simulation results of Area 1. The concerned variables, such as frequency deviation Δf , ACE, tie-line power deviation ΔP_{tie} , control input u, and rate of control input du/dt, are shown in Figures 5 and 6.

From the results of both of the above tests, it is shown that, compared with the controllers presented by Tan and by Anwar, the proposed controller generates smaller overshoot and takes shorter settling time to force the frequency deviation and ACE to zeros. Furthermore, seeing from the control input curves, one can find that the control signal of the proposed controller in this paper meets the prescribed magnitude and rate constraint, while the control signals of Tan's and Anwar's controllers exceed the rate constraint. Here, it should be noted that ΔP_{tie} is always equal to zero since there is no tie-line power deviation between the two symmetrical areas.



Figure 5. Results for load decrease in a single-area system.



Figure 6. Results for load increase in a single-area system.

4.2. Scenario 2: Simulations on a Two-Area System

The anti-windup LFC in Scenario 2 is the same as the controller Equations (11)–(14) in Scenario 1, since the parameters of the area model are identical in both systems. In this scenario, different

load disturbances are set for two tests as: $\Delta P_{L1} = 0.01$ (pu) and $\Delta P_{L2} = 0.02$ (pu) for Test 1; $\Delta P_{L1} = 0.02$ (pu) and $\Delta P_{L2} = -0.01$ (pu) for Test 2. The simulations are performed on the two-area system directly, and the results are shown in Figures 7–10.



Figure 7. Results of Area 1 in a two-area system for Test 1.



Figure 8. Results of Area 2 in a two-area system for Test 1.







Figure 10. Results of Area 2 in a two-area system for Test 2.

From the results in Figures 7–10, it is observed that all the three controllers can drive the system frequency deviation, the ACE and the tie-line power deviation to zero, but the proposed anti-windup controller obtains more favorable performances than the other two controllers at aspect of overshoot and settling time. In addition, unlike Tan's method and Anwar's method, the proposed method avoids the undesired oscillation. Seeing the control input u, we also find that the proposed anti-windup controller can match the magnitude and rate constraint. Therefore, the system can operate in the linear region, and thus the nonlinear GRC is tackled.

By comparing the results in Scenario 1 and Scenario 2, it can be observed that the control performances of Scenario 1 are better than that of Scenario 2. The reason is that, in Scenario 1, the controller only needs to attenuate the local load disturbance, but, in Scenario 2, both the load disturbance in local area and the tie-line power disturbance from the neighboring area are needed to be restrained.

4.3. Scenario 3: Simulations on Three-Area Systems

In Scenario 3, the two-area system shown in Figure 4 is changed into two three-area interconnected systems. One is a chain-type system shown in Figure 11, the other is a delta-type system shown in Figure 12. For simplicity, we make the parameters of each area in the three-area systems be the same to those in the two-area system. In addition, the tie-line synchronizing coefficients are selected as: for the chain-type system, $T_{12} = T_{21} = T_{23} = T_{32} = 0.545$ (pu/Hz), $\gamma_1 = T_{12} = 0.545$ (pu/Hz), $\gamma_2 = T_{21} + T_{23} = 1.09$ (pu/Hz), $\gamma_3 = T_{32} = 0.545$ (pu/Hz); for the delta-type system, $T_{12} = T_{21} = T_{21} = T_{23} = T_{23} = T_{32} = T_{13} = T_{31} = 0.545$ (pu/Hz); for the delta-type system, $T_{12} = T_{21} + T_{23} = 1.09$ (pu/Hz), $\gamma_1 = T_{12} + T_{13} = 1.09$ (pu/Hz), $\gamma_2 = T_{21} + T_{23} = T_{31} = T_{31} = 0.545$ (pu/Hz), $\gamma_1 = T_{12} + T_{13} = 1.09$ (pu/Hz), $\gamma_2 = T_{21} + T_{23} = 1.09$ (pu/Hz), $\gamma_1 = T_{12} + T_{13} = 1.09$ (pu/Hz), $\gamma_2 = T_{21} + T_{23} = 1.09$ (pu/Hz), $\gamma_1 = T_{12} + T_{13} = 1.09$ (pu/Hz), $\gamma_2 = T_{21} + T_{23} = 1.09$ (pu/Hz), $\gamma_3 = T_{31} + T_{32} = 1.09$ (pu/Hz).



Figure 11. Diagram of a three-area chain-type interconnected power system.



Figure 12. Diagram of a three-area delta-type interconnected power system.

For Areas 1 and 3 of the chain-type system, their anti-windup LFC are the same as the one in Scenario 1, namely Equations (11)–(14), since these area models have identical parameters in Equation (3). For Area 2, however, the anti-windup LFC needs to be redesigned, because the parameter γ_2 in the matrix *A* in Equation (3) is not equal to γ_1 . Based on the given parameters, a robust H_{∞} dynamic controller and an AWC are designed for Area 2 as follows:

The robust H_{∞} dynamic controller:

$$\bar{\mathcal{C}}'': \begin{cases} \dot{x}_{\bar{c}} = A_{\bar{c}}'' x_{\bar{c}} + B_{\bar{c}}'' u_{\bar{c}} \\ y_{\bar{c}} = C_{\bar{c}}'' x_{\bar{c}} + D_{\bar{c}}'' u_{\bar{c}} \end{cases}$$
(15)

where:

$$A_{\tilde{c}}^{\prime\prime} = \begin{bmatrix} -9.357 & -2.267 & -2.343 & 5.018 & 535.792 \\ 18.070 & -16.938 & 0.246 & 5.616 & 1531.194 \\ 149.396 & 236.362 & -1083.630 & 183.093 & 381.424 \\ 233.018 & -2806.868 & 1859.898 & -7169.953 & -314.377 \\ 1014222.355 & -978906.090 & -830564.259 & 430071.155 & -251967.406 \end{bmatrix}$$
$$B_{\tilde{c}}^{\prime\prime} = \begin{bmatrix} 154.484 & 127.097 & 2.559 \\ 902.163 & -213.902 & 704.383 \\ -2720.065 & 11933.119 & -116445.141 \\ 222855.373 & -438613.914 & -30897.537 \\ 68477106.358 & 11506520.276 & 1200741.256 \end{bmatrix}$$
$$C_{\tilde{c}}^{\prime\prime} = \begin{bmatrix} -0.040 & 0.028 & 0.030 & -0.062 & -6.652 \\ D_{\tilde{c}}^{\prime\prime} = \begin{bmatrix} -1.532 & -1.569 & -0.002 \end{bmatrix}$$

The AWC:

$$\begin{bmatrix} \dot{x}_{aw} \\ \dot{\phi}_{aw} \end{bmatrix} = \left(\begin{bmatrix} A & B_{u} \\ 0 & 0 \end{bmatrix} + \begin{bmatrix} 0 \\ I \end{bmatrix} K''_{aw} \right) \begin{bmatrix} x_{aw} \\ \phi_{aw} \end{bmatrix}$$
(16)

where $K''_{aw} = [0.908 \ 3.929 \ 1.367 \ -3.777 \ 3.162 \ 6.646].$

Combining the above robust controller and AWC with the bounds of the control input and the linear filter presented in Scenario 1, we obtain the anti-windup LFC of Area 2, namely, Equations (12), (13), (15) and (16).

In summary, for the three-area chain-type system, the anti-windup LFCs are described as: Equations (11)–(14) for Area 1 and 3; Equations (12), (13), (15) and (16) for Area 2. For the three-area delta-type system, the anti-windup LFCs are expressed as Equations (12), (13), (15) and (16) for Area 1, 2, and 3, since the three areas in the delta-type system have the same parameters as Area 2 in the chain-type system.

In Scenario 3, the load disturbances are set as: in the chain-type system, $P_{L1} = 0.01$ (pu) in Area 1, $P_{L2} = -0.01$ (pu) in Area 2, and $P_{L3} = -0.01$ (pu) in Area 3; in the delta-type system, $P_{L1} = 0.02$ (pu) in Area 1, $P_{L2} = 0.01$ (pu) in Area 2, and $P_{L3} = -0.01$ (pu) in Area 3.

The results are shown in Figures 13–15 for the chain-type system and Figures 16–18 for the delta-type system. The simulations reveal that the proposed method can restrain the load disturbances, regulate the frequency of each area, and restore the tie-line power to its scheduled value. In other words, the proposed method can realize the LFC objectives of multi-area interconnected power system. Compared with Tan's method and Anwar's method, the method in this paper obtains better performances in overshoot and settling time. The magnitude and rate of the input signals are especially controlled in the predetermined ranges by the proposed anti-windup controller, which helps to reduce the wear and tear of generators and improve the stability of the closed-loop system.



Figure 13. Results of Area 1 in a three-area chain-type system.



Figure 14. Results of Area 2 in a three-area chain-type system.



Figure 15. Results of Area 3 in a three-area chain-type system.



Figure 16. Results of Area 1 in a three-area delta-type system.



Figure 17. Results of Area 2 in a three-area delta-type system.



Figure 18. Results of Area 3 in a three-area delta-type system.

5. Conclusions

Towards the GRC problem in the LFC, this paper proposes an anti-windup controller design method. In the anti-windup LFC, the H_{∞} dynamic controller is designed to guarantee robust performance against load disturbances and tie-line power disturbances, and the AWC is used to restrict the magnitude and rate of the control input so that the system can operate in the linear region to overcome the GRC. The simulation results show that the proposed anti-windup LFC design method effectively improves the performances against disturbances and GRC. Further work will focus on the coordination method of different LFCs to improve the overall performances of the multi-area interconnected power system.

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