

Article



Influence of the Preformed Coil Design on the Thermal Behavior of Electric Traction Machines

Benedikt Groschup * D, Florian Pauli D and Kay Hameyer

Institute of Electrical Machines (IEM), RWTH Aachen University, Schinkelstraße 4, 52062 Aachen, Germany; florian.pauli@iem.rwth-aachen.de (F.P.); kay.hameyer@iem.rwth-aachen.de (K.H.) * Correspondence: benedikt.groschup@iem.rwth-aachen.de

Abstract: Preformed coils are used in electrical machines to improve the copper slot fill factor. A higher utilization of the machine can be realized. The improvement is a result of both, low copper losses due to the increased slot fill factor and an improved heat transition out of the slot. In this study, the influence of these two aspects on the operational improvement of the machine is studied. Detailed simulation models allow a separation of the two effects. A preform wound winding in comparison to a round wire winding is studied. Full machine prototypes as well as motorettes of the two designs are built up. Thermal finite element models of the stator slot are developed and parameterized with the help of motorette microsections. The resulting thermal lumped parameter model is enlarged to represent the entire electric machine. Electromagnetic finite element models for loss calculation and the thermal lumped parameter models are parameterized using test bench measurements. The developed models show very good agreement in comparison to the test bench evaluation. The study indicates that both, the improvements in the heat transition path and the advantages of the reduced losses in the slot contribute to the improved operational range in dependency of the studied operational point.

Keywords: electrical machines; thermal modeling; preformed coils; insulation systems; pre-shaped conductor

1. Introduction

To further increase the thermal operational limits of high torque low speed electrical machines, high slot fill factors are required. The theoretical maximum copper fill factor using round-wire-windings without insulation is given by the value of an orthocyclic winding with 90.7%. This theoretical value is reduced by the wire insulation and a nonperfect arrangement of the conductors within the notch. Several studies focus to approach high slot fill factors with round wires. Authors in [1] use innovative needle winding and indicate realistic copper fill factors of 65% in the machine. Rectangular conductor designs are a well-known approach to increase the copper slot fill factor in comparison to a round wire design. A state-of-the-art approach is using parallel slot designs with rectangular conductors of the same size in a so called hairpin winding design [2–7]. Only few studies focus on parallel teeth such as [3] where rectangular conductors of the same size are inserted into the notch. Studies with rectangular conductors arranged in a parallel tooth design are more seldom to find [8]. A widely studied field is the loss calculation of ac-copper losses within the winding in two dimensional [5] or three dimensional [4] electromagnetic simulations. Comparison of round wire designs and rectangular wire designs can be found in [2,5]. The authors of both publications focus on the loss comparison of the two designs. Thermal aspects are not studied in detail, even if high slot fill factors cause different thermal resistances within the notch. Investigations focusing on thermal aspects of machines with rectangular wire regularly study different innovative cooling concepts. Direct cooling concepts of the conductors within the notch [7,9] or the end winding [6] can be found. In these studies, authors usually focus on the improved heat extraction of the cooling designs.



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Copyright: © 2021 by the authors. Licensee MDPI, Basel, Switzerland. This article is an open access article distributed under the terms and conditions of the Creative Commons Attribution (CC BY) license (https://creativecommons.org/licenses/by/4.0/). In order to avoid the disadvantage of parallel slots, semi finished coils are used in this study and compressed to fit the geometry of a parallel tooth stator slot. The production process of such coils is described in [10,11].

The semi finished coil has larger radial dimensions than the final tooth of the machine as depicted in Figure 1a. This coil is compressed in a specially designed fixture as described in [10] to match the final size of the slot as shown in Figure 1b. The coil is then impregnated. The final product before inserting into the stator tooth is shown in Figure 1c.

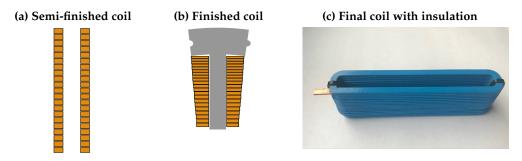


Figure 1. Overview of preformed coil: (**a**) Semi finished coil. (**b**) Finished coil in the stator tooth. (**c**) Images of the final coil with insulation.

Within this study, two different machines are compared to each other. One machine is equipped with classical round wires, the second is equipped with the newly developed preformed coils. In order to have comparable results, the iron core and the entire machine design excluding the coil are kept identical. The machines are designed as directly driven wheel hub drives in an automotive application without using a gearbox. The design requires a relatively low maximum speed of 1200 rpm. The machines are designed with 20 pole pairs, 60 stator slots and a v-shaped permanent magnet arrangement such as depicted in Figure 2.

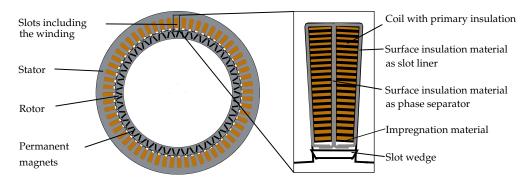


Figure 2. Axial view of the electromagnetic active motor parts and detailed axial view of the slot of the proposed preformed coil design.

A single-tooth winding pattern is used to enable short end windings. The used round wire is the film grade one wire Synwire W210 from SynFlex with a outer diameter of 0.95 mm. The used rectangular wire for the preformed coil has the outer dimensions of $3.4 \text{ mm} \times 1.4 \text{ mm}$ with a corner radius of 0.5 mm. The newly developed technology shows influence on three different categories:

- Insulation system,
- thermal behavior, and
- electromagnetic behavior.

A detailed study of the **insulation system** is given in [8]. The insulation system of the round wire winding is designed to hold a temperature of $180 \degree C$ (thermal class H) while the interturn insulation of the formed coil in its current setup can hold $120 \degree C$ (thermal class E).

An insulation system for the formed coils, which meets the requirements of thermal class H, is however under development.

This study focuses on the enhancement of torque and power density of the drive by analyzing the **thermal behavior**. Microsections of the designs are evaluated to identify the geometrical arrangement of all components within the stator slot, i.e., surface insulation, impregnation, interturn insulation or conductors. The results are used to develop detailed two-dimensional finite element analysis (FEA) models of the stator slot. Lumped parameter thermal network (LPTN) models such as introduced in [12] or [13] are developed for each machine. The FEA models are used to parameterize the thermal resistances of the LPTN models. As all components of the two machines are kept constant between the two machine designs except the winding, only these resistances are adjusted in the LPTN models. The local temperature distribution is analyzed in detail. The LPTN models are validated by thermal transient test bench measurements so that measured temperatures on the test bench, simulated temperatures of the machine LPTN models, and two-dimensional thermal FEA models are coincident in their results. The developed model is used to determine continuous power (S1) for different maximum permitted winding temperatures. The results of this evaluation are validated by test bench measurements of the continuous power (S1).

From a thermal perspective, the **electromagnetic behavior** can be specified by the amount of copper and iron losses that are produced at a specific operational point of the machine, i.e., at defined torque and speed. The approach of substituting the machine winding, which is chosen in the present publication, leaves the iron losses unchanged. The increased copper fill factor leads to a reduction of the copper losses of the machine. From the application of hairpin windings, it is known that large conductor cross sections lead to more pronounced skin- and proximity effects at high speeds [14], which diminishes the advantage of an increased copper fill factor. Electromagnetic finite element simulations and test bench trials are performed to investigate this influence. The efficiency of both machines is simulated and measured for the entire torque-speed range.

2. Simulation Methodology

2.1. Electromagnetic Simulation

The electromagnetic characteristics of both machines are obtained by applying electromagnetic 2D finite element simulations. The torque and the magnetic flux distribution are calcualated in the d-q current reference frame. An exemplary distribution of the magnetic flux density *B* is given in Figure 3. An exemplary operating point with $\hat{I}_d = -105$ A and $\hat{I}_q = 320$ A is used. The local distribution of the losses within the device under test are obtained by applying this data to the post processing process.

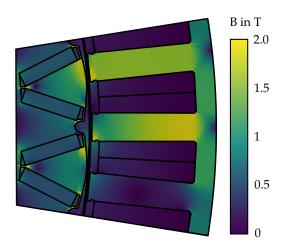


Figure 3. Flux distribution inside the PMSM at an examplary operating point $\hat{l}_d = -105 \text{ A}$ and $\hat{l}_q = 320 \text{ A}$.

The torque-speed map is obtained by applying a maximum torque per ampere (MTPA) control strategy in the base speed region. The copper loss calculation is initially introduced in [8]. Skin and proximity effects are not considered for the machine with conventional round wires due to the small conductor cross section. The copper losses are calculated according to:

$$P_{\text{wind,round}} = \frac{3}{2} \cdot R_{1-1,\text{DC,m,round}} \cdot I^2 , \qquad (1)$$

where *I* is the amplitude of the phase current and $R_{1-1,DC,m,round}$ is the line-to-line resistance. For the preformed coil, skin and proximmity effects are taken into consideration. Therefore, a transient 2D simulation is performed and the ac resistance $R_{1-1,AC,s,pref}(T, n)$ is obtained as a function of torque and speed. As this calculated resistance only considers the part of the coil, which is placed inside the slots, a factor based on the measured dc resistance $R_{1-1,DC,m,pref}$ and the simulated dc resistance $R_{1-1,DC,s,pref}$ is introduced:

$$f_R = \frac{R_{\rm l-l,DC,m,pref}}{R_{\rm l-l,DC,s,pref}}.$$
(2)

The copper losses as a function of torque and speed are calculated by multiplying this factor by the simulated ac resistance:

$$P_{\text{wind,pref,s}} = \frac{3}{2} f_R \cdot R_{1-1,\text{AC,s}}(T,n) \cdot I^2.$$
(3)

Thus, the resistance of the end winding can also be considered. The specific iron losses p_{Fe} are calculated according to the IEM-five-parameter-formula which is introduced in [15–17]:

$$p_{\rm Fe} = a_1 B^{\alpha} f + a_2 B^2 f^2 (1 + a_3 B^{a_4}) + a_5 B^{1.5} f^{1.5} , \qquad (4)$$

where *B* is the magnetic flux density, *f* is the magnetization frequency. α as well as a_1 are material dependent parameters that define hysteresis losses. The eddy current losses are defined by a_2 , excess losses by a_4 . The remaining parameters a_3 , a_5 specify the amount of non-linear losses in the application. The iron losses P_{Fe} of the drive can be derived by an integration of the specific iron losses p_{Fe} over the mass *dm*:

$$P_{\rm Fe} = \int_m p_{\rm Fe} \, dm \,. \tag{5}$$

The loss data to parametrize Equation (4) is taken from the testbench measurements. As only the total losses $P_{\text{loss,tot,m}}$ can be measured, the simulated copper losses $P_{\text{wind,s}}$ are substracted to obtain the ironlosses P_{Fe} :

$$P_{\rm Fe} = P_{\rm tot,m} - P_{\rm wind,s} \,. \tag{6}$$

Friction losses are not considered. As both machines are identical except from the winding, the same set of parameters is used to calculate the iron losses. An overview of the resulting losses for each studied operational point (OP) of this parameterization process can be found in Table 1.

OP -	Т	n	P _{Fe,sta,to}	P _{Fe,sta,yo}	P _{Fe,rot,emp}	P _{Fe,rot,yo}	Pwind,round	P _{wind,pref}
01 -	in Nm	in rpm	in W	in W	in W	in W	in W	in W
1	650	50	32	14	9	2	2699	2277
2	0	800	507	217	155	39	0	0
3	400	800	515	222	152	38	905	894
4	300	1200	461	198	205	51	1275	1440

Table 1. Overview of losses for studied operational points.

OP1 is selected at low speed *n* and high torque *T*. Due to the low magnetization frequency *f* in the soft magnetic material, OP1 shows low iron losses in the stator tooth $P_{\text{Fe,sta,to}}$, the stator yoke $P_{\text{Fe,rot,emp}}$, and the rotor yoke $P_{\text{Fe,rot,yo}}$. The high torque demand in the base speed region leads to high currents in the winding and a high share of copper losses. The round wire design with $P_{\text{wind,round}} = 2699 \text{ W}$ shows increased copper losses by 18.5% in comparison to the preformed coil design with $P_{\text{wind,round}} = 2277 \text{ W}$. OP2 and OP3 are operational points, both placed at a rotational speed *n* of 800 rpm. OP2 has no load, is placed in the base speed region, and thus has no copper losses P_{wind} . OP4 is the operational point with the highest studied speed of n = 1200 rpm. It is interesting to note that the round wire design shows significantly higher winding losses for OP1 (+18.5%), but lower winding losses for OP4 (-11.5%) due to the influence of proximity and skin effects. This difference needs to be kept in mind for the later comparison of the temperature curves. It is interesting to see weather this disadvantage of the preformed coil design is outperformed by the expected improved heat path in the later study.

2.2. Thermal Lumped Parameter Model

For the representation of the thermal behavior of both machines, LPTN models are developed. An assignment of the most important thermal nodes within the network model to the existing geometry is give in Figure 4 using the preformed coil design. The figure includes a longitudinal and a transverse section of the motor as shown in Figure 4a,b respectively.

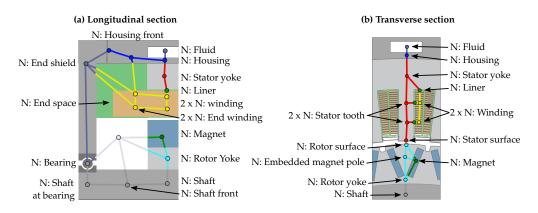


Figure 4. Location of the nodes of the LPTN model: (**a**) Longitudinal section of the geometry. (**b**) Transverse section of the geometry.

The machine is equipped with a water based cooling channel within the housing of the machine. The fluid has a volume flow rate of 7.5 Lmin^{-1} and a heat transfer coefficient of $h_{\text{ho-fluid}} = 4011 \text{ Wm}^{-2} \text{ K}^{-1}$. The thermal resistance between the housing and the fluid $R_{\text{fluid,ho}}$ is calculated to be 1.2 mW K⁻¹. A more detailed description of the LPTN model with the most important nodes and resistances being labeled, is shown in Figure 5.

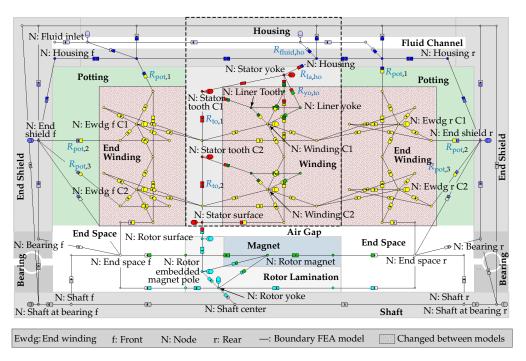


Figure 5. Detailed overview of entiere LPTN model.

Different areas are highlighted in Figure 5. The middle section represents the electromagnetic active length and is the portion that is later modeled in the two-dimensional finite element model. The dotted area marks the thermal resistances and capacities that are changed between the round wire and the preformed coil model. All other parameters of the model are kept constant. The end winding of both machines is potted using a mixture of a mineral-filled epoxy resin Araldit CW 5730 N and a hardener HY 5731 by Bodo Möller Chemie GmbH. Due to the potting, the end winding nodes are directly linked to the end shield and the housing front node of the model. The model is designed symmetrically on both sides of the end winding. The radial connection resistance $R_{\text{pot},1}$ is calculated to be 690 mK W⁻¹. The two axial connection resistances $R_{\text{pot},2}$ and $R_{\text{pot},3}$ are calculated to be 1160 mK W⁻¹. The interface resistance between the housing and the stator lamination $R_{\text{la,ho}}$ shows significant influence on the temperature distribution within the machine. Conductive resistances are calculated using the thickness of the conductive layer *d*, the area of the conductive layer *A* and its thermal conductivity *k*

$$R = \frac{d}{A \cdot k} \,. \tag{7}$$

The resistance $R_{la,ho,1}$ is a sum of the conductive share of the housing $R_{la,ho,1}$, the interface $R_{la,ho,2}$, and the conductive share of the lamination $R_{la,ho,3}$. The second resistance with $R_{la,ho,2} = 3.3 \text{ mK W}^{-1}$ has the largest share of $R_{la,ho} = 4.7 \text{ mK W}^{-1}$ and is frequently discussed in literature. Thus it is described here in detail. The resistance $R_{la,ho,2}$ is influenced by several parameters, such as the surface roughness, the contact pressure between the components, or the material hardness [18]. It can be modeled by an equivalent interface air gap d_{ig} between the components:

$$R_{\rm la,ho} = \frac{d_{\rm ig}}{A_{\rm ig} \cdot k_{\rm air}} \,. \tag{8}$$

The thermal conductivity of air is assumed to be $k_{air} = 31.7 \text{ mW m}^{-1} \text{ K}^{-1}$ and the area of the interface $A_{ig} = 0.095 \text{ m}^2$. Typical literature values for the interface thickness d_{ig} are between 10 µm and 77 µm [18–20] with one outlier in [20] of 95 µm. This range is used for the later described parameter identification process using test bench measurements.

The final value is identified to be 10 µm and thus placed at the lower level of identification range. Some other important resistances in the LPTN are the conductive resistances within the stator lamination. For the stator and rotor lamination, a thermal conductivity of $28 \text{ W m}^{-1} \text{ K}^{-1} \pm 10\%$ is used for the parameterization. The final value is identified to be $30 \text{ W m}^{-1} \text{ K}^{-1}$ leading to a thermal resistance between the stator yoke and the stator tooth of $R_{\text{yo,to}} = 7.3 \text{ mK W}^{-1}$ for the outer part of the tooth of $R_{\text{to},1} = 12 \text{ mK W}^{-1}$ and the inner part of the tooth of $R_{\text{to},2} = 12 \text{ mK W}^{-1}$. The winding and the stator tooth of the model is represented by two radially placed cuiboidal elements as introduced in [21]. The two radially aligned cuboidal elements of the active length are labeled with C1 for the radially outside and C2 for the radially inside element. The two cuboidal elements in the active section are supplemented by two elements for each, the front and the rear section of the motor, i.e., N Ewdg f and N Ewdg r.

The losses according to Table 1 are used for the model. The iron losses of the stator tooth $P_{\text{Fe,sta,to}}$ are inserted into the stator tooth nodes labeled with C1 and C2. $P_{\text{Fe,sta,yo}}$ is applied to stator yoke node. The iron losses $P_{\text{Fe,rot,emp}}$ and $P_{\text{Fe,rot,yo}}$ are applied to the rotor embedded magnet pole node and the rotor yoke respectively. The copper losses $P_{\text{wind,round}}$ and $P_{\text{wind,pref}}$ are distributed between the end winding section and the winding section based on the estimated length of the coil

$$P_{\text{wind},i} = P_{\text{wind}} \cdot \frac{l_i}{l_i + l_{\text{ewdg}}}, \qquad (9)$$

with the active length $l_i = 80$ mm and the length of one end winding turn $l_{ewdg} = 34$ mm.

2.3. Thermal Finite Element Analysis

In order to achieve a well suited network model for the winding section, thermal two-dimensional finite element analysis (FEA) of the slot are performed. The adiabatic boundary of the FEA model is highlighted in the LPTN model description in Figure 5. The LPTN is run with cut resistances along this boundary for comparison and parameterization. The determination of the exact positioning of the components within the slot as well as the penetration of the impregnation material into the cavities is crucial for an exact representation of the slot. Partial stator models that are referred to as motorettes are build up for this purpose. They are first used for the accelerated aging of the winding as described in [8]. After this process, microsections are processed as depicted in Figure 6. A Leica S9i microscope is used for the measurement of the slot of the preformed coil are shown in Figure 6. The results are used for the development of the FEA model as depicted in Figure 7.



Figure 6. Microsection of a partial motor model of the preformed coil design.

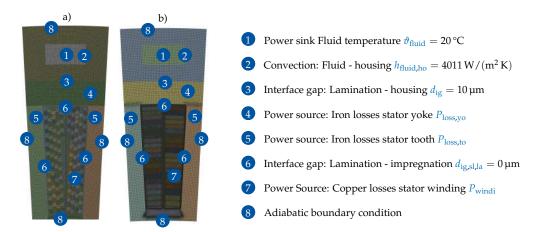


Figure 7. Comparison of mesh settings and boundary conditions for the thermal FEA models. Round wire design (**a**) and preformed coil design (**b**).

No major air pockets can be found in the potting in Figure 6 and thus, no airgap between the lamination stack and the slot liner is modeled (see $d_{ig,sl,la} = 0 \,\mu m$ in Figure 7). Four layers of the phase-to-ground insulation material Nomex Kapton Nomex (NKN) can be found between the two coils in image 1 of Figure 6, i.e., two overlying layers for each coil. The well visible contour is the capton layer. The NKN has a datasheet thickness of $d_{\rm sl} = 170 \,\mu{\rm m}$. This thickness can be confirmed in section 4 of Figure 6. With a measured thickness in section 1 of Figure 6 for four layers of 683 µm, it is obvious that no additional layer of impregnation needs to be inserted in the FEA model between the four layers in the coil divider area. The slot liner is modeled with isotropic conductivity. The value is varied by $\pm 20\%$ between the value of $0.15 \,\mathrm{W \,m^{-1} \,K^{-1}}$ and is identified to be $0.18 \,\mathrm{W \,m^{-1} \,K^{-1}}$ during the parameterization process using test bench measurements as described later in this study. The wire insulation thickness varies significantly in dependency of the location. The maximum measured value is 119 µm as found in section 1 of Figure 6. The minimum value is highlighted by a circle in section 4. No significant remaining wire insulation can be found at this position. The sharp edge is a parasitic effect of the forming process. This effect can lead to a local error-prone wire insulation in combination with the used spray coating process. Values for two layers of wire insulation between the conductors of one coil between 90 µm and 120 µm are measured (see Section 4). This indicates a thickness between 45 µm and 60 µm for the wire insulation thickness. The cross sectional area of one conductor is known with 4.69 mm². The wire insulation thickness with given cross sectional area of the conductor and given slot shape influences the position of the last conductor as measured in section 3 of Figure 6. In order to meet measured distances, an uniformly distributed wire insulation thickness of 52.5 µm is selected for the FEA model.

A heat sink representing the cooling fluid is added to the model with $\vartheta_{\text{fluid}} = 20 \,^{\circ}\text{C}$. The heat transfer coefficient of $h_{\text{fluid,ho}} = 4011 \,\text{W} \,\text{m}^{-2} \,\text{K}^{-1}$ and the interface gap between the stator lamination and the housing with $d_{\text{ig}} = 10 \,\mu\text{m}$ is selected in accordance to the values of the LPTN. The model is surrounded by adiabatic boundary conditions.

3. Measurement Setup

Test bench measurements are performed for the validation of the study as introduced in [8] and depicted in Figure 8.

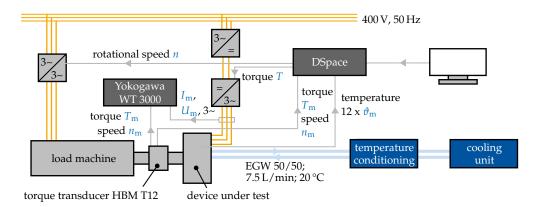


Figure 8. Schematic overview of the test bench setup.

The control system is designed in Matlab-Simulink using d-q lookup tables. The control system is flashed on a dSPace DS1006 system. A dc-link voltage of 300 V for the device under test is used. The tested machine is conditioned with 7.5 L min⁻¹ and 20 °C. The two machines are mechanically connected without a gearbox. A HBM T12 torque transducer is placed in the connecting shaft measuring the torque T_m and the rotational speed n_m of the setup. The currents I_m are measured by LEM sensors. The voltages U_m are measured by a voltage transducer. The measured values are transmitted to a Yokogawa-WT-3000 power analyzer for efficiency evaluation. Temperatures are measured by twelve thermocouples type J per motor. The locations of the sensors are shown in Figure 9. Two sensors for each phase are placed in the end winding region (A: Ewdg). The temperature between the housing and the stator lamination is measured by two sensors, placed in a grove of the stator teeth (B: Lam-Hous). The slot center of the machine is equipped with two sensors (C: Slot Center). The last two sensors are placed in the potting region between the end winding and the housing (D: Potting).

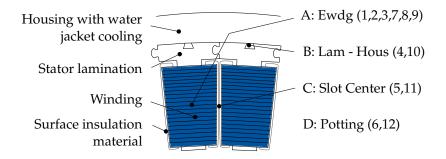


Figure 9. Positioning of the thermocouples type J within the machine.

4. Results

4.1. Comparison of Simulated and Measured Efficiency and Losses

A detailed description of the measurement preparation including results of induced voltage measurements as well as drag torque measurements can be found in [8]. This study also includes a detailed comparison of measured and simulated efficiency. As the losses in the machine are crucial for the thermal modeling, a short review is given. The efficiency of the machines is evaluated using grid points which are placed at steps of 50 N m and 50 rpm. A comparison of the efficiency between measured and simulated losses is given in [8]. The measured and simulated efficiencies fit well in the entire torque speed map. The measured maximum efficiency of the round wire design with 95.1% is similar to the measured efficiency of the preformed coil design in comparison to the round design in the base speed region. Similar behavior was already observed for the simulated losses as shown in Table 1. For high magnetization frequencies f, the advantage of the round to the measure of the simulated store is the preformed coil design in the base of the simulated losses as shown in Table 1.

preformed coil is outperformed due to the increased skin and proximity effects occuring in the winding.

4.2. Influence of Winding Design on Measured Transient Temperature Curves

The parameterization of the models is performed in an iterative process. The simulated losses are used for the parameterization. The thermal 2D-FEA model is adjusted by the parameters that are already introduced in the pervious sections. The maximum and the average temperature ($\vartheta_{wind,max}$ and $\vartheta_{wind,avg}$) of OP1 are calculated and the LPTN model is calibrated to fit these temperatures. The maximum temperature is chosen, because it is crucial for the lifetime and the operational range. The average temperature is chosen, as copper losses of the LPTN model P_{wind} are scaled with average temperature $\vartheta_{\text{wind,avg}}$. OP1 is selected, as the highest winding temperature is expected in this operational point. After calibrating the FEA with the simplified two-dimensional LTPN, the LPTN is transformed to represent the three-dimensional machine. Transient simulations of this model are performed and compared to the test bench measurements. The parameters are changed until measurements, 3D-LPTN as well as 2D-LPTN and 2D-FEA coincide. A comparison of the steady state temperatures for the four operational points between 2D-FEA and 2D-LPTN is given in Table 2. Due to the usage of operational point OP1 for the parameterization, this point fits with almost no deviation. The other operational points all fit well with a deviation of less than 5.5 °C. It is interesting to note that the LPTN model seems to over-predict the temperatures for the operational points with increasing speed and a larger share of iron losses. As the cuboidal model of the winding in the LPTN is a low-discretizised model in comparison to the FEA some deviations seem to be inavoidable.

	Maximum Temperature ϑ_{\max}				Average Temperature ϑ_{avg}			
	OP1	OP2	OP3	OP4	OP1	OP2	OP3	OP4
round wire design: $\vartheta_{\text{LPTN}-2\text{D}} - \vartheta_{\text{FEA}-2\text{D}}$	0.4	3.1	3.5	5.4	0.3	3.1	3.1	4.5
preformed coil design: $\vartheta_{\text{LPTN}-2D} - \vartheta_{\text{FEA}-2D}$	-0.4	3.1	3.3	4.9	0.0	3.3	3.1	4.4

Table 2. Comparison of Steady State Temperatures of Two-Dimensional Thermal LPTN and FEA in $^{\circ}$ C.

A comparison of the transient temperature curves of the two machines for all operational points as introduced in Table 1 is given in Figure 10.

A very good fit of 3D-LPTN-simulated and the measured temperatures is achieved. A deviation of not more than 6 K is observed. The previously identified trend regarding the exact representation of the winding temperature of the LPTN model can be confirmed. The LPTN model seems to overpredict temperatures at OP4 and tends to under-predict temperatures at OP1. As an example, the end winding temperature in OP1 for the round machine is underpredicted by the LPTN model by 5.2 K, whereas the end winding temperature of the preformed coil design in OP4 is overpredicted by 4.9 K. The fit of measured and simulated temperatures was possible under the following boundary conditions:

- Only the input of the copper losses *P*_{wind} is changed between the models of the round wire and the preformed coil as shown in Table 1. All other losses remain constant.
- Only the resistances labeled within the dotted area in Figure 5 are changed between the models.
- The resistances labeled within the dotted area in Figure 5 are changed so that the temperatures between 2D-FEA and 2D-LPTN coincide according to Table 2. The material properties of the impregnation, slot liner and the slot shape are the same in the 2D-FEA and the 2D LPTN.

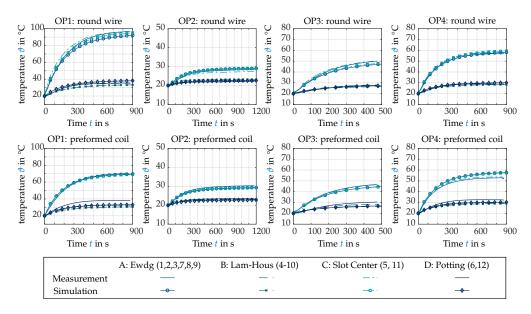


Figure 10. Comparison between LPTN simulation models and test bench measurement data.

It is interesting to compare the temperature curves for the slot center and the end windings for the different operational points. Due to the short end winding, temperatures in the electromagnetic active part of the machine are very similar to the curves of the end winding for each operational point. In OP1, the maximum winding temperature of the round design of around 95 °C is approximately 25 °C higher than the winding temperature of the preformed coil with around 70 °C. The copper losses of the round wire machine are 18.5% higher than those of the preformed coil design. The maximum winding temperature for OP4 is also lower for the preformed coil design by some °C, even if the input losses of the round design are 11.5% lower than those of the preformed coil design. The advantages of the improved heat dissipation paths of the preformed coil design outperform the disadvantage of the increased losses. Following observations can be concluded:

- 1. The 2D-LPTN and the 2D-FEA show good agreement with a maximum temperature deviation of 5.4 K.
- 2. The 3D-LPTN and the transient test bench measurements show a very good agreement with a maximum deviation of 6 K.
- 3. A parameterization of all models was possible under consideration of several boundary conditions that ensure a high degree of comparability.
- 4. The maximum winding temperature of the preformed coil design at the transient operation of OP1 for 15 min is approximately 25 °C lower than the winding temperature of the round wire design.
- 5. The disadvantage of the increased winding losses of the preformed coil design at an operational point with maximum speed (OP4: 15 min) is outperformed by the improved heat transfer path.

4.3. Comparison of the Influence of Reduced Losses and Improved Heat Extraction Path

Some more detailed investigations are performed to separate the two different influences on the maximum temperature of the winding, i.e., the reduction of the losses by the preformed coil design in the base speed region and the improved heat transfer path. The boundary conditions of OP1 are used for this investigation. Three different temperature distributions within the slot are depicted in Figure 11.

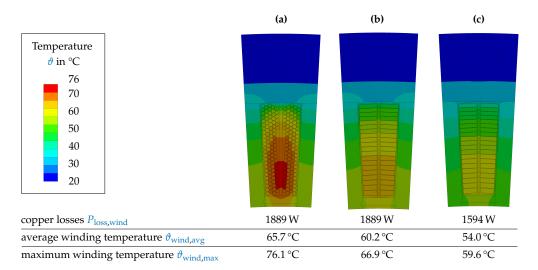


Figure 11. Comparison of temperature distribution in the slot using two-dimensional thermal FEA: (a) Round wire design and high copper losses. (b) Preformed coil design with high copper losses. (c) Preformed coil design with low copper losses.

The first setup (a) uses the geometry and the winding losses of the round design according to Table 1 with the conversion of Equation (9). The second setup (b) is an imaginary setup, where the geometry is changed to the settings of the preformed coil, but the high losses of the round wire are used. The third setup (c) is the real design of the preformed coil with the losses of the preformed coil. The average temperature ϑ_{avg} as well as the maximum temperature in the slot ϑ_{max} are evaluated for the investigation. Two effects can be determined:

- 1. Influence on the average temperature ϑ_{avg} in the slot: The improved heat path (from a to b) shows a reduction of 5.5 K and the reduced losses (from b to c) of 6.2 K. The reduced losses have a slightly higher potential of improvement.
- 2. Influence on the maximum temperature ϑ_{max} in the slot: The improved heat transfer path (from a to b) reduces the maximum temperature by 9.2 K, the reduced losses (from b to c) by 7.3 K. The improved heat path has a more significant influence on the maximum temperature.

The second described influence is a result of the good thermal heat dissipation from the center of the slot to the side. While setup (a) shows iso-temperature lines with an elliptical shape within the slot, (b) and (c) only show horizontal iso-temperature lines in the slot. The preformed coil design enables a more uniform temperature distribution within the slot.

4.4. Influence of Winding Design on the Operational Limits of the Machine

Simulations as well as test bench measurements are performed to investigate the influence of the winding design on the continuous operational limit S1. In the simulation approach, the validated LPTN models are used. The losses from the electromagnetic simulated torque-speed map are used as heat sources in the LPTN model. Steady state simulations are performed in a grid of 50 Nm torque and 40 rpm rotational speed steps. The maximum winding temperature $\vartheta_{S1,s}$ of the steady state simulation is evaluated. The resulting maximum temperatures are plotted in Figure 12. For the measurement approach, the rotational speed beginning from 100 rpm is investigated in steps of 100 rpm. Each rotational speed is set by the load machine on the test bench. The torque demand for the device under test is increased until a maximum steady state temperature of the hottest sensor in the winding of 120 °C is reached. The operational point is held for a time of t = 30 min. Within this period of time, a maximum deviation of the measured temperatures of ± 2 K is permitted. The measurement curve is added to the overview in Figure 12.

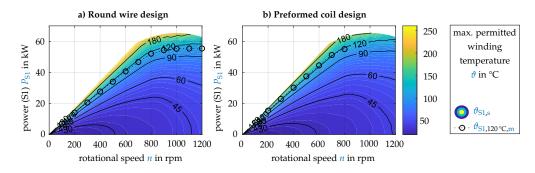


Figure 12. Comparison of continuous operational power (S1) between simulation and measurement: (a) Round wire design (b), Preformed coil design.

The process is repeated for rotational speeds up to 1200 rpm for the round wire design and up to 800 rpm for the preformed coil design. A heat-up of the winding up to 120 °C for speeds larger than 800 rpm is not possible for the preformed coil design with the given lookup tables and restrictions of the test bench setup. The curves of the possible S1-power P_{S1} for a maximum permitted temperature of 120 °C show a good agreement between measurement and simulation. Following observations can be made:

- 1. For the round wire design, the simulated S1-power $P_{S1,120 \circ C,round,s} = 53.5 \text{ kW}$ is 2.7% larger than the measured S1-power of $P_{S1,120 \circ C,round,m} = 52.1 \text{ kW}$ at n = 800 rpm.
- 2. For the preformed coil design, the simulated S1-power of $P_{S1,120 \circ C, \text{pref}, \text{s}} = 55.9 \text{ kW}$ is 1.6% larger than the measured S1-power of $P_{S1,120 \circ C, \text{pref}, \text{m}} = 55.0 \text{ kW}$ at n = 800 rpm.
- 3. According to the measurement, the S1-power $P_{S1,120 \circ C,m}$ of the preformed coil design at n = 800 rpm is increased by 4.5% in comparison to the round wire design.
- 4. According to the simulation, the S1-power $P_{S1,120 \circ C,s}$ of the preformed coil design at n = 800 rpm is increased by 5.3% in comparison to the round wire design.
- 5. Be aware that the given results of the S1-power are calculated based on the temperature limitations caused by the stator winding. Restrictions for the temperature of the permanent magnets in the rotor are likely to occure for high rotational speed [12].

A more significant deviation between measurement and simulation can be observed at maximum speed for the round wire design. At 1200 rpm, the simulation gives a 5.3% higher S1-power than the measurement. One reason for this deviation is the difference between the magnet temperatures between measurement and simulation. The torque of the device under test in the measurement is not controlled in a closed-loop, i.e., the direct and quadrilateral current from the lookup table is controlled in a closed loop. The achievable S1 power is determined at the end of the 30 min time frame. During measurement, the magnet temperature increases. The remanence flux density of the rear earth magnets decrease with increasing temperature. The decreasing rotor flux linkage leads to a decreased permanent magnet torque as shown in [22]. In the simulation, this influence is not considered, because the electromagnetic simulation is performed at room temperature. A more detailed analysis is not possible with the given hardware setup. In order to get a fully validated model, including the permanent magnet temperatures in the rotor, a measurement incorporating a rotor telemetry would be necessary such as performed in [23]. Such investigations could be content of future work.

5. Discussion

A detailed comparison of the conventional round wire design and the innovative preformed coil design is performed in this study. The used losses for the thermal modeling are validated in previous studies [8]. A thermal 2D-FEA model is developed to parameterizte a 2D-LPTN model. The 2-D FEA model is parameterized using microsections of a partial stator prototype. The 2D-LPTN and the 2D-FEA show good agreement with a maximum temperature deviation of 5.4 K. The 2D-LPTN model is extended to a 3D-LPTN model to represent the entire machine. Transient test bench measurements of four operational points are performed for each machine design. The 3D-LPTN and the transient test bench measurements show a very good agreement with a maximum deviation of 6 K. The parameterization of all models is possible under consideration of several boundary conditions that ensure a high degree of comparability. The maximum winding temperature of the preformed coil design at transient operation (OP1: 15 min) is approximately 25 °C lower than the winding temperature of the round wire design. The disadvantage of the increased winding losses of the preformed coil design at a operational point with maximum speed (OP4: 15 min) is outperformed by the improved heat transfer path. A detailed analysis of the local temperature distribution in the slot is performed for steady state temperature distribution of OP1 at 650 Nm and 50 rpm. The influence of the loss reduction and the improvement of the heat extraction path on the winding temperature distribution is performed. The reduction of losses has a slightly higher potential of improving the average temperature (6.2 K) in the slot than the improvement of the heat path (5.5 K). The improved heat path has a more significant influence on the maximum temperature (9.2 K) within the slot than the reduction of losses (7.3 K). The preformed coil design leads to a more uniform temperature distribution within the slot. Both effects significantly contribute to the improved performance of the preformed coil design in comparison to the round wire design. The relevance of the two factors depends on the studied operational point. The maximum continuous power of the preformed coil design and the round wire design is compared. Simulations as well as test bench measurements are performed. The improvement of the S1-power is estimated by the simulation model by 4.5% and by the measurements by 5.3% at 800 rpm and a maximum allowed winding temperature of 120 °C. Measurement and simulation show a very good agreement up to 800 rpm by a maximum deviation of 2.7%. For higher speeds, a maximum deviation of up to 4.5% between measurement and simulation of the round wire design is observed. A reason for this difference is the decreasing remanence flux density of the magnets for increasing magnet temperatures.

An investigation of the designs including rotor telemetry measurements could be content of a future study. The interturn insulation of the formed coil in its current setup can only withstand 120 °C (thermal class E). The analysis of the microsections additionally shows some error-prone locations, where no significant insulation thickness can be found. An insulation system for the formed coils, which meets the requirements of thermal class H, is under development.

6. Conclusions

The study shows that both aspects of the newly developed preformed coil design contribute to gain an improved performance in comparison to the round wire design in dependency of the operational conditions. The reduction of losses, especially in the base speed region of the machine, shows high potential of improvement. The improved heat removal path shows high potential for operational points with high copper losses. The thermal studies demonstrate that the improved heat dissipation path especially contributes to an improved heat removal ability from the middle of the notch to the side, i.e., to the stator teeth not to the stator yoke. This new heat path causes new inter-dependencies that should be taken into account during the design process of machines with preformed coil designs.

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Abbreviations

The following abbreviations are used in this manuscript:

ac alternating current	C	20

- dc direct current
- FEA Finite Element Analysis
- LPTN Lumped Parameter Thermal Network Model
- MTPA Maximum Torque per Ampere
- OP Operational Point

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