

Article

Characteristic Analysis of the Peak Braking Force and the Critical Speed of Eddy Current Braking in a High-Speed Maglev

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Abstract: In the eddy current braking system of high-speed maglev, the peak braking force and the critical speed are key factors determining the performance of eddy current braking force. In this paper, the analytical formula of eddy current braking force is derived by a subdomain method considering the skin effect of the induction plate, and, subsequently, the characteristics of peak braking force and critical speed are analyzed. The analytical model is set up in a 2D Cartesian coordinate system. The Poisson equations in each subdomain are listed by treating the vector magnetic potential as a variable. By combining the boundary conditions between two adjacent subdomains, the expressions of eddy current density and magnetic density in the induction plate are obtained. Then, the analytical formula of the eddy current braking force is obtained by the Ampere force formula. The results of finite-element analysis confirm the validity of the analytical calculation. The methods of improving the performance of eddy current braking force under high speed are proposed by parametric analysis of peak braking force and critical speed, which provides guidance for the design of the eddy current braking system in high-speed maglev.

Keywords: high-speed maglev; eddy current braking; peak braking force; critical speed; subdomain method; finite element verification

1. Introduction

Eddy current braking is an emergency braking system for high-speed maglev, which has the advantages of good controllability, no pollution, and no contact between the train body and rail. Therefore, it is very important to analyze the characteristics of eddy current braking. Referring to References [1,2], it is found that there is a peak point in the characteristic curve of eddy current braking force in relation to speed. The speed corresponding to the peak braking force is called the critical speed. When the speed exceeds the critical speed, the braking force decreases with the increase of speed. In addition, the braking force will become smaller and smaller at higher speeds. If the value of the critical speed is small, the braking force may not meet the braking requirements of the maglev train.

Research has been done to improve the characteristics of eddy current braking [3]. Studies are based on a novel hybrid excitation linear eddy current brake by the magnetic equivalent circuit method, the layer theory approach, and the finite element method. Reference [4] analyzes the impact of air gap length and thickness of the induction plate on the eddy current braking force, but does not specify the effect of these two parameters on the critical velocity. Reference [5] presents a new method for calculating the braking force of a drum-type eddy current brake, which shows that, as the conductivity

of the drum decreases, the speed at which the maximum torque is generated is shifted to high speeds. In Reference [6], the expressions of peak braking force and critical speed are given, which are based on the analysis of permanent-magnet eddy current braking, but the skin effect of the induction plate is not considered in these expressions. Reference [7] analyzes the influence of induction plates made of different materials on peak braking force and critical speed, and concludes that the conductivity of materials has no effect on peak braking force, but would affect the value of the critical speed, which is not analyzed in detail. Neither the peak braking force nor the critical speed has been analyzed in detail in the above literature, so the main objective of this paper is to investigate them.

In this paper, the performance of eddy current braking force, including peak braking force and critical speed, is studied theoretically. First, the subdomain method is used to derive the analytical formula of eddy current braking force, which is verified by finite-element analysis (FEA). Second, the parameters of peak braking force and critical speed are analyzed using the analytical formula. The methods of improving the peak braking force and the critical speed are proposed, which can provide guidance for improving the performance of eddy current braking force under high speed. The analytical formula can clearly reflect the relationship between the braking force and the parameters, and can quickly calculate the value of the peak braking force and the critical speed, which avoids the shortcomings of FEA, such as time-consuming and unable to reflect the mechanism of production of the braking force. The performance analysis of eddy current braking force can guide the optimization design of eddy current braking system of the high-speed maglev line in the laboratory. The speed of the maglev train on this test line is 600 km/h to 800 km/h and its mass is about 1 ton. The eddy current braking system is used as its emergency brake. Because of the high speed, it is very difficult and expensive to design and manufacture the eddy current brake test platform. At the same time, it can be found from Reference [8] and Reference [9] that FEA can accurately verify the validity of the subdomain method calculation. Therefore, this paper only includes FEA-based results rather than experimental verification.

2. Analytical Models and Calculation

To analyze the characteristics of peak braking force and critical speed, the analytical formula of braking force varying with speed needs to be deduced first. Due to the fast calculation speed and high accuracy of the subdomain method [10–13], it is used to deduce this analytical formula, which combines the theoretical formula of electromagnetic field and the boundary conditions of each region of the eddy current braking model.

2.1. *D Simplified Model of Eddy Current Brake*

Figure 1 is a 2D simplified eddy current braking model with one pair of poles. The multi-poles model is the superposition of one pair of poles model. Region I includes the eddy current braking device, which is divided into a core and exciting coils. The material of the core is magnetic steel, and the material of exciting coils is copper. Region II is the vacuum air gap. Region III is the non-magnetic conducting induction plate, and Region IV is the back iron, which can support the induction plate and can close the magnetic circuit. Since the subdomain method can only solve a 2D model, the influence of lateral end effect (in z -direction) of a 3D model, including end flux leakage and end leakage inductance of the exciting coils, cannot be taken into account using the 2D model. And it is assumed that:

- (1) The relative permeability μ_r of region I and region IV is infinite (regardless of magnetic saturation);
- (2) The relative permeability μ_r of the induction plate material in region III is 1, which is consistent with vacuum permeability.

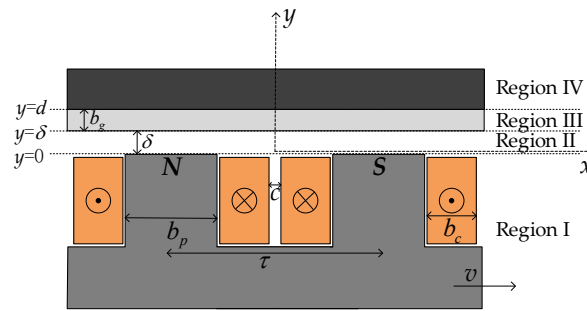


Figure 1. 2D simplified eddy current braking model.

The principle of generating the eddy current braking force is that the relative motion of the induction plate and the eddy current braking device whose *N* pole and *S* pole are arranged alternately produces the eddy current in the induction plate. In addition, the magnetic fields of the eddy current interact with the exciting magnetic field to generate a force, which opposes the direction of motion of the device, and this force is the eddy current braking force [14]. The energy conversion process of eddy current braking is that electric energy is converted into mechanical energy, which is dissipated by thermal energy.

As can be seen from Figure 1, since the device is slotted, the air gap length δ needs to be corrected by the Carter coefficient k_C [15]. The modified air gap length δ' is equal to the product of δ and k_C , which is defined as follows.

$$\begin{cases} k_C = \frac{\tau}{\tau - \gamma\delta} \\ \gamma = \frac{4}{\pi} \left[\frac{\tau - b_p}{2\delta} \arctan\left(\frac{\tau - b_p}{2\delta}\right) - \ln \sqrt{1 + \left(\frac{\tau - b_p}{2\delta}\right)^2} \right] \end{cases} \quad (1)$$

where τ is the pole pitch and b_p is the pole width.

Additionally, the following theoretical calculations use the modified air gap length δ' .

2.2. Expressions of Magnetic Vector Potential

Since the model is solved in a 2D plane, the flux density $\mathbf{B}$ and the magnetic field strength $\mathbf{H}$ of regions I, II, III, and IV have only components in the x and y directions, which are given by the following formula.

$$\begin{cases} \mathbf{B} = (B_x(x, y), B_y(x, y), 0) \\ \mathbf{H} = (H_x(x, y), H_y(x, y), 0) \end{cases} \quad (2)$$

In order to solve the expressions of flux density $\mathbf{B}$, it is necessary to introduce the magnetic vector potential A as the field function [16]. The general expression of the Poisson equation in the above four regions is given by the equation below.

$$\frac{\partial^2 A}{\partial x^2} + \frac{\partial^2 A}{\partial y^2} = -\mu_0 \mu_r \mathbf{J} \quad (3)$$

where $\mathbf{J}$ is the current density vector.

In the whole solution regions, the $\mathbf{J}$ has only the z -component, which is:

$$\mathbf{J} = (0, 0, J_z(x, y)) \quad (4)$$

Therefore, the magnetic vector potential A also has only the z -component, which is:

$$\mathbf{A} = (0, 0, A_z(x, y)) \quad (5)$$

In order to get the concrete expressions of A , the general expressions of the magnetic vector potential A in z direction can be obtained by the variable separation method [17].

$$\begin{cases} A_z(x, y) = \sum_{n=1}^{\infty} A_{zn}(x, y) \\ A_{zn}(x, y) = G(y)_n e^{jn \frac{\pi}{\tau} x} \end{cases} \quad n = 1, 3, 5 \dots \quad (6)$$

where $G(y)_n$ is the function of y and j is the imaginary number unit.

In Figure 1, the exciting current in region I can be equivalent to the linear current density at the interface between region I and region II, according to the principle of equal magnetic potential [18,19]. This equivalence will have influence on the results of magnetic field calculation in region I, but does not affect the accuracy of magnetic field calculation in region II, III, and IV. Additionally, the braking force calculation is carried out in region III, so it can ensure the correctness of the result of the braking force calculation. According to this equivalence, it can be considered that there is no exciting source in region I.

According to the principle of eddy current braking, eddy current can be induced in the induction plate of region III. In order to solve Equation (3), the expression of eddy current density is given by the equation below.

$$J_w = \sigma v B_{(III)y} \quad (7)$$

where σ is the induction plate conductivity, v is the running speed of eddy current braking device, and the direction of v is parallel to the x -axis, according to Figure 1.

Combined with Equation (6) and Equation (7), the expression of Poisson equation in Equation (3) in each region can be given by the following formulas.

$$\begin{cases} \frac{\partial^2 A_{(i)zn}}{\partial y^2} - \left(n \frac{\pi}{\tau}\right)^2 A_{(i)zn} = 0 & i = I, II, IV \\ \frac{\partial^2 A_{(i)zn}}{\partial y^2} - \left(n \frac{\pi}{\tau}\right)^2 A_{(i)zn} = \mu_0 \sigma v \frac{\partial A_{(i)zn}}{\partial x} & i = III \end{cases} \quad (8)$$

where i is the region number.

Combining Equation (6) and Equation (8), the expressions of the n th harmonic of magnetic vector potential in each region can be calculated by the following formulas.

$$\begin{cases} A_{(i)zn} = (C_{(i)n} e^{n \frac{\pi}{\tau} y} + D_{(i)n} e^{-n \frac{\pi}{\tau} y}) e^{jn \frac{\pi}{\tau} x} & i = I, II, IV \\ A_{(i)zn} = (C_{(i)n} e^{\alpha y} + D_{(i)n} e^{-\alpha y}) e^{jn \frac{\pi}{\tau} x} & i = III \end{cases} \quad (9)$$

where $C_{(i)n}$ and $D_{(i)n}$ are the constants to be determined, and the expression of α is given by Equation (10).

$$\alpha = n \frac{\pi}{\tau} \sqrt{1 + \frac{j \mu_0 \sigma v}{n \frac{\pi}{\tau}}} \quad (10)$$

Then, the expressions of flux density B can be calculated by Equations (9) and (10).

2.3. Magnetic Field Boundary Condition

In order to calculate the coefficients $C_{(i)n}$ and $D_{(i)n}$ in Equation (9), connection conditions on different interfaces are given by the following formulas.

$$\begin{cases} B_{(i)y} = B_{(i+1)y} \\ H_{(i)x} - H_{(i+1)x} = K_{(i)(i+1)z} \end{cases} \quad i = I, II, III \quad (11)$$

where $K_{(i)(i+1)z}$ is the z -component of the linear current density at the interface of region i and $i + I$.

Since $K_{(II)(III)z} = 0$, $K_{(III)(IV)z} = 0$, then the distribution of linear current density $K_{(I)(II)z}$ at the interface of region I and region II is analyzed below.

The distribution curve of linear current density at the interface of region I and region II is shown in Figure 2, and the abscissa expressions of the curves are shown in Table 1.

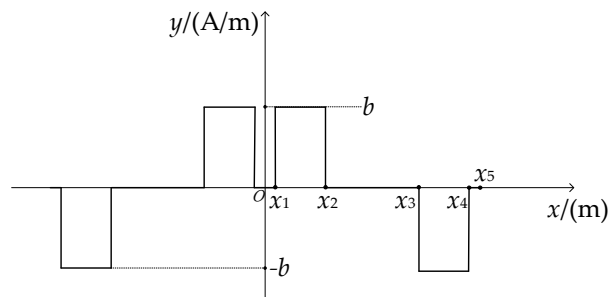


Figure 2. Distribution of equivalent linear current density at the interface of region I and region II.

Table 1. Abscissa expression in Figure 2.

Parameter	Value
x_1	$c/2$
x_2	$c/2 + b_c$
x_3	$\tau - (c/2 + b_c)$
x_4	$\tau - c/2$
x_5	τ

Fourier decomposition of the curve in Figure 2 shows that there are only odd components in the Fourier expression, which is shown below.

$$K_{(I)(II)z} = \sum_{n=1}^{\infty} a_n \cos\left(n \frac{\pi}{\tau} x\right) \quad n = 1, 3, 5 \dots \quad (12)$$

The expression of a_n is shown below.

$$a_n = \frac{2bk}{\pi} \quad (13)$$

The relationship between the linear current density constant b and the electric density J , the cross-sectional area S , the width b_c of the exciting coil, and the full factor k_f of the coil slot is given by the equation below.

$$b = \frac{JSk_f}{b_c} \quad (14)$$

k is the linear current density distribution coefficient, which is given by the formula below.

$$k = \frac{\pi}{\tau} \left[\int_{\frac{c}{2}}^{\frac{c}{2} + b_c} \cos\left(n \frac{\pi}{\tau} x\right) dx - \int_{\tau - (\frac{c}{2} + b_c)}^{\tau - \frac{c}{2}} \cos\left(n \frac{\pi}{\tau} x\right) dx \right] \quad (15)$$

According to Equation (12), and for convenience of calculation, the n th harmonic expression of linear density at the interface of region I and region II is given by the formula below.

$$K_{(I)(II)zn} = a_n e^{jn \frac{\pi}{\tau} x} \quad n = 1, 3, 5 \dots \quad (16)$$

2.4. Eddy Current Braking Force

In order to calculate the expression of braking force, the expressions of eddy current density and flux density in region III need to be calculated.

The n th harmonic expression of eddy current density J_{wn} in the induction plate can be obtained by combining Equation (7) along with Equations (9)–(16).

$$J_{wn} = \frac{-2jn\frac{\pi}{\tau}\mu_0\sigma va_n[e^{(n\frac{\pi}{\tau}-\alpha)\delta'}e^{\alpha y} + e^{(n\frac{\pi}{\tau}+\alpha)\delta'+2\alpha b_g}e^{-\alpha y}]}{(n\frac{\pi}{\tau}+\alpha)[1-e^{2(n\frac{\pi}{\tau}+\alpha b_g)}] + (n\frac{\pi}{\tau}-\alpha)(e^{2\alpha b_g} - e^{2n\frac{\pi}{\tau}\delta'})}e^{jn\frac{\pi}{\tau}x} \quad (17)$$

where δ' is the modified air gap length and b_g is the induction plate thickness.

The magnetic density in region III is the sum of the magnetic density produced by exciting current and eddy current. Because the magnetic field generated by the eddy current in the induction plate does not work on the induction plate itself, there is only the magnetic field generated by the exciting current working on the induction plate in region III when calculating the braking force. The magnetic density in region III can be simplified to that in the static case ($v = 0$). Combined with Equations (9)–(16), in region III, the n th harmonic expression of magnetic density in the y direction is given by the equation below.

$$B_{(III)yn} = -\frac{\partial A_{(III)zn}}{\partial x}\bigg|_{v=0} = -j\mu_0 a_n \frac{e^{n\frac{\pi}{\tau}y} + e^{2n\frac{\pi}{\tau}d}e^{-n\frac{\pi}{\tau}y}}{1 - e^{2n\frac{\pi}{\tau}d}}e^{jn\frac{\pi}{\tau}x} \quad (18)$$

where $d = \delta + b_g$.

Then, according to the ampere force formula [20], the n th harmonic calculation formula of braking force in the induction plate is given by the equation below.

$$F_{Wn} = -l_p w \cdot \operatorname{Re}\left(\int_{\delta}^d J_w(B_{(III)yn})^* dy\right) \quad (19)$$

where $l_p = 2p\tau$, which is the total length of eddy current braking device, p is the number of pole-pairs, w is the vertical paper depth of the device, and $*$ represents the complex conjugate of a variable.

The n th harmonic expression of braking force is calculated using the equation below.

$$F_{Wn} = -\frac{n\frac{\pi}{\tau}l_p w \mu_0 a_n^2 [M \sin(2Nb_g) + N \sinh(2Mb_g)]}{2[(MY_1 - NY_2 + n\frac{\pi}{\tau}Y_3)^2 + (MY_2 + NY_1 + n\frac{\pi}{\tau}Y_4)^2]} \quad (20)$$

$$\begin{cases} M = \sqrt{\frac{(n\frac{\pi}{\tau})^2 + n\frac{\pi}{\tau}[(n\frac{\pi}{\tau})^2 + \mu_0^2 \sigma^2 v^2]^{\frac{1}{2}}}{2}} \\ N = \sqrt{\frac{-(n\frac{\pi}{\tau})^2 + n\frac{\pi}{\tau}[(n\frac{\pi}{\tau})^2 + \mu_0^2 \sigma^2 v^2]^{\frac{1}{2}}}{2}} \\ Y_1 = \cosh(n\frac{\pi}{\tau}\delta') \sinh(Mb_g) \cos(Nb_g) \\ Y_2 = \cosh(n\frac{\pi}{\tau}\delta') \cosh(Mb_g) \sin(Nb_g) \\ Y_3 = \sinh(n\frac{\pi}{\tau}\delta') \cosh(Mb_g) \cos(Nb_g) \\ Y_4 = \sinh(n\frac{\pi}{\tau}\delta') \sinh(Mb_g) \sin(Nb_g) \end{cases} \quad (21)$$

Therefore, the expression of total braking force is given by the equation below.

$$F_W = \sum_{n=1}^{\infty} F_{Wn} \quad n = 1, 3, 5 \dots \quad (22)$$

In this paper, the expression of eddy current braking force (22) has been deduced, which is the key to analyze the peak braking force and the critical speed.

3. Finite Element Verification

According to the design requirement of a maglev project, a set of rated parameters is selected to design an FEA model of eddy current braking to verify the accuracy of the analytical formula of the braking force. The design parameters of the analytical and FEA model are shown in Table 2.

Table 2. Basic parameters of the eddy current braking model.

Parameter	Value
number of pole pairs p	6
maximum operating speed v_{max}	272 m/s
exciting current density J	12 A/mm ²
static air gap magnetic density B_0	1.5 T
the fill factor k_f	0.687
height of exciting coil h_c	115 mm
width of exciting coil b_c	15.3 mm
pole pitch τ	100 mm
air gap length δ	10 mm
height of yoke h_s	120 mm
height of pole h_p	120 mm
width pole b_p	69.2 mm
thickness of induction plate b_g	5 mm
thickness of back iron b_s	30 mm
model depth w	200 mm
induction plate conductivity σ	$10^6 \text{ S}\cdot\text{m}^{-1}$
setting simulation time	6 ms

Note: The model is simulated 160 times in the setting simulation time of 6 ms, which means the model is simulated every 0.0375 ms.

Since the linear brake is analyzed in this paper, there will be entry and exit effects when the brake moves. However, in Reference [6], it is mentioned that, if the brake exceeds four or more poles, the effects are insignificant. In this paper, the brake has 12 poles, so the entry and exit effects of the device are neglected in the analytical calculation.

According to the model parameters given in Table 2, the FEA simulation model of eddy current braking is set up in Ansoft Maxwell, as shown in Figure 3.

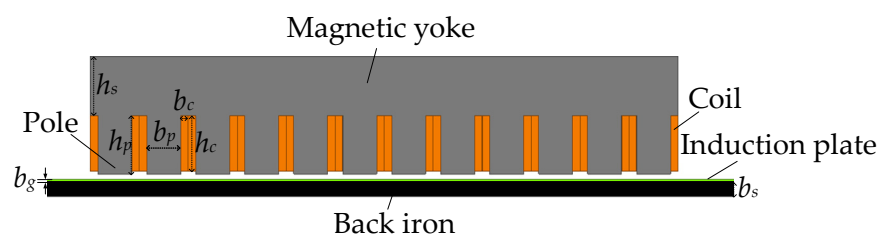


Figure 3. FEA model of eddy current braking.

In Figure 3, the upper part is the eddy current braking device, which is composed of a yoke, poles, and exciting coils. The lower part is the rail, which is composed of an induction plate and a back iron. And it is assumed that:

- (1) The magnetic field is simulated in 2D, and the Cartesian coordinate system and the International Unit System are used;
- (2) The lateral end effect has been neglected. The magnetic field distributes uniformly along the z -axis. The current density vector J and the magnetic potential vector A have only z -axis components.

In the FEA software, the eddy current braking model is simulated by the transient solver to verify the accuracy of the braking force analytical formula given in Equations (20), (21), and (22). Furthermore, Figures 4 and 5 are the static magnetic line distribution and the static magnetic density distribution of the FEA simulation, respectively.

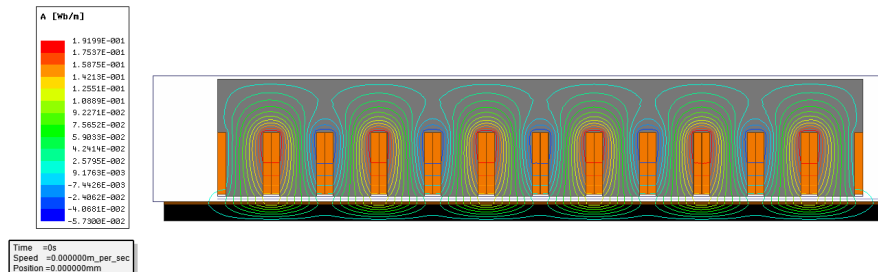


Figure 4. Distribution field diagram of the static magnetic line.

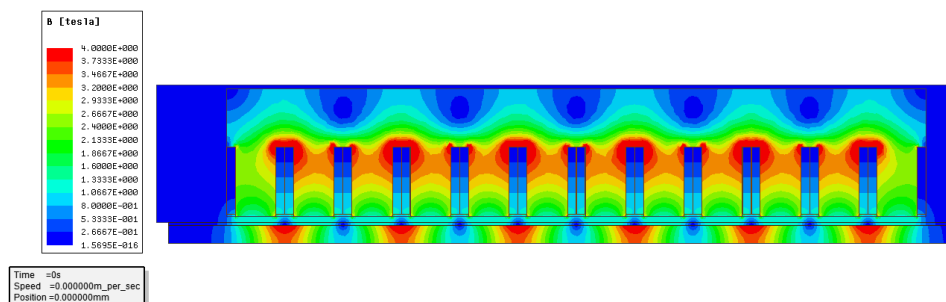


Figure 5. Distribution field diagram of static magnetic density.

Figures 4 and 5 can verify the accuracy of the model and its simulation. Additionally, it can also be seen from Figure 5 that the air gap magnetic density of the model is slightly less than 1.5 T, which is due to the saturation of magnetic circuit in the core that cannot be neglected in the FEA simulation.

Figure 6 shows the analytical and simulation curves of the variation of eddy current braking force with speed.

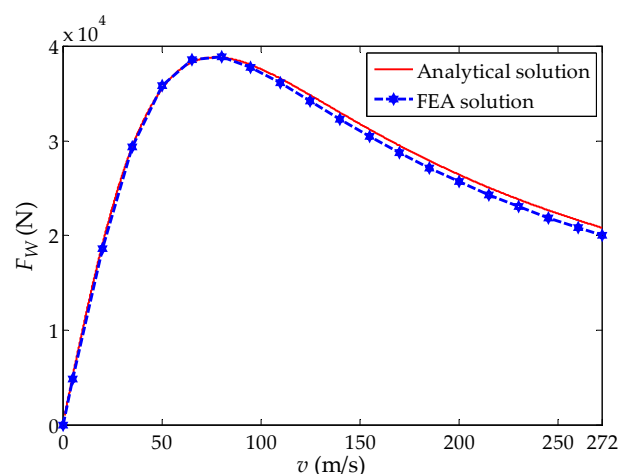


Figure 6. Eddy current braking force varying with speed.

It can be seen from Figure 6 that the analytical solution is in good agreement with the FEA solution, but the analytical result is slightly higher than the FEA solution at high speeds. This is due to the assumption that the relative permeability of the ferromagnetic region is infinite in the analytical calculation, while the loss of the ferromagnetic region increases at high speed in the FEA.

Then, the braking force decreases in the FEA at high speed. From Figure 6, the correctness of the braking force analytical formula is verified by this simulation.

Additionally, it takes about 3 h to run one working point (corresponding to one simulation point in Figure 6, and there are 20 FEA simulation points in Figure 6) whose setting simulation time is 6 ms by FEA. In contrast, the result of one working point can be obtained immediately by the subdomain method calculation. In addition, the running time of simulation is also related to the accuracy of the model partition. The finer the partition, the longer the running time of simulation will be.

4. Parameter Analysis

Equation (22) can be regarded as a function $F_W(v)$ of eddy current braking force and speed. Since the skin effect of the induction plate is considered in this paper, the analytical formula of braking force becomes very complicated, which makes it difficult to obtain the expressions of the peak braking force and critical speed point by differentiating the function $F_W(v)$ with respect to v . Therefore, the dichotomy method is used to calculate the root derivative of function $F_W(v)$, which is the critical speed v_m . Then the peak braking force F_{Wm} can be obtained.

Let the first derivative of function $F_W(v)$ be $F'_W(v)$. The root of $F'_W(v)$ is v_m , which is the critical speed and is also the stationary point of $F_W(v)$. We take the velocity interval as $[v_1, v_2]$ in the velocity range from 0 m/s to 272 m/s to make the following formula.

$$F'_W(v_1) \cdot F'_W(v_2) < 0 \quad (23)$$

If

$$\begin{cases} v_c = \frac{v_1+v_2}{2} \\ F'_W(v_c) = 0 \end{cases} \quad (24)$$

then $v_m = v_c$, otherwise

$$\begin{cases} F'_W(v_1) \cdot F'_W(v_c) < 0 \\ F'_W(v_2) \cdot F'_W(v_c) < 0 \end{cases} \quad (25)$$

Judging which inequality in Equation (25) is correct, then the new velocity interval $[v_1, v_c]$ or $[v_c, v_2]$ is redefined to calculate by dichotomy until it reaches the condition of Equation (26).

$$|v_a - v_b| < \zeta \quad (26)$$

where v_a and v_b are the endpoints of the velocity interval calculated by multiple dichotomies, and ζ is the given accuracy.

Then v_a or v_b is the stationary point v_m of $F_W(v)$ and the peak braking force F_{Wm} can be obtained by substituting v_m into Equation (22).

Using the calculated value of critical speed and the peak braking force, the characteristics of the peak braking force and the critical speed of different working conditions are analyzed.

4.1. Effect of the Air Gap Length on the Peak Braking Force and the Critical Speed

The air gap length is varied to study its impact on the peak braking force and critical speed while keeping other rated parameters unchanged. Figures 7 and 8 show the analytical and simulation curves of peak braking force F_W and critical speed v_m varying with the air gap length δ , respectively.

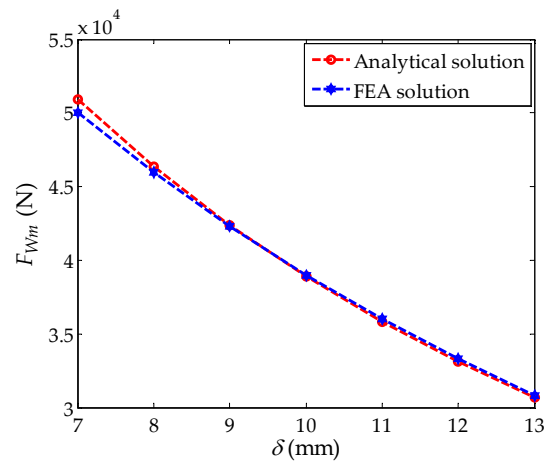


Figure 7. The peak braking force varying with the air gap length.

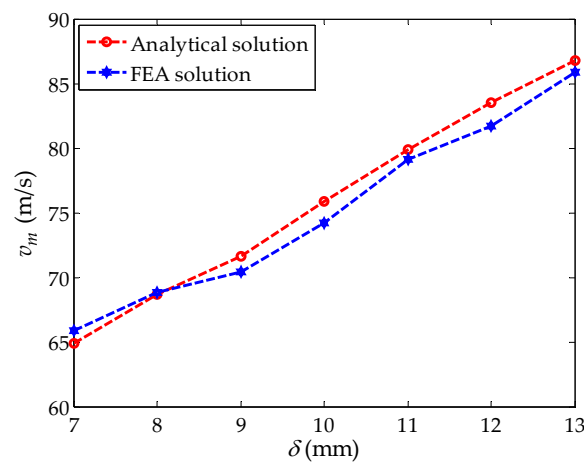


Figure 8. The critical speed varying with the air gap length.

As can be seen from Figures 7 and 8, a shorter air gap length is preferred since it leads to higher peak braking force values, while the critical speed represents the opposite. Therefore, the relationship between peak braking force and critical speed is mutually restricted by the change of the air gap length. When the air gap changes, one of the factors increases and the other decreases. Therefore, the appropriate air gap length should be selected according to the demand of actual working conditions to satisfy the value of the peak braking force and the critical speed.

4.2. Effect of the Pole Pitch on the Peak Braking Force and the Critical Speed

The influence of the pole pitch on peak braking force and critical speed is studied. The analytical and simulation curves of peak braking force and critical speed varying with the pole pitch are given in Figures 9 and 10, respectively.

From Figures 9 and 10, it can be seen that the variation tendency of the analytical solution is basically consistent with that of the FEA solution. Similar to the effect of air gap length on peak braking force and critical speed, the relationship between peak braking force and critical speed is also mutually restricted by the change of the pole pitch. The peak braking force increases with the increase of the pole pitch, but the critical speed decreases. The influence of the pole pitch on peak braking force and critical speed can guide the selection of the pole pitch.

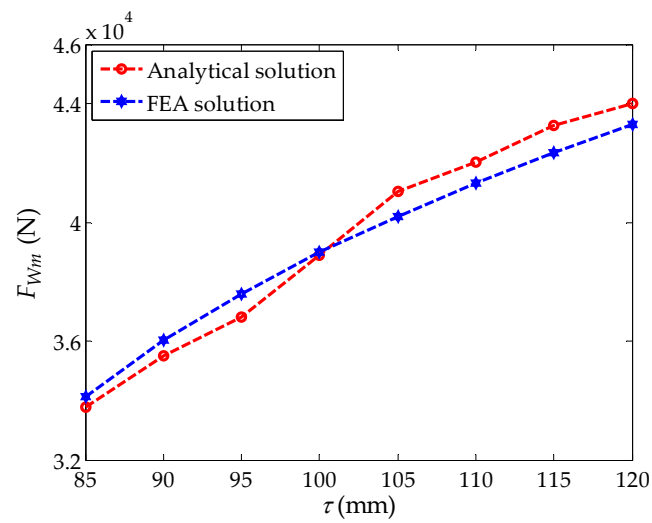


Figure 9. The peak braking force varying with the pole pitch.

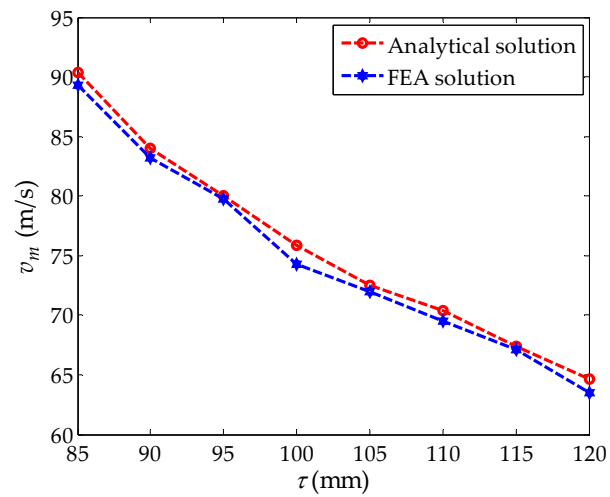


Figure 10. The critical speed varying with the pole pitch.

4.3. Effect of the Thickness of the Induction Plate on Peak Braking Force and Critical Speed

The influence of the thickness of the induction plate on peak braking force and critical speed is studied. The analytical and simulation curves of peak braking force and critical speed varying with the thickness of the induction plate are given in Figures 11 and 12, respectively.

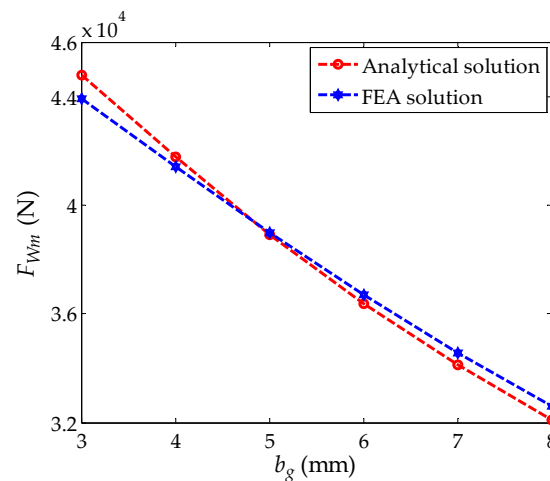


Figure 11. The peak braking force varying with the induction plate thickness.

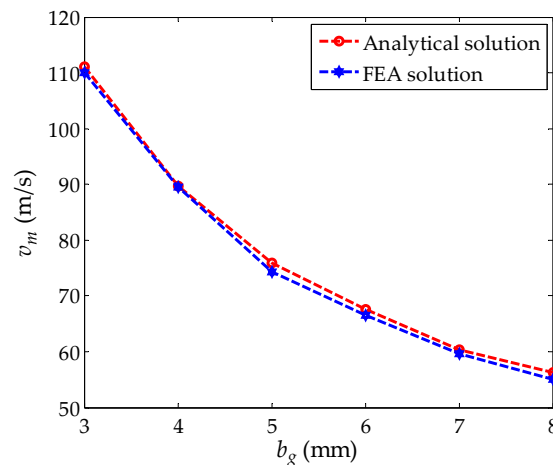


Figure 12. The critical speed varying with the induction plate thickness.

From Figures 11 and 12, it can be seen that the peak braking force and critical speed increase with the decrease of induction plate thickness. In addition, the critical speed increases in a trend of quadratic function. This is because the eddy current in the induction plate decreases and the eddy current magnetic field decreases as the thickness of the induction plate decreases. Therefore, if the eddy current magnetic field is to reach its original strength, it needs to increase the speed. Therefore, the critical speed increases with the thickness of the induction plate decreases. At the same time, the effective air gap length d decreases and the air gap magnetic field increases with the decrease of the thickness of the induction plate. Therefore, the peak braking force increases accordingly. In order to improve the performance of eddy current braking force at high speeds, the selection of a thinner induction plate can increase the peak braking force and the critical speed simultaneously.

4.4. Effect of Induction Plate Conductivity on Peak Braking Force and Critical Speed

The analytical and simulation curves of peak braking force and critical speed varying with the induction plate conductivity are given in Figures 13 and 14, respectively.

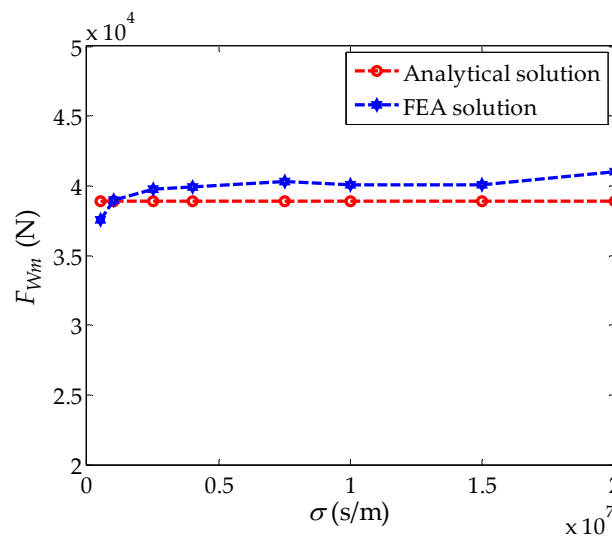


Figure 13. The peak braking force varying with the induction plate conductivity.

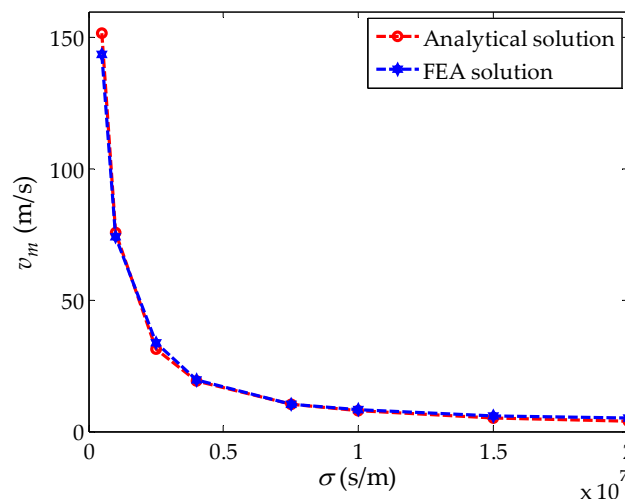


Figure 14. The critical speed varying with the induction plate conductivity.

As can be seen from Figure 13, the induction plate conductivity has no influence on the peak braking force. Furthermore, as can be seen from Figure 14, the critical speed is strictly an inverse function of the conductivity. Similar to the effect of induction plate thickness, the decrease of the conductivity of the induction plate also weakens the eddy current magnetic field. If the eddy current magnetic field is to reach its original strength, it needs to increase the speed. Therefore, the critical speed increases when the conductivity of the induction plate decreases. In this paper, the range of the induction plate conductivity is 5×10^5 s/m to 2×10^7 s/m in the simulation. In this range, choosing the induction plate material with lower conductivity can improve the value of critical speed, and then can improve the performance of braking force at high speed.

5. Error Analysis

Here is the error analysis of the analytical solution and the FEA solution. Table 3 shows the maximum error of the analytical solution and the FEA solution corresponding to the peak braking force and the critical speed with different variables.

Table 3. Maximum error of the analytical solution and FEA solution.

Variable	Peak Braking Force	Critical Speed
air gap length	1.89%	2.25%
pole pitch	2.17%	2.25%
induction plate thickness	2.01%	2.16%
induction plate conductivity	5.09%	21.4%

Among the three parameters of the air gap length, the pole distance, and the thickness of induction plate, the maximum error between the analytical solution and the FEA solution is less than 3%. Regarding induction plate conductivity, the maximum error between the analytical solution and the FEA solution of peak braking force is 5.09%, but the maximum error for the critical speed is 21.4%, which is relatively large. This is because, with the increase of conductivity, the value of the critical speed becomes very small. A small error between the analytical solution and the FEA solution can also cause a large value in percentage. The average error for the critical speed is 7.36% regarding plate conductivity, which is less than 10%. In conclusion, the errors between analytical solution and the FEA solution are within a reasonable range. The main reasons for these errors are that the saturation of the ferromagnetic region is not taken into account and the lateral end effect of the model is neglected in the analytical calculation.

6. Conclusions

This paper focuses on these two characteristics of eddy current braking force and has revealed how various design parameters can influence them. The analytical formula of eddy current braking force in the high-speed maglev exciting eddy current braking system is obtained by the subdomain method, and the accuracy of this formula is verified by FEA. To increase the braking force at high speeds, the appropriate air gap and pole pitch should be selected according to the demand in actual working conditions. The plate thickness should be as thin as possible and the conductivity should be as small as possible in the range of the induction plate conductivity selected in this paper while meeting other engineering requirements. The results provided in this paper can provide theoretical guidance for the preliminary design and optimization of the eddy current braking system in the high-speed maglev test line.

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